

FREIGHT TRAIN OPTIMAL TRAJECTORY CALCULATION
BY LINEAR PROGRAMMING

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NOMENCLATURE

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
A	$(3n+1+i+\tau_n+\omega_n) \times (3n+1+i+\tau_n+\omega_n)$ constant system matrix of the augmented discrete system: $x(k+1) = Ax(k) + Bu(k)$	-	63
$a_i(t)$	air flow rate in the brake cylinder pipe of the i^{th} train member	$\text{in}^3/\text{sec.}$	26
$a_{n+1}(t)$	air flow rate in the pipe between the train brake pipe and auxiliary reservoir of train member J	$\text{in}^3/\text{sec.}$	29
B	$(3n+1+i+\tau_n+\omega_n) \times (1+2)$ constant control matrix of the augmented discrete system: $x(k+1) = Ax(k) + Bu(k)$	-	63
$b(k)$	state variable constraint parameter	-	77

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$b_i(t)$	air braking effort developed at the i^{th} train member; $b_i(t) = e_i p_i(t)$	lbs.	16
C_i	fluid capacitance of brake cylinder of the i^{th} train member	$\text{in}^5/\text{lb.}$	26
C_{n+1}	fluid capacitance of auxiliary reservoir of train member J	$\text{in}^5/\text{lb.}$	29
c	((1+2)K+1) - dimensional cost vector: $c = \{0 \ 0 \ 0 \ \dots \ 0 \ 1\}^T$		74
$D(k)$	(3n+1+l+w _n +τ _n) X (1+2)K matrix defined by: $D(k) = \{A^{k-1} B \ A^{k-2} B \ \dots \ A B \ B \ 0 \ \dots \ 0\}$ where 0 is a (3n+1+l+τ _n +w _n) - dimensional null column vector		76
d	direction vector normal to the separating hyperplane defined as follows in the euclidian space E^n : $\langle d, x \rangle = \langle d, d \rangle$, where $\langle d, x \rangle$ denotes the inner product of the n-dimensional vectors d and x		72

SYMBOL

DEFINITION

UNITS

FIRST
APPEARS
ON PAGE:

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$d(s_i(t))$	the effective grade at the position of the i^{th} member	%	39
d_2	direction vector used to define velocity constraints	-	88
$d_{3 +1}$	direction vector used to define objective constraints	-	87
d_L	direction vector used to define tractive effort constraints	-	88
d_M	direction vector used to define dynamic braking constraints	-	88
$d^*(t)$	disturbance input to the system (ie. the time dependent function giving the effective grade at the head end of the train = actual grade + equivalent grade due to curvature)	%	40
$d^*(k)$	discrete time equivalent of $d^*(t)$	%	59
E^n	euclidian space of dimension n	-	72
E^+	the half-space region in E^n defined by: $E^+ = \{x: \langle d, x \rangle > \langle d, d \rangle\}$	-	72

(x)

SYMBOL

DEFINITION

UNITS

FIRST
APPEARS
ON PAGE:

E^-	the half-space region in E^n defined by: $E^- = \{x: \langle d, x \rangle < \langle d, d \rangle\}$		72
e_i	constant of proportionality between brake cylinder pressure and braking effort at the i^{th} train member	in^2	27
F	$\bar{m} \times (3n+1+1+\bar{\tau}_n, \bar{\omega}_n)$ terminal constraint matrix of the augmented discrete system: $x(k+1) = Ax(k) +$ $Bu(k); Fx(k) = f$		71
f	\bar{m} - dimensional terminal constraint vector of the augmented discrete system: $x(k+1) = Ax(k) + Bu(k);$ $Fx(k) = f$		71
$f_i^*(t)$	tractive/dynamic braking effort developed by the i^{th} train member (identically zero for all t if train member i is not a powered locomotive)	lbs.	17

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$F_i(t)$	force of combined draft gears between train member i and $(i+1)$	lbs.	16
g	acceleration due to gravity	ft/sec ²	39
$H(k)$	control variable constraint parameter		78
h_i	length of the i^{th} train member	ft.	19
i	the member number of the locomotive nearest train member i which is capable of effecting a reduction in brake pipe pressure	-	24
l	index indicating member number of train; $l=1$ indicates the leading member in the direction of train motion	-	16
j	the member number of the car furthest from any locomotive capable of effecting an increase in brake pipe pressure	-	29

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
J	Index indicating locomotive number (note that it is not necessarily true that $l = j$)		50
K	total number of discrete time intervals, each of duration T, in the interval (t_0, t_f)		57
KIPS	kilopounds	lbs	91
k	discrete time interval number		57
L''	tractive effort linear constraint parameter	lb-secs/ft	66
L'	tractive effort linear constraint parameter	lbs.	66
l	total number of locomotives in the train (∴ no. of cars = n-1)		50
M'	dynamic braking linear constraint parameter	lbs-secs/ft	66
\bar{m}	dimension of terminal constraint vector f		71

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
M''	dynamic braking linear constraint parameter	lbs.	66
m_i	the effective mass of the i^{th} train member (i.e. the actual mass of the member plus some allowance for the rotational inertia of the wheelsets)	slugs	16
n	total number of members in the train, including locomotives	-	29
$\underline{0}$	$(3n+1)$ - dimensional null column vector	-	64
$\underline{0}$	$(3n+1+\overline{\tau}_n + \overline{\omega}_n)$ - dimensional null column vector	-	76
$\underline{0}_{31}$	3-dimensional null column vector	-	50
$\underline{0}_{13}$	3-dimensional null row vector	-	50
$\underline{0}_{33}$	3X3 null matrix	-	50
P_i	maximum horsepower rating of locomotive member i	hp	33

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$p_i(t)$	brake cylinder pressure at the i^{th} train member	lbs/in ²	26
$p_{n+1}(t)$	reduction in auxiliary reservoir pressure of train member i (when $p_{n+1}(t) = 0$, all auxiliary reservoirs are completely recharged)	lbs/in ²	29
$q(t)$	(3n+1) - dimensional state vector of the continuous system: $\dot{q} = \Delta q + \Omega r$	-	46
$q(k)$	(3n+1)-dimensional state vector of the discrete system: $q(k+1) = \Delta * q(k) + \Omega * r(k)$	-	58
$q'(k)$	$(1 + \bar{\tau}_n + \bar{\omega}_n)$ - dimensional vector appended to $q(k)$ to represent the time delays in the system	-	62
$r(t)$	(2n+1) - dimensional control vector of the continuous system: $\dot{q} = \Delta q + \Omega r$	-	47
$r(k)$	(2n+1) - dimensional control vector of the discrete system: $q(k+1) = \Delta * q(k) + \Omega * r(k)$	-	58

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$r^*(t)$	reduction in equalizing reservoir pressure effected by the engineman, resulting from varying automatic brake valve handle positions	lb/in ²	22
$r_i^*(t)$	reduction in brake pipe pressure at the i^{th} train member	lbs/in ²	24
R_C	curve resistance	lbs.	41
R_{F+A}	friction and air resistance	lbs.	43
R_G	grade resistance	lbs.	39
R_i	fluid resistance of pipe connecting auxiliary reservoir and brake cylinder of the i^{th} train member	lbs-secs/in ⁵	26
R_{n+1}	fluid resistance of pipe connecting auxiliary reservoir and train brake pipe at train member J	lbs-secs/in ⁵	29
S_0	reference position of train	ft.	16

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$s_i(t)$	position of the i^{th} train member relative to s_0	ft.	16
T	discrete time interval duration	secs.	57
t	the independent variable time (t_0, t_f denote initial and final times, respectively)	secs.	16
t'	dummy variable		57
U	$\{(1+2)K + 1\}$ - dimensional constant offset vector		86
u	$\{(1+2)K + 1\}$ -dimensional LP vector defined by: $u = \{u(0), u(1), \dots, u(K-1)\}^T$		74
u'	$u' = u + U$		86
$u(k)$	$(1+2)$ - dimensional control vector of the augmented discrete system: $x(k+1) = Ax(k) + Bu(k)$		60
u	number of scalar control constraints		80

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
v'	minimum permissible train speed	ft/sec.	67
v''	maximum permissible train speed	ft/sec.	67
$v_i(t)$	velocity of the i^{th} train member	ft/sec.	16
w_i	total weight of the i^{th} member	lbs.	33
w_i	zero-force length of the draft gears and couplers between train member i and $(i+1)$	ft.	19
$x(k)$	$(3n+1+l+\bar{\tau}_n+\bar{\omega}_n)$ - dimensional state vector of the augmented discrete system: $x(k+1) = Ax(k) + Bu(k)$		60
x_0	initial condition of x ; $x_0 = x(0)$		71
\bar{x}	number of state trajectory constraints or minimax objectives		80

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$y_i(t)$	total resistance to motion of the i^{th} train member due to grades, curves, wind and friction	lbs.	17
Z	upper limit to dynamic braking level which applies to locomotives with extended range over the speed range 6-25	mph	lbs.
$z_i(t)$	extension (+) or compression (-) of draft gears between train members i and $(i+1)$	ft.	19
α_i	spring constant of the combined draft gears between train members i and $(i+1)$	lbs/ft	19
β_i	damping coefficient of the combined draft gears between train members i and $(i+1)$	lbs-secs/ft.	19
Y	response time of the automatic brake valve and equalizing reservoir	secs.	23

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
Δ	(3n+1) X (3n+1) constant system matrix of the continuous system: $\dot{q} = \Delta q + \Omega r$		48
Δ^*	(3n+1) X (3n+1) constant system matrix of the discrete system: $q(k+1) = \Delta^* q(k) + \Omega^* r(k)$		58
δ	transmission speed of the brake pipe reduction signal in the train line	ft/sec.	23
ξ	low speed dynamic braking coefficient (ie., constant of porportionality between velocity of locomotive and dynamic braking limit at low speeds)	lbs-secs/ft	35
η	constant of proportionality between the time delayed equalizing reservoir reduction appearing at the i^{th} train member and the supply pressure of that member's auxiliary reservoir.		24

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
θ	vertical angle of the track gradient	deg.	39
λ_i	velocity coefficient in linearized version of the Davis formula for resistance, due to air and friction of member i	secs/ft.	43
λ_{0i}	zero-velocity constant in linearized version of the Davis formula for resistance due to air and friction of member i ($\lambda_0 = \lambda_{0i}$ is assumed constant for all members of the train).		43
μ	coefficient of friction between wheel and rail (assumed constant for each train member)		33
π	dummy scalar variable which is to be minimized in the LP problem corresponding to the original minimax problem		74

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$\rho_C\{s_i(t)\}$	track curvature at position $s_i(t)$.	deg.	39
$\rho_G\{s_i(t)\}$	track gradient at position $s_i(t)$	%	38
σ_i	time constant of brake cylinder and connecting pipe system of the		
	th train member ($\sigma_i = R_i C_i$)	secs.	27
σ_{n+1}	time constant of auxiliary reservoir and connecting pipe system of train member J		
	($\sigma_{n+1} = R_{n+1} C_{n+1}$)	secs.	30
τ_i	overall time delay which occurs between the reduction made by the engineman and the initiation of braking action at member i.		
	($\tau_i = \gamma + \frac{1}{\delta} s_i - s_i $)	secs.	23
$\bar{\tau}_i$	number of time intervals, each of duration T, represented by the time delay τ_i (ie. $\bar{\tau}_i = \tau_i/T$)		59

SYMBOL	DEFINITION	UNITS	FIRST APPEARS ON PAGE:
$\phi(t)$	state transition matrix of the system (for system matrix A: $\phi(t) = e^{At}$)	-	57
Ω	(3n+1) X (2n+1) constant control matrix of the continuous system: $\dot{q} = \Delta q + \Omega r$	-	50
Ω^*	(3n+1) X (2n+1) constant control matrix of the discrete system: $q(k+1) = \Delta^* q(k) + \Omega^* r(k)$	-	58
ω_i	the time delay between the appearance of the effective grade $d^*(t)$ at $s_0(t)$ and $s_i(t)$	secs.	40
$\bar{\omega}_i$	the number of time intervals, each of duration T, represented by the time delay ω_i (i.e., $\bar{\omega}_i = \omega_i / T$)	-	59

ABSTRACT

Freight train separations and derailments attributable to adverse longitudinal dynamics of the train moving over undulating track have become more frequent with the advent of longer and heavier trains in recent years. This thesis addresses the problem of attempting to compute the optimal control trajectory for a specific train over a specific track section, wherein the optimality criterion used is the minimization of the maximum coupler force appearing in the train over the duration of its operation.

A mathematical model of the train (including a representation of its braking systems) is first formulated, based on a third-order model for each train member. The discrete equivalent of this continuous model, incorporating the time delays inherent in the system, is then derived. The optimization problem is then reformulated as a linear programming problem and solved on a digital computer.

The complementary relationship of this work to the research in this area currently underway at the Canadian Institute of Guided Ground Transport at Queen's University, Kingston, Ontario, is also delineated.

The procedure developed in this thesis appears to meet the stated objective of designing a method of pre-computing the "best" way to operate a specific train over a particular territory.

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CHAPTER 1
INTRODUCTION

1.1 Background:

Modern freight trains, particularly those in bulk commodity service, are characterized by very high tonnage, great length and increasing speed of operation. This trend has been due to the necessity of increasing both the capacity of single-track main line operations and the utilization of the motive power fleet. Failure to realize these operational objectives would result in the requirement for capital expenditures of such magnitude that rail transport costs for these commodities would soon become prohibitive.

However, the move towards operation of heavier and longer freight trains has unfortunately been marked by a significant increase in the frequency of train separations and derailments. The experience of the Southern Railway ⁽⁴³⁾ is typical evidence of the correlation between train uncoupling rate and train length. One can only conclude that there must be even more frequent occurrences of excessive inter-car forces which, although not resulting in such catastrophic events as train derailments or separations, certainly contribute to the costly incidence of car and lading damage. Estimates of this cost run as high as a quarter of a billion dollars. ⁽⁶⁰⁾

Although railbed, track and equipment failures are a major

cause of train derailments, a very significant portion of such incidents can be attributed mainly to the adverse dynamic effects of a train in motion, variously described as "train action", "slack action", "run-ins" and "run-outs." The suitability of such descriptive terms becomes immediately apparent when one considers the characteristics of a moving n-car train. Rather than being rigidly connected together, each car in the train is coupled with its neighbour in such a manner as to allow relative displacements between successive cars of approximately 0.5 feet. Hence, as a result of various unbalanced external forces, the individual cars comprising the train are likely to undergo relative velocities and accelerations. Indeed, a 200 car train can vary in length by anywhere up to 100 feet during motion.

Inter-car forces well in excess of 500,000 pounds resulting from such slack action in trains have been reported. (69) A serious problem currently confronting railways is to devise a method of operating trains such that these excessive slack action forces are minimized.

1.2 Prior Work:

A specific train moving over a particular section of track may be considered as a system with a specified set of parameter values being subjected to a specified set of input or control functions. In

the context of this system viewpoint, the problem defined above may be considered to be an optimization problem, wherein one is concerned with the selection of parameter values and/or control functions which will lead to optimum performance, as defined by an appropriate criterion.

Wilson (50) identifies the most important elements of this system, which may be divided into two groups, as follows:

System Parameters

- i) track condition (ie., effective coefficient of friction between wheel and rail)
- ii) freight car characteristics
- iii) locomotive characteristics
- iv) braking system characteristics
- v) inter-car coupling and draft gear characteristics
- vi) train marshalling (ie., sequential ordering of locomotives and cars)

System Control Functions

- i) track profile (ie., gradient and curvature as a function of time)
- ii) locomotive throttle action, as a function of time
- iii) train braking action, as a function of time

Practically speaking, the parameters (i) to (iv) and control function (i) are fixed; that is, the system designer is usually

not free to select these parameter values and control function. Thus, the only system elements available for design in the optimization process are the system parameters (v) and (vi) and the system control functions (ii) and (iii).

Therefore, all prior work on the above-defined problem, whether empirical or analytical in nature, may be classified as effectively either parameter optimization or optimal control approaches.

1.2.1. Parameter Optimization:

It is intuitively evident that the optimum selection of control functions from some class of functions is a more difficult analytical or empirical problem than the optimum selection of parameter values from some admissible set. Perhaps for this reason, most prior work on the slack action problem has in fact been in the area of parameter optimization.

Parker (29) for example, has given an empirical method for estimating the optimum marshalling of locomotives in the train.

Train simulations have been used extensively in attempts to arrive at optimal draft gear characteristics and train marshalling policies. The work of Wilson (50) and Roggeveen (34 - 38) is representative of the efforts that have been made in this direction.

The effects of draft gear characteristics on both freight car lading and longitudinal train motions has also been the subject of much analytical work. For example, Pipes (32) has treated the analysis of longitudinal motions of trains by the electrical analog. Frudenstein (14) has completed a dynamic analysis which relates the principle design parameters of the draft gears to the forces experienced by a resilient lading.

No attempt will be made here to compile an exhaustive history of work in the area of parameter optimization. The American Association of Railroads (AAR) has undertaken a comprehensive ten year program of research on track/train dynamics. The first phase of this program will include the preparation of an extensive bibliography of work in this area and this reference is expected to be available very shortly.

1.2.2 Optimal Control:

Despite much work in the area of parameter optimization, it is generally concluded that train separations caused by excessive coupler forces resulting from slack actions are still very much a problem.

For example, the Southern Railway has decided to abandon using all motive power on the front of long unit trains; instead, they are distributing locomotives through the train by adopting

remote-control techniques. Even so, the use of mid-train locomotive units has not in itself eliminated slack action problems. (67)

Parker (29) gives an excellent case study of train operation with remote-controlled locomotives and he too concludes that skillful handling of the train control functions is the most important factor in avoiding slack action problems.

In a 1968 panel discussion (67), S.H. Fillion, Chief Engineer for Waugh Equipment, a major producer of draft gears, expressed doubts that upgrading of the knuckle, coupler, yoke and draft gear can be the whole solution. Slack, he said, must be controlled because "if a run-in is going to buckle an essentially unstable train, then stronger inter-car hardware is not going to solve, and may actually complicate, the problem by causing structural failure or derailment."

Again, Wilson (50) recognizes the need to establish "optimal operating programs for a given train on a given route," and that this implies development of optimal brake and locomotive throttle functions such that they be "coordinated to yield a combined energy input and dissipation system which provides the best attainable performance."

Many train handling rules have been empirically evolved as a result of extensive operating experience. Reference (53) contains

an excellent example of the practical train braking and throttle control procedures that have resulted from experimental field tests by CP Rail. Goldstone (15) reports that the Southern Railway is even instrumenting one coupler in some of its trains with a strain gauge in an attempt to provide the train engineer with some input regarding the internal dynamics of his moving train.

There have been relatively few analytical approaches to the problem of designing optimal braking and throttle control functions for a specific train over a specific route.

Ichikawa (17) has successfully applied optimization methods for determining the optimal control law which minimizes energy consumption for a train operating from one station to the next. Unfortunately, he has had to assume an extremely simple, lumped-mass mathematical model for the train as a whole in order to be able to solve the resulting bounded state variable problem using the calculus of variations. This approach is simply impractical when a more realistic (i.e., more complex) train model is assumed.

To the author's knowledge, by far the most promising research on the problem of designing optimal control functions for freight trains is the work of McLane and Peppard (25), currently in progress at the Canadian Institute of Guided Ground Transport (CIGGT) at Queen's University, Kingston, Ontario. The objective of this work is, through the application of linear regulator theory, to develop

a feedback controller which minimizes coupler forces and velocity deviations from some scheduled velocity. The model for the longitudinal dynamics of a train is somewhat deficient in that the air braking system of the train is not represented. Nevertheless, Dr. McLane has indicated to the author that the model will be altered so as to include this important aspect of train operation. When this is accomplished, a practical regulator capable of compensating for model inaccuracies and such system noise as wind and variable track conditions should result.

Although there is a good possibility that the work currently underway at the CIGGT will result in an implementable train regulator, there still remains unresolved the problem of designing the optimal "scheduled velocity" about which this regulator should function. It is precisely this problem which is the subject of this thesis - to develop a procedure for designing for a specific train an ideal velocity schedule (or more accurately, an optimal system trajectory) which may then serve as the reference input for an on-board train regulator such as that being developed by McLane and Peppard.

One very desirable feature of such a design procedure is that the track profile be taken into account directly in the process of determining the optimal trajectory of the system. This track profile, consisting of the gradient and curvature as functions of

distance, is one of the most important inputs to the system. Whereas McLane and Peppard are attempting to design a regulator which is capable of operating over non-level terrain by treating gradient and curvature effects as just additional noise to the system, it would seem a better approach to take advantage of the fact that this input is known in the process of deriving the optimal system trajectory. Intuitively, if grades and curves are accounted for in the specification of the "scheduled velocity" as a function of time, then the regulator would have a much better chance of succeeding at compensating for the many other true system noises, such as wind, slippery rail conditions, etc.

1.3 Scope of Thesis and Brief Outline of Approach:

Athans ⁽¹⁾ gives an excellent outline of a systematic design process for controlling a non-linear uncertain system about a desired trajectory through the use of the stochastic linear-quadratic-gaussian (LQG) problem. * In his paper, Athans identifies thirteen distinct steps in the design process, comprising the following six separate functions:

- Part A: Deterministic Modelling
- Part B: Stochastic Modelling
- Part C: Linearization Modelling
- Part D: Control Problem Calculations
- Part E: Filtering Calculations
- Part F: Construction of Linearized Dynamic Compensator.

* I am indebted to Dr. McLane for bringing this paper to my attention.

Athans also gives the structure of an overall control system which combines pre-computed signals with on-line measurements generated by the linear dynamic compensator. This structure is repeated here as Figure 1.1 for reference.

Whereas McLane and Peppard are concerned with Parts C, D and F, resulting in the construction of a proportional - integral - derivative dynamic compensator, this thesis addresses the problem of pre-computing the optimal trajectory of the system as defined by the set of vector functions $\{u_0(t), x_0(t), y_0(t)\}$. With reference to Athans' paper, the objective of this thesis is to provide the pre-computed ideal open-loop control input and system output, by accomplishing the following three steps of Part A:

Part A: Deterministic Modelling -

Step 1:

Obtain deterministic model of actuators and plant. In state variable form, this yields the vector differential equation:

$$\dot{x}(t) = f \{ x(t), u(t) \} \dots\dots\dots (1-1)$$

Step 2:

Obtain deterministic model of plant and sensors:

$$y(t) = g \{ x(t) \} \dots\dots\dots (1-2)$$

Step 3:

Determine ideal input - state - output response:

$u_0(t)$: ideal (open-loop) input

$x_0(t)$: ideal state response

$y_0(t)$: ideal output response

As pointed out by Athans, the modelling process (steps 1 and 2 combined) is extremely critical to the ultimate success of the control design, and much of this thesis will be concerned with this modelling process.

The usual procedure for accomplishing step 3 is to formulate an optimal control problem of the following form:

"Given the system described by equation (1-1), find an admissible control vector $u_0(t)$ which transfers the system from some initial state $x_0(t_0)$ to some final state $x_0(t_f)$, such that $x_0(t_f)$ is a subset of S , where S is a target set of final states, and such that some cost functional defined by the designer is minimized."

It will be shown that the traditional means of obtaining a solution to the problem just formulated will not work in this instance, primarily because of the extremely high dimensionality of the system. A procedure first outlined by Enns (12) will then be employed to obtain a solution. The approach of Enns is to reformulate the given problem as a standard linear programming (LP) problem by an appropriate discretization of the system equation (1-1); the resulting LP problem may then be solved by any of the standard methods commonly available. An important attribute of this solution method is that the dimension of the LP problem is relatively independent of the dimensionality of the system (1-1).

The thesis then concludes with a worked example for a small sample train of three cars (or three sections), and a discussion of the results.

In essence, the problem under consideration in this thesis is to devise a method for controlling excessive inter-car forces caused by relative accelerations between successive cars that may result from the slack-action of a train in motion.

CHAPTER 2.

PHYSICAL SYSTEM AND MODEL FORMULATION

2.1 Assumptions Regarding Overall System Model:

In this chapter, a model of the overall physical system will be formulated. The model so derived will be a one-dimensional representation in which only longitudinal motion and forces will be considered directly.

To be sure, the actual train undergoes vertical displacements as a result of track gradients; but it is only the horizontal components of the forces due to gravity that we are interested in and the effect of these forces is represented in the model as resistance to longitudinal motion. Also, the track itself is assumed to be continuous in that the impact of wheels on rail joints is neglected.

Similarly, lateral forces are certainly encountered between the flanges of the car wheels and the rail guideways, particularly through curved track sections, but the effect of these forces is again represented in the model simply as resistance to longitudinal motion.

Additional assumptions will be noted as each of the elements of this physical system is modelled in the following sections.

2.2. Model for Individual Train Members:

Roggeveen (34) has shown in freight car impact tests that a two-lump model of an individual train member must be assumed in order to depict the coupler force versus time characteristic reasonably accurately for all coupler force levels. However, he has also shown that a first-order lumped model gives a sufficiently accurate representation of the impact of successive train members up to a certain critical coupler force level, corresponding to the point where the spring of the inter-car draft gear is completely compressed. At force levels exceeding this critical value, the two train members are effectively in solid metal-to-metal contact, and it is in this region that the first-order model fails and a second-order representation becomes necessary.

However, the end objective of this whole exercise is to minimize the peak coupler forces in the train. Making the assumption that the optimal control derived in this dissertation will succeed in keeping coupler forces at least below this critical level (approximately 200,000 pounds for average draft-gears) permits us to assume the simpler first-order model. By way of justifying this assumption, Wilson (50) reports excellent agreement between test data from instrumented trains and computer predictions from a model which assumes a first-order representation for each train member.

Let us define the following nomenclature:

t : the independent variable time (in seconds)

i : the subscript indicating the member number in the train; $i = 1$ indicates the leading member in the direction of train motion.

m_i : the effective mass (in slugs) of the i^{th} train member; this is the actual mass of the member plus some allowance for the rotational inertia of the wheelsets (see Taylor (46), for instance).

$s_i(t)$: the position (in feet) of the i^{th} train member, relative to some reference position S_0 .

$v_i(t)$: the velocity (in ft/sec) of the i^{th} train member.

$b_i(t)$: the air braking effort (in pounds) developed at the i^{th} train member.

$F_i(t)$: the force (in pounds) on the i^{th} and $(i + 1)^{\text{th}}$ train members from the combined draft gears between them.

$f_i^*(t)$: the tractive or dynamic braking effort (in pounds) developed by the i^{th} train member (Identically zero for all t if train member i is not a powered locomotive).

$y_i(t)$: the total resistance to motion (in pounds) of the i^{th} train member due to grades, curves, wind and friction.

Considering the model depicted in Figure 2.1, then from Newton's Second Law the equations of motion for the i^{th} train member are:

$$\dot{s}_i(t) = v_i(t) \dots \dots \dots (2-1a)$$

$$\dot{v}_i(t) = \frac{1}{m_i} \{ F_{i-1}(t) - F_i(t) - y_i(t) - b_i \dot{v}_i(t) + f_i^*(t) \} \dots (2-1b)$$

where $\dot{(\cdot)} \equiv \frac{d}{dt} (\cdot)$

2.3 Model for Draft Gears and Couplers between Train Members:

Smith (45) lists over twenty different types of draft gears, but for our purposes we shall assume that the important characteristics of the combined draft gears between any two successive train members

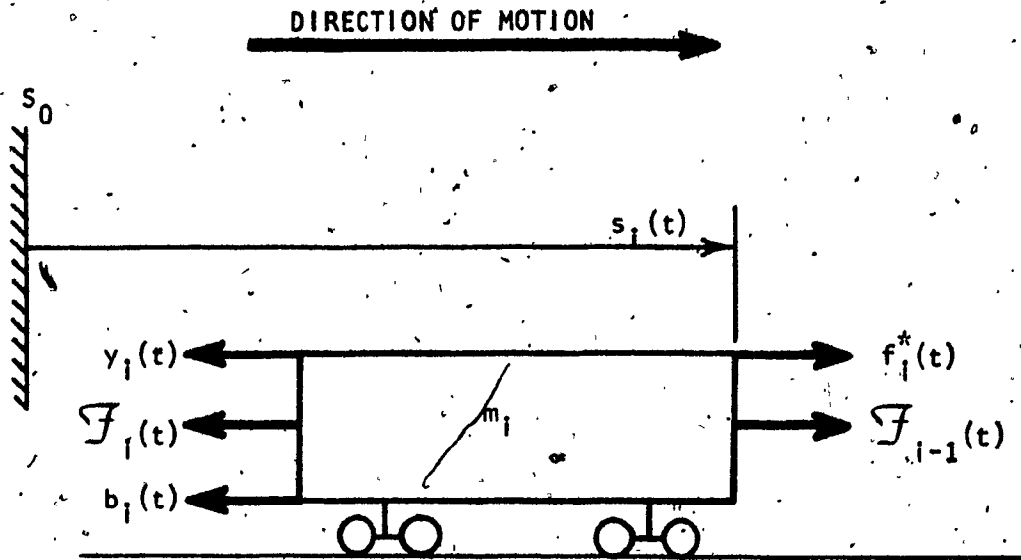


Figure 2.1: Model of the i^{th} member of the train.

can be modelled by a translational pure spring and viscous damper arranged in parallel, as in Figure 2.2 wherein the following additional symbols are defined:

α_i : the spring constant (in pounds/foot) of the combined draft gears between train members i and $(i + 1)$.

β_i : the damping coefficient (in pounds-seconds/foot) of the combined draft gears between train members i and $(i + 1)$.

h_i : the length (in feet) of train member i .

w_i : the zero-force length (in feet) of the draft gears between train member i and $(i + 1)$.

$z_i(t)$: the extension or compression (in feet) of the draft gears between train members i and $(i + 1)$.
Note that $\{ z_i(t) + w_i \}$ represents the total separation between the trailing wall of member i and the leading wall of member $(i + 1)$.

