

DYNAMIC ANALYSIS OF SHEAR WALLS AND DISCRETELY
CONNECTED PANEL STRUCTURES USING A SUPER FINITE
ELEMENT TECHNIQUE

Hans-Peter Huttelmaier

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Hans-Peter Huttelmaier

ABSTRACT

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Hans-Peter Huttelmaier, Ph.D.
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The modularity employed by the building industry, apparent in the increased use of large-panel construction, invites the application of substructuring techniques in analysis and design. This results in an efficient and economical use of the finite element method.

The aim of this research is the development of a versatile substructuring technique for dynamic analysis, applicable to shear wall-type structures. Intended applications are precast panel structures and their seismic response.

First, the theoretical background for dynamic substructuring is reviewed. This is followed by the development of a substructuring scheme which is described as a multilevel super-finite element technique, based on a series of static condensations. The "panel element" thus derived consists of a rectangular substructure with interior nodes eliminated and flexible boundary node arrangement. Inertia

characteristics are represented by either a consistent or a lumped mass matrix. A substructure stress-displacement matrix allows full stress recovery. The present coded version for the panel element is attached to a multipurpose linear analysis program (SAP4) in the form of SUBROUTINE PANEL.

Applications in free vibration analysis and stress computation demonstrate the versatility of the scheme as well as a reduction in computational effort. Accuracy is tested with respect to mass representation. It is found that a homogeneous mass distribution may require a consistent substructure mass matrix; however, if concentrated masses exist (e.g., floor masses in multi-storey structures), a lumped mass representation may suffice.

The seismic behaviour of a 12-storey panel structure is investigated in a parametric study, representing an additional application of the substructure technique. A modal analysis was used for this purpose. Forces in vertical joints are transmitted through discrete connectors located at floor levels, whereas horizontal joints are assumed as rigid and continuous. Response is evaluated for structural discontinuities such as monolithic storeys and missing panels, with emphasis on connector forces. The investigation concludes with a comparison of dynamic analysis and static Code loading.

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TABLE OF CONTENTS

TABLE OF CONTENTS

	PAGE
ABSTRACT	i
ACKNOWLEDGEMENTS	iii
LIST OF TABLES	viii
LIST OF FIGURES	x
CHAPTER I	INTRODUCTION
1.1	General 1
1.2	Review of Previous Work 3
1.2,1	Static substructuring 3
1.2.2	Dynamic substructuring 5
1.3	Present Study 8
CHAPTER II	SOLUTION OF EQUILIBRIUM EQUATIONS IN DYNAMIC ANALYSIS USING SUBSTRUCTURING TECHNIQUES
2.1	Introduction 11
2.2	Reduced Equilibrium Equations in Dynamic Analysis 11
2.3	Review of Rayleigh-Ritz Analysis 15
2.4	Static Condensation 18
2.5	Component Mode Synthesis 22
2.6	Super-Finite Element Technique 27
2.6.1	Reduction of Static Equilibrium Equations 28
2.6.2	Reduction of Dynamic Equilibrium Equations 29
2.7	Concluding Remarks 31

	PAGE
CHAPTER III DEVELOPMENT OF A PANEL SUPER ELEMENT	
3.1 Introduction	34
3.2 Numerical Considerations for the Reduction Procedure	36
3.2.1 Block elimination	38
3.2.2 Step elimination	40
3.2.3 Comparison of the two procedures	42
3.3 The Multilevel Growth Scheme	45
3.3.1 General scheme	45
3.3.2 Description of the suggested scheme	46
3.3.2.1 Level zero	47
3.3.2.2 Level one	48
3.3.2.3 Level two	49
3.3.2.4 Level three	50
3.3.2.5 Level four	52
3.4 Stress Recovery	53
3.5 Storage Scheme	57
3.6 Discussion and Concluding Remarks	58
CHAPTER IV APPLICATIONS USING PANEL ELEMENTS	
4.1 Introduction	74
4.2 Plate Bending Vibrations	76
4.2.1 Single panel element as cantilever plate	77
4.2.2 Single panel element as free-free plate	81
4.2.3 Assembly of panel elements as cantilever plate	82
4.3 Structural Problems in Membrane Action	83
4.3.1 Single bay shear wall with varied floor mass intensity	84
4.3.2 Free vibration of a coupled shear wall	87
4.3.3 Static analysis of a coupled shear wall	89
4.3.4 Stress recovery for cantilever beam	90
4.4 Summary	91

CHAPTER V	SEISMIC BEHAVIOUR OF A 12-STOREY PRECAST PANEL STRUCTURE	
5.1	Introduction	116
5.2	Components of Panel Structures	120
5.2.1	Overall structural functioning	120
5.2.2	Panels	122
5.2.3	Joints	122
5.3	Description of Structure and Analytic Considerations	124
5.3.1	Geometric and material properties	124
5.3.2	Connector model and connector stiffnesses	125
5.3.3	Summary of assumptions	127
5.3.4	Response spectrum analysis	129
5.4	Parametric Investigation of Dynamic Behaviour	135
5.4.1	Effect of uniform connector shear stiffness	137
5.4.1.1	Overall structural response	138
5.4.1.2	Forces in connectors	141
5.4.1.3	Stresses in horizontal joints	142
5.4.1.4	Final remarks	144
5.4.2	Effect of irregular distribution of connector shear stiffness, including monolithic storeys	146
5.4.2.1	Non-uniform distribution of connector stiffness	147
5.4.2.2	Monolithic storeys	148
5.4.2.3	Connector forces in structures with monolithic storeys	151
5.4.2.4	Transition in stiffness	152
5.4.3	Effect of missing panels	154
5.4.3.1	Connector forces	156
5.4.3.2	Final remarks	158
5.4.4	Asymmetric floor mass distribution	158
5.4.5	Connector axial stiffness	160