It should be noted that the non-linear effect (namely dead-zone) of slack in the draft gears, as described by Roggeveen (37),

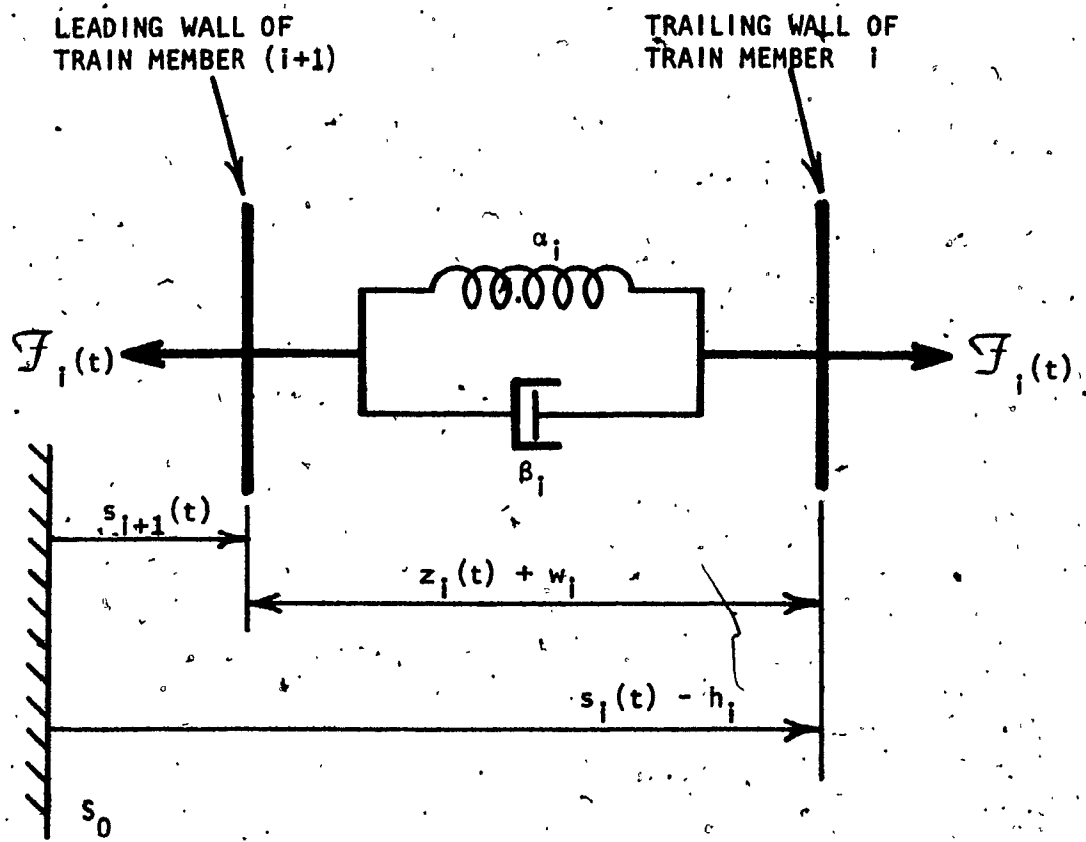


Figure 2.2: Model of the combined draft gears between train members i and (i+1)

is specifically not included in this model. Opinion is currently divided on the question of whether or not this non-linearity is an important factor which should be represented in the model; the current efforts of McLane and Peppard (25) in this regard perhaps will settle this issue.

From Figure 2.2, we have:

$$z_i(t) = s_i(t) - s_{i+1}(t) - h_i - w_i \dots\dots\dots(2-2)$$

and $\dot{z}_i(t) = \dot{s}_i(t) - \dot{s}_{i+1}(t) = v_i(t) - v_{i+1}(t) \dots\dots(2-3)$

since h_i and w_i are constants.

Thus, the mathematical model of the inter-car connection is:

$$F_i(t) = \alpha_i z_i(t) + \beta_i \dot{z}_i(t) \\ = \alpha_i z_i(t) + \beta_i \{ v_i(t) - v_{i+1}(t) \} \dots\dots\dots(2-4)$$

2.4 Model for Train Air Brake System:

Reference (65) contains an excellent equipment glossary and overall description of the train air brake system. Other relevant material dealing with this important aspect of train operation is given in the bibliography.

To quote from reference (65) :

"The primary source for slowing and stopping trains is the compressed air stored at each car in the train. This air is admitted to the brake cylinder, translated to a mechanical force acting on the brake rigging, and in turn forces the brake shoes against the wheels of the train to retard their rotation. The amount of braking is controlled by the reduction of brake pipe pressure, as determined by the automatic brake valve handle position in the locomotive. This brake pipe reduction throughout the train causes the control valve at each car to respond. The control valve then meters the stored air in the brake cylinders in proportion to the amount of drop in brake pipe pressure."

Let $r^*(t)$ represent the reductions in brake pipe pressure (lbs/in²) called for by the engineer as a result of varying automatic brake valve handle positions. There are definite restrictions on the manner in which this brake valve handle position can be varied, but these constraints will be dealt with later.

Now consider the following sequence of events which occurs after the engineer has initiated the braking action by moving the brake valve handle from the release position:

- i) the locomotive equalizing reservoir pressure reduces;
- ii) this in turn causes the train-brake pipe pressure at the locomotive to reduce;
- iii) this pressure drop is then transmitted through the brake pipe of the train until it reaches the control valve on the i^{th} train member;
- iv) the control valve then meters the supply of air from the auxiliary reservoir to the brake cylinder of the i^{th} train member.

Rather than model in detail the above sequence of events, let τ_i represent the overall time delay which occurs between events (i) and (iv). Then, τ_i is known to be of the form:

$$\tau_i = \gamma + \frac{1}{\delta} \left| s_i(t) - s_i(t) \right| \dots \dots \dots (2-5)$$

where symbols are defined as follows:

γ : response time (seconds) of the automatic brake valve and equalizing reservoir.

δ : transmission speed (feet/second) of the pressure drop signal in the brake pipe of the train.

i : the member number of the locomotive nearest train member i which is capable of effecting a reduction in brake pipe pressure.

(Typical values for γ and δ are 2 seconds and 500 feet/second, respectively).

The reduction in brake pipe pressure at the i^{th} train member, $r_i^*(t)$, is therefore given by: $r_i^*(t) = r^*(t - \tau_i)$. Thus, the auxiliary reservoir supply pressure to the brake cylinder pipe can be represented as $\eta \cdot r^*(t - \tau_i)$, where τ_i is defined by eq. (2-5) and η is a constant of proportionality between the time delayed equalizing reservoir pressure drop appearing at the control valve of the i^{th} train member and the supply pressure of that member's auxiliary reservoir.

The braking force generated at the wheels of member i is proportional to the pressure in its brake cylinder. In essence, the brake cylinder behaves approximately like a pure fluid capacitance in which energy is stored because of the compressibility of the fluid. The brake cylinder is supplied with air from the auxiliary reservoir through a long, relatively narrow, brake cylinder pipe. Hence, we shall assume the simple model for brake action as illustrated in Figure 2.3.

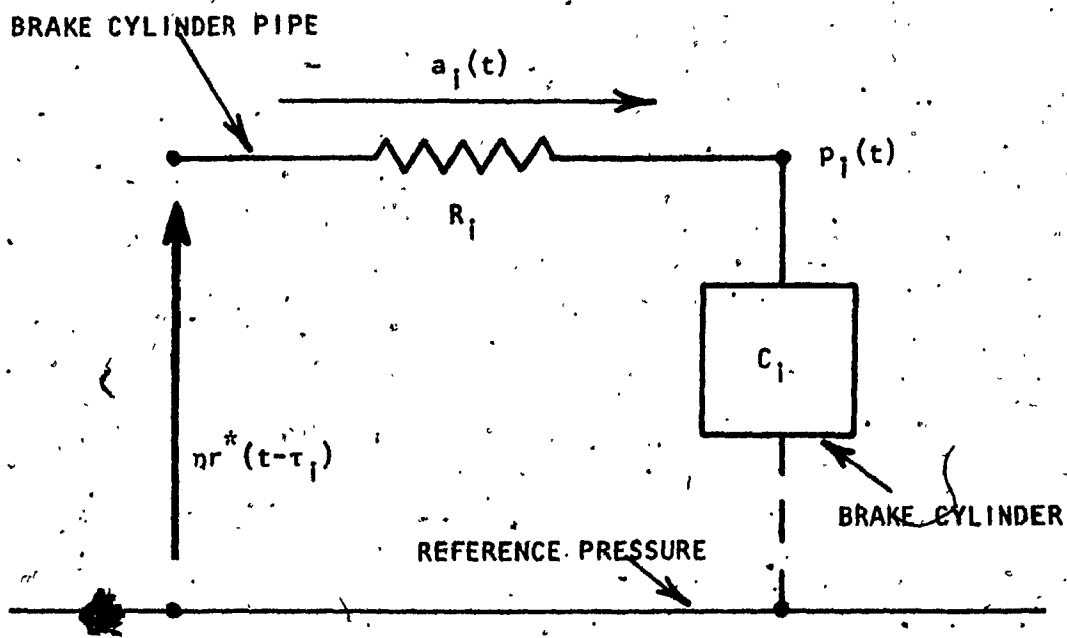


Figure 2.3: Model of brake action at train member i .

The new nomenclature of Figure 2.3 is defined as follows:

C_i : fluid capacitance (in 5 /lb.) of the brake cylinder at train member i .

R_i : fluid resistance (lb-secs/in 5) of the brake cylinder pipe at train member i .

$a_i(t)$: air flow rate (in 3 /sec) in the brake cylinder pipe of the i^{th} train member.

$p_i(t)$: brake cylinder pressure (lbs/in 2) at the i^{th} train member.

Note that the auxiliary reservoir supply pressure is modelled as an ideal pressure source. In actual fact, the auxiliary reservoir is a fixed volume container, but since its volume is significantly greater than that of the brake cylinder, this representation is a reasonable approximation.

From Figure 2.3, we obtain the mathematical model for brake action at train member i as:

$$a_i(t) = C_i \dot{p}_i(t) = \frac{1}{R_i} (\eta \cdot r^* (t - \tau_i) - p_i(t)) \dots \dots (2-6)$$

and \therefore
$$\dot{p}_i(t) = \frac{1}{C_i} (\eta \cdot r^* (t - \tau_i) - p_i(t)) \dots \dots (2-7)$$

where $\sigma_i = R_i C_i$ is the time constant (seconds) of the braking system.

Now define e_i to be the constant of proportionality (in^2) between brake cylinder pressure and the braking force developed at the i^{th} train member. Therefore, e_i takes into account such factors as the brake rigging action, the coefficient of friction between the brake shoes and wheels of member i , and the coefficient of friction between wheel and rail. Thus, the braking effort $b_i(t)$ of Figure 2.1 is given by:

$$b_i(t) = e_i p_i(t) \dots\dots\dots (2-8)$$

As mentioned previously, in order for the braking action model defined by eqs. (2-7) and (2-8) to be valid, $r^*(t)$ must be suitably constrained so as to represent realistic train air brake applications and releases.

For instance, when an application of the train air brakes is called for, the minimum and maximum automatic brake valve handle positions are known (65) to correspond to equalizing reservoir reductions of 7 and 23 psi respectively. Thus we have the first constraint on $r^*(t)$:

$$\left. \begin{array}{l} \text{either } r^*(t) = 0 \\ \text{or } 7 \leq r^*(t) \leq 23 \end{array} \right\} \dots\dots\dots (2-9)$$

In addition, although brake application increases are permitted at any time, partial brake releases are not valid. That is, if Δt is some small time increment, then:

$$\left. \begin{array}{l} \text{either } r^*(t + \Delta t) = 0 \\ \text{or } r^*(t) \leq r^*(t + \Delta t) \leq 23 \end{array} \right\} \dots\dots\dots(2-10)$$

One final constraint on $r^*(t)$ is required, in order to avoid undesired brake releases in the actual train. Peterson (30) gives an excellent account of the causes of such undesired releases, and illustrates that these can be attributed to inappropriate brake applications made under "false gradient" conditions.

Some explanation of terminology is in order before proceeding further. When the automatic brake valve handle is moved to the release position, this causes the brake valve to restore air pressure to the brake pipe. The increasing brake pipe pressure then causes the equipment at each member to move to release, thereby exhausting the brake cylinder pressure and recharging the auxiliary reservoir. Now, "true gradient" may be defined as the difference between the head end and rear car brake pipe pressures when the train is fully charged, and is primarily due to leakage in the piping of the system. Whenever the train is not fully charged (i.e., the recharging function alluded to previously has not been completed), a "false gradient" is said to exist and is defined as the difference between the actual gradient at that point in time and

the true gradient for that particular train.

In order to represent this final constraint on $r^*(t)$, it will therefore be necessary to represent in the model this recharging of the auxiliary reservoir farthest from the locomotive capable of supplying air to the brake pipe. To this end, let us define the following nomenclature:

n : the total number of members in the train, including locomotives.

J : the member number of the car furthest from any locomotive capable of effecting an increase in brake pipe pressure.

C_{n+1} : fluid capacitance (in $^5/\text{lb}$) of auxiliary reservoir of train member J .

R_{n+1} : fluid resistance (lb-secs/in 5) of pipe connecting auxiliary reservoir and brake pipe at train member J .

$a_{n+1}(t)$: air flow rate (in $^3/\text{sec}$) in the pipe connecting the auxiliary reservoir and brake pipe at train member J .

$P_{n+1}(t)$: reduction in auxiliary reservoir pressure (lbs/in 2) at train member J ; as such, $p_{n+1}(t)$ is also the

"false" gradient existing at any time t . When $p_{n+1}(t) = 0$, all auxiliary reservoirs in the train are completely recharged.

The model of this recharging function of the J^{th} member's auxiliary reservoir is therefore assumed as in Figure 2.4.

From Figure 2.4 we have:

$$p_{n+1}(t) = \frac{1}{\sigma_{n+1}} \{ r^*(t) - p_{n+1}(t) \} \dots \dots \dots (2-11)$$

where $\sigma_{n+1} = R_{n+1} C_{n+1}$ is the time constant of the recharging system (secs).

Peterson (30) shows that in order to avoid undesired train brake releases, brake applications required must be increased by an amount equal to any false gradient existing at the time of application.

Thus, the final constraints on $r^*(t)$ may be expressed as:

$$\left. \begin{array}{l}
\text{if } r^*(t) = 0, \text{ then} \\
\text{either: } r^*(t + \Delta t) = 0 \\
\text{or: } 7 + p_{n+1}(t) \leq r^*(t + \Delta t) \leq 23
\end{array} \right\} \dots \dots \dots (2-12)$$

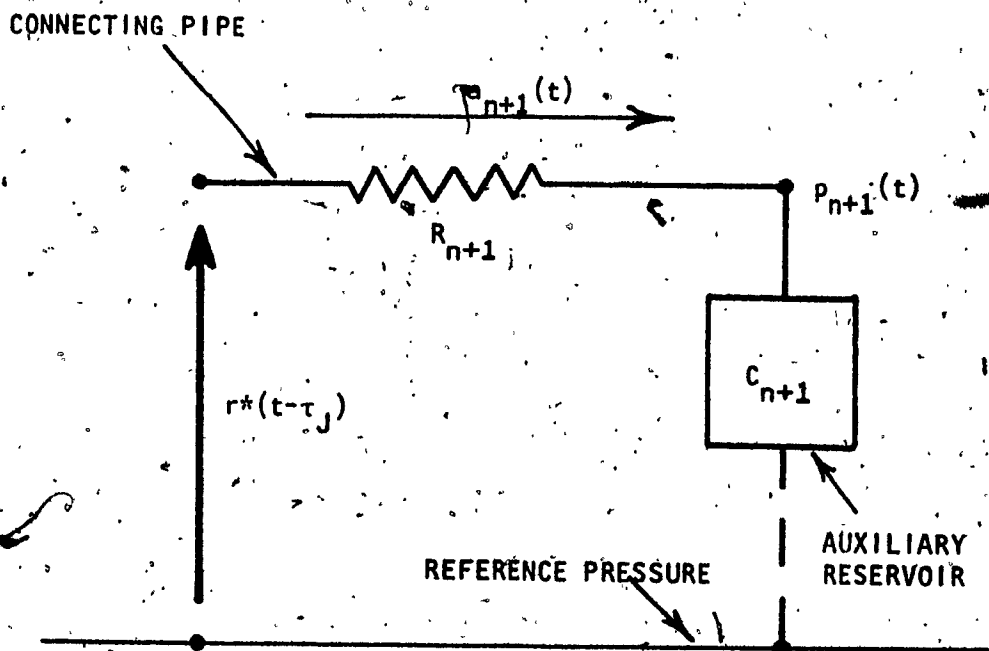


Figure 2.4:

Model of train recharging action. When $r^*(t)$ is brought to zero then brake release is initiated and when $p_{n+1}(t) = 0$, then train is fully recharged. Thus, $p_{n+1}(t)$ is a measure of the "false" gradient in the train at any time t .

For convenience, the constraints (2-9), (2-10) and (2-12) are summarized here as follows:

$$\left. \begin{array}{l}
 \text{I) } \quad \text{if } r^*(t) = 0, \text{ then} \\
 \quad \text{either: } r^*(t + \Delta t) = 0 \\
 \quad \quad \text{or: } p_{n+1}(t) + 7 \leq r^*(t + \Delta t) \leq 23 \\
 \\
 \text{II) } \quad \text{if } r^*(t) \neq 0, \text{ then} \\
 \quad \text{either: } r^*(t + \Delta t) = 0 \\
 \quad \quad \text{or: } r^*(t) \leq r^*(t + \Delta t) \leq 23
 \end{array} \right\} \dots\dots (2-13)$$

2.5 Representation of Motive Power Constraints:

Each locomotive in the train represents a potential source of both tractive and braking effort. As in Figure 2.1, let $f_i^*(t)$ represent the force output (lbs) of the locomotive member i at any time t . Then for $f_i^*(t) > 0$ the unit is "motoring", that is, applying tractive force; and for $f_i^*(t) < 0$ the unit is in "dynamic braking", that is, applying braking force. Obviously, if $f_i^*(t) = 0$, then the unit is idling. In order to represent real locomotives, $f_i^*(t)$ must be suitably constrained, as follows.

2.5.1 Motoring Constraints (when $f_i^*(t) > 0$):

The theoretical maximum force that a locomotive can transmit to the rail is limited by two factors. These are (1) the adhesion

limit (coefficient of friction) between the wheel and the rail, beyond which wheel slip will occur, and (ii) the actual power of the locomotive:

i) Adhesion Limit:

If W_i is the effective weight of the locomotive (lbs) and μ is the tractive adhesion coefficient (dimensionless), then:

$$f_i^*(t) \leq \mu \cdot W_i \dots\dots\dots (2-14)$$

ii) Motive Power Limit:

If P_i is the maximum horsepower rating of the locomotive, and $v_i(t)$ is the velocity of the locomotive (ft/sec), then from reference (61):

$$f_i^*(t) \leq \frac{451 P_i}{v_i(t)} \dots\dots\dots (2-15)$$

where the constant 451 is derived from 550 ft-lbs/sec per horsepower X 0.82 transmission efficiency of the electro-mechanical drive system.

2.5.2. Dynamic Braking Constraints (when $f_i^*(t) < 0$):

There are basically four constraints on the braking force, which can be generated by locomotive member 1:

i) Adhesion Limit:

As in motoring, the maximum braking force is limited by the adhesion between wheel and rail:

$$f_i^*(t) \geq -\mu W_i \dots\dots\dots(2-16)$$

ii) Motor Heating Limit:

In dynamic braking, the grid current (and hence the braking horsepower developed through I^2R loss (59, 66) in the braking resistor grids) is limited. For instance, from performance curves (65) for the SD-40 locomotive, this limit is known to be as follows:

$$f_i(t) \geq -\frac{638 P_i}{v_i(t)} \dots\dots\dots(2-17)$$

iii) Motor Field Current Limit:

At low speeds, the motor armature current is less, as the wheels are turning slowly. Hence, the braking horsepower is less, since I^2R is smaller due to the smaller grid current I . This limit on the available braking effort is of the form (65):

$$f_i^*(t) \geq - \xi v_i(t) \dots \dots \dots (2-18)$$

where the value of ξ depends on whether dynamic braking is standard or extended range (see reference (65) for an explanation of these terms).

Typical values for a 4 axle, 2250 horsepower unit are $\xi_{std} = 1110 \text{ lb-secs/ft.}$, and $\xi_{ext} = 3960 \text{ lb-secs/ft.}$

iv) Extended Range Dynamic Braking Limit:

If extended range dynamic braking is available, then over the speed range of approximately 9 to 37 ft/sec, the following constraint applies:

$$f_i^*(t) \geq - Z \dots \dots \dots (2-19)$$

Typically, $Z = 40,000$ lbs. for a GP-40 unit (4 axles, 2250 horsepower), and $Z = 60,000$ lbs. for an SD-40 unit (6 axles 3000 horsepower).

2.5.3. Summary of Motive Power Constraints:

For convenience, the constraints on $f_i^*(t)$ are summarized here as follows:-

$$\left. \begin{aligned}
 &|f_i^*(t)| \leq \mu W_i \\
 &-\frac{638P_i}{v_i(t)} \leq f_i^*(t) \leq \frac{451P_i}{v_i(t)} \\
 &f_i^*(t) \geq -\xi v_i(t) \\
 &f_i^*(t) \geq -Z
 \end{aligned} \right\} \dots\dots\dots (2-20)$$

It should be noted that some or all of the constraints of (2-30) apply for any particular class of power, depending on the specific values of μ , P_i , ξ and Z . However, in the most general problem, all these constraints may apply, in which case the allowable region for $f_i^*(t)$ has the form represented by the hatched area in Figure 2.5.

Of course, if dynamic braking is not permitted for any particular train, then $f_i^*(t) \geq 0$ must be added as the final constraint.

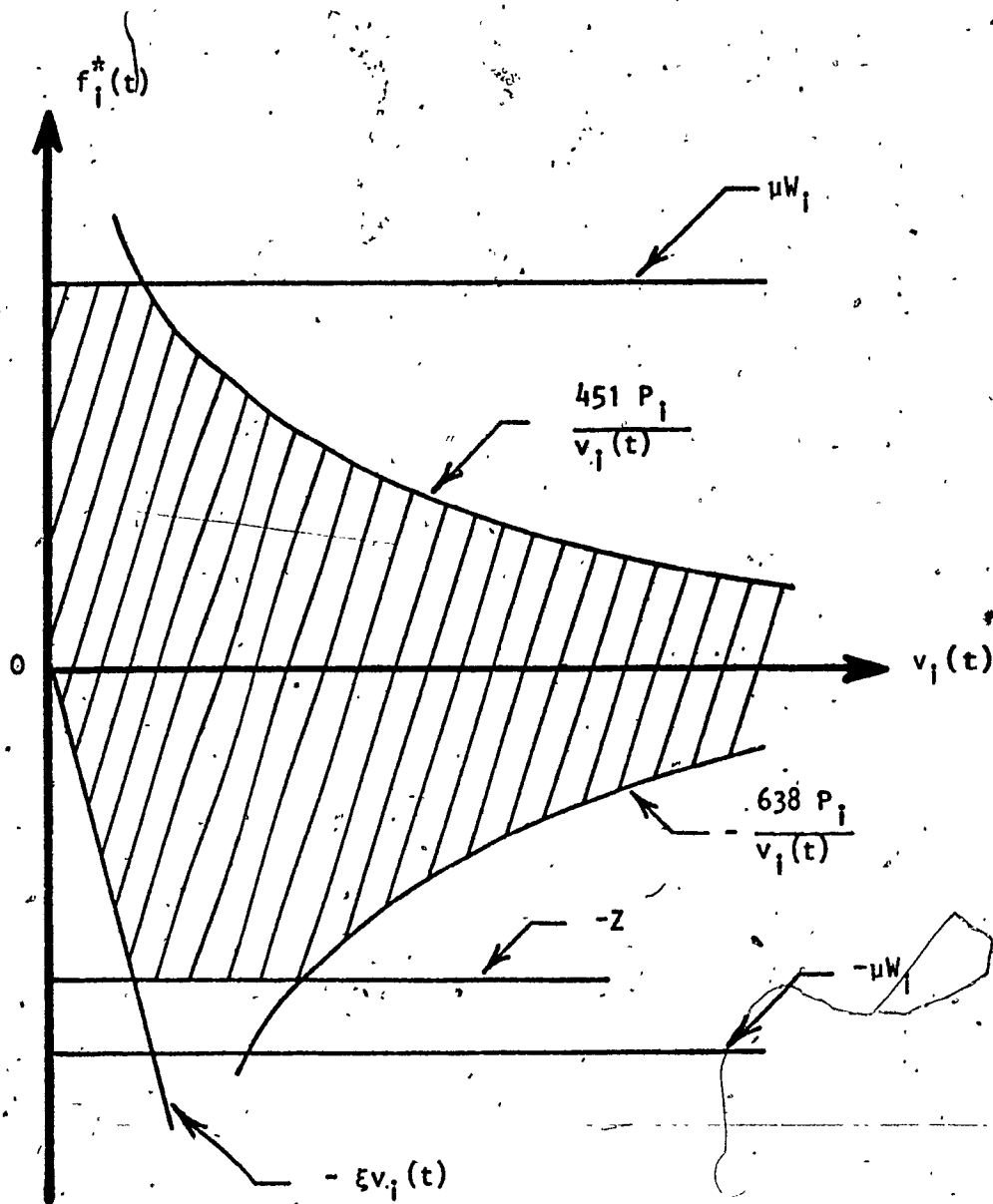


Figure 2.5: Motive power constraints; hatched area is allowable region.

In addition, $f_i^*(t)$ within the allowable region of Figure 2.5 must be quantized, since the engineman's throttle has only eight distinct positions and each corresponds to a specific power output. For instance, McLane and Peppard (25) treat the quantization of locomotive tractive effort, and reference (53) contains an example of this in the case of a locomotive in dynamic braking. However, the approach taken here will be to assume that the allowable region is continuous and then effect the required quantization after the optimal control has been obtained.

2.6 Representation of the Resistance to Motion Function, $y_i(t)$,
- for the i^{th} Member:

The function $y_i(t)$ is taken to represent the total resistance to motion of the i^{th} train member due to grades, curves, friction and air resistance. Let us consider each component of $y_i(t)$ in the following sections.

2.6.1. Grade Resistance:

If the member i is ascending a grade (slope), its mass is being raised vertically. Let $\rho_G\{s_i(t)\}$ be the grade, expressed in per cent, at position $s_i(t)$. Since the grade $\rho_G\{s_i(t)\}$ gives the vertical rise over 100 ft. of track, then

$$\tan \theta = \frac{\rho_G\{s_i(t)\}}{100} \dots\dots\dots(2-21)$$

where θ is the angle of the grade in degrees. Also, for a member of weight W_i , ascending a grade of θ degrees, we have for the grade resistance, R_G , in lbs:



$$R_G = W_i \sin \theta \dots\dots\dots(2-22)$$

However, since θ is quite small, we have $\sin \theta = \tan \theta = 0.01 \rho_G \{s_i(t)\}$ by eq. (2-21). Hence:

$$R_G = 0.01 W_i \cdot \rho_G \{s_i(t)\} = 0.01 g m_i \rho_G \{s_i(t)\} \dots\dots(2-23)$$

where g is the acceleration due to gravity (ft/sec^2).

2.6.2. Curve Resistance:

The effect of varying degrees of curvature on resistance to motion has been determined empirically ⁽⁴⁶⁾, and most simply stated, one degree of curvature offers approximately the same resistance to motion as a 0.04 per cent grade, that is 0.8 lbs/ton/degree of curvature. Hence, the effect of curve resistance is represented in the model by constructing an "effective" grade $d \{s_i(t)\}$ which is the sum of the actual grade $\rho_G \{s_i(t)\}$ and the equivalent grade due to curvature. Let $\rho_C \{s_i(t)\}$ represent the actual curvature (degrees) as a function of position on the track. Then we have:

$$d[s_i(t)] = \rho_G |s_i(t)| + 0.04 \rho_C |s_i(t)| \dots \dots \dots (2-24)$$

As eq. (2-24) now stands, the effective grade is expressed as an explicit function of position. Unfortunately, this is an inconvenient representation, since the functions ρ_G and ρ_C cannot be expressed in closed form and, as we shall see, $s_i(t)$ is one of the state variables of the system. To circumvent this difficulty, we shall arbitrarily assume that the effective grade can be defined as an explicit function of time. Let:

$$d^*(t - \omega_i) = d[s_i(t)] \dots \dots \dots (2-25)$$

where ω_i is the time delay (seconds) which occurs before the grade input to the system appears at the i^{th} member. It will be found convenient to define $d^*(t)$ with respect to some position $s_0(t)$ one member length ahead of the first train member. If z_0 , h_0 and w_0 denote the parameter values of this imaginary "zeroth" member, then we have for $i = 1, 2, \dots, n$:

$$s_i(t) = s_0(t) - \sum_{j=0}^{i-1} (z_j + h_j + w_j) \dots \dots \dots (2-26)$$

and thus for a train travelling at constant speed we have for ω_i :

$$\omega_i = \frac{\sum_{j=0}^{i-1} (z_j + h_j + w_j)}{v_i(t)} \dots\dots\dots (2-27)$$

Since the objective of this whole exercise is to reduce the maximum absolute value of the z_i 's, a reasonable approximation for ω_i is:

$$\omega_i \approx \frac{\sum_{j=0}^{i-1} (h_j + w_j)}{v_i(t)} \dots\dots\dots (2-28)$$

Thus, we see that the specification of $d^*(t)$ and ω_i for a particular train over a specific piece of track will probably require an iterative approach, since some beforehand assumptions regarding the trajectory of $v_i(t)$ must be made. It is for this reason that the approximation sign is used in (2-25).

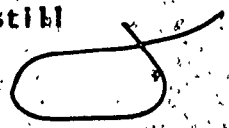
To summarize this and the preceding section, if R_C is the resistance due to curvature, then the sum of R_G and R_C for any member i is taken to be given by:

$$R_G + R_C = 0.01 g m_i d^*(t - \omega_i) \dots\dots\dots (2-29)$$

2.6.3. Friction and Air Resistance:

To quote from reference (61):

"A railway vehicle moving upon level, tangent track, in still



air and at constant speed encounters certain resistances that must be overcome by the tractive effort of the locomotive.

These resistances include:

i) Rolling friction between wheel and rail:

This varies with the surface condition of the rail under load, the horizontal contour of the railhead, and contour and condition of the wheel tread. This can be considered a constant for a given quality of track.

ii) Journal bearing friction:

This varies with the weight of each axle and, at low speed, the type, design and lubrication of the bearing.

iii) Train dynamic losses:

These include flange effects which are associated with lateral motion and the resulting friction and impact of the wheel flanges against the gage side of the rail. They vary with speed, rail alignment, and the tracking effect of the trucks. Also, there are miscellaneous losses due to sway, concussion, buffing and slack action.

iv)

Air resistance:

This varies directly with the cross-sectional area of the vehicle, its length and shape, and the square of its speed. It is also influenced by zones of turbulence related to shape."

Davis (10) gives an empirical formula for computing the total of the four classes of resistance noted above. This formula is a second degree polynomial in $v_i(t)$, the speed of the i^{th} member. However, it is reasonable to use a linearized form of the Davis formula as follows (25):

$$R_{F+A} = gm_i \{ \lambda_{0i} + \lambda_{1i} v_i(t) \} \dots\dots\dots (2-30)$$

where we have:

R_{F+A} : friction and air resistance (lbs) of the i^{th} trafo member.

λ_{0i} : zero-velocity constant (dimensionless) in linearized version of Davis formula for resistance due to friction and air resistance.

λ_{1i} : velocity coefficient (secs/ft) in linearized version of Davis formula.

Hence, the total resistance to motion of the i^{th} train member, $y_i(t)$, is given by:

$$\begin{aligned}
 y_i(t) &= R_G + R_F + A \\
 &= 0.01 gm_i d^*(t-\omega_i) + gm_i \{ \lambda_{0i} + \lambda_i v_i(t) \}, \dots (2-31)
 \end{aligned}$$

2.7. Overall Train Model:

By combining equations (2-1b), (2-3), (2-4), (2-7), (2-8) and (2-31), one derives the following system equations for the i^{th} member of the train:

$$\begin{aligned}
 \dot{v}_i(t) &= \frac{\alpha_{i-1}}{m_i} \cdot z_{i-1}(t) + \frac{\beta_{i-1}}{m_i} \cdot v_{i-1}(t) \\
 &- \frac{\alpha_i}{m_i} \cdot z_i(t) - \left(\frac{\beta_{i-1} + \beta_i}{m_i} + g\lambda_i \right) \cdot v_i(t) \\
 &- \frac{e_i}{m_i} \cdot p_i(t) + \frac{\beta_i}{m_i} \cdot v_{i+1}(t) - g\lambda_{0i} \\
 &- 0.01 gd^*(t-\omega_i) + \frac{1}{m_i} \cdot f_i^*(t) \\
 \dot{p}_i(t) &= -\frac{1}{\sigma_i} \cdot p_i(t) + \frac{n}{\sigma_i} \cdot r^*(t-\tau_i) \\
 z_i(t) &= v_i(t) - v_{i+1}(t)
 \end{aligned} \dots (2-32)$$

Thus, equations (2-32) constitute the mathematical model for train member i ; however, a number of special situations exist which require modification of this system:

I) $i=1$:

In this situation there is obviously no coupler force on the leading draft gear of the first member, and therefore α_{i-1} and β_{i-1} of eqns. (2-32) should be set to zero for $i=1$.

II) Member i is not a powered locomotive

In this case, $f_i^*(t) = 0$ for all t .

III) $i=n$:

In this instance, there is obviously no coupler force on the trailing draft gear of the last member, and therefore α_i and β_i should be set to zero for $i=n$. Also, the last equation of system (2-32) should be deleted since $z_i(t)$ is not defined for $i=n$.

Now, equation (2-1a) with $i=1$, equation (2-11), and equations (2-32) for each train member, together constitute the overall train model.

2.8. State Variable Representation of Train:

Consideration of the system equations for the train, as derived in the preceding sections, immediately suggests a convenient set of physical variables (see Schultz and Melsa (39), for instance) which may be used to define the state vector of the system. Therefore,

let:

$$\begin{bmatrix} s_1(t) \\ v_1(t) \\ p_1(t) \\ z_1(t) \\ \vdots \\ v_i(t) \\ p_i(t) \\ z_i(t) \\ \vdots \\ v_n(t) \\ p_n(t) \\ p_{n+1}(t) \end{bmatrix} = \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ \vdots \\ q_{3i-1} \\ q_{3i} \\ q_{3i+1} \\ \vdots \\ q_{3n-1} \\ q_{3n} \\ q_{3n+1} \end{bmatrix} = q(t) \dots\dots\dots (2-33)$$

where q is the $(3n+1)$ - dimensional state vector of the n member system.

Let us further define:

$f_1^*(t)$		r_1
⋮		⋮
$f_j^*(t)$		r_j
⋮		⋮
$f_i^*(t)$		r_i
$r^*(t-\tau_1)$		r_{i+1}
⋮		⋮
$r^*(t-\tau_1)$	=	r_{i+1}
⋮		⋮
$r^*(t-\tau_n)$		r_{n+1}
$0.01 d^*(t-\omega_1)+\lambda_{01}$		r_{1+n+1}
$0.01 d^*(t-\omega_2)+\lambda_{02}$		r_{2+n+1}
⋮		⋮
$0.01 d^*(t-\omega_i)+\lambda_{0i}$		r_{i+n+1}
⋮		⋮
$0.01 d^*(t-\omega_n)+\lambda_{0n}$		r_{2n+1}

$= r(t) \dots \dots \dots (2-34)$

where r is the $(2n+1)$ - dimensional control vector of the system of n members, 1 of which are powered locomotives. Note that the last n elements of r are treated as control variables, although in fact they are actually pre-defined inputs.

With the state and control vectors defined as in (2-33) and (2-34), the state variable representation of the train is given by:

$$\dot{q} = \Delta q + \Omega r \quad \dots\dots\dots (2-35)$$

where Δ is the $(3n+1) \times (3n+1)$ constant system matrix:

$$\Delta = \begin{bmatrix} \Delta_{11} & \Delta_{12} & 0_{33} & \dots & 0_{33} & 0_{31} \\ 0_{33} & \dots & \Delta_{i,i-1} & \Delta_{i,i} & \Delta_{i,i+1} & 0_{33} & \dots & 0_{33} & 0_{31} \\ \dots & \dots & \dots & \dots & \dots & \dots & \dots & 0_{33} & \Delta_{n,n-1} & \Delta_{n,n} & 0_{31} \\ 0_{13} & \dots & \dots & \dots & \dots & \dots & \dots & 0_{13} & 0_{13} & 0_{13} & \Delta_{n+1,n+1} \end{bmatrix} \quad \dots\dots\dots (2-36)$$

$$\Delta_{i,i-1} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & \frac{\beta_{i-1}}{m_i} & 0 \\ 0 & 0 & 0 \end{bmatrix} \quad i = 2, 3, \dots, n \quad \dots \dots \dots (2-37a)$$

$$\Delta_{i,i} = \left\{ \begin{array}{l} \begin{bmatrix} 0 & 1 & 0 \\ 0 & -\left(\frac{\beta_1}{m_1} + g\lambda_1\right) & -\frac{e_1}{m_1} \\ 0 & 0 & -\frac{1}{\alpha_1} \end{bmatrix} \quad i = 1 \\ \begin{bmatrix} 0 & -1 & 0 \\ \frac{\alpha_{i-1}}{m_i} & -\left(\frac{\beta_{i-1} + \beta_i}{m_i} + g\lambda_i\right) & -\frac{e_i}{m_i} \\ 0 & 0 & -\frac{1}{\alpha_i} \end{bmatrix} \quad i = 2, 3, \dots, n-1 \\ \begin{bmatrix} 0 & -1 & 0 \\ \frac{\alpha_{n-1}}{m_n} & -\left(\frac{\beta_{n-1}}{m_n} + g\lambda_n\right) & -\frac{e_n}{m_n} \\ 0 & 0 & -\frac{1}{\alpha_n} \end{bmatrix} \quad i = n \end{array} \right\} \dots \dots \dots (2-37b)$$

$$\Delta_{i,i+1} = \begin{bmatrix} 0 & 0 & 0 \\ -\frac{\alpha_i}{m_i} & \frac{\beta_i}{m_i} & 0 \\ 0 & 0 & 0 \end{bmatrix} \quad i = 1, 2, \dots, n-1 \quad \dots \dots \dots (2-37c)$$

$$\Delta_{n+1,n+1} = -\frac{1}{\alpha_{n+1}} \dots \dots \dots (2-37d)$$

and 0_{33} , 0_{31} and 0_{13} are null matrices of order 3×3 , 3×1 and 1×3 respectively.

Also, Ω in (2-35) is the $(3n+1) \times (2n+1)$ constant control matrix:

$$\Omega = \{\Omega_1 \dots \Omega_j \dots \Omega_l \Omega_{l+1} \dots \Omega_{l+1} \dots \Omega_{n+1} \Omega_{1+n+1} \dots \Omega_{l+n+1} \dots \Omega_{2n+1}\} \dots (2-38)$$

where Ω_1 through Ω_{2n+1} are defined as follows:

I) Ω_1 through Ω_l :

Each of these $(3n+1)$ - dimensional column vectors is associated with a powered locomotive in the train.

Note that the subscript of Ω_l indicates only that there are l such units in the train, and not necessarily that the l^{th} unit is also the l^{th} member of the train; that is, the l powered locomotives could be dispersed throughout the train rather than grouped together at the head end.

Let the j^{th} locomotive be the j^{th} member of the train. Then the only non-zero element of Ω_j is the $(3l-1)^{th}$, which has a value of $\frac{1}{m_j}$.

II) Ω_{i+1} through Ω_{n+1} :

Each of these n $(3n+1)$ - dimensional column vectors is associated with the braking system for each train member. For the i^{th} member, if $i \neq J$, then the only non-zero element of Ω_{i+1} is the $(3i)^{\text{th}}$, which has a value of $\frac{\eta}{\sigma_i}$. If $i = J$, then Ω_{i+1} has one other non-zero element, the $(3n+1)^{\text{th}}$, which has a value of $\frac{1}{\sigma_{n+1}}$.

III) Ω_{i+n+1} through Ω_{2n+1} :

Each of these n $(3n+1)$ - dimensional column vectors is associated with the total resistance to motion at each train member due to grades, curves, friction and air resistance. For the i^{th} member, then, the only non-zero element of Ω_{i+n+1} is the $(3i-1)^{\text{th}}$, which has a value of $-g$.

In summary, with the definitions (2-33), (2-34), (2-36), (2-37) and (2-38), then the state variable representation of the train is given by system (2-35) along with the constraints indicated by (2-13) and (2-20).

CHAPTER 3

FORMULATION AND A METHOD OF SOLUTION
OF THE OPTIMAL CONTROL PROBLEM

3.1. The Optimal Control Problem:

Referring to the problem description given in section 1.1, we recall that our objective is to devise a control procedure which minimizes the maximum coupler force experienced anywhere in the train during its operation over a given section of track. This objective will be realized if we can minimize the maximum draft gear extension or compression, since coupler forces are largely proportional to this variable.

$$\text{If } \max_{t_0 \leq t \leq t_f} \left[|z_1(t)|, |z_2(t)|, \dots, |z_1(t)|, \dots, |z_{n-1}(t)| \right]$$

represents the maximum draft gear extension or compression experienced anywhere in the train between the start and end of operation at times t_0 and t_f respectively, then the optimal control problem may be stated precisely as follows:

"Given the linear system (2-35), it is desired to find an admissible control vector $r(t)$ which transfers the system from its initial state $q(t_0)$ to some desired final state $q(t_f)$ such that the performance measure $\text{MAX}_{t_0 < t < t_f} [|z_1(t)|, |z_2(t)|, \dots, |z_{n-1}(t)|]$ is minimized."(3-1)

3.2. Possible Approaches to the Problem:

The specific problem (3-1) has the following characteristics:

- i) The state vector of system (2-35) is of very high dimension; for example, the representation of a 100 member train would require 301 state variables.
- ii) There are time delays in the control vector of system (2-35), as is evident in the definition of r given by (2-34). As a matter of fact, the usual description of such a system would be:

$$\dot{q}(t) = \Delta q(t) + \sum_{j=0}^n \{ \Omega^j r^j(t-\tau_j) + \Omega^{j+1} r^{j+1}(t-\omega_j) \}$$

where Ω^j , Ω^{j+1} , r^j and r^{j+1} are control matrices and vectors of appropriate dimensions. Olbrot (28) illustrates a procedure for determining

the absolute controllability of linear systems with time delays in control, and Sebakhly and Bayoumi (41) give a simplified criterion for the controllability of such systems. However, for the present, it will be convenient to retain the system description given by (2-35) and in particular the control vector definition (2-34).

- (iii) The performance measure selected is not of the standard integral form.
- (iv) The control vector r is rather severely constrained. In particular, the first l control variables are constrained as in (2-20), the next n as in (2-13), and the final n control variables are actually pre-determined inputs or forcing functions whose value over time is fixed a priori and not subject to manipulation by the optimization process.
- (v) We do not seek the optimal control law $r\{q(t)\}$; rather, we seek the optimal control function $r(t)$. That is, it would be sufficient for our purposes to determine the optimal open-loop control, rather than the optimal feedback control.

This is consistent with the discussion in section 1.3 regarding the relationship of this work to that of McLane and Peppard (25).

Consideration of these problem characteristics in relation to the available optimization techniques (Pierre (31) provides an excellent catalogue, for example) led the author to the opinion that the only hope for a practical solution method would be to apply some suitable mathematical programming technique to a discrete representation of problem (3-1). For instance, the two point boundary value problem which results from the application of variational techniques is intractable for such large dimensional systems. Similarly, the application of dynamic programming, particularly the state increment form of Larson (22), would have been a very elegant approach, had not Bellman's "curse of dimensionality" made it impractical.

Canon et al (7), Bantzig (9) and Polak (33) provide rigorous treatments of the applicability of mathematical programming (MP) to optimal control problems, and Torng (47), Zadeh and Whalen (52) give specific examples of linear programming (LP) applications in particular.

Wheeler (49) demonstrates a fairly straightforward procedure for transforming any linear optimal control problem into either a linear, quadratic, or convex programming problem. Unfortunately, the method retains the state variables in the MP formulation, so

that for other than very low-order systems the matrices involved become extremely large. As an example, application of Wheeler's transformation of the discrete problem analogous to (3-1) for a 100 member train over 100 discrete time intervals would result in an LP matrix containing in the order of two billion elements.

In a procedure outlined by Enns (12,13) and Lack (21), a transformation from state space to control space permits an LP formulation with the crucial property that the dimension of the LP tableau is not directly dependent on the number of state variables. This solution method, to be described in section 3.4, will be applied to a discretized form of problem (3-1).

3.3. Discrete Representation of the Problem:

In order to apply an LP technique, the set of control trajectory values to be optimized must obviously be finite. A discrete form of problem (3-1) must therefore be derived.

3.3.1. System Equations

Disregarding for the present the time delays present in some elements of the control vector defined in (2-34), let us consider the system (2-35). This representation implies that the control

vector is a continuously varying function of time, whereas, in the practical operation of the actual train, this will not be the case. Rather, it is much more realistic to assume that any particular control vector value will be held constant over some minimum time period. Hence, the control variables are actually piecewise-constant signals, and this fact is emphasized by writing:

$$r(t) = r(kT), \quad \left\{ \begin{array}{l} kT < t < (k+1)T \\ k = 0, 1, \dots, K-1 \end{array} \right\} \dots\dots (3-2)$$

where $T = t_f/K$ is the minimum time interval over which the control signals are assumed to remain constant, and K is an appropriately specified integer parameter.

It should be stressed that $r(kT)$ is not merely an approximation of $r(t)$; rather, the representation (3-2) is an exact equivalence so long as K is appropriately selected.

With $r(t)$ defined as in (3-2) we no longer require the continuous time response, and in fact need only obtain the values of the state vector at the end of each time interval. To this end, let us first consider the continuous solution of the state equations (2-35), as given in any standard reference (see, for instance, Schwarz and Friedland (40)):

$$q(t) = \phi(t-t_0)q(t_0) + \int_{t_0}^t \phi(t-t')\Omega r(t') dt' \dots\dots (3-3)$$

where t' is a dummy variable and $\phi(t) = e^{\Delta t}$ is the state transition matrix of the system (2-35).

Now, letting $t_0 = kT$ and $t = (k+1)T$ in (3-3), we have:

$$q\{(k+1)T\} = \phi(T)q(kT) + \int_{kT}^{(k+1)T} \phi\{(k+1)T-t'\} \Omega r(t') dt' \dots (3-4)$$

With r as defined in (3-2):

$$q\{(k+1)T\} = \phi(T)q(kT) + \left[\int_{kT}^{(k+1)T} \phi\{(k+1)T-t'\} \Omega dt' \right] r(kT) \dots (3-5)$$

The integral term in (3-5) has the same value for all k , and

therefore:

$$q\{(k+1)T\} = \phi(T)q(kT) + \left[\int_0^T \phi(T-t') \Omega dt' \right] r(kT) \dots (3-6)$$

For notational convenience, let us omit the argument T and

define:

$$\Delta^* = \phi(T) \dots (3-7)$$

$$\Omega^* = \int_0^T \phi(T-t') \Omega dt' \dots (3-8)$$

Then we have the set of $(3n+1)$ difference equations:

$$q(k+1) = \Delta^* q(k) + \Omega^* r(k), \quad k = 0, 1, \dots, K-1 \dots (3-9)$$

Enns (12) gives the following equivalent definitions for

Δ^* and Ω^* :

$$\Delta^* = \sum_{n=0}^{\infty} \frac{1}{n!} (\Delta T)^n \dots\dots\dots (3-10)$$

$$\Omega^* = T \left[\sum_{n=0}^{\infty} \frac{1}{(n+1)!} (\Delta T)^n \right] \cdot \Omega \dots\dots\dots (3-11)$$

Equations (3-10) and (3-11) are useful for calculating Δ^* and Ω^* for small values of T, and Johnson and Phillips (18) have derived a practical method of evaluating Ω^* for large T.

The algebraic recursion relation (3-9) is thus an exact discretized representation of system (2-35) for a suitably chosen value of K.

3.3.2. Time Delays in Control:

We may now turn our attention to the time delays in the control vector r. Recalling definitions (2-34) and (3-2), we may write:

$$r_{i+1}(t) = r^*(t - \tau_i) = r^*(k - \bar{\tau}_i) \dots\dots\dots (3-12)$$

$$\begin{aligned} r_{i+n+1}(t) &= 0.01d^*(t - \omega_i) + \lambda_{0i} \\ &= 0.01d^*(k - \bar{\omega}_i) + \lambda_0 \dots\dots\dots (3-13) \end{aligned}$$

for $i = 1, 2, \dots, n$, and where K has been specified so that $\bar{\tau}_i = \tau_i/T$ and $\bar{\omega}_i = \omega_i/T$ are integers. Also, $\lambda_0 = \lambda_{0i}$ has been

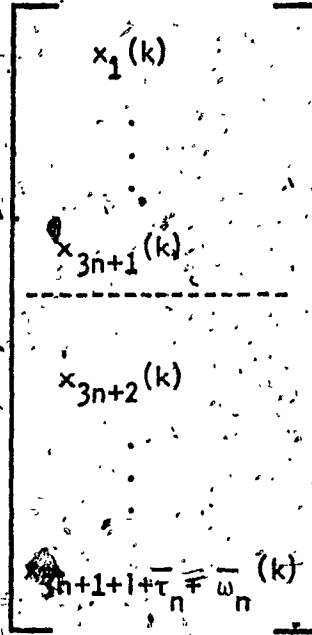
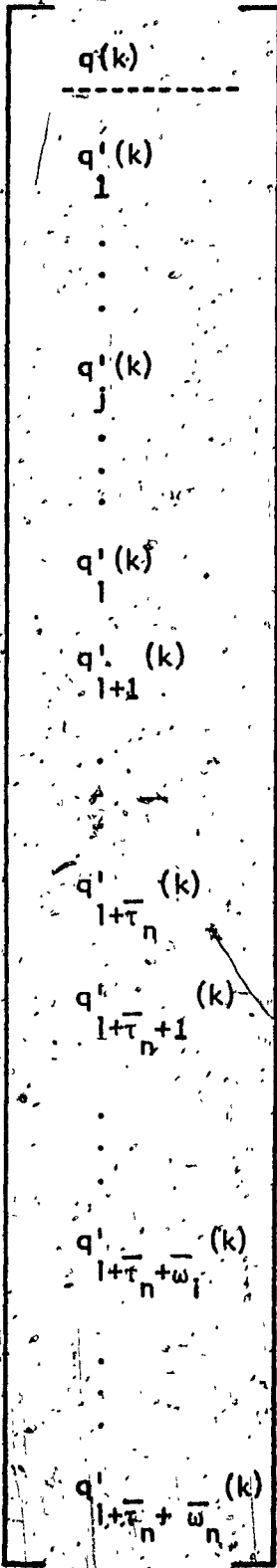
assumed constant for all members of the train.

Therefore r_{i+1} is merely the control variable $r^*(k)$ delayed in time by $\bar{\tau}_i$ intervals. Similarly, r_{i+n+1} is the control variable $\{0.01d^*(k)+\lambda_0\}$ delayed in time by $\bar{\omega}_i$ intervals. Since (3-9) is a set of difference equations, we may take advantage of relationships (3-12) and (3-13) to reduce the dimension of the control vector by incorporating the system time delays in the definitions of the system matrix and the state vector (see Noton (26) for example).

To this end, let us define the $(1+2)$ - dimensional control vector $u(k)$ as follows:

$$\begin{bmatrix} f_1^*(k) \\ \vdots \\ f_j^*(k) \\ \vdots \\ f_1^*(k) \\ r^*(k) \\ 0.01d^*(k)+\lambda_0 \end{bmatrix} = \begin{bmatrix} u_1(k) \\ \vdots \\ u_j(k) \\ \vdots \\ u_1(k) \\ u_{1+1}(k) \\ u_{1+2}(k) \end{bmatrix} = u(k) \dots\dots (3-14)$$

Let us further define the $(3n+1+\bar{\tau}_n+\bar{\omega}_n)$ - dimensional state vector $x(k)$ by augmenting the vector q of definition (2-33) as follows:



$x(k) \dots (3-15)$

where $q'(k)$ is the vector appended to $q(k)$ and is defined by the following relationships:

$$\begin{aligned}
 q'_1(k+1) &= f_1^*(k) = u_1(k) \\
 q'_j(k+1) &= f_j^*(k) = u_j(k) \\
 q'_l(k+1) &= f_l^*(k) = u_l(k) \\
 q'_{l+1}(k+1) &= q'_{l+2}(k) \\
 q'_{l+\tau_i}(k+1) &= q'_{l+\tau_i+1}(k) \\
 q'_{l+\tau_n-1}(k+1) &= q'_{l+\tau_n}(k) \\
 q'_{l+\tau_n}(k+1) &= r^*(k) = u_{l+1}(k) \\
 q'_{l+\tau_n+1}(k+1) &= q'_{l+\tau_n+2}(k) \\
 q'_{l+\tau_n+\omega_l}(k+1) &= q'_{l+\tau_n+\omega_l+1}(k) \\
 q'_{l+\tau_n+\omega_l-1}(k+1) &= q'_{l+\tau_n+\omega_l}(k) \\
 q'_{l+\tau_n+\omega_l}(k+1) &= 0.01d^*(k) + \lambda_0 = u_{l+2}(k)
 \end{aligned}$$

.....(3-16)

Note that a time delay of T has been introduced in the control variable $u_j(k) = f_j^*(k)$, $j = 1, 2, \dots, l$. This will facilitate the expression of constraints (2.20), since $v_i(k)$ and $f_j^*(k-1)$ are now both in state space and, as we shall see, linear state variable constraints can readily be accommodated in the LP formulation. This delay may either be regarded as representative of the reaction time of the engineman and the diesel locomotive, or may be discounted entirely by treating $f_j^*(k-1)$ as the actual system input when the optimal control trajectory has been found.

With $x(k)$ and $u(k)$ defined as in (3-15) and (3-14), the relationships (3-16) may be included in the state variable description of the system (3-9) by writing:

$$x(k+1) = Ax(k) + Bu(k), \quad k = 0, 1, \dots, K-1 \quad \dots\dots\dots (3-17)$$

where the $(3n+1+l+\tau_n+\bar{w}_n) \times (3n+1+l+\tau_n+\bar{w}_n)$ constant system matrix A is given by:

$$A = \begin{bmatrix} A_{11} & | & A_{12} \\ \hline A_{21} & | & A_{22} \end{bmatrix} \quad \dots\dots\dots (3-18)$$

and the various sub-matrices of A are defined as follows:

i) A_{11} is the $(3n+1) \times (3n+1)$ matrix:

$$A_{11} = \Delta^* \dots \dots \dots (3-19)$$

ii) A_{12} is the $(3n+1) \times (1+\bar{\tau}_n + \bar{\omega}_n)$ matrix:

$$A_{12} = \begin{bmatrix} \Omega_1^* & \dots & \Omega_j^* & \dots & \Omega_1^* & \Omega_{n+1}^* & \dots & \underline{0} & \dots & \dots \\ \Omega_{i+1}^* & \dots & \underline{0} & \dots & \Omega_{1+1}^* & \dots & \underline{0} & \dots & \Omega_{2n+1}^* & \dots \\ \underline{0} & \dots & \Omega_{1+n+1}^* & \dots & \underline{0} & \dots & \Omega_{1+n+1}^* & \dots & \dots & \dots \end{bmatrix} \dots \dots \dots (3-20)$$

In which Ω_1^* through Ω_{2n+1}^* are the column vectors of the matrix Ω^* .

Note particularly the inverted order of Ω_{1+1}^* through Ω_{n+1}^* , and of Ω_{1+n+1}^* through Ω_{2n+1}^* . Note also that if $\bar{\tau}_n > n$ there are

$(\bar{\tau}_n - n)$ $(3n+1)$ - dimensional null vectors $\underline{0}$ dispersed through the sequence Ω_{1+1}^* through Ω_{n+1}^* . Similarly, if $\bar{\omega}_n > n$ there are

$(\bar{\omega}_n - n)$ $\underline{0}$ vectors dispersed through the sequence Ω_{1+n+1} through

Ω_{2n+1}^* . There are no $\underline{0}$ vectors for $(\bar{\tau}_n + \bar{\omega}_n) \leq 2n$.

iii) A_{21} is an $(1+\bar{\tau}_n + \bar{\omega}_n) \times (3n+1)$ null matrix

iv) $A_{22} = [a_{ij}]$ is the $(1+\bar{\tau}_n + \bar{\omega}_n) \times (1+\bar{\tau}_n + \bar{\omega}_n)$ matrix:

$$a_{ij} = \left\{ \begin{array}{l} 1, \quad i=(l+1), \dots, (l+\bar{\tau}_n-1); j=(l+2), \dots, (l+\bar{\tau}_n) \\ 1, \quad i=(l+\bar{\tau}_n+1), \dots, (l+\bar{\tau}_n+\bar{\omega}_n-1); j=(l+\bar{\tau}_n+2), \dots, (l+\bar{\tau}_n+\bar{\omega}_n) \\ 0, \quad \text{otherwise} \end{array} \right\} \dots\dots\dots (3-21)$$

and where the $(3n+1+l+\bar{\tau}_n+\bar{\omega}_n) \times (l+2)$ constant control matrix

$B = [b_{ij}]$ is defined by:

$$b_{ij} = \left\{ \begin{array}{l} 1, \quad i=(3n+2), \dots, (3n+1+l); j=1, \dots, l \\ 1, \quad i=(3n+1+l+\bar{\tau}_n); j=(l+1) \\ 1, \quad i=(3n+1+l+\bar{\tau}_n+\bar{\omega}_n); j=(l+2) \\ 0, \quad \text{otherwise} \end{array} \right\} \dots\dots\dots (3-22)$$

Thus, (3-17) constitutes a discretization of system (2-35)

in which the time delays in control are represented by additional state variables.

3.3.3. System Constraints:

I) The constraints (2-20) on $u_j(k) = f_j^*(k), j=1, \dots, l,$

cannot be expressed in this form since only linear constraints can be accommodated by an LP technique. Instead, the following approximations will be considered as the constraints on the tractive or braking effort:

$$\left. \begin{aligned}
 &|x_{3n+1+j}(k)| \leq \mu W_j \\
 &x_{3n+1+j}(k) + L'x_2(k) \leq L'' \\
 &-x_{3n+1+j}(k) + M'x_2(k) \leq M'' \\
 &x_{3n+1+j}(k) + \epsilon x_2(k) \geq 0 \\
 &x_{3n+1+j}(k) \geq -Z
 \end{aligned} \right\} \begin{aligned}
 & \\
 & \\
 &j=1, \dots, l \\
 & \dots \dots \dots (3-23)
 \end{aligned}$$

where L' , L'' , M' and M'' are positive constants chosen so as to best approximate the constraints (2-20) over the speed range of interest, and where $x_{3n+1+j}(k) = u_j(k-1) = f_j^*(k-1)$. Since usually only a specific speed range is desired, not all of the constraints

(3-23) may need to be applied. For example, if the speed of the train (as defined by $x_2(k) = v_1(k)$) is restricted to between V' and V'' in Figure 3.1, then only the indicated subset of (3.23) is required.

If a single train speed is specified rather than a range as in Figure 3.1, then obviously only absolute bounds on $u_j(k)$, $j=1, \dots, l$, need be applied.