	PAGE
5.4.6 Comparison with code static loading	162
5.4.7 Extension to non linear analysis	167
5.5 Summary of Results	170
CHAPTER VI SUMMARY AND CONCLUDING REMARKS	221
REFERENCES	225
NOTATION	234
APPENDIX A Operation Count and Storage Requirement for Block and Step Elimination	239
APPENDIX B Flow Chart for Reduction Algorithm Using Step Elimination	243
APPENDIX C Instructions for Input and Description of Program Arrangement to Conform with SAP4	245
APPENDIX D FORTRAN Listing of SUBROUTINE PANEL	263

LIST OF TABLES

LIST OF TABLES

NUMBER	DESCRIPTION	PAGE
3.1	Growth patterns	60
3.2	Level One elements	61
3.3	Level Two elements	62
3.4	Coupling in Level Four (Example 2) numbering of degrees of freedom	63
3.5	Dynamic storage arrays for matrices	64
4.1	Comparison of frequencies - cantilever plate	94
4.2	Comparison of normalized mode shapes - cantilever plate	95
4.3	Mode shapes compared to Ref. [41] - cantilever plate	96
4.4	Frequencies and mode shapes compared to Ref. [41] - cantilever plate and differ- ent panel elements	97
4.5	Comparison of frequencies - free-free plate	98
4.6	Frequencies and mode shapes compared to Ref. [41] - cantilever plate as panel assemblies	99
4.7	Natural periods compared to full finite element model - homogeneous single bay shear wall	101
4.8	Mode shapes compared to full finite element model - single bay shear wall with concentrated floor masses	102
4.9	Natural periods compared to full finite element model - coupled shear wall	103

NUMBER	DESCRIPTION	PAGE
5.1	Comparison of connector shear stiffness definition	176
5.2	Absolute response values	177
5.3	Forces in horizontal joints due to vertical dead load and lateral seismic load for different connector shear stiffness, $k_v = 10^n$ kips/ft	178
5.4	Linear stiffness transitions	179
5.5	Smooth stiffness transitions	180
5.6	Ratios for fundamental period and top deflection for structure with panels missing at different floor levels	181
5.7	Force reduction factors μ_v along structure height	182
A.1	Operation count and storage requirement for block eliminations	242

LIST OF FIGURES

LIST OF FIGURES

FIGURE	DESCRIPTION	PAGE
2.1	Typical examples of super finite elements (a) line elements, (b) rectangular elements, and (c) trapezoidal elements	33
3.1	Storage schemes for stiffness and mass matrix	65
3.2	Growth scheme of the level-four panel element	66
3.3	Numbering of 4-node elements; (a) nodal numbering, (b) degrees of freedom for plane stress, and (c) degrees of freedom for plate bending	67
3.4	Coupling in level three (example 1); (a) level one and level two elements, and (b) level three elements	68
3.5	Coupling in level three (example 2); (a) level one and level two elements, and (b) level three element	69
3.6	Coupling in level four (example 1)	70
3.7	Coupling in level four (example 2); (a) nodal numbering for level three element, and (b) nodal numbering for level four element before reduction of internal nodes	71
3.8	Storage scheme for stress-displacement matrix	72
3.9	Example for stress point locations; (a) basic 4-node element, and (b) panel element	73
4.1	Panel element models for free vibration in plate bending; (a) cantilever plate, and (b) free-free plate	104
4.2	Lumping of panel mass in lateral direction; (a) consistent lumping, and (b) lumping by tributary areas	105

FIGURE	DESCRIPTION	PAGE
4.3	Comparison of mode shapes for cantilever plate	106
4.4	Single bay shear wall; (a) full finite element model, and (b) - (d) using panel elements (1 ft. = 0.305 m.)	107
4.5	Effect of floor mass intensity on difference in natural periods compared to the full finite element model	108
4.6	Coupled shear wall for dynamic analysis; (a) full finite element model, and (b) panel substructure representation (1 ft = 0.305 m)	110
4.7	Coupled shear wall for static analysis; (a) full finite element model (from [44]), and (b) panel substructure representation (1 ft = 0.305 m)	111
4.8	Lateral deflection under static loading; left wall (1 ft = 0.305 m)	112
4.9	Isobars of σ_y stresses due to static loading in y lower left pier (1 kip/ft ² = 47.9 kN/m ²)	113
4.10	Cantilever beam under static loading; (a) full finite element model, and (b) panel substructure (1 in = 2.54 cm, 1 kip = 4.45 kN)	114
4.11	Normal stress σ_y and shear stress σ_{xy} at different cross y - sections of cantilever beam (1 in = 2.54 cm, 1 ksi = 6.9 MN/m ²).	115
5.1	Elevation of panel structure	183
5.2	Connector element: (a) idealization, and (b) degrees of freedom	184
5.3	Panel elements and storey arrangements ((a) Type(101); (b) Type(120); (c) Type (101,100/2))	185
5.4	Definition of panel stiffness	186
5.5	Acceleration response spectra for 5 per cent damping	187

FIGURE	DESCRIPTION	PAGE
5.6	Normalized natural periods and modal participation factors as functions of connector shear stiffness (1 kip/ft = 14.6 kN/m)	188
5.7	Normalized base shear and overturning moment as functions of connector shear stiffness (1 kip/ft = 14.6 kN/m)	189
5.8	Normalized top and connector vertical displacements as functions of connector shear stiffness (1 kip/ft = 14.6 kN/m).	190
5.9	Distribution of horizontal storey shears for different connector stiffness k_v normalized to; (a) base shear of monolithic wall, and (b) actual base shear (1 kip/ft = 14.6 kN/m)	191
5.10	Envelopes of lateral displacements for different connector stiffnesses k_v normalized to; (a) top displacement of monolithic wall, and (b) actual top displacement (1 kip/ft = 14.6 kN/m)	192
5.11	Normalized connector shear forces and axial-to-shear force ratios for different stiffness k_v (1 kip/ft = 14.6 kN/m)	193
5.12	Normal forces in horizontal joints for 1 g peak acceleration (El Centro Spectrum) and different connector shear stiffness k_v (1 kip/ft = 14.6 kN/m)	194
5.13	Structures with monolithic storeys: (a) monolithic top storey, (b) monolithic top and mid-height storey, and (c) monolithic mid-height storey	195
5.14	Effect of distribution of connector stiffness k_v on normalized connector shear forces (1 kip/ft = 14.6 kN/m).	196
5.15	Effect of monolithic storeys at top and mid-height on normalized fundamental period (1 kip/ft = 14.6 kN/m)	197