II) The constraints (2-13) on $u_{l+1}(k) = r^*(k)$ are difficult to express simultaneously. Instead, the problem will first be run with the constraint:

$$0 \leq u_{l+1}(k) \leq 23, k=0, 1, \dots, K-1 \quad \dots \dots \dots (3-24)$$

Then by examining the optimal trajectory of $u(k)$ and $x(k)$, one or the other of the following constraints will be applied for values of k specified on the basis of the trajectory of $x_{3n+1}(k)$:

$$\left\{ \begin{array}{l} \text{either } u_{l+1}(k) = 0 \\ \text{or } u_{l+1}(k-1) \leq u_{l+1}(k) \leq 23 \end{array} \right\} \quad (k \text{ as specified}) \dots (3-25)$$

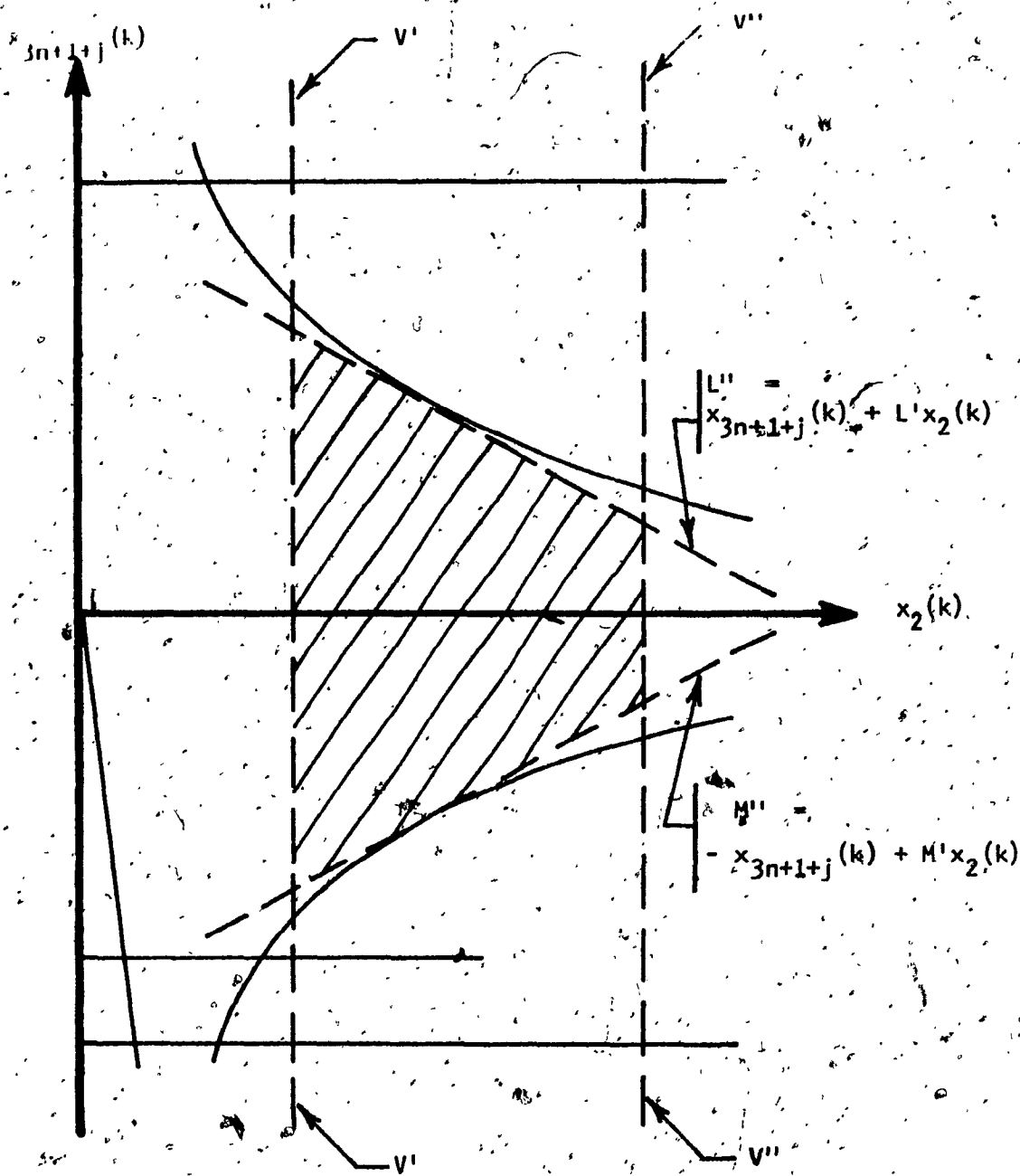


Figure 3.1: Approximation of constraints (2-20) by linear constraints (3-23). Hatched area is allowable region for the indicated V' and V'' . Solid lines are constraints (2-20) from Figure 2.5.

With constraints (3-25) specified for appropriate values of k, a new/optimal trajectory should then be calculated.

(III) Train speed will usually be restricted by one of the following constraints:

$$\left[\begin{array}{l} \text{either } x_2(k) = V' \\ \text{or } V' \leq x_2(k) \leq V'' \end{array} \right] \quad (k \text{ as specified}) \dots\dots\dots(3-26)$$

(IV) As indicated in section 3.2(iv), the control variable $u_{1+2}(k)$ is actually fixed a priori by the specification of the effective grade $d^*(k)$ as a function of k.

$$u_{1+2}(k) = 0.01d^*(k) + \lambda_0 \quad \dots\dots\dots(3-27)$$

If the first of the constraints (3-26) is specified, then $x_1(k) = s_1(k)$ is known and hence $d^*(k)$ as well as $u_{1+2}(k)$ may be defined exactly. However, if the second of the constraints (3-26) is applied to the system then the following iterative approach is required:

- i) guess at the probable trajectory of $x_1(k)$.
- ii) based on (i), specify $d^*(k)$ and hence $u_{1+2}(k)$.

(iii) calculate the optimal trajectories,

(iv) examine $d^*(k)$ and $x_1(k)$ from step (iii) and re-specify $d^*(k)$ accordingly.

(v) repeat steps (iii) and (iv) until a satisfactory correlation of $x_1(k)$ and $d^*(k)$ is obtained.

3.3.4. Restatement of the Problem

"Given the linear discrete-time system (3-17),

it is desired to find a control vector $u(k)$ which transfers the system from its initial

state $x(0)$ to some desired final state $x(K)$

such that the specified constraints (3-23)

through (3-27) are met, and such that the

performance measure $\text{MAX}_{0 \leq k \leq K} \left\{ |x_4(k)|, |x_7(k)|, \dots, |x_{3i+1}(k)|, \dots, |x_{3n-2}(k)| \right\}$ is minimized.

..... (3-28)

3.4. A Method of Solution

Lack (21) and Enns (12, 13) give a straightforward procedure for calculating the optimal control trajectory for problems such as (3-28) by linear programming. The reader is referred to reference (3.5) in particular for an excellent detailed description

of the method, but a brief summary of the procedure in terms of problem (3-28) is presented here for convenience.

3.4.1. Basic Solution Method:

Consider the system:

$$x(k+1) = Ax(k) + Bu(k), \quad k = 0, 1, \dots, K-1 \quad \dots \dots \dots (3-17)$$

where $x(k)$ is the $(3n+1+l+\bar{\tau}_n+\bar{\omega}_n)$ - dimensional state vector, $u(k)$ is the $(l+2)$ - dimensional control vector, and A and B are the constant matrices of the system as defined by (3-18) and (3-22) respectively.

The initial condition:

$$x(0) = x_0 \quad \dots \dots \dots (3-29)$$

and the terminal condition:

$$Fx(K) = f \quad \dots \dots \dots (3-30)$$

must be satisfied, where f is an \bar{m} - dimensional vector and F is an $\bar{m} \times (3n+1+l+\bar{\tau}_n+\bar{\omega}_n)$ matrix.

The optimality criterion is specified by defining a region E^+ in state space and requiring the state trajectory to stay as far from this region as possible. That is, we wish to find the control which maximizes the distance of closest approach of the trajectory $x(k)$ to the region E^+ while meeting conditions (3-29) and (3-30). The region E^+ is the half-space in E^n defined by:

$$E^+ = \{ x: \langle d, x \rangle > \langle d, d \rangle \} \dots\dots\dots (3-31)$$

where d is a vector normal to the separating hyperplane, as illustrated for the two dimensional case in Figure 3.2. (The half-space E^- is defined analogously). The notation $\langle d, x \rangle$ indicates the inner (scalar) product of the vectors d and x ; geometrically, $\langle d, x \rangle$ may be considered to be the "projection" of the vector x on the direction vector d .

For the specific problem (3-28), we ideally require $2(n-1)$ such d vectors to be defined, each specifying the "danger region" for the state variable x_{3i+1} , $i = 1, 2, \dots, n-1$ in turn. This point will be discussed further in chapter 5.

Enns shows that the maximization of the minimum distance from the trajectory $x(k)$ to the region E^+ is equivalent to the minimax problem:

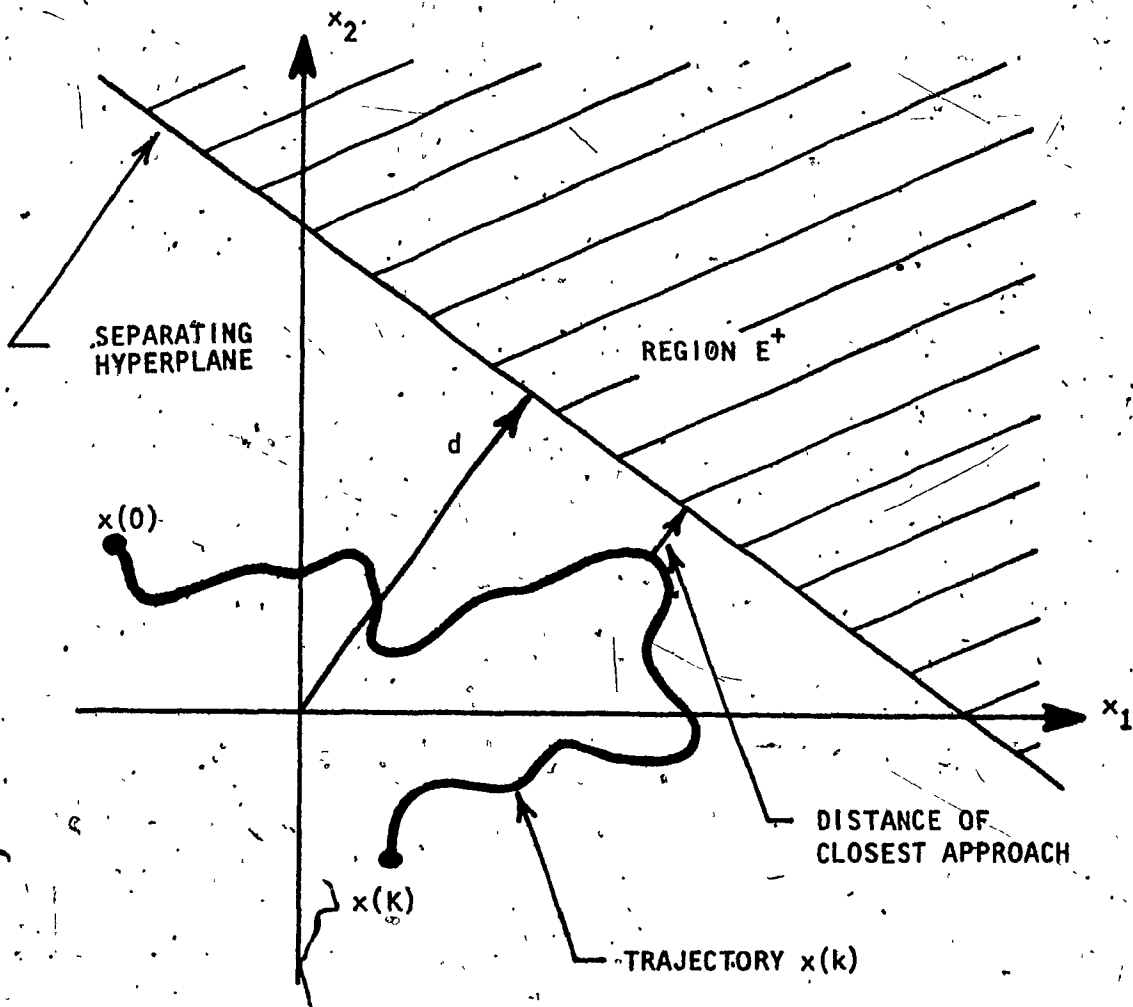


Figure 3.2: Illustration of optimality criterion for state space of dimension 2.

$$\left\{ \begin{array}{l} \text{Find } u(k) \text{ so as to:} \\ \max_{u(k)} \min_k \left[\frac{\langle d, d-x(k) \rangle}{\langle d, d \rangle} \right] \end{array} \right\} \dots\dots\dots(3-32)$$

The key to transforming problem (3-32) to an LP problem is to introduce a dummy variable π (a scalar), and the set of constraints:

$$\langle d, x(k) \rangle \leq \pi, \quad k = 0, 1, \dots, K \quad \dots\dots\dots(3-33)$$

Then (3-32) is equivalent to the constrained minimization problem:

$$\left\{ \begin{array}{l} \min_{u(k)} \pi, \text{ subject to (3-33)} \end{array} \right\} \dots\dots\dots(3-34)$$

Accomplishing (3-34) amounts to "squashing" the state trajectory with a hyperplane normal to the d vector and parallel to the boundary of the region E^+ .

To put the problem (3-34) into an LP form, define the $\{(1+2) \cdot K+1\}$ - dimensional vector u and the constant $\{(1+2) \cdot K+1\}$ - dimensional vector c by:

$$u = \begin{bmatrix} u(0) \\ u(1) \\ \vdots \\ u(k-1) \\ \vdots \end{bmatrix} \dots\dots\dots (3-35)$$

$$c = \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} \dots\dots\dots (3-36)$$

where it is understood that $u(k)$ is defined by (3-14)

Now, the crucial operation in this procedure is the transformation from state space to control space accomplished by writing the state vector x at any time k as a linear function of u by repeated use of (3-17) with (3-29):

$$\begin{aligned}
 x(1) &= Ax(0) + Bu(0) = Ax_0 + Bu(0) \\
 x(2) &= Ax(1) + Bu(1) = A^2x_0 + ABu(0) + Bu(1) \\
 x(k) &= Ax(k-1) + Bu(k-1) \\
 &= A^kx_0 + A^{k-1}Bu(0) + A^{k-2}Bu(1) + \dots + ABu(k-2) + Bu(k-1)
 \end{aligned}$$

or:

$$x(k) = A^kx_0 + \sum_{j=0}^{k-1} A^{k-1-j} Bu(j) \dots\dots\dots (3-37)$$

By defining the $(3n+1+\bar{r}_n + \bar{m}_n) \times (1+2)K$ matrix $D(k)$ as:

$$D(k) = \begin{bmatrix} A^{k-1}B & A^{k-2}B & \dots & AB & B & 0 & \dots & 0 \end{bmatrix} \dots\dots\dots (3-38)$$

then (3-37) may be written:

$$x(k) = A^k x_0 + \begin{bmatrix} D(k) \\ 0 \end{bmatrix} \cdot u \dots\dots\dots (3-39)$$

and hence the condition (3-30) as:

$$\begin{bmatrix} F & D(k) \\ 0 \end{bmatrix} \cdot u = f - FA^k x_0 \dots\dots\dots (3-40)$$

and the constraints (3-33) as:

$$\begin{bmatrix} \langle d, D(k) \rangle \\ -1 \end{bmatrix} \cdot u \leq - \langle d, A^k x_0 \rangle, \quad k = 0, 1, \dots, K \dots\dots (3-41)$$

where $\langle d, D(k) \rangle$ indicates the inner product of d with each of the columns of $D(k)$, the result being a row vector.

Thus the control problem (3-32) is shown by Enns to be equivalent to the problem:

$$\left\{ \begin{array}{l} \text{Find } u \text{ to } \min \langle c, u \rangle \\ \text{subject to } (3-40) \text{ and } (3-41) \end{array} \right\} \dots\dots\dots (3-42)$$

This is the standard form of linear programming and the optimal control may be found directly by the simplex algorithm (see Künzi (20); for instance).

A distinct advantage of this method is that the problem (3-42) may be readily extended to include the linear constraints associated with problem (3-28) directly as additional LP constraints.

3.4.2 Extension of the Problem to Include State Variable Constraints:

Engs shows that any linear equality or inequality relation for the state variables may be included as additional LP constraints by means of (3-37) and (3-38). In general, the constraint:

$$\langle d, x(k) \rangle \begin{bmatrix} \leq \\ = \\ \geq \end{bmatrix} b, \quad k = 0, 1, \dots, K \quad \dots \dots \dots (3-43)$$

may be formulated as the set of (K+1) scalar LP constraints:

$$\left. \begin{aligned} & 0 \begin{bmatrix} \leq \\ = \\ \geq \end{bmatrix} b - \langle d, x_0 \rangle \\ \left[\langle d, D(1) \rangle \mid 0 \right] u \begin{bmatrix} \leq \\ = \\ \geq \end{bmatrix} b - \langle d, Ax_0 \rangle \\ \left[\langle d, D(k) \rangle \mid 0 \right] u \begin{bmatrix} \leq \\ = \\ \geq \end{bmatrix} b - \langle d, A^k x_0 \rangle \\ \left[\langle d, D(K) \rangle \mid 0 \right] u \begin{bmatrix} \leq \\ = \\ \geq \end{bmatrix} b - \langle d, A^K x_0 \rangle \end{aligned} \right\} \dots \dots \dots (3-44)$$

Note that these constraints may be time-varying; that is $b = b(k)$ in general.

3.4.3. Extension of the Problem to Include Control Variable.

Constraints:

Again, Enns shows that any linear equality or inequality constraint on the controls may be included as additional LP constraints. Although constraints on the control variables of the type (3-43) may be similarly expressed, the more usual situation is that the constraints are simple bounds on the absolute value of a particular control variable. This type of constraint:

$$|u_i(k)| \leq H(k) \dots\dots\dots (3-45)$$

is equivalent to the two linear constraints:

$$\left. \begin{aligned} u_i(k) &\leq H(k) \\ -u_i(k) &\leq H(k) \end{aligned} \right\} \dots\dots\dots (3-46)$$

and hence may be readily included in problem (3-42) as additional LP constraints. The even simpler constraint :

$$u_i(k) = H(k) \dots\dots\dots (3-47)$$

may obviously be incorporated in the LP formulation as well. Constraint (3-47) is the means by which a particular control variable is treated as a fixed input. The notation $H(k)$ has been used in (3-45) to (3-47) to emphasize the fact that the inequality or equality constraint on the control variable may be time varying.

3.5. Linear Programming Problem Equivalent to the Control Problem:

Providing that a suitable set of direction vectors d can be defined, we have thus transformed the original control problem (3-1) to the linear programming problem:

"Find u to $\min_u \langle c, u \rangle$ subject to constraints (3-40), and (3-41), in addition to constraints (3-44), (3-46) and (3-47) as required."(3-48)

A remarkable feature of this method is that the dimension of the state vector of problem (3-28), the discrete equivalent of (3-1), does not enter into the dimension of the equivalent LP problem (3-48). In particular, if the:

- i) number of control variables = (1+2)

- ii) number of scalar control constraints = \bar{u}
- iii) number of state trajectory constraints or minimax objectives = \bar{x}
- iv) number of state variable terminal constraints = \bar{m}
- v) number of discrete time intervals = K

then, as illustrated by Enns (12), the dimensions of the corresponding LP matrix are as follows:

$$\left\{ \begin{array}{l} \text{Number of rows} = \{(\bar{x} + \bar{u})K + \bar{x} + \bar{m}\} \\ \text{Number of columns} = \{(1+2)K + 1\} \end{array} \right\} \dots\dots\dots (3-49)$$

Thus the limiting factor is now the total number of constraints and discrete time intervals rather than the dimension of state space. This characteristic will be elaborated in Chapter 5, but let us first illustrate the technique with a small example.

CHAPTER 4

EXAMPLE: A THREE-MEMBER TRAIN

To illustrate the solution technique, consider the simple case of a train consisting of just three "members." However, in order that reasonably large coupler forces will be realized, we shall assume that the leading train member actually comprises two 2,000-hp locomotives and ten freight cars, and that the other two members each comprise twelve freight cars. The gross weight of each freight car and locomotive is taken to be 80 tons and 190 tons respectively, and the overall length of each vehicle is assumed to be 44 feet. The velocity at some given initial position is taken to be 45 mph, and we require the train to travel an additional mile in 80 seconds. It should be noted that although we are requiring the average speed to be equal to the initial speed, we are not specifying what the velocity trajectory should be over the run time interval, except that a speed limit of 60 mph is assumed to apply over the given territory.

The parameter values of this 4000 hp, 3100 ton train are given in Table 4.1.

$$m_1 = (2 \times 190) + (10 \times 80) = 1180 \text{ tons} = 73,292 \text{ slugs}$$

$$m_2 = m_3 = (12 \times 80) = 960 \text{ tons} = 59,627 \text{ slugs}$$

$$\alpha_1 = \alpha_2 = 500,000 \text{ lbs/ft.}$$

$$\beta_1 = \beta_2 = 15,000 \text{ lbs-sec/ft.}$$

$$e_1 = e_2 = e_3 = 1440 \text{ in.}^2$$

$$\lambda_0 = 0$$

$$\lambda_1 = 0.000054 \text{ secs/ft.}$$

$$\lambda_2 = \lambda_3 = 0.000045 \text{ secs/ft.}$$

$$\sigma_1 = \sigma_2 = \sigma_3 = 15 \text{ secs.}$$

$$\sigma_4 = 60 \text{ secs.}$$

$$\tau_1 = 4 \text{ secs.}$$

$$\tau_2 = 8 \text{ secs.}$$

$$\tau_3 = 12 \text{ secs.}$$

$$\omega_1 = 8 \text{ secs}$$

$$\omega_2 = 16 \text{ secs.}$$

$$\omega_3 = 24 \text{ secs.}$$

$$\eta = 2.5$$

Table 4.1: Train Parameter Values

The w_i 's of Table 4.1 are calculated from (2-28), where
 $h_i = w_i = 12 \times 44 = 528$ ft., for $i = 1, 2, 3$.

Let the effective grade under consideration be such that its maximum frequency of change in value is once per 350 feet. At the given speed limit of 88 ft/sec., this implies that the effective grade input to each member will be constant for at least four seconds. Since it is reasonable to require the other inputs to the system (i.e., air brakes and throttle/dynamic brake applications) to be held constant for at least this maximum time period as well, then it is concluded that a discrete time interval of four seconds, would be a satisfactory assumption in this case. We therefore require the number of time intervals to be $K^*t_f/T = 80/4 = 20$.

Let the effective grade as a function of discrete time interval k be specified as in Figure 4.1.

Note from Table 4.1 that λ_0 has been assumed to be zero. Thus the control variable $u_3(k)$, as defined by (3-27), is given simply as $0.01d^*(k)$. This assumption for λ_0 was made merely to simplify the example; any other value would just serve to bias this function.

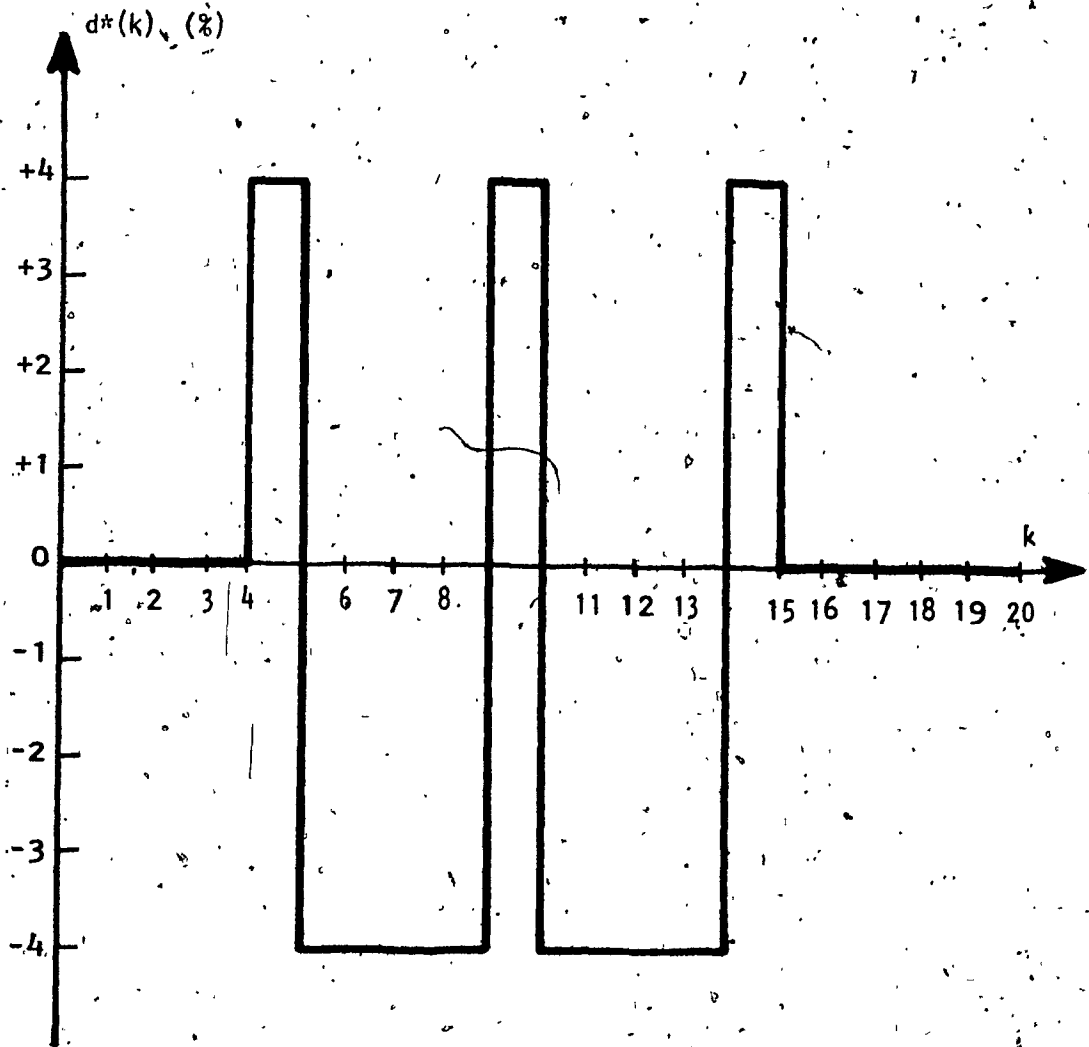


Figure 4.1: Effective grade as a function of discrete time interval k . $d^*(k)$ is specified relative to $s_0(k)$ (see discussion in section 2.6.2).

Let us now cast this problem in terms of the previous developments in Chapter 3.

For this case, we have $n = 3$, $l = 1$, $\bar{\tau}_n = \tau_3/T = 3$, and $\bar{\omega}_n = \omega_3/T = 6$. Thus the dimension of the state vector $x(k)$ is 20, and that of the control vector $u(k)$ is 3.

As the initial condition, x_0 of the train, we shall assume that the starting position is considered to be zero, the velocities of the three members are equal, the draft gears are neither extended nor compressed and the brake cylinders have all been released for some previous time sufficient to ensure that no false gradient presently exists. Then for x_0 we have:

$$x_0 = [0 \ 66 \ 0 \ 0 \ 66 \ 0 \ 0 \ 66 \ 0 \ 0 \ 0 \ \dots \ 0]^T \dots\dots\dots(4-1)$$

Now consider for a moment the 61 - dimensional vector u , as defined by (3-35):

$$u = \begin{bmatrix} u_1(0) & u_2(0) & u_3(0) & u_1(1) & u_2(1) & u_3(1) & \dots & u_1(19) & u_2(19) & u_3(19) & \pi \end{bmatrix}^T \dots\dots\dots(4-2)$$

In the LP formulation in Chapter 3, it was assumed that all the LP variables (ie., the elements of u) were positive. Except for $u_2(k)$, this is not the case for our train; however, we can establish

definite lower bounds for each of the other elements of u .

For example, since we have specified that the average speed over the 80 second run should be 45 mph, we may be certain that the train speed will never drop below, say, 20 mph. Hence, by applying inequality (2-17), we know that $u_1(k) \geq -87,000$ for all k . Similarly, from the given effective grade specifications, we know that $u_3(k) \geq -0.04$ for all k . Therefore, u may be offset appropriately so that the resultant vector contains only positive elements. To this end, let u' be defined as:

$$u' = u + U \dots\dots\dots(4-3)$$

where U is a constant offset vector, given in this example as:

$$U = [87,000 \ 0 \ .04 \ 87000 \ 0 \ .04 \ \dots \ 87000 \ 0 \ .04]^T \dots(4-4)$$

By substituting u' for u as the LP vector, the requirement for positive LP variables is met. The LP problem may then be solved, and the true solution subsequently may be derived simply by subtracting U from the optimal u' obtained.

If a 61 - dimensional cost vector c is defined as in (3-36), then our problem is to find u to minimize $\langle c, u' \rangle$ subject to the following constraints:

1) Objective Constraints:

Define a set of 20 - dimensional direction vectors d_{3i+1} for $i = 1, 2$, where the $(3i+1)^{th}$ element is 1, and the remainder are zero. Then, from (3-41) and (4-3), we have the set of 42 objective constraints:

$$\begin{bmatrix} \langle d_{3i+1}, D(k) \rangle & | & -1 \end{bmatrix} u' \leq \begin{bmatrix} \langle d_{3i+1}, D(k) \rangle & | & -1 \end{bmatrix} U - \langle d_{3i+1}, A^k x_0 \rangle \dots \dots \dots (4-5)$$

for $i = 1, 2$ and $k = 0, 1, \dots, 20$.

(ii) Terminal Constraint:

The only terminal constraint being set for this train ($\bar{m} = 1$) is that its position at the end of the 80 second time interval should be one mile more than its initial position. From (3-40) and (4-3), this constraint is expressed by:

$$\begin{bmatrix} FD(20) & | & 0 \end{bmatrix} u' = \begin{bmatrix} FD(20) & | & 0 \end{bmatrix} U + f - FA^{20} x_0 \dots \dots \dots (4-6)$$

where the 20 dimensional row vector $F = [1 \ 0 \ 0 \ \dots \ 0]$ and $f =$

III.) Velocity Constraints:

Define a 20 - dimensional vector d_2 whose second element is equal to .1 and the remainder are 0. Then from (3-44) and (4-3), we have the following 20 velocity constraints:

$$\left[\langle d_2, D(k) \rangle \mid 0 \right] u^1 \leq \left[\langle d_2, D(k) \rangle \mid 0 \right] U + V_{MAX} - \langle d_2, A^k x_0 \rangle \dots\dots\dots(4-7)$$

for $k = 1, 2, \dots, 20$ and where $V_{MAX} = 88$ for this example.

IV.) Tractive Effort Constraints:

With reference to Figure 3.1, reasonable values for V^I and V^{II} would be 30 and 60 mph respectively. By applying equality (2-15), the parameters of constraints (3-23b) and (3-23c) over this speed range are found to be as follows: $L^I = 465.91$, $L^{II} = 61,500$, $M^I = 659.1$ and $M^{II} = 87,000$. Define d_L as a 20-dimension vector whose second element is equal to L^I , whose eleventh element is +1, and whose remaining elements are 0. Similarly, define d_M whose second element is equal to M^I , whose eleventh element is -1 and whose remaining elements are 0. Then, from (3-44) and (4-3), the following 40 tractive effort constraints apply:

$$\left[\langle d_L, D(k) \rangle \mid 0 \right] u^1 \leq \left[\langle d_L, D(k) \rangle \mid 0 \right] U + L^{II} - \langle d_L, A^k x_0 \rangle \dots\dots\dots(4-8)$$

$$\left[\langle d_M, D(k) \rangle \mid 0 \right] u^1 \leq \left[\langle d_M, D(k) \rangle \mid 0 \right] U + M^{II} - \langle d_M, A^k x_0 \rangle \dots\dots\dots(4-9)$$

for $k = 1, 2, \dots, 20$.

V) Effective Grade Constraints

From (3-47), (4-3) and (4-4), we have the following 20 grade constraints:

$$u'_3(k) = 0.04 + 0.01 d^*(k) \dots\dots\dots(4-10)$$

for $k = 0, 1, \dots, 19$ and for $d^*(k)$ as specified in Figure

4.1 (relative to $s_0(k)$ - see discussion in section 2.6.2).

VI) Air Brake Constraints

Applying (3-24), we have the following 20 control variable constraints:

$$u'_2(k) \leq 23 \dots\dots\dots(4-11)$$

for $k = 0, 1, 2, \dots, 19$.

The above defined problem was solved in 2.7 minutes on the Computer Science Canada UNIVAC 1108 computer by employing a system of programs called UMPIRE (Unified Mathematical Programming System Incorporating Refinements and Extensions - see reference (57) and (58)).

The complete solution procedure, including all job control statements, input information, problem definition, and output calculations, is documented in Appendix A for reference.

For convenience, the optimal control trajectory and the resultant force levels in the couplers as a function of k are presented in Figures 4.2 and 4.3 respectively.

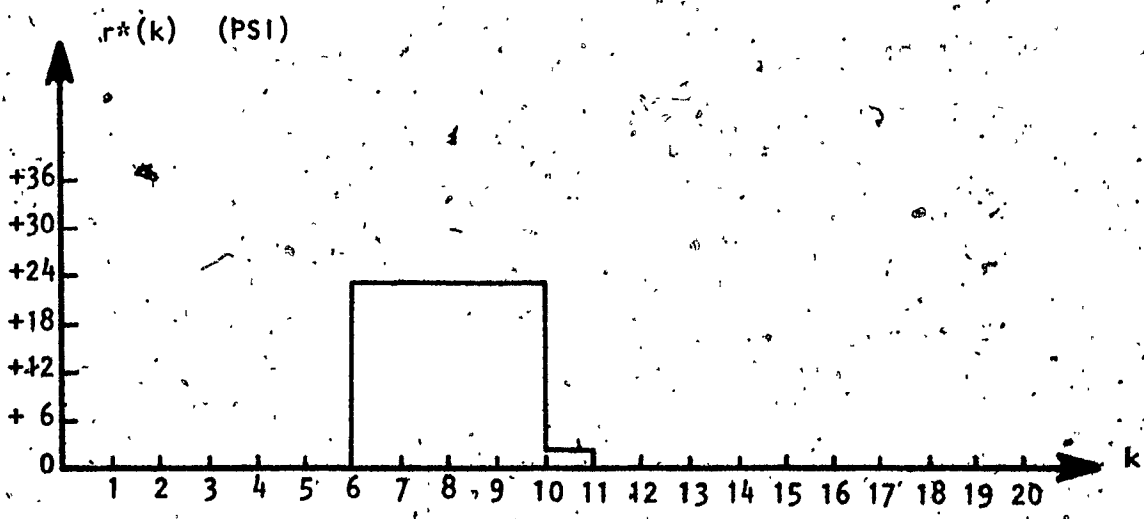
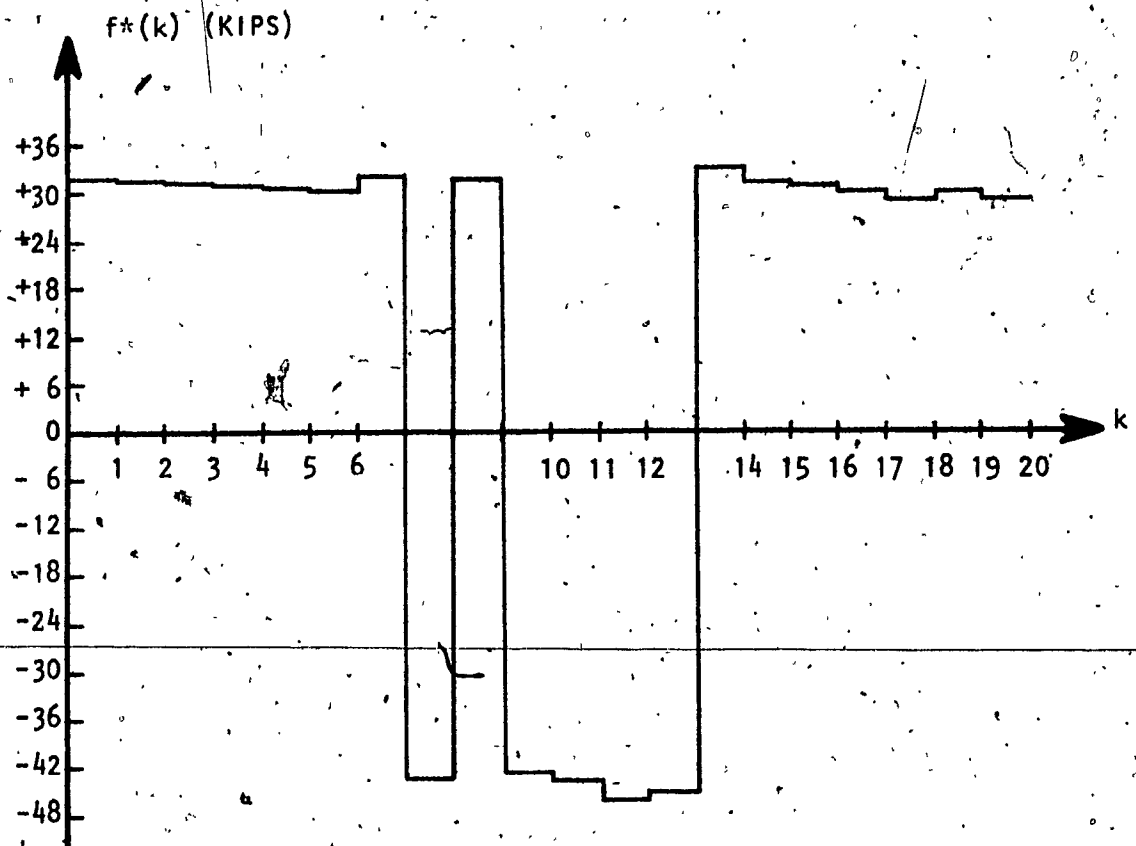


Figure 4.2: Optimal control trajectories u_1 and u_2 (i.e., f^* and r^* respectively). Optimal u_3 is as specified by (3-27) and Figure 4.1.

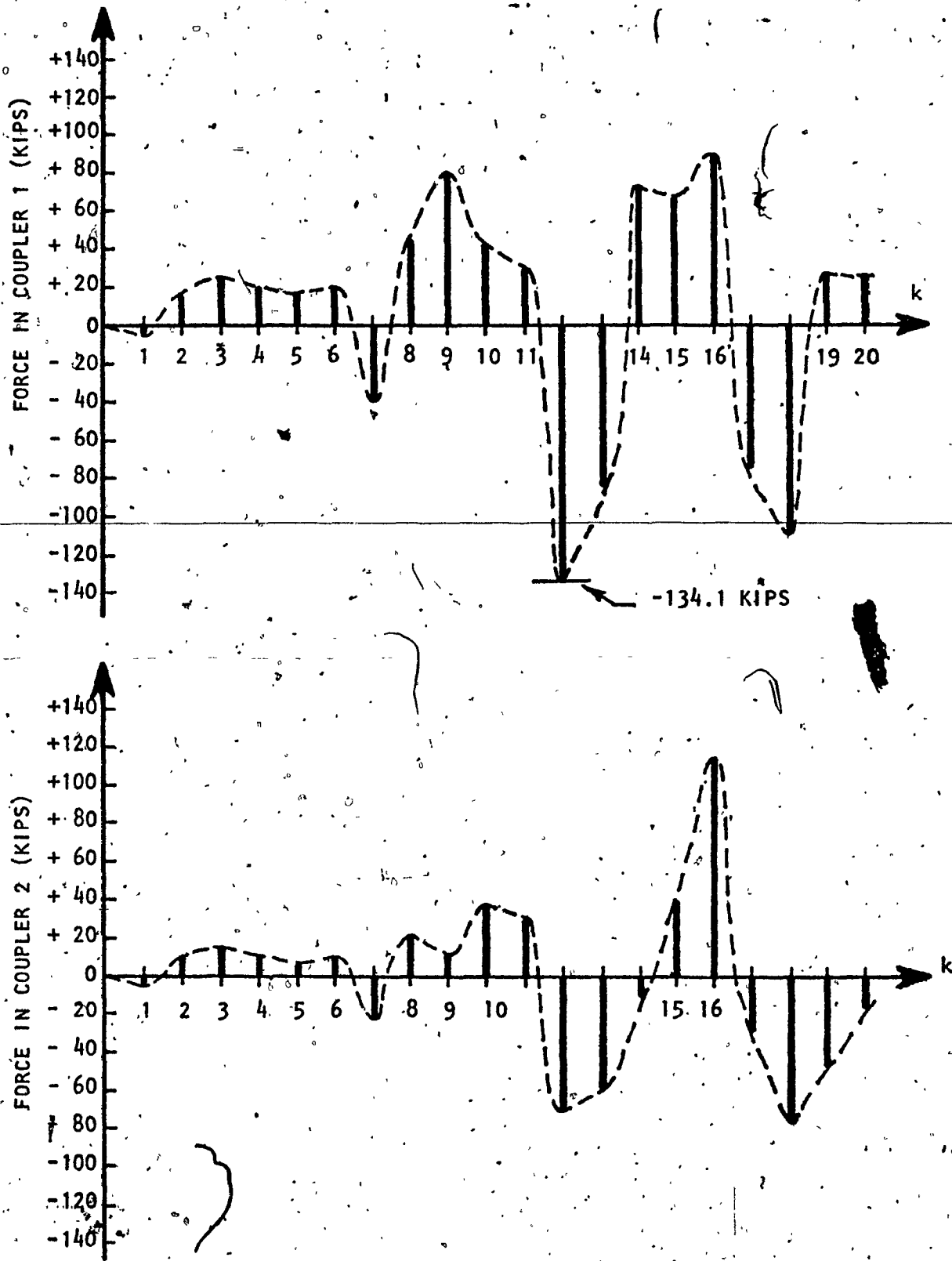


Figure 4.3: Force levels in couplers resulting from the application of the optimal control depicted in Figure 4.2. Exact values are only known at discrete times k ; dashed lines are force envelopes.

CHAPTER 5

DISCUSSION OF RESULTS AND SUGGESTIONS

FOR FUTURE WORK:

A brief discussion of the salient features of the example problem presented in Chapter 4, as well as the solution obtained, is now in order:

Firstly, as noted in Chapter 3, $2(n-1)$ direction vectors are ideally required to specify the "danger region" for each of the variables x_{3i+1} , for $i = 1, 2, \dots, (n-1)$ in turn. This is because each x_{3i+1} can take on negative as well as positive values (i.e., the draft-gears can be compressed as well as extended). In essence, with reference to Figure 3.2, two hyperplanes are required for each x_{3i+1} in order to minimize its absolute value. However, in order to simplify this example, only the positive values of x_{3i+1} were chosen to be minimized, and therefore, the level of train run-ins was not included directly in the objective constraints. Nevertheless, as shown in Figure 4.3, the maximum coupler force in compression exceeded the maximum coupler force in tension by only 17.6 kips. For practical problems, additional constraints of the form (4-5) should be included in the LP formulation, with direction vectors defined such that the $(3i+1)$ th element equals -1 and the remainder are 0. In this manner, the

absolute value of coupler forces will be minimized.

Although only two couplers were considered in this problem, the 36 - car 3100 - ton train was subjected to an extremely violent sequence of grade changes, as per Figure 4.1. In fact, this one mile track section is considerably more severe than that likely to be found in the main line of any Canadian railroad. Yet the maximum force levels in these two couplers was only 134 and 116 kips respectively, and both figures are well below the generally accepted critical level of about 250 kips.

However, even though the control trajectory of Figure 4.2 is optimal from the viewpoint of the mathematical problem posed, this solution could not be practically implemented. The reason for this is that there are two characteristics of the solution which conflict with known real constraints of the actual physical system. These are as follows:

- i) The air brake release action depicted at time interval 10 is shown as only being a partial release. As discussed in section 2.4, this is not possible in the real system. The problem of course is that we have not completely specified the constraints (2-13) in the formulation of the optimization problem.

- 2) The instantaneous switching action of the locomotive throttle to dynamic braking at time intervals 7 and 9 is similarly not feasible. In the actual physical system, in essence, the traction motors are being asked to instantaneously become generators, and this of course is not possible; the throttle must first be moved to the idle position for some minimal time period (usually 5 seconds) before moving to the dynamic braking region. The problem again, of course, is that this physical restraint was not incorporated as a mathematical constraint in the formulation of the optimization problem.

An obvious and direct way of circumventing these difficulties would be to simply add the appropriate constraints to the LP problem statement. Therefore, it is suggested that the design of these constraint statements is the first extension of this work which should be undertaken.

Alternatively, the most practical way of overcoming these deficiencies in the optimal trajectory might simply be to alter the obtained solution by applying common sense. In the case of our example, for instance, one might make the following modifications:

- i) Make the air brake reduction $r^*(k)$ go to zero in the tenth interval;
- (ii) Rather than switching from braking to tractive effort and back to braking in the 7th, 8th and 9th intervals respectively, make $f^*(k)$ go to zero (idle) for the 7th and 8th intervals before going into braking.

These changes would make the resultant control trajectories physically implementable. To be sure, these controls would be sub-optimal. However, a simulation of the system with the modified inputs (a relatively simple matter, since the difference equation model of the system has been obtained) would be able to quantify the degradation from optimal. Intuitively, from the trivial nature of the changes required, this deterioration should not be too severe. Additional work in this area to test this hypothesis would be well worthwhile.

What is the largest real problem that could be practically solved with the approach outlined in this dissertation? To answer this question, consider again the three separate phases of the solution procedure:

- A) obtain discrete model of train
- B) formulate LP problem
- C) solve LP problem.

Taking these in reverse order, the largest LP problem which UMPIRE can handle is 8,000 rows and 262,000 columns (57). In consideration of this fact and (3-49), one sees that the following inequalities must hold:

$$(\bar{x} + \bar{u}) \cdot K + \bar{x} + \bar{m} \leq 8,000 \quad \dots\dots\dots (5-1)$$

$$(1+2) \cdot K + 1 \leq 262,000 \quad \dots\dots\dots (5-2)$$

Generally speaking, $(\bar{x} + \bar{u}) > (1+2)$ so that (5-1) will usually be the limiting constraint. (In order to take full advantage of the capabilities of UMPIRE, some work should be directed towards the formulation of the dual LP problem). Since K will usually be at least two orders of magnitude larger than \bar{x} , \bar{u} , or \bar{m} , then an approximate expression of the limiting constraint for phase C is:

$$K \leq \frac{8000}{(\bar{x} + \bar{u})} \quad \dots\dots\dots (5-3)$$

Let us now digress for a moment and consider the difficulties associated with Phase A, i.e., obtaining the discrete model of the

train. Although the number of state variables does not directly affect the dimension of the LP matrix, it certainly does have considerable import to the problem of calculating the state transition matrix $\phi(t)$. The evaluation of the Taylor series approximation of (3-10) becomes extremely costly as the dimension of the continuous system matrix Δ grows increasingly large. The author is not aware of what the practical limit for the dimension of Δ might be before the calculation of $\phi(t)$ becomes infeasible on the basis of cost and/or accuracy. However, it would seem reasonable to assume that a 61 X 61 matrix would not present too much difficulty. Thus, a 20-member train would certainly seem capable of being treated. By considering a 100-car train to be made up of 5-car members, a discrete model for such a train may readily be obtained.

The total number of state and control variable constraints for the example of Chapter 4 was equal to 7, and this number is not likely to exceed 15 for practical trains. Therefore, from (5-3), we have that the maximum value for K should be in the order of 500. It is therefore mandatory to keep K below this approximate limit.

Consider now the effect of the choice of K on T. Obviously, for any particular length of track and overall time interval,

$\{t_0, t_f\}$, these quantities are inversely proportional. Hence, for too small K , T may become so large as to make the accurate representation of the effective grade input impossible. In addition, the control may be too "coarse" for too large T . One is therefore forced to conclude that the limit on K implies a limit on the length of train run which can be optimized.

As an example, for average train speeds of 30 mph and $T = 4$ secs., we have $K = 30$ per mile. Hence, optimization over track lengths of about 15 miles would seem to be possible.

In summary, the problem of optimizing the operation of a 100-car train over a 15 mile track section so that the maximum force levels in every fifth coupler are minimized would certainly seem to be feasible. By making the initial condition of the next track section correspond to the terminal values of the preceding section, effectively any track length may be considered.

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APPENDIX A

DOCUMENTATION OF DIGITAL COMPUTER PROCEDURE
USED TO SOLVE THE PROBLEM OUTLINED
IN CHAPTER 4

0 RUIF0 THESIS OAK002
LOG NUMBER= THESIS

59997501000

OAK00PR2

0 ASG R=30000
FSTRID ASSIGNMENTS* R= 2/17 \$00000

0X ASG M C=H+D
* TAPE ASSIGNMENTS* C= 4/00 * H= 4/82 * O= 4/03 *

0 ASG A=SMTPAK
FSTRID ASSIGNMENTS* A= 2/00 SMTPAK

0 ASG B=SUNPLB
FSTRID ASSIGNMENTS* B= 2/01 SUNPLB

SCIENTIFIC SERVICE

COMPUTER SCIENCE CENTER

CSC: INVENTORY
 LHMFGG - LHMFGG MATRIX GENERATOR
 PROGRAM NO. 3023 VERSION 5.3

NOTATION

SUFFIXES

C N = INDEX FOR OBJECTIVE CONSTRAINTS

C I = INDEX FOR LP VECTOR

C K = INDEX FOR TIME INTERVAL NUMBER

C Z = DUMMY INDEX FOR LP VARIABLE PI

N NMAX 2

I IMAX 60

K KMAX 20

Z DMAX 1

VARIABLES

U = VECTOR SET OF LLBYNK CONTROL VARIABLES

PI = DUMMY LP VARIABLE WHICH IS BEING MINIMIZED

U(I) ' * II'

PI(Z) ' * * 2'

EXTERNAL VALUES

FIRST CALL PREPIO

XXXX

DECLARATIONS

PARAMETER NCARS=3, NLCOCS=1, JL=1, TAUN=3, OMEGAN=6, NSUM=10, NT=4, NK=20

PARAMETER MBAR=1

PARAMETER L=2, NCARS1=NLCOCS, NCARS2=1, LL=NLCOCS+2, KT=2*NT-1

PARAMETER MW=NLCOCS+TAUN, MW=MM+OMEGAN, INCTAU=TAUN/NCARS

PARAMETER ITCOGA=OMEGAN/NCARS, LESS1=NCARS-1, PLUS1=NCARS+1

PARAMETER I=I, K=K, LL=LL

REAL A(MM,MM), B(MM,LL), D(MM,LL,LL), D(MM,LL,LL,LL), AKXO(MM,NK), XO(MM), M11

REAL X1(MM), X2(MM), PUBIAS(LL), GRADE(NK), ALPHA(LESS1), L1, L11, NI

CSCX CS005Y 19 JUN 75 20:28 USER GAK002 THESIS
CARDS IN 491 OUT 400 LINES 2827 PROCESSING TIME 161.56 SECONDS

FORCE LEVELS IN COUPLERS AS A FUNCTION OF K (T = 4.000 SECONDS)

K CPLR 1 CPLR 2

1	-.85157+03	-.47946+03
2	.17406+05	.98866+04
3	.24393+05	.14647+05
4	.18137+05	.89690+04
5	.15644+05	.67829+04
6	.18055+05	.90240+04
7	-.38764+05	-.22630+05
8	.44239+05	.21549+05
9	.80390+05	.11510+05
10	.41752+05	.39705+05
11	.20445+05	.77950+05
12	-.13415+06	-.70077+05
13	-.84059+05	-.57636+05
14	.71341+05	-.79235+04
15	.67809+05	.39391+05
16	.88981+05	.11648+06
17	-.77153+05	-.22458+05
18	-.10216+06	-.77397+05
19	.24923+05	-.44758+05
20	.26267+05	-.18515+05

*** MAXIMUM COUPLER FORCE EXPERIENCED IN TRAIN = -134.1 KIPS ***

OPTIMAL SYSTEM TRAJECTORY (T = 4.000 SECONDS)

K	X1(KT)	X2(KT)	X3(KT)	X4(KT)	X5(KT)	X6(KT)	X7(KT)	X8(KT)	X9(KT)	V10(KT)
0	.00000	.65000+02	.00000	.00000	.66000+02	.00000	.00000	.66000+02	.00000	.00000
1	.26318+03	.65592+02	.00000	-.17031-02	.65590+02	.00000	-.95975-03	.65587+02	.00000	.00000
2	.52603+03	.65778+02	.00000	.35613-01	.65824+02	.00000	.19973-01	.65680+02	.00000	.00000
3	.74989+03	.66054+02	.00000	.48785-01	.66053+02	.00000	.29375-01	.66056+02	.00000	.00000
4	.10245+04	.66300+02	.00000	.36274-01	.66279+02	.00000	.17938-01	.66259+02	.00000	.00000
5	.13200+04	.66504+02	.00000	.31288-01	.66504+02	.00000	.13566-01	.66503+02	.00000	.00000
6	.15065+04	.66715+02	.00000	.36109-01	.66724+02	.00000	.18056-01	.66731+02	.00000	.00000
7	.18498+04	.65126+02	.00000	-.77529-01	.64984+02	.00000	-.45261-01	.64814+02	.00000	.00000
8	.21140+04	.66677+02	.13459+02	.88478-01	.66979+02	.00000	.33099-01	.67314+02	.00000	.00000
9	.23787+04	.65306+02	.23768+02	.16078-00	.65222+02	.13459+02	.23019-01	.65226+02	.00000	.00000
10	.26444+04	.67446+02	.31664+02	.83504-01	.67473+02	.23768+02	.77410-01	.67376+02	.13459+02	.14833+01
11	.29103+04	.65605+02	.37711+02	.56889-01	.65700+02	.31664+02	.15592-00	.65668+02	.23768+02	.28710+01
12	.31673+04	.67244+02	.29919+02	-.26829-00	.62640+02	.37711+02	-.14015-00	.62069+02	.31664+02	.41692+01
13	.34200+04	.67412+02	.22916+02	-.16812-00	.63673+02	.29919+02	.11527+00	.63991+02	.37711+02	.54836+01
14	.36712+04	.61639+02	.17552+02	.14268-00	.61864+02	.22916+02	-.15847-01	.62222+02	.29919+02	.51505+01
15	.39258+04	.65493+02	.13444+02	.13562-00	.65521+02	.17552+02	.78763-01	.65462+02	.22916+02	.44183+01
16	.41893+04	.66046+02	.17297+02	.17796-00	.66334+02	.13444+02	.23297-00	.66332+02	.17552+02	.45076+01
17	.44447+04	.67143+02	.78866+01	-.15431-00	.66589+02	.10297+02	-.56917-01	.66016+02	.13444+02	.42760+01
18	.47263+04	.69128+02	.60406+01	-.21632-00	.69197+02	.78866+01	-.15479-00	.69301+02	.10297+02	.30449+01
19	.50023+04	.68526+02	.46266+01	.49845-01	.68706+02	.60406+01	-.89517-01	.69057+02	.78866+01	.36905+01
20	.52790+04	.70075+02	.35437+01	.52533-01	.70042+02	.46266+01	-.37031-01	.69979+02	.60406+01	.34525+01

OPTIMAL CONTROL TRAJECTORY (T = 4.000 SECONDS)

K	U1(KT)	U2(KT)	U3(KT)
0	.30959+05	.00000	.00000
1	.30884+05	.00000	.00000
2	.30760+05	.00000	.00000
3	.30649+05	.00000	.00000
4	.30537+05	.00000	.40000-01
5	.30436+05	.00000	-.40000-01
6	.31195+05	.2000+02	-.40000-01
7	-.3071+05	.2000+02	-.40000-01
8	.31091+05	.2000+02	-.40000-01
9	-.42519+05	.2000+02	.40000-01
10	-.43693+05	.1769+01	-.40000-01
11	-.45300+05	.00000	-.40000-01
12	-.45199+05	.00000	-.40000-01
13	.32786+05	.00000	-.40000-01
14	.31038+05	.00000	.40000-01
15	.30764+05	.00000	.00000
16	.30245+05	.00000	.00000
17	.29327+05	.00000	.00000
18	.29568+05	.00000	.00000
19	.28849+05	.00000	.00000

1	0524-02531
2	10103-10110
POSIT / CODE	
0	10311-10315
1	02532-02567
MSGINT / CODE	
0	10316-10527
1	02568-02747
MXSA / CODE	
0	10528-10577
1	02748-02756
MXADJ / CODE	
0	10530-10536
1	02757-02762
CHECK / CODE	
0	10537-10538
1	02763-02772
MOLINE / CODE	
0	10539-10542
1	02773-03056
MOMAD / CODE	
0	10543-10550
1	03057-03058
MOMDS / CODE	
1	03059-03060
2	10551-10552
MYTAGS / CODE	
0	10553-10624
MYBLK2 / CODE	
0	10625-10626
MYBLK1 / CODE	
0	10627-11241
MGCINT / CODE	
0	11242-11252
MGTABS / CODE	
0	11253-11254
DRUM LENGTH 020624	

END OF ALLOCATION
THIS ALLOCATION WAS DONE ON 19 JUN 73 AT 23:24:00

NFTVS /CODE	1	016331-016353
NOTIFS /CODE	1	016354-016356
	2	101607-101646
NFTMS /CODE	1	016637-017639
	2	101647-101765
NLIMS /CODE	1	017635-021511
	2	101766-102136
NIIMS /CODE	1	021512-021542
	2	102137-102171
NIIMTS /CODE	1	021643-023127
	2	102172-102224
NEIPS /CODE	0	102225-102311
NOBGS /CODE	1	023130-023172
	2	102312-102321
NFLOTS /CODE	1	023173-023256
	2	102322-102351
NEPPT /CODE	0	102352-102615
	1	023257-023341
NXNLT /CODE	0	102616-102652
	1	023342-023711
RNREAD /CODE	0	102653-102773
	1	023712-024615
ITDGED /CODE	0	102774-103030
	1	024616-024747
NEXPIS /CODE	1	024750-025003
	2	103031-103031
NVSNL /CODE	0	103032-103102
	1	025004-025233
IRXSPY /CODE		