FIGURE	DESCRIPTION	PAGE
5.16	Effect of monolithic storeys at top and mid-height on normalized base shear and overturning moment (1 kip/ft = 14.6 kN/m).	198
5.17	Effect of monolithic storeys at top and mid-height on normalized top deflection (1 kip/ft = 14.6 kN/m)	199
5.18	Effect of monolithic storeys at top and at top and mid-height on deflection envelopes (1 kip/ft = 14.6 kN/m)	200
5.19	Normalized connector shear forces in structures with monolithic storeys for different connector stiffness k_v (1 kip/ft = 14.6 kN/m)	201
5.20	Comparison of linear and smooth stiffness transitions for normalized connector shear forces in structures with monolithic top storey; (a) transition over 3 storeys, and (b) transition over 4 storeys ($k_v = 10^4$ kips/ft (14.6 x 10 ⁴ kN/m))	202
5.21	Effect of length of smooth stiffness transition on normalized connector shear forces in structure with monolithic mid-height storey ($k_v = 10^4$ kips/ft (14.6 x 10 ⁴ kN/m)).	203
5.22	Structures with a panel missing at different levels in wall 2	204
5.23	Structures with a panel missing at different levels in wall 3	205
5.24	Effect of the level of panels missing in wall 3 on normalized horizontal storey shears ($k_v = 10^5$ kips/ft (14.6 x 10 ⁵ kN/m)).	206
5.25	Effect of the level of missing panels on normalized lateral displacements ($k_v = 10^5$ kips/ft (14.6 x 10 ⁵ kN/m))	207
5.26	Effect of a panel missing at level 2-3 in wall 2 on normalized connector forces ($k_v = 10^5$ kips/ft (14.6 x 10 ⁵ kN/m)).	208

FIGURE	DESCRIPTION	PAGE
5.27	Effect of a panel missing at level 6-7 in wall 2 on normalized connector forces ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))	209
5.28	Effect of a panel missing at level 10-11 in wall 2 on normalized connector forces ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))	210
5.29	Effect of a panel missing at level 2-3 in wall 3 on normalized connector forces ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))	211
5.30	Effect of a panel missing at level 6-7 in wall 3 on normalized connector forces ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))	212
5.31	Effect of a panel missing at level 10-11 in wall 3 on normalized connector forces ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))	213
5.32	Effect of asymmetric floor mass distribution on connector axial to shear force ratios (1 kip/ft = 14.6 kN/m)	214
5.33	Influence of connector axial stiffness k_a on normalized fundamental period (1 kip/ft = 14.6 kN/m)	215
5.34	Influence of connector axial stiffness k_a on normalized connector shear forces and axial-to-shear force ratios ($k_v = 10^4$ kips/ft (14.6×10^4 kN/m))	216
5.35	Comparison of base shears for static and dynamic analyses (NBCC-1977) and force reduction factor μ_v (1 kips/ft = 14.6 kN/m, 1 kip = 4.45 kN).	217
5.36	Comparison of horizontal storey shears for static and dynamic analyses (NBCC-1977), (1 kip = 4.45 kN)	218
5.37	Comparison of maximum connector shear forces for static and dynamic analyses (NBCC-1977) and associated force reduction factors (1 kip/ft = 14.6 kN/m, 1 kip = 4.45 kN)	219

FIGURE	DESCRIPTION	PAGE
5.38	Comparison of connector shear force distributions for dynamic and normalized static analyses (1 kip = 4.45 kN)	220

CHAPTER I
INTRODUCTION

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INTRODUCTION

1.1 GENERAL

Tall structures, in particular those represented by shear wall or coupled shear wall systems, have traditionally been analyzed using approximate methods. This implies uniform structural patterns not allowing irregularities in the stiffness or mass distribution along the height of the structure.

Modern digital computers and the advancement of the finite element method allow the analysis of complex civil engineering structures within high accuracy; this includes shear wall-type structures of arbitrary three-dimensional arrangement. Detailed information for structural response can be obtained, such as complete deformation shapes, or the stress concentrations around openings, for example.

However, the prohibitive computational costs for large-size structures urged the reorganization of solution schemes of the conventional finite element analysis, apparent through the appearance of substructuring techniques. Structures with many hundreds, or even thousands, of degrees of freedom, can thus be analyzed economically, particularly

when repetitive structural patterns are present, as occurs in many modern highrise structures.

Savings are particularly noticeable for dynamic analysis, since the solution of the dynamic equilibrium equations - in many cases identical to the solution of an eigenvalue problem - is particularly costly for large structural systems. Reduction of an equation set in dynamic analysis, i.e., "dynamic substructuring" requires, however, greater analytical effort than in the case of static analysis, the actual effort depending on the desired level of refinement in the representation of the mass. In some cases, crude lumping may result in good approximations of dynamic properties. In other cases a "consistent mass matrix" or even "component modes" may need to be applied in a substructuring technique in order to achieve satisfactory results.