8 007 RMAIN

STARTING ADDRESS 014000

COPE LIMITS 014000 030402 100000 112554

REMAIN / CODE

0 100000-100031
1 014000-014132

✓ STOPS / CODE

NTIMP / CODE

1 014133-014230
2 100032-100032

NIOTM / CODE

1 014231-014337
2 100033-100067

SYSM / CODE

NTABS / CODE

NTAB22 / *****

0 100070-100174

NSAUFFS / CODE

1 014440-014504
2 100175-101203

NIERS / CODE

1 014505-014657
2 101204-101332

NSAUFFS / CODE

1 014660-014701

EDITS / CODE

0 101333-101417

NIERS / CODE

1 014702-015257
2 101420-101552

NIOUTS / CODE

1 015260-016330
2 101553-101606

END CUR

1. TEF H

2. TRM M

3. ERS

4. IN R

END FILE UNIT R

END CUR

AGEIDUM - STOP

22:23:38 19 JUN 73

OPTIMAL TRAJECTORY CALCULATION FOR A THREE-SECTION FREIGHT TRAIN

BASIC OR N/Z VARIABLE NAME	VARIABLE VALUJF	ROW NAME	SHADOW PRICE	ORIGINAL RHS
* VEL 16	21.000466	* TELO20	.00000	-104.893499
* VEL 17	20.975812	* BRAK01	.00000	23.000000
* VEL 18	18.906125	* GRAD01	.766745	.000000
* VEL 19	17.422947	* BRAK02	.000000	23.000000
* VEL 20	17.881572	* GRAD02	-422882	.000000
* TEL008	157.019435	* BRAK03	.000000	23.000000
* TEL010	155.709790	* GRAD03	-1.112752	.000000
* TEL011	160.007530	* BRAK04	.000000	23.000000
* TEL012	165.2982390	* GRAD04	-580080	.000000
* TEL013	165.613000	* BRAK05	.000000	23.000000
* TEL001	113.351101	* GRAD05	-2.183796	.000000
* TEL002	113.079330	* BRAK06	.000000	23.000000
* TEL003	112.625112	* GRAD06	.000000	.000000
* TEL004	112.215756	* BRAK07	.000112	23.000000
* TEL005	111.007504	* GRAD07	.000000	.000000
* TEL006	111.437596	* BRAK08	.001091	23.000000
* TEL007	114.216499	* GRAD08	.000000	.000000
* TEL009	113.374569	* BRAK09	.000000	23.000000
* TEL014	120.044869	* GRAD09	.000000	.000000
* TEL015	113.643324	* BRAK10	.000000	23.000000
* TEL016	112.638457	* GRAD10	.000000	.000000
* TEL017	110.735859	* BRAK11	.000000	23.000000
* TEL018	107.373808	* GRAD11	.000000	.000000
* TEL019	100.255067	* BRAK12	.000000	23.000000
* TEL020	105.625004	* GRAD12	.000000	.000000
* BRAK01	23.000000	* BRAK13	.000000	23.000000
* BRAK02	23.000000	* GRAD13	.000000	.000000
* BRAK03	23.000000	* BRAK14	.000000	23.000000
* BRAK04	23.000000	* GRAD14	.000000	.000000
* BRAK05	23.000000	* BRAK15	.000000	23.000000
* BRAK06	23.000000	* GRAD15	.000000	.000000
* BRAK11	21.231148	* BRAK16	.000000	23.000000
* BRAK12	23.000000	* GRAD16	.000000	.000000
* BRAK13	23.000000	* BRAK17	.000000	23.000000
* BRAK14	23.000000	* GRAD17	.000000	.000000
* BRAK15	23.000000	* BRAK18	.000000	23.000000
* BRAK16	23.000000	* GRAD18	.000000	.000000
* BRAK17	23.000000	* BRAK19	.000000	23.000000
* BRAK18	23.000000	* GRAD19	.000000	.000000
* BRAK19	23.000000	* BRAK20	.000000	23.000000
* BRAK20	23.000000	* GRAD20	.000000	.000000

EOF

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OPTIMAL TRAJECTORY CALCULATION FOR A THREE-SECTION FREIGHT TRAIN

BASIC OR N/Z VARIABLE NAME	VARIABLE VAL/JF	ROW NAME	SHADOW PRICE	ORIGINAL RHS
+ OC1 02	.143108	+ VEL 08	.00000	16.590010
+ OC1 03	.178235	+ VEL 09	.00000	13.738256
+ OC1 04	.142899	+ VEL 10	.00000	10.824807
+ OC1 05	.130267	+ VEL 11	.00000	7.961069
+ OC1 06	.144540	+ VEL 12	.00000	5.146317
+ OC1 07	.264983	+ VEL 13	.00000	2.336219
+ OC1 08	.091353	+ VEL 14	.00000	-.468000
+ OC1 10	.056140	+ VEL 15	.00000	-3.251416
+ OC1 11	.137652	+ VEL 16	.00000	-6.071653
+ OC1 12	.412271	+ VEL 17	.00000	-8.756823
+ OC1 13	.317570	+ VEL 18	.00000	-11.486033
+ OC1 14	.025603	+ VEL 19	.00000	-14.199243
+ OC1 15	.040431	+ VEL 20	.00000	-16.893003
+ OC1 16	.030027	+ TEH101	.00000	253.139060
+ OC1 17	.330623	+ TEH102	.00000	255.221050
+ OC1 18	.384524	+ TEH103	.00000	255.735460
+ OC1 19	.130633	+ TEH104	.00000	255.885250
+ OC1 20	.170863	+ TEH105	.00000	254.422840
+ OC2 01	.153307	+ TEH106	.00000	253.123590
+ OC2 02	.213654	+ TEH107	.00000	250.108730
+ OC2 03	.218214	+ TEH108	.00000	247.321100
+ OC2 04	.275652	+ TEH109	.00000	244.469340
+ OC2 05	.194580	+ TEH110	.00000	241.555890
+ OC2 06	.207414	+ TEH111	.00000	238.692160
+ OC2 07	.076039	+ TEH112	.00000	235.877400
+ OC2 08	.182032	+ TEH113	.00000	233.067300
+ OC2 09	.148657	+ TEH114	.00000	230.262190
+ OC2 10	.126456	+ TEH115	.00000	227.479670
+ OC2 11	.047107	+ TEH116	.00000	224.719430
+ OC2 12	.302302	+ TEH117	.00000	221.974260
+ OC2 13	.282139	+ TEH118	.00000	219.244150
+ OC2 14	.199059	+ TEH119	.00000	216.531840
+ OC2 15	.157393	+ TEH120	.00000	213.837180
+ OC2 16	.017518	+ TEL001	.00000	-65.592027
+ OC2 17	.278334	+ TEL002	.00000	-63.510031
+ OC2 18	.368917	+ TEL003	.00000	-62.985625
+ OC2 19	.314329	+ TEL004	.00000	-62.845833
+ OC2 20	.264030	+ TEL005	.00000	-64.308239
+ VEL 02	22.248758	+ TEL006	.00000	-65.607899
+ VEL 03	21.992647	+ TEL007	.00000	-68.622355
+ VEL 04	21.742824	+ TEL008	.000659	-71.409088
+ VEL 05	21.503686	+ TEL009	.000000	-74.261741
+ VEL 06	21.296930	+ TEL010	.000000	-77.175192
+ VEL 07	22.914974	+ TEL011	.000000	-80.038030
+ VEL 08	21.359816	+ TEL012	.000000	-82.853682
+ VEL 09	22.691216	+ TEL013	.000000	-85.663780
+ VEL 10	20.521743	+ TEL014	.000000	-88.468000
+ VEL 11	22.303273	+ TEL015	.000000	-91.251414
+ VEL 12	24.740837	+ TEL016	.000000	-94.011652
+ VEL 13	24.597406	+ TEL017	.000000	-96.756820
+ VEL 14	26.329574	+ TEL018	.000000	-99.486031
+ VEL 15	22.579174	+ TEL019	.000000	-102.199242

OPTIMAL TRAJECTORY CALCULATION FOR A THREE-SECTION FREIGHT TRAIN

RHS. NO. = 1 COST ROW NO. = 1 PROBLEM OPTIMAL

BASIC OR N/Z VARIABLE NAME	VARIABLE VALUE	ROW NAME	SHADOW PRICE	ORIGINAL RHS
Z OBJ				
U 01	152347	Z OBJ	1.00000	.00000
U 03	117958.55500	+ OC1 00	.00000	.00000
U 04	.940080	+ OC2 00	.00000	.00000
U 06	117834.360000	+ OC1 01	.00000	.001703
U 07	.040000	+ OC1 02	.00000	.108719
U 09	117760.360000	+ OC1 03	.00000	.028379
U 10	.040000	+ OC1 04	.00000	-.045752
U 12	117680.605000	+ OC1 05	.00000	.030219
U 13	.040000	+ OC1 06	.00000	.069000
U 15	117537.151000	+ OC1 07	.00000	.121608
U 15	.040000	+ OC1 08	.00000	.136084
U 16	117436.150000	+ OC1 09	1.00000	.106517
U 18	.040000	+ OC1 10	.00000	.099146
U 19	118194.040000	+ OC1 11	.00000	.111067
U 20	23.000000	+ OC1 12	.00000	.114181
U 21	.040000	+ OC1 13	.00000	.109528
U 22	43920.928000	+ OC1 14	.00000	.108400
U 23	23.000000	+ OC1 15	.00000	.110366
U 24	.040000	+ OC1 16	.00000	.110917
U 25	118090.500000	+ OC1 17	.00000	.110229
U 26	23.000000	+ OC1 18	.00000	.110110
U 27	.040000	+ OC1 19	.00000	.110486
U 28	44881.382000	+ OC1 20	.00000	.110633
U 29	23.000000	+ OC2 01	.00000	.000960
U 30	.040000	+ OC2 02	.00000	.063007
U 31	43307.005000	+ OC2 03	.00000	.000072
U 32	1.768851	+ OC2 04	.00000	.038887
U 33	.040000	+ OC2 05	.00000	-.053356
U 34	81700.436000	+ OC2 06	.00000	-.034270
U 36	.040000	+ OC2 07	.060000	.056154
U 37	41801.256000	+ OC2 08	.00000	.078837
U 39	.040000	+ OC2 09	.00000	.052429
U 40	119785.957000	+ OC2 10	.00000	.045345
U 42	.040000	+ OC2 11	.00000	.056045
U 43	119031.332000	+ OC2 12	1.00000	.058830
U 45	.040000	+ OC2 13	.00000	.054561
U 46	117764.002000	+ OC2 14	.00000	.053501
U 48	.040000	+ OC2 15	.00000	.055265
U 49	117244.503000	+ OC2 16	.00000	.055738
U 51	.040000	+ OC2 17	.00000	.055081
U 52	116326.751000	+ OC2 18	.00000	.054043
U 54	.040000	+ OC2 19	.00000	.055257
U 55	116567.592000	+ OC2 20	.00000	.055362
U 57	.040000	A TC	.00000	-1049.659000
U 58	115849.327000	+ VEL 02	.00000	24.489071
U 60	.040000	+ VEL 03	.00000	25.004376
P1 1	.152347	+ VEL 04	.00000	25.154168
+ OC1 00	.152347	+ VEL 05	.00000	23.691760
+ OC2 00	.152347	+ VEL 06	.00000	22.392501
OC1 01	.154050	+ VEL 07	.00000	19.373643

AGENTIA - UNRAVL

22:23:35 19 JUN 73

TOTAL ITERS	NO. ETAS	ROW IDENT.	CURRENT VALUE	RHS NO.	C/V NO.	PROB. STATE	REMOVED VECTOR	CHOSEN VECTOR	NO. NEG D/J'S	NO. OF INFEAS	D/J VALUE (OR THETA)	TIME
55	37	1	63.276646	1	0	I	155	164	3	1	-.025079	41.6
56	38	1	-.226031	1	0	F	177	184	3	0	-.001528	41.7
57	39	1	-.225888	1	0	F	164	176	3	0	.000054	41.8
58	40	1	-.152347	1	0	F	91	71	1	0	-.000066	41.8
58	31	1		1	0	0	0	0	0	0	-.000000	42.0

TOTAL ITEMS	NO. ETAS	ROW IDENT.	CURRENT VALUE	RHS NO.	C/V NO.	PROB. STATE	REMOVED VECTOR	CHOSEN VECTOR	NO. NEG D/J'S	NO OF INFEAS	D/J VALUE (OR THETA)	TITE
M	1	1	192.900300	1	0	I	0	0	0	27	.000000	36.4
M	2	1	190.136200	1	0	I	64	145	21	27	-.002576	36.4
M	3	1	1613.671400	1	0	I	65	148	21	26	-.002541	36.4
M	4	1	1433.671400	1	0	I	66	151	21	26	-.002541	36.4
M	5	1	1433.133900	1	0	I	12	205	21	18	-8.000000	36.4
M	6	1	1259.501100	1	0	I	67	154	17	17	-.002429	36.4
M	7	1	1092.500500	1	0	I	68	157	17	16	-.002372	36.4
M	8	1	932.163360	1	0	I	64	160	17	15	-.002315	36.4
M	9	1	904.576910	1	0	I	44	163	17	14	-.002258	36.4
M	10	1	850.007190	1	0	I	70	155	38	13	-14.150000	36.4
M	11	1	850.705480	1	0	I	173	173	29	13	-.101770	36.4
M	12	1	850.709580	1	0	I	155	170	29	13	-.104617	36.4
M	13	1	850.505670	1	0	I	170	191	22	13	-.042960	36.7
M	14	1	855.760020	1	0	I	191	173	17	13	-.086029	37.5
M	15	1	855.025620	1	0	I	194	194	17	13	-.031000	37.5
M	16	1	854.620580	1	0	I	197	197	17	13	-.017093	37.5
M	17	1	854.497630	1	0	I	200	200	17	13	-.005693	37.5
M	18	1	853.489830	1	0	I	188	188	18	13	-.043917	37.7
M	19	1	852.539840	1	0	I	185	185	18	13	-.041005	37.7
M	20	1	852.207640	1	0	I	173	182	18	13	-.034730	37.7
M	21	1	785.165350	1	0	I	182	175	13	12	-.001532	37.8
M	22	1	771.085540	1	0	I	94	172	13	12	-.001517	37.8
M	23	1	720.179980	1	0	I	73	155	28	11	-18.311741	37.8
M	24	1	715.132480	1	0	I	155	176	21	11	-.225875	38.0
M	25	1	710.787500	1	0	I	176	169	13	11	-.001569	38.0
M	26	1	644.902830	1	0	I	179	179	13	10	-5.622135	38.0
M	27	1	643.332750	1	0	I	173	173	16	10	-.253680	38.1
M	28	1	572.177460	1	0	I	75	178	11	9	-.001519	38.1
M	29	1	555.515440	1	0	I	173	181	11	9	-.001814	38.2
M	30	1	522.703350	1	0	I	17	155	26	8	-25.66929	38.2
M	31	1	499.171090	1	0	I	155	170	17	8	-.173660	38.3
M	32	1	499.462930	1	0	I	96	171	17	8	-20.027098	38.5
M	33	1	451.707470	1	0	I	83	202	10	7	-.001517	38.5
M	34	1	378.630680	1	0	I	170	166	10	6	-.001573	38.5
M	35	1	377.300950	1	0	I	71	75	10	6	-.020973	38.5
M	36	1	377.129870	1	0	I	95	94	13	6	-.011033	38.6
M	37	1	377.071130	1	0	I	194	194	13	6	-.009099	38.7
M	38	1	323.045840	1	0	I	82	199	9	5	-.002554	38.7
M	39	1	323.520220	1	0	I	197	197	9	5	-.001524	38.8
M	40	1	323.483100	1	0	I	200	200	9	5	-.013792	38.8
M	41	1	264.059320	1	0	I	94	196	7	4	-.001062	39.8
M	42	1	242.441370	1	0	I	101	194	7	4	-.001535	39.8
M	43	1	242.011320	1	0	I	80	155	23	3	-.001517	39.8
M	44	1	201.300100	1	0	I	185	185	14	3	-69.550496	40.0
M	45	1	200.103490	1	0	I	155	167	14	3	-.065705	40.0
M	46	1	198.279750	1	0	I	188	188	8	3	-.112006	40.0
M	47	1	195.440420	1	0	I	182	182	8	3	-.083206	40.1
M	48	1	195.247880	1	0	I	167	101	8	3	-.079071	40.1
M	49	1	195.775200	1	0	I	155	155	23	3	-.006701	40.2
M	50	1	195.663160	1	0	I	155	173	9	3	-.509958	40.2
M	51	1	130.730740	1	0	I	77	190	4	2	-.005547	40.4
M	52	1	75.972200	1	0	I	93	187	4	2	-.001540	40.4
M	53	1	64.311239	1	0	I	78	155	27	1	-.001546	40.4
M	54	1	63.629384	1	0	I	179	179	8	1	-39.007021	40.6
											-.009646	41.6

AGENDUM - NORMAL

22:23:27 10 JUN 73

THIS PROBLEM HAS:-

61 COLUMNS

144 ROWS COST ROW IS 1

1 RIGHT HAND SIDES - 1 USED

2573 ELEMENTS

THE DENSITY IS 29.29 PER CENT

PHASE ONE OF INPUT TOOK
ROW 25 IS REDUNDANT
ROW 26 IS REDUNDANT

39.23 SECONDS

PRESOLVE FOUND-

1 INALL ROWS

40 SUB ROWS

INPUT TOOK 35.13 SECONDS

KINPUT=13,KEY=3,ROPRINT=1

AGETIDUM - INPUT

22:22:51. 19 JUN 73

MAXIMUM ADDRESS ASKED FOR = 63777

OPTIMAL TRAJECTORY CALCULATION FOR A THREE-SECTION FREIGHT TRAIN

ROW ID

MATRIX

FIRST B

EOF

CSC FOREY
UNPAG1 - UNP IRE/AGENIUM
PROGRAM NO. 3020 VERSION-07

ALBIS W'VACH

8 XBT CUR

1. OUT R

2. TRM R

3. TRM C

END CUR

CONTROL MATRIX OF DISCRETE SYSTEM FOR T = 4.000 SECONDS

.92533-04	-.13287-01	-.12196-01	-.11625-01	-.10006+03	-.79058+02	-.77736+02
.19323-04	-.92991-02	-.90772-02	-.89499-02	-.85329+02	-.39752+02	-.43215+02
.00000	.58519-00	.00000	.00000	.00000	.00000	.00000
.12236-05	-.10909-02	.61911-03	.70258-03	-.28853+01	.12621+01	.16267+01
.20721-04	-.90772-02	-.91925-02	-.91066-02	-.88862+02	-.40476+02	-.36958+02
.00000	.00000	.55518-00	.00000	.00000	.00000	.00000
.68906-06	-.57128-03	-.40781-03	.12104-02	-.16249+01	-.95736-00	.25840+01
.22526-04	-.89499-02	-.91066-02	-.93492-02	-.53116+02	-.38958+02	-.36220+02
.00000	.00000	.00000	.58518-00	.00000	.00000	.00000
.00000	.00000	.00000	.64493-01	.00000	.00000	.00000

INTEGRAL OF SYSTEM TRANSITION MATRIX FOR T = 4.000 SECONDS

.40000+01	.31100+01	-.79723-01	-.28674+01	.24572+01	-.73177+01	-.13363+01	.24141+01	-.69750-01	.00000
.00000	.14089+01	-.55789-01	-.61179-00	.12355+01	-.54463-01	-.34453-00	.13432+01	-.53699-01	.00000
.00000	.00000	.35111+01	.00000	.00000	.00000	.00000	.00000	.00000	.00000
.00000	.89679-01	-.65445-02	-.81416-01	-.39228-01	.36346-02	-.12936-00	-.50559-01	.42155-02	.00000
.00000	.15187+01	-.54463-01	.32895-00	.12580+01	-.55155-01	-.24951-00	.12109+01	-.54639-01	.00000
.00000	.00000	.00000	.00000	.00000	.35111+01	.00000	.00000	.00000	.00000
.00000	.50503-01	-.34277-02	-.12936-00	.29756-01	-.30469-02	-.57310-01	-.60315-01	.72624-02	.00000
.00000	.16310+01	-.53699-01	.42396-00	.12109+01	-.54639-01	.67348-00	.11257+01	-.56095-01	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.35111+01	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.34695+01

SYSTEM TRANSITION MATRIX FOR T = .0000 SECONDS

.10000+01	.14089+01	-.55789-01	-.61177-00	.12335+01	-.54463-01	-.34453-00	.13432+01	-.53689-01	.00000
.00000	.40823-00	-.23961-01	.74902-00	.27008-00	-.26207-01	.90256-00	.31551-00	-.28857-01	.00000
.00000	.00000	.76593-00	.00000	.00000	.00000	.00000	.00000	.00000	.00000
.00000	-.11079+00	-.13256-02	.59264-01	-.22518-01	.69172-03	-.95015-01	.13229-00	.93998-03	.00000
.00000	.33197-00	-.26207-01	.18802-00	.40219-00	-.26705-01	-.39554-00	.25963-00	-.25600-01	.00000
.00000	.00000	.00000	.00000	.00000	.76593-00	.00000	.00000	.00000	.00000
.00000	-.13230-00	-.76373-03	-.95015-01	.47170-01	-.51348-03	.77010-01	.85116-01	.14555-02	.00000
.00000	.38751-00	-.28857-01	-.11093+01	.25963-00	-.25600-01	-.71374-00	.34631-00	-.23447-01	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.76593-00	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.93551-00

INTEGRAL OF SYSTEM TRANSITION MATRIX FOR T = .250 SECONDS

.25000-00	.29698-01	-.49285-04	-.16491-01	.14960-02	-.13921-05	-.67598-03	.51176-04	-.53464-07	.00000
.00000	.22776-00	-.59021-03	-.19006-00	.21151-01	-.35995-04	-.12115-01	.10394-02	-.12325-05	.00000
.00000	.00000	.24793-00	.00000	.00000	.00000	.00000	.00000	.00000	.00000
.00000	.27860-01	-.47293-04	.21407-00	-.26084-01	.55746-04	.18609-01	-.17759-02	.23858-05	.00000
.00000	.25997-01	-.35995-04	.21873-00	.19819-00	-.66222-03	-.21595-00	.25761-01	-.43963-04	.00000
.00000	.00000	.00000	.00000	.00000	.24793-00	.00000	.00000	.00000	.00000
.00000	.17759-02	-.19409-05	.18609-01	.25753-01	-.59302-04	.21060-00	-.27529-01	.57687-04	.00000
.00000	.12776-02	-.12325-05	.14892-01	.25761-01	-.43963-04	.23084-00	.22292-00	-.74495-03	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.24793-00	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.24948-00

SYSTEM TRANSITION MATRIX FOR T = .250 SECONDS

.10000	.22776-00	-.58021-03	-.19006-00	.21151-01	-.35996-04	.12116-01	.10394-02	-.12323-05	.00000
.00000	.76825-00	-.49361-02	-.13764+01	.21414-00	-.50840-03	-.16865-00	.17173-01	-.25020-04	.00000
.00000	.00000	.98347-00	.00000	.00000	.00000	.00000	.00000	.00000	.00000
.00000	.20176-00	-.54421-03	-.59121-00	-.17704-00	.62622-03	.70384-00	-.24721-01	.42731-04	.00000
.00000	.26322-00	-.50840-03	.14846+01	.47712-00	-.47423-02	-.14459+01	.25924-00	-.61919-03	.00000
.00000	.00000	.00000	.00000	.00000	.98347-00	.00000	.00000	.00000	.00000
.00000	.24721-01	-.54763-04	.20384-00	.17243-00	-.61825-03	.55320-00	-.19716-00	.66098-03	.00000
.00000	.21109-01	-.25020-04	.20730-00	.25929-00	-.61919-03	.16532+01	.71924-00	-.53365-02	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.98347-00	.00000
.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.99544-00

TRUNCATION ERROR IN SERIES APPROXIMATION OF STM AND ITS INTEGRAL FOR T = .25n SECONDS

.0000	-.40190-07	-.18872-09	-.23014-06	.63241-07	.46747-09	.24658-06	-.43104-07	-.23560-09	.00000
.0000	-.20091-06	.90123-09	.97185-06	.41566-06	-.20415-08	-.10595-05	-.21474-06	.14567-08	.00000
.0000	.00000	.45575-24	.00000	.00000	.00000	.00000	.00000	.00000	.00000
.0000	-.14246-06	-.65629-09	-.81614-06	.29774-06	.16711-08	.88911-06	-.15529-06	-.86436-09	.00000
.0000	.51092-06	-.20415-08	-.24967-05	-.10711-05	.52603-08	.27394-05	.56018-06	-.27512-08	.00000
.0000	.00000	.00000	.00000	.00000	.45575-24	.00000	.00000	.00000	.00000
.0000	.15530-06	.70317-09	.88911-06	-.32568-06	-.16323-08	-.98194-06	.17139-06	.96791-09	.00000
.0000	-.26395-06	.10567-08	.13022-05	.56018-06	-.27512-08	-.14372-05	-.29622-06	.14525-08	.00000
.0000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.45575-24	.00000
.0000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.00000	.53463-30

SYSTEM MATRIX OF CONTINUOUS SYSTEM

.0000	.1000+01	.0000	.0000	.0000	.0000	.0000	.0000	.0000	.0000
.0000	-.20640-00	-.19647-01	-.68220+01	.20456-00	.0000	.0000	.0000	.0000	.0000
.0000	.0000	-.66667-01	.0000	.0000	.0000	.0000	.0000	.0000	.0000
.0000	.1000+01	.0000	.0000	-.1000+01	.0000	.0000	.0000	.0000	.0000
.0000	.25156-00	.0000	.83855+01	-.50458-00	-.24150-01	-.83855+01	.25156-00	.0000	.0000
.0000	.0000	.0000	.0000	.0000	-.66667-01	.0000	.0000	.0000	.0000
.0000	.0000	.0000	.0000	.1000+01	.0000	.0000	-.1000+01	.0000	.0000
.0000	.0000	.0000	.0000	.25156-00	.0000	.83855+01	-.25301-00	-.24150-01	.0000
.0000	.0000	.0000	.0000	.0000	.0000	.0000	.0000	-.66667-01	.0000
.0000	.0000	.0000	.0000	.0000	.0000	.0000	.0000	.0000	-.16667-01

TITLE OPTIMAL TRAJECTORY CALCULATION FOR A THREE-SECTION FREIGHT TRAIN

UNITS 6 13 5 15 99999.0 0.000001

NMAX=2

IMAX=60

KMAX=20

OMAX=1

DATA

0 102667-102723
1 024783-025074

HEXPS/CODE
1 025075-025130
2 102724-102724

MGINT/CODE
0 102725-105036
1 025131-026770

HALPS/CODE
1 026771-036645
2 105037-105207

HXALT /CODE
0 105210-105244
1 030648-031015

HXSEA YCODE
0 105245-105264
1 031016-031104

HXADJ /CODE
0 105265-105303
1 031105-031175

CHECH/CODE
0 105304-105342
1 031176-031300

HGLPIE/CODE
0 105343-105410
1 031301-031424

HGNFAD/CODE
0 105411-105505
1 031425-031720

HYBLK2/*****
0 105506-105511

HYBLK1/*****
0 105512-111655

HGCCMT/*****
0 111656-111770

HGTABS/*****
0 111771-112020

DRUM*LENGTH 022120

END OF ALLOCATION
THIS ALLOCATION WAS DONE ON 19 JUNI 73 AT 22:21:47

HOUTS /CODE
1 015275-015345
2 101545-101600

NFTVS /CODE
1 015346-015370

NHTPS /CODE
1 015371-015653
2 101601-101640

REHTS /CODE
1 016654-017651
2 101641-101757

MGOUT /CODE
1 01760-102060
2 017452-021200

COE1 /CODE
1 102061-102332
2 021203-022454

MGELAT /CODE
1 02333-102402
2 022455-022611

CHECKC /CODE
1 102403-102451
2 022612-023120

SCALFC /CODE
1 102452-102452
2 023121-023145

HCODS /CODE
1 023146-023175
2 102453-102453

HHTPS /CODE
1 023176-023326
2 102454-102506

HHTPS /CODE
1 023327-024613
2 102507-102507

INFLTS /CODE
1 024614-024677
2 102442-102571

AEXPS /CODE
1 102572-102656

ADBCS /CODE
1 024700-024742
2 102657-102666

IT08CD /CODE

8 XGT 8HAIN

STARTING ADDRESS 014000

COPE LIMITS 014000 031720 100000 112020

REMAIN / CODE

0 100000-100016
1 014000-014132

REOITS / CODE

1 014133-014215
2 100017-100020

REOITS / CODE

1 014316-014424
2 100021-100055

SYSSH / CODE

NTADS / CODE

NTARZZ / *****

0 100056-100162

NSUFFS / CODE

1 014425-014471
2 100163-101171

REMHDS / CODE

1 014472-014521
2 101172-101175

NSTORES / CODE

NIERS / CODE

0 014522-014674
1 101176-101324

IPAUSS / CODE

1 014675-014716

FOITS / CODE

0 101325-101411

NIERS / CODE

1 014717-015274
2 101412-101544


```

00353 119* C
00354 120* FUNCTION F07(K)
00357 121* F07 = 0.0
00360 122* KK = LL*(HK-K)
00361 123* DO 10 I1 = 1,K
00364 124* DO 10 I11 = 1,LL
00367 125* KK = KK + LL*(I1-1) + I11
00370 126* IO F07 = F07 + (0(2*KK) + D(11*KK)/LI)*UBIAS(III)
00373 127* MK = IK-K+1
00375 128* F07 = F07 + LI1/LI - (AKXO(2*KK) + AKXO(11*KK)/LI)
00376 129* RETURN
00377 130* C
00378 131* FUNCTION F08(K)
00381 132* F08 = 0.0
00384 133* KK = LL*(HK-K)+I
00387 134* IF (KK.GT.LLBYKK) GO TO 10
00390 135* F08 = D(2*KK) - D(11*KK)/M1
00393 136* IO RETURN
00396 137* C
00400 138* FUNCTION F09(K)
00403 139* F09 = 0.0
00406 140* KK = LL*(HK-K)
00409 141* DO 10 I1 = 1,K
00412 142* DO 10 I11 = 1,LL
00415 143* KK = KK + LL*(I1-1) + I11
00418 144* IO F09 = F09 + M11/M1 - (AKXO(2*KK) - AKXO(11*KK)/M1)
00421 145* MK = IK-K+1
00424 146* F09 = F09 + M11/M1 - (AKXO(2*KK) - AKXO(11*KK)/M1)
00427 147* RETURN
00430 148* C
00433 149* FUNCTION F10(I,K)
00436 150* F10 = 0.0
00439 151* IF (I.NE.K*LL) RETURN
00442 152* F10 = 1.0
00445 153* RETURN
00448 154* C
00451 155* FUNCTION F11(I,K)
00454 156* F11 = 0.0
00457 157* IF (I.NE.K*LL-1) RETURN
00460 158* F11 = 1.0
00463 159* RETURN
00466 160* END
00469 161* END COMPILATION ** 0 MESSAGES **