Development of new analytical techniques - or adapting available ones to present needs - is a basic requirement in any technological process. The modularity applied by the modern building industry invites the use of substructuring techniques. Moreover, in the development of these techniques, consideration should be given to increased industrialization evident in the form of large-panel constructions, where continuous or discrete panel-to-panel connections may exist. Thus, a substructuring technique should possess the

versatility to be applicable not only to the more conventional "continuous" type of construction but also to precast panel structures where substructures may be employed to represent actual physical panels interconnected at selected points.

Finally, the need to include dynamic procedures in structural analysis programs, namely, to include appropriate mass representation in the substructuring technique, becomes increasingly apparent as building codes begin to treat the true dynamic nature of wind and earthquake loadings.

1.2 REVIEW OF PREVIOUS WORK

In a discussion of substructuring techniques one may consider static substructuring and dynamic substructuring as two different but interrelated developments. Previous work performed in static substructuring, also known as static condensation, will be presented first. An additional review of research applied to shear wall and panel structures is given in Section 5.1.

1.2.1 Static Substructuring

During the early development stages of finite element analysis, static condensation was applied for a single element to eliminate internal degrees of freedom. The use of complete polynomials to express displacement shapes, or in another

case, creation of quadrilateral elements assembled from triangular elements, result in internal nodes [1,2]. As a subsequent development, static condensation was applied to eliminate element degrees of freedom originated by incompatible modes [3]. In all the above cases more refined element behaviour is obtained without increasing the number of degrees of freedom.

The concept of eliminating selected degrees of freedom in a complete structural assembly became popular mainly for frame structures, where degrees of freedom of secondary importance were removed. Eliminated degrees of freedom relate mostly to small deformations (e.g., axial or rotational), or to points where there exist no applied external loads. Primarily work which may be summarized as "matrix structural analysis"[4,5,6] made use of this approach.

A further step in the development of static substructuring is characterized by the more systematic approach where internal degrees of freedom of individual structural components are eliminated; thus, the total structure is composed of substructures and is represented only by the nodal points at the substructure boundaries. These reduced subassemblies are also described as super finite elements or macro elements. Their use is particularly effective when repetitively applied. Some of the more recent work is mentioned below.

Egeland and Araldsen [7] described a general purpose program which allows generation of a super finite element library. This program was utilized to provide large savings in the analysis of ship hulls. A more systematic procedure with automatic partitioning of the structure into substructures performed according to available core memory, was designed by Utku [8]. This scheme allows efficient solution of large structures even on small size computers.

An efficient and refined special purpose program for linear analysis of complex reinforced concrete walls was developed by Peterson and Popov [9,10]. Rectangular substructures, defined by the user, may be divided into any finite element mesh; different material properties as well as openings can be included. Arbitrary loading and flexibility in desired displacement and stress output contribute to the versatility of this program.

1.2.2 Dynamic Substructuring

Most of the above work, although designed for static analysis, may be used for a dynamic analysis if appropriate lumped masses are introduced and if the solution of the equations is extended to a dynamic problem. This approach is used effectively for many practical engineering problems. However, adequate mass representation depends to a large

extent on the experience of the analyst.

Guyan [11] introduced a more refined mass formulation for the reduced subsystem, by subjecting the mass matrix to the same reduction constraints as the stiffness matrix. Thus, inertia characteristics of a substructure can be expressed by an "energy consistent", fully occupied mass matrix which results in an improved representation of the dynamic substructure properties. For an ideal situation, i.e., when complete and identical displacement shapes exist for the element stiffness and mass matrices, [12], upper bound solutions are obtained for the structural system, since the procedure is then equivalent to a Rayleigh-Ritz analysis [13].

Derivation of the consistent substructure mass matrix has been presented by many authors [6,7,15,23]. Applications have been reported by Egelund and Araldsen [7] who describe the use as well as the limitation of the consistent mass matrix, incorporated in a program using the super finite element technique. Goodno and Gere [14,15] investigated concrete shear cores using three-dimensional floor-to-floor subassemblies modelled by a consistent mass formulation. However, a lumped mass representation resulted in the same accuracy, with less computational effort.

The component mode synthesis can be considered as a separate development in dynamic substructuring. In this

scheme additional displacement shapes represented by normal modes of the constrained substructure are superimposed on the consistent formulation. The method was originated by Hurty [16,17] and later summarized by Hurty and Hou [18]. In more recent work, Holze and Boresi [19] applied the method to plate vibrations, with emphasis on numerical aspects. The component mode synthesis is rarely used for multistorey structures; however, Barber and Blotter [20] demonstrated economic application compared to standard dynamic analysis. Their investigation includes free vibration analysis of 4 and 6-storey structures divided into two components or substructures.

The close relationship between dynamic substructuring and the solution of the structural eigenvalue problem should be emphasized, since the reduction of the dynamic equilibrium equations to a more compact form is an intermediate step toward the solution of the eigenproblem. Moreover, regarding the subspace iteration technique, the two procedures are equivalent, since a similar mathematical formulation exists. The subspace iteration technique developed by Bathe and Wilson [21,22] is an efficient scheme for computation of the significant lowest eigenpair; thus, it can be described as concealed dynamic substructuring.

1.3 PRESENT STUDY

The purpose of this study is the development of a versatile and efficient substructuring procedure for rectangular structural patterns which includes a generation of mass properties. Thus, a more economical dynamic analysis than usually available from general purpose programs can be performed. The class of structures to be investigated includes shear wall type structures and, in particular, large-panel structures where discrete connections may exist. In future applications of the procedure it is expected to obtain further insight into the response characteristics of panelized structures subjected to dynamic excitation.

The general scope of this work involves the development of a computer subroutine attached as a substructure option to an efficient and widely used multipurpose program, rather than the creation of an independent program. The popular linear static and dynamic analysis program SAP4 [24] was chosen for this purpose.

The following Chapters describe the development of a super finite element technique and present applications which include a detailed parametric study of a large-panel structure subjected to earthquake excitation. The content of individual Chapters is outlined below.

In Chapter II, different methods for substructuring in dynamic analysis are reviewed. Mathematical formulations of the procedures are presented as congruence transformations based on a Rayleigh-Ritz analysis.

Chapter III describes the substructuring technique used in this study. It is formulated as a multilevel substructuring scheme for the creation of a rectangular "panel element". Stiffness and mass, as well as stress-displacement, matrices are derived. The major considerations focus on:

- (1) Numerical procedures for the reduction algorithm, and
- (2) Description of the multilevel growth scheme including the necessary input identifications.

The stress recovery procedure and the storage scheme for dynamic arrays are also discussed.

Chapter IV presents applications of the panel element in free vibration analysis, as well as a stress recovery using static analysis. Plate bending vibrations are investigated for different arrangements of panel elements, using consistent and lumped mass models. Shear wall problems demonstrate the influence of concentrated floor masses on the accuracy and the versatility with which the panel element is applied.

In Chapter V a large-panel precast structure consisting of 12 storeys is investigated for earthquake loading using elastic response spectrum analysis. Horizontal joints are assumed continuous, whereas discrete connectors exist in vertical joints at floor levels. Overall structural response with emphasis on connector forces is evaluated for

- (1) Uniformly distributed stiffness and inertia properties, and
- (2) Structural discontinuities over the height of the structure. This includes rigid or monolithic storeys, missing panels and non-uniform floor mass distribution.

The results obtained from a parametric investigation of this structure are summarized in Section 5.5.

Chapter VI contains a brief summary of this study, together with concluding remarks.

CHAPTER II

SOLUTION OF EQUILIBRIUM EQUATIONS IN DYNAMIC
ANALYSIS USING SUBSTRUCTURING TECHNIQUES

CHAPTER II

SOLUTION OF EQUILIBRIUM EQUATIONS IN DYNAMIC
ANALYSIS USING SUBSTRUCTURING TECHNIQUES2.1 INTRODUCTION

In this Chapter, the theoretical aspects necessary for derivation of a reduced set of dynamic equilibrium equations are discussed. Several methods of substructuring applied for this purpose are summarized in the following Sections, namely static condensation, component mode synthesis and the super finite element technique, where the latter is a procedure based on static condensation. However, for a presentation of the general formulation, a summary of the Rayleigh-Ritz method is given first.

2.2 REDUCED EQUILIBRIUM EQUATIONS IN DYNAMIC ANALYSIS

In this Section the equilibrium equations for a discretized system and its equivalent reduced set are identified.

The governing matrix equation for a dynamic finite element system is expressed by [25],[26],[27]

$$[M]\{\ddot{u}\} + [C]\{\dot{u}\} + [K]\{u\} = \{P\} \quad (2.1)$$

This equation states equilibrium of all forces at time t , with $\{u\}$, $\{\dot{u}\}$ and $\{\ddot{u}\}$ being the time-dependent state vectors expressing displacements, velocities and accelerations.

Forces consist of inertia forces $\{F_I\} = [M]\{\ddot{u}\}$, viscous,

damping forces $\{F_D\} = [C]\{\dot{u}\}$, elastic forces $\{F_E\} = [K]\{u\}$ and the time-dependent load vector $\{P\}$. Coefficient matrices $[M]$, $[C]$ and $[K]$ represent mass, damping and stiffness matrices respectively. If the coefficient matrices remain constant the problem is a linear one, and for its solution there exist basically two methods: direct integration and mode superposition. The latter is preceded by a solution of the eigenvalue problem.

The major concern in this work is the reduction of system equation (2.1) prior to its solution. This is generally achieved by creating subregions or substructures, and instead of working with the original finite element assembly, only those degrees of freedom representing boundaries of the subregions enter into the main equation.

A new set of equations which represents the reduced system is written as

$$[\bar{M}]\{\ddot{y}\} + [\bar{C}]\{\dot{y}\} + [\bar{K}]\{y\} = \{\bar{P}\} \quad (2.2)$$

where

$$[\bar{M}] = [T]^T[M][T] \quad (2.3a)$$

$$[\bar{C}] = [T]^T[C][T] \quad (2.3b)$$

$$[\bar{K}] = [T]^T[K][T] \quad (2.3c)$$

and

$$\{\bar{P}\} = [T]^T\{P\} \quad (2.3d)$$

Transformation matrix $[T]$, of order n by q , thus transforms the original equation set of order n by n into a new reduced set of order q by q . The nodal displacements $\{y\}$ relating to the reduced set are then expressed by

$$\{u\} = [T]\{y\} \quad (2.4)$$

For a solution of the original system (Eq. (2.1)) by first solving the free vibration problem assuming harmonic time-varying displacements, one arrives at the eigenvalue problem of the original system:

$$[K]\{\phi\} - \omega^2[M]\{\phi\} = \{0\} \quad (2.5)$$

where

ω is the circular frequency.

and

$\{\phi\}$ represents modal displacement shapes.

Subjecting the coefficient matrices $[K]$ and $[M]$ in Eq. (2.5) to the same constraints as applied in Eqs. (2.3a) and (2.3c) results in the reduced eigenvalue problem

$$[\bar{K}]\{a\} - \rho[\bar{M}]\{a\} = \{0\} \quad (2.6)$$

The new eigenpairs ρ_i and $\{a\}_i$ represent approximations to frequencies and mode shapes of the unreduced system. Approximated normalized modal displacements of eliminated degrees of freedom can be retrieved from Eq. (2.4) by inserting $\{a\}_i$ for back substitution.

The reduced eigenvalue problem as stated in Eq. (2.6) is the core equation in dynamic substructuring. The reduced equation set must result in good approximations of the first significant frequencies and mode shapes in order to consider a substructuring technique.

For the solution of a reduced system the following assumption is made: subregions stay linear during solution phase and constant damping exists within a subregion. In this study Rayleigh damping is assumed; damping is therefore excluded in the description and demonstration of reduction procedures.