```

00229	59a	ENTRY C10(DUMMY)
00225	60a	COF1 = F10(I,K)
00227	61a	RETURN
00230	62a	
00231	63a	
00232	64a	ENTRY C19(DUMMY)
00233	65a	COF1 = F11(I,K)
00234	66a	RETURN
00235	67a	
00236	68a	
00237	69a	FUNCTION F01(H,I,K)
00238	70a	F01 = 0.0
00239	71a	KK = LL*(IK-K)
00240	72a	IF (KK.GT.LLBYNK) GO TO 10
00241	73a	F01 = D(3*H+1)KK
00242	74a	RETURN
00243	75a	
00244	76a	FUNCTION F02(H,I,K)
00245	77a	F02 = 0.0
00246	78a	KK = LL*(IK-K)
00247	79a	DO 10 I1 = 1,K
00248	80a	DO 10 I2 = 1,LL
00249	81a	KKK = KK+LL*(I1-I2)+I2
00250	82a	F02 = F02 + D(3*H+1)KKK + UBIAS(I1,I2)
00251	83a	KK = IK-K+1
00252	84a	F02 = F02 - AKX0(3*H+1)KK
00253	85a	RETURN
00254	86a	
00255	87a	FUNCTION F03(Z)
00256	88a	F03 = 0.0
00257	89a	DO 10 I1 = 1,K
00258	90a	DO 10 I2 = 1,LL
00259	91a	KK = LL*(IK-K)+I1
00260	92a	F03 = F03 + D(1)KK + UBIAS(I1,I2)
00261	93a	F03 = F03 + FV - AKX0(1)I1
00262	94a	RETURN
00263	95a	
00264	96a	FUNCTION F04(I,K)
00265	97a	F04 = 0.0
00266	98a	KK = LL*(IK-K)+I
00267	99a	IF (KK.GT.LLBYNK) GO TO 10
00268	100a	F04 = D(2)KK
00269	101a	RETURN
00270	102a	
00271	103a	FUNCTION F05(K)
00272	104a	F05 = 0.0
00273	105a	KK = LL*(IK-K)
00274	106a	DO 10 I1 = 1,K
00275	107a	DO 10 I2 = 1,LL
00276	108a	KKK = KK + LL*(I1-I2) + I2
00277	109a	F05 = F05 + D(2)KKK + UBIAS(I1,I2)
00278	110a	KK = IK-K+1
00279	111a	F05 = F05 + VMAX - AKX0(2)KK
00280	112a	RETURN
00281	113a	
00282	114a	FUNCTION F06(I,K)
00283	115a	F06 = 0.0
00284	116a	KK = LL*(IK-K)+I
00285	117a	IF (KK.GT.LLBYNK) GO TO 10
00286	118a	F06 = D(2)KK + D(1)KK/I
00287	119a	RETURN
00288	120a	

01 PR55* COF1
 COMPLETED BY LEVEL 23.5 CSGX FORTRAN V ON 19 JUN 73 AT 22:21
 FUNCTION COF1(DUMMY)
 INCLUDE M6COMM

00101	1*	ENTRY C01(DUMMY)
00102	2*	COF1 = -X0(4)
00103	3*	RETURN
00103b	4*	
00135	5*	ENTRY C02(DUMMY)
00137	6*	COF1 = -X0(4)
00140	7*	RETURN
00140	8*	
00141	9*	ENTRY C03(DUMMY)
00143	10*	COF1 = F01(N,I,K)
00144	11*	RETURN
00144	12*	
00145	13*	ENTRY C04(DUMMY)
00147	14*	COF1 = F02(N,K)
00150	15*	RETURN
00150	16*	
00151	17*	ENTRY C05(DUMMY)
00153	18*	COF1 = D(1,I)
00154	19*	RETURN
00154	20*	
00155	21*	ENTRY C06(DUMMY)
00157	22*	COF1 = F03(I)
00160	23*	RETURN
00160	24*	
00161	25*	ENTRY C07(DUMMY)
00163	26*	COF1 = F04(I,K)
00164	27*	RETURN
00164	28*	
00165	29*	ENTRY C08(DUMMY)
00167	30*	COF1 = F05(K)
00170	31*	RETURN
00170	32*	
00171	33*	ENTRY C09(DUMMY)
00173	34*	COF1 = F06(I,K)
00174	35*	RETURN
00174	36*	
00175	37*	ENTRY C10(DUMMY)
00177	38*	COF1 = F07(K)
00200	39*	RETURN
00200	40*	
00201	41*	ENTRY C11(DUMMY)
00204	42*	COF1 = F08(I,K)
00204	43*	RETURN
00205	44*	
00207	45*	ENTRY C12(DUMMY)
00210	46*	COF1 = F09(K)
00210	47*	RETURN
00210	48*	
00211	49*	ENTRY C13(DUMMY)
00213	50*	COF1 = F10(I,K)
00214	51*	RETURN
00214	52*	
00215	53*	ENTRY C14(DUMMY)
00217	54*	COF1 = URAS(3) + 0.01*GRADE(K)
00220	55*	RETURN
00220	56*	
00221	57*	ENTRY C15(DUMMY)
00223	58*	COF1 = URAS(2) + 23.0
00224	59*	RETURN


```

00406 184*  NAMPOW = 6HTELO
00407 185*  DO 1200 K = 1, KMAX
00412 186*  CALL ITORCD(NAMPOW,K ,24,12)
00413 187*  CALL RAPI(NAMROW,PI)
00414 188*  PIFLO ( K ) =PI
00415 190*  1200 CONTINUE
00415 191*  C-ROW(S) FOR GPAD
00417 193*  C
00420 194*  NAMROW = 6HGRAD
00421 195*  DO 1270 K = 1, KMAX
00422 196*  CALL ITORCD(NAMROW,K ,24,12)
00425 197*  CALL RAPI(NAMROW,PI)
00426 198*  PGFAD ( K ) =PI
00426 199*  1270 CONTINUE
00426 200*  C
00426 201*  C-ROW(S) FOR BPAK
00426 201*  C
00430 202*  NAMPOW = 6HBRAK
00431 203*  DO 1340 K = 1, KMAX
00434 204*  CALL ITORCD(NAMPOW,K ,24,12)
00435 205*  CALL RAPI(NAMROW,PI)
00436 206*  PUPAK ( K ) =PI
00437 207*  1340 CONTINUE
00437 208*  C
00437 209*  C-FIELD EOF MARK
00437 210*  C
00441 211*  CALL RWSOL(999)
00441 212*  C
00442 213*  RETURN
00443 214*  END

```

END COMPILATION ** 0 MESSAGES **

Address	Port	Label	Description
00330	124	C-READ PI VALUES	
00330	125	C	
00331	126	CALL RWSQL(LUNTY)	
00331	127	C	
00331	128	C-ROW(S) FOR OBJ	
00331	129	C	
00331	130	C	
00332	131	NAMROW = 6H0BJ	
00332	132	CALL RWPI(NAMROW,PI)	
00332	133	C	
00332	134	C-ROW(S) FOR OC1000	
00332	135	C	
00333	136	NAMROW = 6HOC1000	
00333	137	CALL RWPI(NAMROW,PI)	
00333	138	POC100=PI	
00333	139	C	
00336	140	C-ROW(S) FOR OC2000	
00336	141	C	
00336	142	NAMROW = 6HOC2000	
00336	143	CALL RWPI(NAMROW,PI)	
00336	144	POC200=PI	
00336	145	C	
00341	146	C-ROW(S) FOR OC	
00341	147	C	
00342	148	NAMROW = 6HOC	
00342	149	DO 920 N = 1, NMAX	
00342	150	DO 910 K = 1, KMAX	
00342	151	CALL ITORCD(IAMROW,K	,12, 6)
00342	152	CALL ITORCD(IAMROW,K	,24,12)
00342	153	CALL RWPI(NAMROW,PI)	
00342	154	POC (I, NMAX) = PI	
00342	155	910 CONTINUE	
00342	156	920 CONTINUE	
00342	157	C	
00342	158	C-ROW(S) FOR TC	
00342	159	C	
00361	160	NAMROW = 6HTC	
00361	161	CALL RWPI(NAMROW,PI)	
00361	162	PTC = PI	
00361	163	C	
00363	164	C-ROW(S) FOR VEL	
00363	165	C	
00364	166	NAMROW = 6HVEL	
00364	167	DO 1060 K = 1, KMAX	
00364	168	CALL ITORCD(IAMROW,K	,24,12)
00364	169	CALL RWPI(NAMROW,PI)	
00364	170	PVEL (K) = PI	
00364	171	1060 CONTINUE	
00364	172	C	
00373	173	C-ROW(S) FOR TEHI	
00373	174	C	
00375	175	NAMROW = 6HTEHI	
00375	176	DO 1130 K = 1, KMAX	
00375	177	CALL ITORCD(IAMROW,K	,24,12)
00375	178	CALL RWPI(NAMROW,PI)	
00375	179	PTHEI (K) = PI	
00375	180	1130 CONTINUE	
00375	181	C	
00375	182	C-ROW(S) FOR TELO	
00375	183	C	

CALL RBETA(IAMROW,BETA)
SOC100=BETA

C-ROW(S) FOR OC2000

NAMROW = 6HOC2000
CALL RBETA(IAMROW,BETA)
SOC200=BETA

C-ROW(S) FOR OC

NAMROW = 6HOC

DO 360 N = 1, KMAX
DO 350 K = 1, KMAX
CALL ITORCD(IAMROW,K ,12,6)
CALL RBETA(IAMROW,K ,20,12)
CALL RBETA(IAMROW,BETA)
SOC (I,K) =BETA

350 CONTINUE
360 CONTINUE

C-ROW(S) FOR VEL

NAMROW = 6HVEL

DO 430 K = 1, KMAX
CALL ITORCD(IAMROW,K ,20,12)
CALL RBETA(IAMROW,BETA)
SVFL (K) =BETA

430 CONTINUE

C-ROW(S) FOR TEHI

NAMROW = 6HTEHI = 1, KMAX
DO 500 K = 1, KMAX
CALL ITORCD(IAMROW,K ,20,12)
CALL RBETA(IAMROW,BETA)
STEHI (K) =BETA

500 CONTINUE

C-ROW(S) FOR FELO

NAMROW = 6HTELO

DO 570 K = 1, KMAX
CALL ITORCD(IAMROW,K ,20,12)
CALL RBETA(IAMROW,BETA)
STELO (K) =BETA

570 CONTINUE

C-ROW(S) FOR RPAK

NAMROW = 6HBRAK

DO 640 K = 1, KMAX
CALL ITORCD(IAMROW,K ,20,12)
CALL RBETA(IAMROW,BETA)
SHPAK (K) =BETA

640 CONTINUE

C-BACKTRACK DRPAVEL
CALL RNSOL(O)

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BT FRS: RHEAD

COMPILE BY LEVEL 23.8 CSCX FORTRAN V ON 19 JUN 73 AT 22:21
SUBROUTINE RHEAD(KUNIT)

C-THIS SR READS ONE SOLUTION INTO CORE

00101	1*	INCLUDE MGCOMM, LIST	MGCOMM	*INCLUDED*
00101	2*	PARAMETER NCARS=3, NLOCOS=1, JL=3, TAUNE=3, OMEGAN=6, NSUM=10, NT=4, NK=2	MGCOMM	*INCLUDED*
00101	3*	PARAMETER MBAR=1	MGCOMM	*INCLUDED*
00101	4*	PARAMETER L=2, NCARS=NLOCOS, M=3, NCARS=1, LL=NLOCOS+2, KT=2, NT=1	MGCOMM	*INCLUDED*
00101	5*	PARAMETER M=2, NLOCOS=TAUNE, MM=MM+OMEGAN, INC=TAUNE, NCARS	MGCOMM	*INCLUDED*
00101	6*	PARAMETER IICOGA=OMEGAN/NCARS, LESSI=NCARS-1, PLUSI=NCARS-1	MGCOMM	*INCLUDED*
00101	7*	PARAMETER LLBYNK=LL	MGCOMM	*INCLUDED*
00101	8*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	9*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	10*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	11*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	12*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	13*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	14*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	15*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	16*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	17*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	18*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	19*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	20*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	21*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	22*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	23*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	24*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	25*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	26*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	27*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	28*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	29*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	30*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	31*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	32*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	33*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	34*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	35*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	36*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	37*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	38*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	39*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	40*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	41*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	42*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	43*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	44*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	45*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	46*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	47*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	48*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	49*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	50*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	51*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	52*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	53*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	54*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	55*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	56*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	57*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	58*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	59*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	60*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	61*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	62*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	63*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	64*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	65*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	66*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	67*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	68*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	69*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	70*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	71*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	72*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	73*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	74*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	75*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	76*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	77*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	78*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	79*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	80*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	81*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	82*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	83*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	84*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	85*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	86*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	87*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	88*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	89*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	90*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	91*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	92*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	93*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	94*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	95*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	96*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	97*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	98*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	99*	COMMON/NGTABS/	MGCOMM	*INCLUDED*
00101	100*	COMMON/NGTABS/	MGCOMM	*INCLUDED*

```

00143      4*      KURVUL = 8
00144      5*      KCOMM = 0
00145      6*      MCFLECE=1
00146      7*      NAMELIST/PARAM/ KCOMM,KURVUL,ISELEC
00147      8*      READIS,PAPAM,ENDEIO)
00152      9*      10 IF (KCOMM) 50,30,40
00155     10*     30 CONTINUE
00156     11*     CALL MHEAD(IFEED)
00157     12*     IF (IFEED .EQ. 0) CALL MGINPT
00161     13*     W (IERRDP .LE. 0) GO TO 50
00163     14*     WRITE (KPRINT,32)
00165     15*     FORMATTED IERRDP .GT. 0 IN RMMAIN. JOB TERMINATED.)
00166     16*     40 CONTINUE
00167     17*     READ(KCOMM) MGCCEV
00175     18*     READ(KCOMM) MGCCEV
00203     19*     READ (KCOMM) A10,X0,UBIAS,ALPHA,T1
00232     20*     CALL MGET(ISELEC,KURVUL)
00233     21*     CALL MIFAD(KURVUL)
00234     22*     CALL MPRINT
00235     23*     GOTO 50
00236     24*     END
END COMPILATION.**
n MESSAGES **

```

*NEW

BY PRS: * REMAIN

COMPILED BY LEVFL-23.8 C5CX FORTRAN V OH 19 JUN 73 AT 22:21

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00100 1* C-MAIN PROGRAM P*HAIIR
00101 INCLUDE MGCOMH, LIST
00102 PARAMEFEP ICARS=3, ILOCOS=1, JL=3, IAU=3, MEGAN=5, NSUM=10, NT=4, NK=2, MGCOMH *INCLUDED*
00103 PARAMEFEP I*PAR=1
00104 PARAMEFEP L=2, ICARS=NLOCOS, M=3, NCARS=1, LL=INLOCOS+2, RT=2, *NT-1
00105 PARAMEFEP M=EM+ILOCOS+TAUN, MM=MM+OMEGAN, INCTAU=TAUN/ICARS
00106 PARAMEFEP IICOGA=OMEGAN/NCARS, LESS=NCARS-1, PLUS=TAUN/ICARS+J
00107 PARAMEFEP LLBY=IK*LL*NK
00110 COM, IOR/I*IGTARS/
00111 COM, IOR/I*IGTARS/
00112 * XXXX
00113 X SPARFF
00114 COM, IOR/I*MGOUT/
00115 1 *KCAD, KPR, IIT, MATRIX,
00116 2 BIG, SMALL, IEP, ROR, LISTOP,
00117 3 KITTLE(12), MGS, BCU(50),
00118 4 IARAD, IIA, COL, IAR, IOR, KELEM (2), KCOMH
00119 1 *K, I, K, Z
00120 2 IMAX, IMAX, KMAX, DMAX
00121 X SPARCA
00122 * INTEGER
00123 1 I, I, K, Z
00124 2 IMAX, IMAX, KMAX, DMAX
00125 X SPARCA
00126 COM, IOR/I*MGTARS/ MG IOR, MGLUS( 12) 2*
00127 DIMENSION: MGCEOV(74), MGTEOV(
00128 EQUIV ALFICE (MGCEOV, XREAD), (MGTEOV, XXXX )
00129 DATA KRFAD, KPR, IIT, MATRIX, SMALL, BIG/5, 6, 13, I, NE=6, 9999, 0/
00130 REAL A(NM), MM, B(I, J), R(I, J), LL, D(NM), LL, RYRK), AKXO(MM, NK), XO(MM), M11
00131 REAL X1(MM), X2(MM), UPTAS(LL), GRADE(NK), ALPHA(LESS), L1, L11, M1
00132 COM, IOR/I*YILK1/ A, B, D, AKXO, XO, X1, X2, UBTAS, GRADE, ALPHA, VMAX, L1, L11
00133 COM, IOR/I*YILK2/ M1, M11, FVAT1
00134 END
00135 INCLUDE P*COMM, LIST
00136 COM, IOR/I*P*WTARS/
00137 1 IORJ, IIRHS, OBUVAL, NOBJF
00138 COM, IOR/I*P*WTARS/
00139 1 RU ( 60 )
00140 2 BPI ( 1 )
00141 * P*DJMA
00142 COM, IOR/I*P*WTARS/
00143 1 SOC ( 2 ) 20 1
00144 2 POC ( 2 ) 20 1
00145 3 SVEL ( 20 )
00146 4 PVEL ( 20 )
00147 5 STEHI ( 20 )
00148 6 RYERI ( 20 )
00149 7 STELO ( 20 )
00150 8 PTELO ( 20 )
00151 9 POPAD ( 20 )
00152 COM, IOR/I*P*WTARS/
00153 1 SIZAK ( 20 )
00154 2 PURAK ( 20 )
00155 * P*DJIB
00156 COM, IOR/I*P*WTARS/
00157 * SOBJ, SOC100, POC100, SOC200, POC200, PTC
00158 END

```

```

00712 179* DO 190 I = 1, NLOCOS
00715 180* K = M + I
00716 181* 190 B(K,I) = 1.0
00720 182* B(MH,LOCOS + 1) = 1.0
00721 183* B(MH,LOCOS + 2) = 1.0
00722 184* WRITE (6,1007) T1
00725 185* WRITE (6,100) ((B(I,K),K=1,LL),I=1,MMH)
00736 186* DO 200 I = 1,MMH
00741 187* X(I) = X0(I)
00742 188* DO 200 II = 1,LL
00745 189* B(II,II) = B(II,II)
00750 190* KK = LL*(BK-1)
00751 191* DO 205 I = 1,MMH
00754 192* DO 205 II = 1,LL
00757 193* K = KK+II
00760 194* 205 D(I,K) = B(II,II)
00763 195* DO 200 I = 1,JK
00766 196* CALL MXMT (A,X1,X2,MMH,MMH,1,MMH,MMH)
00767 197* CALL MXMT (A,01,B2,MMH,MMH,LL,MMH,MMH)
00770 198* DO 210 II = 1,MMH
00773 199* X(II) = X2(II)
00774 200* DO 210 III = 1,LL
00777 201* B(III,III) = B2(II,III)
01002 202* K = JK-II
01003 203* KK = LL*(BK-I-1)
01004 204* DO 220 II = 1,MMH
01007 205* AKXO(II,K) = X2(II)
01010 206* IF (K,LT,0) GO TO 220
01012 207* DO 219 III = 1,LL
01015 208* KKK = KK+III
01016 209* D(III,KKK) = B2(II,III)
01017 210* 219 CONTINUE
01021 211* 220 CONTINUE
01026 212* WRITE (6,1014)
01029 213* DO 221 K = 1,5
01031 214* K1 = 12*(K-1)+1
01032 215* K3 = 12*K
01033 216* 221 WRITE (6,1012) ((D(I,KK),KK=K),K2),I=1,MMH)
01045 217* WRITE (6,1015)
01047 218* WRITE (6,1013) ((AKXO(I,K),K=1,10),I=1,MMH)
01060 219* WRITE (6,1013) ((AKXO(I,K),K=1,20),I=1,MMH)
01071 220* RETURN
01072 221* END

```

END COMPILATION ** n MESSAGES **
 RELOCATABLE
 09 FEB 71 11:08:44 1 01112604 24 1 (DELETED)
 0 01112634 14 3

```

00430 1104 WRITE (6,1000) ((DEL2(I,K),K=1,M),I=1,M)
00441 1204 DO 70 I = 1,M
00444 1214 DO 70 K = 1,M
00447 1224 DO 70 I = 1,M
00452 1234 DO 70 I = 1,M
00455 1244 CALL MXMLT(DELTA,DELTA,DEL,M,M,M,M,M)
00456 1254 DO 90 K = 1,M
00461 1264 DO 90 KK = 1,M
00464 1274 DO 80 DELTAD(K,KK) = DEL(K,KK)
00470 1284 WRITE (6,1003) T1
00473 1294 WRITE (6,1000) ((DELTAD(I,K),K=1,M),I=1,M)
00507 1314 DO 90 I = 1,M
00512 1324 DEL(I,II) = 0.0
00513 1334 DELTA(I,II) = 0.0
00514 1344 IF (I,II) GO TO 90
00516 1354 DEL(I,I) = 1.0
00517 1364 DELTA(I,II) = 1.0
00520 1374 DO 100 I = 1,M
00523 1384 DO 100 I = 1,M
00526 1394 CALL MXMLT(DEL1,DELTA,DEL3,M,M,M,M,M)
00527 1404 DO 95 II = 1,M
00532 1414 DO 95 III = 1,M
00535 1424 DELTA(II,III) = DEL3(II,III)
00542 1434 DO 100 CALL MXMLT(DEL2,DEL,DELTA,M,M,M,M,M)
00543 1454 WRITE (6,1004) T1
00546 1464 CALL MXMLT(DELTA,DELTA,OMEGA,OMEGA,M,M,L,M,M)
00547 1474 WRITE (6,1005) T1
00560 1484 WRITE (6,700) ((OMEGAD(I,K),K=1,L),I=1,M)
00574 1504 DO 120 I = 1,M
00577 1514 DO 120 K = 1,M
00602 1524 A(I,K) = DELTAD(I,K)
00605 1534 DO 130 I = 1,NILOCOS
00610 1544 K = I + 1
00611 1554 DO 130 II = 1,M
00614 1564 A(II,K) = OMEGAD(II,I)
00617 1574 KKK = M + NILOCOS - INCTAU + 1
00620 1584 DO 140 I = 1,NIICARS
00623 1594 K = KKK + INCTAU + 1
00624 1604 KK = NIICARS + NILOCOS + 1 - I
00625 1614 DO 140 II = 1,M
00630 1624 A(II,K) = OMEGAD(II,KK)
00633 1634 KKK = M + NILOCOS + TAUN - INCOGA + 1
00634 1644 DO 150 I = 1,NIICARS
00637 1654 K = KKK + NIICARS + 1
00640 1664 KK = L + 1 - I
00641 1674 DO 150 II = 1,M
00644 1684 A(II,K) = OMEGAD(II,KK)
00647 1694 DO 170 I = 2,TAUN
00652 1704 K = M + NILOCOS + I
00653 1714 I70 A(K-I,K) = 1.0
00655 1724 DO 180 I = 2,MEGAR
00660 1734 K = M + 1 - I
00661 1744 A(K-I,K) = 1.0
00663 1754 WRITE (6,1006) T1
00666 1764 WRITE (6,1000) ((A(II,K),K=1,20),I=1,MMH)
00677 1774 WRITE (6,1011)
00791 1784 WRITE (6,1090) ((A(II,K),K=1,20),I=1,MMM)

```


1006 FORMAT (1H1, //T16, ' TRUNCATION ERROR IN SERIES APPROXIMATION OF S

1011 FORMAT (1H1, //T16, ' SECONDS', //T16, ' ')

1012 FORMAT (13E10, 3)

1013 FORMAT(10E10, 3)

1014 FORMAT(1H1, ' O-MATRIX FOLLOWS')

1015 FORMAT(1H1, ' AKKO MATRIX FOLLOWS')

T1 = T*(KT + 1)

DELTA(1,2) = 1/0

DELTA(M,2) = -1.0 / SIGMA(NCARS + 1)

DO 10 I = 1, NCARS

DELTA(1,3) = -1.0 / MASS(I)

DELTA(1,3) = -1.0 / SIGMA(I)

IF (I.EQ.1) GO TO 2

DELTA(1,3) = 1.0

DELTA(1,3) = -1.0

DELTA(1,3) = -1.0 / MASS(I)

DELTA(1,3) = -1.0 / MASS(I)

IF (I.EQ. NCARS) GO TO 6

DELTA(1,3) = -1.0 / MASS(I)

DELTA(1,3) = -1.0 / MASS(I)

IF (I.EQ.1) GO TO 10

DELTA(1,3) = -1.0 / MASS(I)

GO TO 10

2 DELTA(1,3) = -1.0 / MASS(I)

6 DELTA(1,3) = -1.0 / MASS(I)

10 CONTINUE

WRITE(6, 1000) ((DELTA(I,K), K=1, M), I=1, M)

DO 20 K = 1, NLOCOS

I = LOCOS(K)

20 OMEGA(1,3) = 1.0 / MASS(I)

DO 30 I = 1, NCARS

K = I + NCARS + NLOCOS

30 OMEGA(1,3) = EIA / SIGMA(I)

I = NLOCOS + JL

OMEGA(M,1) = 1.0 / SIGMA(NCARS+1)

DO 40 I = 1, NCARS

K = I + NCARS + NLOCOS

40 OMEGA(1,3) = -1.0 / MASS(I)

WRITE(6, 1002)

WRITE(6, 700) ((OMEGA(I,K), K=1, M), I=1, M)

DO 50 I = 1, M

DO 60 I = 1, NISUM

TI = T / TI

CALL MXADD(DELTA, DEL1, M, M, M)

CALL MXSCA(DELTA, M, M, M, TT)

CALL MXADD(DELTA, DEL2, M, M, M)

CALL MXM(TDELTA, DELTA, DEL, M, M, M, M, M)

DO 60 K = 1, M

60 DELTA(K, KK) = DEL(K, KK)

WRITE (6, 1003) T

WRITE (6, 1000) ((DELTA(I,K), K=1, M), I=1, M)

WRITE (6, 1003) T

WRITE (6, 1000) ((DEL(I,K), K = 1, M), I=1, M)

WRITE (6, 1004) T

WRITE (6, 1004) T

BY FRIS MGINTP

COMPILED BY LEVEL 21.8 CSXK FORTRAN V ON 19 JUN 73 AT 22:21

SUBROUTINE JMSUFR

C-THIS S-R READS CURRENT MAXIMUMS FROM FILE NUMBER

INCLUDE MGCONH

NAMELIST/NGSUF/

1 IMAX ,IMAX ,KMAX ,DMAX

X SPAPEA

READ(13,NGSUF)

RETURN

ENTRY MGINTP

C-THIS S-R READ EXTERNAL VALUES FROM CARDS AND PRINT THE DATA

CALL PREPIO

-CARDS

C-XXXX

IF (MGSUCH(1).EQ.2) GOTO 110

CALL MGLINE(1)

WRITE(KPINT, 115)

115 FORMAT(2X,

* 6HXXX /2X,

110 CONTINUE

125 EDENAT(286, F5.0)

CALL CHFCIR(KCH,IN,6HXXXX , 1)

IF IGE JCH(1).EQ.21 GOTO 120

WRITE(KPINT, 135) KCH,IN, XXXX

135 FORMAT(2X,2A6, F7.0)

120 CONTINUE

STOP

110 CONTINUE

125 EDENAT(286, F5.0)

CALL CHFCIR(KCH,IN,6HXXXX , 1)

IF IGE JCH(1).EQ.21 GOTO 120

WRITE(KPINT, 135) KCH,IN, XXXX

135 FORMAT(2X,2A6, F7.0)

120 CONTINUE

STOP

110 CONTINUE

125 EDENAT(286, F5.0)

CALL CHFCIR(KCH,IN,6HXXXX , 1)

IF IGE JCH(1).EQ.21 GOTO 120

WRITE(KPINT, 135) KCH,IN, XXXX

135 FORMAT(2X,2A6, F7.0)

120 CONTINUE

STOP

110 CONTINUE

125 EDENAT(286, F5.0)

CALL CHFCIR(KCH,IN,6HXXXX , 1)

IF IGE JCH(1).EQ.21 GOTO 120

WRITE(KPINT, 135) KCH,IN, XXXX

135 FORMAT(2X,2A6, F7.0)

120 CONTINUE

STOP

110 CONTINUE

125 EDENAT(286, F5.0)

CALL CHFCIR(KCH,IN,6HXXXX , 1)

IF IGE JCH(1).EQ.21 GOTO 120

WRITE(KPINT, 135) KCH,IN, XXXX

135 FORMAT(2X,2A6, F7.0)

120 CONTINUE

STOP