The external force vector $\{P\}$ is not reduced if nodal forces are applied only at retained degrees of freedom. To retrieve eliminated displacements and hence stresses, after solution Eq. (2.4) can be applied. An alternative method, of retaining stresses more directly, will be presented in Chapter III.

Application of substructuring techniques, namely static condensation, component mode synthesis and the super finite element technique, all described in the following Sections, leads to a reduced set of equations, as presented by Eq. (2.2) or (2.6). The theoretical basis of these reduction procedures lies in a Rayleigh-Ritz analysis. A review of this method, therefore, is presented next.

2.3 REVIEW OF RAYLEIGH-RITZ ANALYSIS

The deformation-based finite element method, for static as well as for dynamic analysis, is basically recognized as a Rayleigh-Ritz (or Ritz) analysis [28,29]. This applies to a continuum which is reduced to a discrete system, as well as to a discrete system which is further reduced to fewer degrees of freedom. The second case applies to the procedures presented in the following Sections. The Rayleigh Ritz method is therefore discussed for a discrete dynamic system.

The unreduced generalized eigenvalue problem of order n is restated from Section 2.1:

$$[K]\{\phi\} - \omega^2[M]\{\phi\} = \{0\} \quad (2.5)$$

In a Ritz analysis approximations are derived for the first p significant solutions ω_i and $\{\phi\}_i$, $i = 1, p$, by assuming basis vectors $\{\psi\}_i$, $i = 1, q$, where $p < q < n$.

The first p approximate solution shapes are then represented by linear combinations as

$$\{v\} = \sum_{i=1}^q c_i \{\psi\}_i \quad (2.7)$$

where

c_i are the Ritz coordinates which determine the participation of a basis vector in a deformation shape.

A set of chosen basis vectors must approximate the least dominant subspace spanned by the exact solution vectors.

To arrive at the reduced eigenvalue problem, Rayleigh's quotient ρ is stated as

$$\rho(\phi) = \frac{\{\phi\}^T [K] \{\phi\}}{\{\phi\}^T [M] \{\phi\}} \quad (2.8)$$

where $\rho(\phi)$ is the minimum over all possible vectors $\{\phi\}$ and represents an upper bound on frequencies [13, 28, 30]. Substituting the deformation shapes $\{v\}$ from Eq. (2.7) for $\{\phi\}$ in Eq. (2.8); leads to

$$\rho(v) = \frac{\sum_{i=1}^q \sum_{j=1}^q c_i c_j \bar{K}_{ij}}{\sum_{i=1}^q \sum_{j=1}^q c_i c_j \bar{M}_{ij}} \quad (2.9)$$

with

$$\bar{k}_{ij} = \{\psi\}_i^T [K] \{\psi\}_j \quad (2.10a)$$

and

$$\bar{m}_{ij} = \{\psi\}_i^T [M] \{\psi\}_j \quad (2.10b)$$

Extremizing $\rho(v)$ with respect to the coefficients c_i yields

$$[\bar{K}]\{a\} - \rho[\bar{M}]\{a\} = \{0\} \quad (2.6)$$

which is the reduced eigenvalue problem as stated in Section 2.1.

Coefficient matrices $[\bar{K}]$ and $[\bar{M}]$ consist of elements \bar{k}_{ij} and \bar{m}_{ij} , respectively. The reduced system is of order q and is solved for q eigenpairs $\rho_i, \{a\}_i$, $i = 1, q$, where ρ_i represents an upper bound solution for frequencies, and vector $\{a\}_i$ represents the Ritz coordinates. The reduced matrices $[\bar{K}]$ and $[\bar{M}]$ can thus be derived in a congruence transformation from the unreduced matrices:

$$[\bar{K}] = [T]^T [K] [T] \quad (2.3c)$$

and

$$[\bar{M}] = [T]^T [M] [T] \quad (2.3a)$$

where

$$[T] = [\{\psi\}_1, \{\psi\}_2, \dots, \{\psi\}_q] \quad (2.11)$$

The transformation matrix $[T]$ thus consists of the chosen basis vectors and is of order n by q .

Matrices $[\bar{K}]$ and $[\bar{M}]$ may be visualized as projected operators on q -dimensional space, represented by $[T]$, or alternatively in a physical interpretation their elements may be seen as energy terms, expressed by Eq. (2.10). If transformation matrix $[T]$ of Eq. (2.11) is also applied to damping matrix and load vector, the complete dynamic equilibrium equation is reduced, as expressed by Eq. (2.2).

Static condensation and component mode synthesis, discussed in the following Sections, are both Ritz transformations and differ only in the choice of basis vectors in matrix $[T]$. It should be mentioned that the subspace iteration technique [21,24] used for the solution of the generalized eigenvalue problem, consists of a sequence of Ritz transformations, applied in an iterative manner.

2.4 STATIC CONDENSATION

In order to derive Ritz transformation shapes used in static condensation, Eq. (2.5) which may in this case represent the full system or only a substructure is re-written in partitioned form [31,32]:

$$\left(\begin{array}{cc} [K_{11}] & [K_{12}] \\ [K_{21}] & [K_{22}] \end{array} - \omega^2 \begin{array}{cc} [M_{11}] & [M_{12}] \\ [M_{21}] & [M_{22}] \end{array} \right) \begin{Bmatrix} \{x_1\} \\ \{x_2\} \end{Bmatrix} = \{0\} \quad (2.12)$$

This system of equations consists of two sets, unknown in displacement vectors $\{x_1\}$ and $\{x_2\}$, where $\{x_2\}$ defines displacements which are to be eliminated. Using the second equation set, subvector $\{x_2\}$ can be expressed in terms of subvector $\{x_1\}$ in the following manner:

$$\begin{aligned} \{x_2\} = & - ([K_{22}] - \omega^2 [M_{22}])^{-1} ([K_{21}] - \\ & - \omega^2 [M_{21}]) \{x_1\} \end{aligned}$$

or

$$\{x\} = \begin{Bmatrix} \{x_1\} \\ \{x_2\} \end{Bmatrix} = [T(\omega)] \{x_1\} \quad (2.13)$$

with

$$[T(\omega)] = \begin{bmatrix} [I] & \\ - ([K_{22}] - \omega^2 [M_{22}])^{-1} ([K_{21}] - \omega^2 [M_{21}]) & \end{bmatrix} \quad (2.14)$$

Equation (2.13) is equivalent to Eq.(2.4); however, transformation matrix $[T]$ is expressed as a function of ω (Eq.(2.14)). Since frequency ω is not known prior to the solution, Eq.(2.14) can only be applied in an iterative fashion and is impractical for large systems [31]. A common convention in static condensation, therefore, is to

assume $\omega = 0$. This results in the simplified transformation matrix

$$[T(0)] = [T] = \begin{bmatrix} [I] \\ -[K_{22}]^{-1}[K_{21}] \end{bmatrix} \quad (2.15)$$

Inertia forces inside subregions, discussed in more detail in Sections 2.5 and 2.6, are neglected in Eq. (2.15). In physical terms, transformation matrix $[T]$ from Eq. (2.15) represents deformation shapes generated by unit displacements at the retained nodes when degrees of freedom to be eliminated are released.

Applying Eq. (2.15) to stiffness and mass matrices, according to Eqs. (2.3a) and (2.3c), results in the reduced stiffness matrix

$$[\bar{K}] = [K_{11}] - [K_{12}][K_{22}]^{-1}[K_{21}] \quad (2.16)$$

and in the reduced mass matrix

$$\begin{aligned} [\bar{M}] = & [M_{11}] - [M_{12}][K_{22}]^{-1}[K_{21}] - [K_{12}][K_{22}]^{-1}[M_{21}] \\ & + [K_{21}][K_{22}]^{-1}[M_{22}][K_{22}]^{-1}[K_{12}] \end{aligned} \quad (2.17)$$

This simultaneous reduction of stiffness and mass matrices was first derived by Guyan [11] and was subsequently described and applied by several authors as a technique for dynamic substructuring [6,7,15,23].

If a lumped mass model is used, i.e., $[M_{12}] = [M_{21}]^T = [0]$, Eq. (2.17) may be simplified to

$$[\bar{M}] = [M_{11}] + [K_{21}][K_{22}]^{-1}[M_{22}][K_{22}]^{-1}[K_{12}] \quad (2.18)$$

The reduced mass matrix $[\bar{M}]$ in Eqs. (2.17) and (2.18) will always be consistent and full, where consistent is meant in the sense of energy consistent with stiffness matrix $[\bar{K}]$, since identical deformation shapes are used for reduction of both matrices. In mathematical terms the procedure is equivalent to a Gauss elimination on stiffness matrix $[K]$. Numerical aspects for efficient application of Eqs. (2.16) through (2.18) are discussed in Chapter III.

For many engineering applications it is common practice to assume masses of eliminated degrees of freedom as zero. The complete equation set (Eq. 2.12) is then written as:

$$\left(\begin{bmatrix} [K_{11}] & [K_{12}] \\ [K_{21}] & [K_{22}] \end{bmatrix} - \omega^2 \begin{bmatrix} [M] & [0] \\ [0] & [0] \end{bmatrix} \right) \begin{Bmatrix} \{x_1\} \\ \{x_2\} \end{Bmatrix} = \{0\} \quad (2.19)$$

Applying the reduction leads to

$$([\bar{K}] - \omega^2 [M]) \{x_1\} = \{0\} \quad (2.20)$$

where $[\bar{K}]$ is stated in Eq. (2.16) and $[M]$ represents diagonal mass terms associated with the retained degrees of

freedom. Assumption of zero masses at eliminated degrees of freedom is the only case where the reduced system leads to exactly the same results as the original system.

Internal inertia forces, related to eliminated degrees of freedom and neglected in Eq. (2.15), will be considered in the component mode synthesis, summarized in the following section.

2.5 COMPONENT MODE SYNTHESIS

The component mode synthesis [16,17,33], leads to an improved representation of mass characteristics of a substructure. A short summary of the method, mainly according to [34] is given below. Formulations presented for one component, use the following subscripts: B refers to retained boundary nodes, I represents eliminated interior nodes and N represents a transformation by normal modes.

The unreduced component matrices are partitioned according to boundary and interior degrees of freedom

$$[K] = \begin{bmatrix} [K_{BB}] & [K_{BI}] \\ [K_{IB}] & [K_{II}] \end{bmatrix} \quad (2.21a)$$

and

$$[M] = \begin{bmatrix} [M_{BB}] & [M_{BI}] \\ [M_{IB}] & [M_{II}] \end{bmatrix} \quad (2.21b)$$

A transformation is then performed using Eqs. (2.3a) and (2.3c) with transformation matrix

$$[T'] = \begin{bmatrix} [I] & [0] \\ [T_S] & [T_N] \end{bmatrix} \quad (2.22)$$

Matrix $[T']$ is composed of submatrix

$$[T_I] = \begin{bmatrix} [I] \\ [T_S] \end{bmatrix} = \begin{bmatrix} [I] \\ -[K_{II}]^{-1}[K_{IB}] \end{bmatrix} \quad (2.23)$$

which corresponds to Eq. (2.15) and represents a static condensation, with $[T_S] = -[K_{II}]^{-1}[K_{IB}]$. The submatrix

$$[T_N] = [\{\phi_I\}_1, \{\phi_I\}_2, \dots, \{\phi_I\}_p] \quad (2.24)$$

contains the first p normal modes, derived from the solution of the eigenvalue problem:

$$([K_{II}] - \omega^2[M_{II}])\{\phi_I\} = \{0\} \quad (2.25)$$

represented by the internal degrees of freedom.