```

0512 2390 DO 31200 K = 1, KMAX
0515 2404 CALL ITORCD(IAMROW,K ,12, 6)
0516 2414 CALL ITORCD(IAMROW,K ,25,12)
0517 2424 CALL MFLMT( C09(DUMMY), 0, 0 )
0520 2434 31290 CONTINUE
0522 2444 C
0522 2454 C ROW(S) FOR TC
0522 2464 C
0522 2474 C NAMPOW = 6HTC
0524 2484 CALL MFLMT( C06(DUMMY), 0, 0 )
0525 2504 C
0525 2514 C ROW(S) FOR VEL
0525 2524 C
0526 2534 C NAMPOW = 6HVEL
0527 2544 DO 31440 K = 1, KMAX
0532 2554 CALL ITORCD(IAMROW,K ,24,12)
0533 2564 CALL MFLMT( C09(DUMMY), 0, 0 )
0534 2574 31440 CONTINUE
0534 2584 C
0534 2594 C ROW(S) FOR TEHI
0534 2604 C
0536 2614 C NAMPOW = 6HTEHI
0537 2624 DO 31510 K = 1, KMAX
0543 2634 CALL ITORCD(IAMROW,K ,25,12)
0544 2654 CALL MFLMT( C12(DUMMY), 0, 0 )
0544 2664 31510 CONTINUE
0544 2674 C
0544 2684 C ROW(S) FOR TELO
0546 2694 C
0546 2704 C NAMPOW = 6HTELO
0547 2714 DO 31580 K = 1, KMAX
0552 2724 CALL ITORCD(IAMROW,K ,24,12)
0553 2734 CALL MFLMT( C15(DUMMY), 0, 0 )
0554 2744 31580 CONTINUE
0554 2754 C
0554 2764 C ROW(S) FOR GRAD
0554 2774 C
0554 2784 C NAMPOW = 6HGRAD
0556 2794 DO 31650 K = 1, KMAX
0562 2804 CALL ITORCD(IAMROW,K ,24,12)
0563 2814 CALL MFLMT( C16(DUMMY), 0, 0 )
0564 2824 31650 CONTINUE
0564 2834 C
0564 2844 C ROW(S) FOR BRAK
0566 2854 C
0566 2864 C NAMPOW = 6HBRAK
0572 2874 DO 31720 K = 1, KMAX
0573 2884 CALL ITORCD(IAMROW,K ,25,12)
0574 2894 CALL MFLMT( C17(DUMMY), 0, 0 )
0576 2904 31720 WRITE(MATRIX,24)
0600 2914 RETURN
0601 2924 EJD

```

END COMPILATION ** n MESSAGES **

00432	179*	C	OBJ	ROW	
00432	180*	C			
00432	181*	C			
00433	182*				
00434	183*				
00436	184*				
00437	185*				
00437	186*	C			
00437	187*	C			
00437	188*	C			
00440	189*				
00441	190*				
00441	191*				
00441	192*				
00444	193*	C			
00444	194*	C			
00444	195*	C			
00445	196*				
00446	197*				
00450	198*				
00451	199*				
00451	200*	C			
00451	201*	C			
00451	202*	C			
00452	203*				
00453	204*				
00455	205*				
00460	206*				
00463	207*				
00468	208*				
00468	209*				
00466	210*				
00470	211*				
00472	212*				
00473	213*				
00473	214*				
00473	215*	C			
00473	216*	C			
00475	217*				
00475	218*	C			
00475	219*	C			
00475	220*	C			
00477	221*				
00500	222*				
00501	223*				
00501	224*	C			
00501	225*	C			
00501	226*	C			
00502	227*				
00503	228*				
00503	229*	C			
00503	230*	C			
00503	231*	C			
00504	232*				
00505	233*	C			
00505	234*	C			
00505	235*	C			
00505	236*	C			
00506	237*				
00507	238*				

NAMROW = 6H0BJ
 IFIZ = .NE. 1160 TO 16020
 CALL MGFLMT(1.0 , 0.0 , 0.0)
 16020 CONTINUE

OC1000 ROW
 NAMROW = 6HOC1000
 IFIZ = .NE. 1160 TO 16040
 CALL MGFLMT(-1.0 , 0.0 , 0.0)
 16040 CONTINUE

OC2000 ROW
 NAMROW = 6HOC2000
 IFIZ = .NE. 1160 TO 16060
 CALL MGFLMT(-1.0 , 0.0 , 0.0)
 16060 CONTINUE

OC ROW
 NAMROW = 6HOC
 IFIZ = .NE. 1160 TO 16080
 DO 16078 K = 1, NMAX
 DO 16076 K = 1, KMAX
 CALL ITRCDD(NAMROW,K , 12, 6)
 CALL ITRCDD(NAMROW,K , 12, 12)
 CALL MGFLMT(-1.0 , 0.0 , 0.0)
 16076 CONTINUE
 16078 CONTINUE

FIRST B
 WRITE(MATRIX,23)
 C-SET MGID TO 1001 FOR FIRST B SECTION

MGID = 1001
 NAMCOL = 6H
 NAMID = 6H
 ROW(S) FOR OC1000

NAMROW = 6HOC1000
 CALL MGFLMT(COL(DUMMY), 0 , 0)
 ROW(S) FOR OC2000

NAMROW = 6HOC2000
 CALL MGFLMT(COL(DUMMY), 0 , 0)
 ROW(S) FOR OC
 NAMROW = 6HOC
 DO 31300 JL = 1, NMAX

00334	1190	NAMROW = 6H0C	
00335	1200	DO 938 H	= 1, KMAX
00340	1210	DO 938 K	= 1, KMAX
00343	1220	CALL ITORCD(NAMROW,N	,12, 6)
00344	1230	CALL ITORCD(NAMROW,K	,24,12)
00345	1240	CALL MGE LMT(C03(DUMM Y), 0, 0)	
00346	1250	938 CONTINUE	
00350	1260	938 CONTINUE	
00350	1270	E	
00350	1280	C	
00350	1290	C	
00352	1300	C	
00353	1310	NAMROW = 6HTC	
00353	1320	CALL MGE LMT(C05(DUMM Y), 0, 0)	
00353	1330	C	
00353	1340	C	
00353	1350	C	
00354	1360	NAMROW = 6HVEL	
00355	1370	DO 978 K	= 1, KMAX
00360	1380	CALL ITORCD(NAMROW,K	,24,12)
00362	1390	CALL MGE LMT(C08(DUMM Y), 0, 0)	
00362	1400	978 CONTINUE	
00362	1410	C	
00362	1420	C	
00362	1430	C	
00364	1440	NAMROW = 6HTEH	
00365	1450	DO 998 K	= 1, KMAX
00370	1460	CALL ITORCD(NAMROW,K	,24,12)
00371	1470	CALL MGE LMT(C11(DUMM Y), 0, 0)	
00372	1480	998 CONTINUE	
00372	1490	C	
00372	1500	C	
00374	1510	NAMROW = 6HTELO	
00375	1520	DO 1018 K	= 1, KMAX
00400	1530	CALL ITORCD(NAMROW,K	,24,12)
00401	1540	CALL MGE LMT(C18(DUMM Y), 0, 0)	
00402	1550	1018 CONTINUE	
00402	1560	C	
00402	1570	C	
00402	1580	C	
00404	1590	GRAD .FON	
00405	1600	NAMROW = 6HGRAD	
00410	1610	DO 1038 K	= 1, KMAX
00414	1620	CALL ITORCD(NAMROW,K	,24,12)
00412	1630	CALL MGE LMT(C18(DUMM Y), 0, 0)	
00412	1640	1038 CONTINUE	
00412	1650	C	
00412	1660	C	
00412	1670	C	
00414	1680	NAMROW = 6HBRAC	
00415	1690	DO 1058 K	= 1, KMAX
00420	1700	CALL ITORCD(NAMROW,K	,24,12)
00421	1710	CALL MGE LMT(C19(DUMM Y), 0, 0)	
00422	1720	1058 CONTINUE	
00424	1730	1598 CONTINUE	
00424	1740	C	
00424	1750	C	
00424	1760	C	
00427	1770	PI VARIABLES	
00427	1780	NAMCOL = 6HP	
00427	1790	DO 81078 Z	= 1, DMAX
00432	1800	CALL ITORCD(NAMCOL,Z	,18, 6)

00227	59*	NARROW = 6HVEL	
00230	60*	DO 610 K	
00233	61*	CALL ITORCD(IIAMROW,K	,24,12)
00234	62*	IFLAG = 6H	
00235	63*	WRITE(MATRIX,25) IFLAG,NARROW	
00241	65*	610 CONTINUE	
00241	66*	C	
00241	67*	ROW(S) FOR TEHI	
00243	68*	C	
00244	69*	NARROW = 6HTEHI	
00247	70*	DO 620 K	
00250	71*	CALL ITORCD(IIAMROW,K	,24,12)
00251	72*	IFLAG = 6H	
00255	73*	WRITE(MATRIX,25) IFLAG,NARROW	
00255	74*	680 CONTINUE	
00255	75*	C	
00255	76*	POW(S) FOR TEL0	
00257	77*	C	
00260	78*	NARROW = 6HTELO	
00263	79*	DO 750 K	
00264	80*	CALL ITORCD(IIAMROW,K	,24,12)
00265	81*	IFLAG = 6H	
00271	82*	WRITE(MATRIX,25) IFLAG,NARROW	
00271	83*	750 CONTINUE	
00271	84*	C	
00271	85*	ROW(S) FOR GRAD	
00273	86*	C	
00274	87*	NARROW = 6HGRAD	
00277	88*	DO 820 K	
00300	89*	CALL ITORCD(IIAMROW,K	,24,12)
00301	90*	IFLAG = 6H	
00305	91*	WRITE(MATRIX,25) IFLAG,NARROW	
00305	92*	820 CONTINUE	
00305	93*	C	
00305	94*	POW(S) FOR BRAK	
00305	95*	C	
00307	96*	NARROW = 6HBRAK	
00310	97*	DO 390 K	
00313	98*	CALL ITORCD(IIAMROW,K	,24,12)
00314	99*	IFLAG = 6H	
00315	100*	WRITE(MATRIX,25) IFLAG,NARROW	
00321	101*	890 CONTINUE	
00321	102*	C	
00321	103*	MATRIX	
00323	104*	C	
00323	105*	WRITE(MATRIX,22)	
00323	106*	C	
00323	107*	SET NGRID TO 0 FOR MATRIX SECTION	
00323	108*	C	
00325	109*	MGRID = 0	
00326	110*	NAMID = 6H	
00326	111*	C	
00326	112*	U VARIABLES	
00326	113*	C	
00327	114*	NAMCOL = 6H1	
00330	115*	DO 1590 I	
00333	116*	CALL ITORCD(IIAMCOL,I	,12,12)
00333	117*	C	
00333	118*	OC	
00333	119*	C	
00333	120*	ROW	
00333	121*	C	

```

08 PR5.8 NGOUT
COMPILED BY LEVEL 23.8 CSCK FORTRAN V ON 19 JUN 73 AT 22:21
00101 1a SUBROUTINE 4GOUT
00193 2a INCLUDE MGCONM
00135 3a 20 FORMAT(1H,12A6)
00136 4a 21 FORMAT(6HROW ID ,72X)
00137 5a 22 FORMAT(6HMATRIX ,72X)
00180 6a 23 FORMAT(7HFIRST B,72X)
00181 7a 24 FORMAT(6HNEOF ,72X)
00182 8a 25 FORMAT(6X,2A6)
00183 9a 32 FORMAT(12X,6HSETEND)
00183 10a C
00184 11a WRITE(MATRIX,20) KTITLE
00184 12a C
00184 13a C ROW ID
00184 14a C -----
00182 15a WRITE(MATRIX,21)
00182 16a C
00152 17a C--SET MGINID TO -1000 FOR ROW ID SECTION
00152 18a C
00154 19a MGINID = -1000
00154 20a C
00154 21a C ROW(S) FOR OBJ
00154 22a C
00155 23a NARROW = 6HOBJ Z
00156 24a IFLAG = 6H
00157 25a WRITE(MATRIX,25) IFLAG,NARROW
00157 26a C
00157 27a C ROW(S) FOR OC1000
00157 28a C
00163 29a NARROW = 6HOC1000
00164 30a IFLAG = 6H
00165 31a WRITE(MATRIX,25) IFLAG,NARROW
00165 32a C
00165 33a C ROW(S) FOR OC2000
00165 34a C
00171 35a NARROW = 6HOC2000
00172 36a IFLAG = 6H
00173 37a WRITE(MATRIX,25) IFLAG,NARROW
00173 38a C
00173 39a C ROW(S) FOR OC
00173 40a C
00177 41a NARROW = 6HOC
00200 42a DO 470 N = 1, NMAX
00203 43a DO 460 K = 1, KMAX
00206 44a CALL ITOBCE(MNARROW,H ,12, 6)
00207 45a CALL ITOBCE(MNARROW,K ,25,12)
00210 46a IFLAG = 6H
00211 47a WRITE(MATRIX,25) IFLAG,NARROW
00215 48a 460 CONTINUE
00217 49a 470 CONTINUE
00217 50a C
00217 51a C ROW(S) FOR TC
00217 52a C
00221 53a NARROW = 6HTC
00222 54a IFLAG = 6H
00223 55a WRITE(MATRIX,25) IFLAG,NARROW
00223 56a C
00223 57a C ROW(S) FOR VEL
00223 58a C

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BT FR578 MGHAIN
COMPILED BY LEVEL 23.8 CSCK FORTRAN V ON 19 JUN 73 AT 22:21
00100 10 C-MAIN PROGRAM MGHAIN
00101 20 INCLUDE MGCMM
00134 30 CALL MGHED(IFEHD)
00135 40 IF(IFEHD.EQ.0) CALL MGINPT
00137 50 IF(IEERR.EQ.0) GO TO 20
00141 60 WRITE (KPIHT,10)
00143 70 FORMAT(4H IEERROR ,6T, 0 IN MGHAIN. JOB TERMINATED.)
00146 80 STOP
00145 90
00146 100 CALL MGOIT
00150 110 IF(MATRIX.EQ.6) GO TO 50
00151 120 END FILE MATRIX
00151 120 REWIND MATRIX
00152 130 IF(KCOMM.EQ.0) STOP
00154 140 WRITE(KCOMM)MGCEQV
00162 150 WRITE(KCOMM)MGTEQV
00170 160 WRITE (KCOMM) A'B,XO,UBIAS,ALPHA,T1
00170 170 END FILE KCOMM
00220 180 REWIND KCOMM
00221 190 STOP
00222 200 END
ENL COMPILATION ** 0 MESSAGES **

```


BI POP P COM

UNIVAC 1108 PROCEDURE DEFINITION PROCESSOR DATED MAY 5, 1969

THIS PROC ELEMENT PROCESSED ON 19 JUN 73 AT 22:21:20

PROC ORIGIN 1 ENTRY POINT 1 RWCOMM FCOPY

COMMON/RWTABS/
1 IOBJ IRHS OBJVAL NOBJE

COMMON/RWTABS/

1 RU 1 60)

2 BPI 1 1)

* RWDUMA

COMMON/RWTABS/

1 SOC 1 2)

2 POC 1 2)

3 SVEL 1 20)

4 PVEL 1 20)

5 STEH 1 20)

6 PYEH 1 20)

7 STELO 1 20)

8 PLELO 1 20)

9 PGRAD 1 20)

COMMON/RWTABS/

1 SRRAK 1 20)

2 PBRAB 1 20)

* RWDUMR

COMMON/RWTABS/

* SOBJ 150C100,POC100,SOC200,POC200,PTC

END

PROCESSING TIME = 0 SECONDS

BI POP, MGCOM

UNIVAC 1104 PROCEDURE DEFINITION PROCESSOR DATED MAY 5, 1969
THIS PROCEDURE WAS PROCESSED ON 19 JUN 73 AT 22:21:19

1. HPOC ORIGIN 1 ENTRY POINT 1 MGCOM F COPY

```

2. PARAMETER NCARS=3, NLOCOS=1, JL=3, TAUN=3, OMEGAN=6, HSUM=10, HT=4, H=20
3. PARAMETER MBAR=1
4. PARAMETER L=2, NCARS=NLOCOS, NE3=NCARS+1, LL=NLOCOS+2, KJ=2+HT-1
5. PARAMETER MM=H+NLOCOS+TAUN, MM=MM+OMEGAN, INC TAUN/NCARS
6. PARAMETER INCOG=OMEGAN/NCARS, LESS1=NCARS-1, PLUS1=NCARS+1
7. PARAMETER LLBYNK=LL*MK
8. COMMON/MGTABS/
9. COMMON/MGTABS/
10. * XXXX
11. X SPAREC
12. COMMON/MGCONT/
13. 1 *KREAD, *KPRINT, MATRIX,
14. 2 BIG, *SMALL, *MERROR, LISTOP,
15. 3 *TITLE(1), *MGSWCH(50),
16. 4 *MAMID, *MARCOL, *NAMROW, *KELEM (2), *KCOMM
17. COMMON/MGTABS/
18. 1 H, I, K, Z
19. 2 NMAX, IMAX, KMAX, DMAX
20. X SPAREA
21. INTEGER
22. 1 N, K, Z
23. 2 NMAX, IMAX, KMAX, DMAX
24. X SPAREA
25. COMMON/MGTABS/ MGIND, MGLUS( 12)
26. DIMENSION MGCEV(74), MGTEQV( 24)
27. EQUIVALENCE (MGCEV, KREAD), (MGTEQV, XXXX )
28. DATA KREAD, KPRINT, MATRIX, SMALL, BIG/5, 6, 1571, 0E-6, 9999.07
29. REAL A(MM, MM), R(MM, LL), D(MM, LL), YNK, AKXO(MM, MK), XO(MM) /M1
30. REAL XI(MM), X2(MM), UBIAS(LL), GRADE(MK), ALPHA(LFSS), LI, LI1, M1
31. COMMON/MYFLK1/ A, B, D, AKXO, XO, X1, X2, UBIAS, GRADE, ALPHA, VMAX, LI, LI1
32. COMMON/MYBLK2/ M1, M11, FV, T1
33. END

```

PROCESSING TIME = 1 SECONDS

8 XOT CUR

1. IN B

END FILE --- UNIT B

2. IN A

END FILE --- UNIT A

END CUR

95 COMMON STORAGE LOCATIONS IN HG

426 COMMON STORAGE LOCATIONS IN RW

15

1

1

FUNCTION F08(I,K)

F08 = 0.0

KK = LL*(NK-K)+I

IF (KK.GT.LL*YK) GO TO 10

F08 = D(2*KK) - D(11*KK)/M1

10 RETURN

C

FUNCTION F09(K)

F09 = 0.0

KK = LL*(NK-K)

DO 10 II = 1,K

DO 10 III = 1,LL

KKK = KK + LL*(II-1) + III

10 F09 = F09 + (D(2*KKK) - D(11*KKK)/M1)*SUBJAS(III)

KK = NK-K+1

F09 = F09 + M1/M2 - LAKKO(2*KK) - AKKO(11*KK)/M1

RETURN

C

FUNCTION F10(I,K)

F10 = 0.0

IF (I.NE.K*LL) RETURN

F10 = 1.0

RETURN

C

FUNCTION F11(I,K)

F11 = 0.0

IF (I.NE.K*LL-1) RETURN

F11 = 1.0

RETURN

END

FUNCTION F05(K)

F05 = 0.0

KK = LL*(NK-K)

DO 10 II = 1,K

DO 10 III = 1,LL

KKK = KK + LL*(II-1) + III

10 F05 = F05 + D(2,KKK)*UBIAS(III)

KK = NK-K+1

F05 = F05 + VMAX - AKXO(2,KK)

RETURN

FUNCTION F06(I,K)

F06 = 0.0

KK = LL*(NK-K)+I

IF (KK.GT.LL*NYKI) GO TO 10

F06 = D(2,KK) + D(11,KK)/LI

10 RETURN

FUNCTION F07(K)

F07 = 0.0

KK = LL*(NK-K)

DO 10 II = 1,K

DO 10 III = 1,LL

KKK = KK + LL*(II-1) + III

10 F07 = F07 + (6(2,KKK) + D(11,KKK)/LI)*UBIAS(III)

KK = NK-K+1

F07 = F07 + LII/LI - (AKXO(2,KK) + AKXO(11,KK)/LI)

RETURN

IF (KK.GT.LL)YK) 60 TO 10
F01 = D(3*N+1,K)
10 RETURN

FUNCTION F02(H,K)
F02 = 0.0
KK = LL*(NK-K)

DO 10 II = 1,K
DO 10 III = 1,LL

KKK = KK*LL*(II-1)*III

10 F02 = F02 + D(3*N+1,KKK)*(BIAS(III))

KK = NK-K+1

F02 = F02 - AKX0(3*N+1,KK)

RETURN

FUNCTION F03(Z)

F03 = 0.0

DO 10 II = 1,NK

DO 10 III = 1,LL

KK = LL*(II-1)+III

10 F03 = F03 + D(1,KK)*(BIAS(III))

F03 = F03 + FV - AKX0(1,1)

RETURN

FUNCTION F0A(I,K)

F0A = 0.0

KK = LL*(NK-K)+I

IF (KK.GT.LL)YK) 60 TO 10

F0A = D(2,KK)

10 RETURN

SUM(I) C14 * U(I) .EQ. C16

FOR ALL K

BRAK

SUM(I) C19 * U(I) .LE. C17

FOR ALL K

ELEMENTS

C01 = -X0(N)

C02 = -X0(7)

C03 = F01(N,I,K)

C04 = F02(N,K)

C05 = D(1,1)

C06 = F03(2)

C08 = F04(I,K)

C09 = F05(K)

C11 = F06(I,K)

C12 = F07(K)

C14 = F08(L,K)

C15 = F09(K)

C16 = UBIAS(3) * 0.01 * GRADE(K)

C17 = UBIAS(2) + 23.0

C18 = F10(I,K)

C19 = F11(I,K)

FUNCTIONS

FUNCTION F01(N,I,K)

F01 = 0.0

KK = LL * (NK - K) + I

PROBLEM

MINIMIZE

DUMMY LP VARIABLE PI

* OBJ *****
1.0 * PI(1)

SUBJECT TO

* OC1080 *****
-1.0 * PI(1) .LE.C01

* OC2000 *****
-1.0 * PI(1) .LE.C02

* OC *****
SUM(I) C03 * U(I) - 1.0 * PI(1) .LE.C03

FOR ALL N,K

* TC *****
SUM(I) C05 * U(I) .EQ.C06

* VEL *****
SUM(I) C05 * U(I) .LE.C09

FOR ALL K

* TEHI *****
SUM(I) C11 * U(I) .LE.C12

FOR ALL K

* TELO *****
SUM(I) C14 * U(I) .LE.C15

FOR ALL K

* GRAD *****

KKK = KK+III
D(II, KKK) = D2(II, III)

219 CONTINUE

220 CONTINUE

WRITE (6, I14)

DO 221 K = 1, 5

K1 = 12*(K-1)+1

K2 = 12*K

221 WRITE (6, I12) ((D(I, KKK), KK=K1, K2), I=1, MMM)

WRITE (6, I13)

WRITE (6, I13) ((AKXO(I, K), K=1, 10), I=1, MMM)

WRITE (6, I13) ((AKXO(I, K), K=11, 20), I=1, MMM)

RETURN

WRITE (6,1000) ((A(I,K),K=1,20),I=1,MMN)

DO 190 I = 1,MLCOS

K = M + I

190 B(K,I) = 1.0

B(MN,MLCOS + 1) = 1.0

B(MN,MLCOS + 2) = 1.0

WRITE (6,1007) T1

WRITE (6,300) ((B(I,K),K=1,LL),I=1,MMN)

DO 200 I = 1,MMN

XI(I) = X0(I)

DO 200 II = 1,LL

200 B(I,II) = B(II,II)

KK = LL*(IK-1)

DO 205 I = 1,MMN

DO 205 II = 1,LL

K = KK+II

205 D(I,K) = B(II,II)

DO 220 I = 1,PK

CALL MEXPL(A,X1,X2,MMN,MMN,I,MMN,MMN)

CALL MEXPL(A,U1,82,MMN,MMN,LL,MMN,MMN)

DO 210 II = 1,MMN

XI(II) = X2(II)

DO 210 III = 1,LL

210 B(II,III) = B2(II,III)

K = NK+II

KK = LL*(NK-1)

DO 220 II = 1,MMN

MSQ(II,K) = X2(II)

IF (KK.LT.0) GO TO 220

DO 219 III = 1,LL

C SET UP A AND B MATRICES

DO 120 I = 1,M

DO 120 K = 1,M

120 A(I,K) = DELTAD(I,K)

DO 130 I = 1,NLOCOS

K = I + M

DO 130 II = 1,M

130 A(II,K) = OMEGAD(II,I)

KKK = M + NLOCOS - I(CTAU + I

DO 140 I = 1,NCARS

K = KKK + I(CTAU * I

KK = NCARS + NLOCOS + I - I

DO 140 II = 1,M

140 A(II,K) = OMEGAD(II,KK)

KKK = M + NLOCOS + TAUH - INCOGA + I

DO 150 I = 1,NCARS

K = KKK + I(INGA * I

KK = L + I - I

DO 150 II = 1,M

150 A(II,K) = OMEGAD(II,KK)

DO 170 I = 2,TAUW

K = M + NLOCOS + I

170 A(K-1,K) = 1.0

DO 180 I = 2,OMEGAN

K = M + I

180 A(K-1,K) = 1.0

WRITE (6,1006) T1

WRITE (6,1000) ((A(I,K),K=1,10),I=1,MMH)

WRITE (6,1011)

80 DELTA(K,KK) = DEL(K,KK)

WRITE (6,1005) T1

WRITE(6,1000) ((DELTA(I,K),K=1,M),I=1,M)

C CALCULATE INTEGRAL OF STM FOR T1 IN DELTA

DO 90 I = 1,M

DO 90 II = 1,M

DEL(I,II) = 0.0

DELTA(K,II) = 0.0

IF II.NE.111 GO TO 90

DEL(I,I) = 1.0

DELTA(I,I) = 1.0

90 CONTINUE

DO 100 I = 1,KT

CALL MXMULT(DEL1,DELTA,DEL3,M,M,M,R)

DO 95 II = 1,M

DO 95 III = 1,M

95 DELTA(III,III) = DEL3(II,III)

100 CALL MXADD(DEL,DELTA,DEL,M,M,M)

CALL MXMULT(DEL2,DEL,DELTA,M,M,M,M,M)

WRITE (6,1004) T1

WRITE(6,1000) ((DELTA(I,K),K=1,M),I=1,M)

C CALCULATE DISCRETE CONTROL MATRIX

CALL MXMULT(DELTA,OMEGA,OMEGA,M,M,L,M,M)

WRITE (6,1005) T1

WRITE(6,700) ((OMEGA(I,K),K=1,L),I=1,M)

CALCULATE DEL2 = INTEGRAL OF SYM FOR T

DO 50 I = 1,M

50 DELTAD(I,I) = 1.0

DO 60 J = 1,NISUM

TI = I

TV = T / TI

CALL MKADD(DELTA,DEL1,DEL1,M,M,M)

CALL MKSCA(DELTA,M,M,M,TT)

CALL MKADD(DELTA,DEL2,DEL2,M,M,M)

CALL MKPLT(DELTA,DELTA,DEL,M,M,M,M,M)

DO 60 K = 1,H

DO 60 KK = 1,H

60 DELTAD(K,KK) = DFL(K,KK)

WRITE (6,1003) T

WRITE (6,1000) ((DELTAD(I,K),K=1,M),I=1,M)

WRITE (6,1003) T

WRITE (6,1000) ((DELT(I,K),K=1,M),I=1,M)

WRITE (6,1004) T

WRITE (6,1000) ((DEL2(I,K),K=1,M),I=1,M)

CALCULATE 5TH FOR TI

DO 70 I = 1,M

DO 70 K = 1,M

70 DELTAD(I,K) = DEL(I,I,K)

DO 80 I = 1,NIT

CALL MKPLT(DELTA,DELTA,DEL,M,M,M,M,M)

DO 80 K = 1,H

DO 80 KK = 1,H

```

IF (1.EQ.NCARS) GO TO 6
4 DELTA(3*I-1,3*I+1) = -ALPHA(I) / MASS(I)
  DELTA(3*I-1,3*I+2) = BETA(I) / MASS(I)
IF (1.EQ.1) GO TO 10
  DELTA(3*I-1,3*I-1) = -(BETA(I-J)+BETA(I))/MASS(I)-G*LAMBDA(I)
  GO TO 10
2 DELTA(3*I-1,3*I-1) = -BETA(I) / MASS(I) - G*LAMBDA(I)
  GO TO 4
6 DELTA(3*I-1,3*I-1) = -BETA(I-1) / MASS(I) - G*LAMBDA(I)
10 CONTINUE
WRITE(6,1001)
WRITE(6,1000) ((DELTA(I,K),K=1,N),I=1,M)
C
C SET UP OMEGA MATRIX
C
DO 20 K = 1,NLOCOS
  I = LOCON(K)
20 OMEGA(3*I-1,K) = 1.0 / MASS(I)
DO 30 I = 1,NCARS
  K = I + NLOCOS
30 OMEGA(3*I,K) = ETA / SIGMA(I)
  I = NLOCOS + 1
  OMEGA(M,I) = 1.0 / SIGMA(NCARS+1)
DO 40 I = 1,NCARS
  K = I + NCARS + NLOCOS
40 OMEGA(3*I-1,K) = -6
  WRITE(6,1002)
  WRITE(6,700) ((OMEGA(I,K),K=1,L),I=1,M)
C
C CALCULATE DELTA = STM FOR T

```



```

      * T = 0.0773 SECONDS'////////
1004 FOPMAT (IH1//T31,'INTEGRAL OF SYSTEM TRANSITION MATRIX FOR T =',
      *F7.3' SECONDS'////////)
1005 FOPMAT (IH1//T14,'CONTROL MATRIX OF DISCRETE SYSTEM FOR T =',
      *F7.3' SECONDS'////////)
1006 FOPMAT (IH1//T28,'AUGMENTED SYSTEM MATRIX OF DISCRETE SYSTEM FOR
      * T =',F7.3' SECONDS'////////)
1007 FOPMAT (IH1//41H,'AUGMENTED CONTROL MATRIX OF DISCRETE SYSTEM FO
      *R T =',F7.3' SECONDS'////////)
1008 FOPMAT (IH1//T16,'TRUNCATION ERROR IN SERIES APPROXIMATION OF S
      *T M AND ITS INTEGRAL FOR T =',F7.3' SECONDS'////////)
1011 FOPMAT (IH1//T10,'D
1012 FOPMAT (IPE10,3)
1013 FOPMAT(10E10,3)
1014 FOPMAT(IH1,'D MATRIX FOLLOWS')
1015 FOPMAT(IH1,'AKXO MATRIX FOLLOWS')
      TI = T*(KT + 1)
C
C   SET UP DELTA MATRIX
      DELTA(1:2) = 1.0
      DELTA(M,N) = -1.0 / SIGMA(INCAR + 1)
      DO 10 I = 1,NCARS
      DELTA(3*I-1,3*I) = -E(I) / MASS(I)
      DELTA(3*I,3*I) = -1.0 / SIGMA(I)
      IF (I.EQ.1) GO TO 2
      DELTA(3*I-2,3*I-4) = 1.0
      DELTA(3*I-2,3*I-1) = -1.0
      DELTA(3*I-1,3*I-4) = BETA(I-1) / MASS(I)
      DELTA(3*I-1,3*I-2) = ALPHA(I-1) / MASS(I)

```


COMMON/RYBLK1/ A,B,D,AKXO,FXO,X1,X2,UBIAS,GRADE,ALPHA,VMAX,LI,LI1

COMMON/RYBLK2/ MI,MII,EV,TJ

DATA MODIFICATIONS

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