Applying the congruence transformation to $[K]$ and $[M]$, using $[T']$ (and Eqs. 2.3a,c), leads to the reduced matrices

$$[\bar{K}] = \begin{bmatrix} [\bar{K}_{BB}] & [0] \\ [0] & [\bar{K}_{NN}] \end{bmatrix} \quad (2.26)$$

and

$$[\bar{M}] = \begin{bmatrix} [\bar{M}_{BB}] & [\bar{M}_{BN}] \\ [\bar{M}_{NB}] & [\bar{M}_{NN}] \end{bmatrix} \quad (2.27)$$

The submatrices in $[\bar{K}]$ are defined as follows:

$$\begin{aligned} [\bar{K}_{BB}] &= [T_I]^T [K] [T_I] \\ &= [K_{BB}] - [K_{IB}] [K_{II}]^{-1} [K_{BI}] \end{aligned}$$

and

$$[\bar{K}_{NN}] = [T_N]^T [K_{II}] [T_N]$$

Due to the special nature of $[T']$, the off-diagonal matrices in $[\bar{K}]$ are zero. Submatrices in $[\bar{M}]$ are:

$$[\bar{M}_{BB}] = [T_I]^T [M] [T_I]$$

which, when expanded corresponds to Eq. (2.17), and

$$\begin{aligned} [\bar{M}_{BN}] &= [\bar{M}_{NB}]^T = [T_S]^T [M_{II}] [T_N] + [M_{BI}] [T_N] \\ &= -[K_{BI}] [K_{II}]^{-1} [M_{II}] [T_N] + [M_{BI}] [T_N] \end{aligned}$$

representing mixed modes.

If the mode shapes of the constrained component are orthonormalized with respect to $[M_{II}]$; i.e.,

$$\{\phi_I\}_i [M_{II}] \{\phi_I\}_j = \delta_{ij}$$

with

$$\delta_{ij} = \begin{cases} 0, & \text{for } i \neq j \\ 1, & \text{for } i = j \end{cases}$$

the reduced equations are simplified to

$$[\bar{K}] = \begin{bmatrix} [K_{BB}] & & & \\ & \omega_1^2 & & \\ & & \ddots & \\ & & & \omega_p^2 \\ & & & & P \end{bmatrix} \quad (2.28a)$$

and

$$[\bar{M}] = \begin{bmatrix} [M_{BB}] & [M_{BN}] \\ [M_{NB}] & [I] \end{bmatrix} \quad (2.28b)$$

Component frequencies ω_1 to ω_p were derived from the internal eigenvalue problem (Eq.(2.25)). The fact that $[\bar{M}_{BN}] = [\bar{M}_{NB}]^T \neq [0]$, creates a "generalized inertia coupling"

[35] which leads to a higher occupancy in the mass matrix as compared to the stiffness matrix.

Matrices $[\bar{K}]$ and $[\bar{M}]$ are assembled into the system equation for solution. Internal degrees of freedom can be visualized as p internal nodal points with one degree of freedom each, representing generalized coordinates or amplitudes of the normal modes.

In summary, the component mode synthesis consists basically of a static condensation and additionally imposed generalized displacements represented by the p lowest normal modes of the constrained components. These additional degrees of freedom allow a good approximation of internal inertia forces. A component or a substructure can therefore be of any dimension, as long as all significant normal modes are included in the analysis. Selection of the p lowest modes, however, may involve difficulties [36]. It is also noted that during the solution of the eigenvalue problem of Eq. (2.25) all interior degrees of freedom need to be kept in core. Component mode synthesis, therefore, fails when advantage is taken of repetitive patterns inside the component, as is done in multilevel substructuring.

2.6 SUPER FINITE ELEMENT TECHNIQUE

The super finite element technique can be applied for static as well as for dynamic analysis. [7,15,28] In a static analysis this method may be regarded as straightforward, whereas in a dynamic analysis several considerations concerning internal inertia forces are necessary, partially discussed in Sections 2.3 and 2.4.

A super element is a multilevel substructure subjected to a series of static condensations. A first-level element, for example, is initially composed of basic elements (i.e., zero level elements) which were derived by assuming continuous displacement fields. The displacement fields of a first level element are then described by a linear combination of (discrete) nodal displacements at eliminated degrees of freedom originally introduced by zero level elements. First level degrees of freedom are now the basis for a second level element. The procedure is continued to any multilevel stage and can thus be described as a series of transformations from one set of variables to a reduced set of super variables [7]. In Fig.-2.1, typical examples for multilevel substructuring are given for one - and two - dimensional finite elements.

2.6.1 Reduction of Static Equilibrium Equations

Static condensation of stiffness matrix $[K]$ applied in a multilevel scheme implies the creation of transformation matrix $[T]$ (Eq. (2.15)) at each level. The series of transformations, beginning with zero level and ending with the m^{th} level, equation may be summarized as

$$\begin{aligned} [K_0]\{X_0\} &= \{P_0\} \\ &\vdots \\ [\bar{K}_m]\{\bar{X}_m\} &= \{\bar{P}_m\} \end{aligned} \quad (2.29)$$

The procedure is initiated by a transformation from zero to first level equations; i.e.,

$$[\bar{K}_1] = [T_0]^T [K_0] [T_0] \quad (2.30a)$$

$$\{\bar{P}_1\} = [T_0]^T \{P_0\} \quad (2.30b)$$

and

$$\{\bar{X}_1\} = [T_0]^T \{X_0\} \quad (2.30c)$$

It can be shown that the m^{th} level assembly furnishes exactly the same results as the fully assembled equation set. A formal mathematical proof is given in [7]. A short proof, which also provides insight into the numerical procedure, is given by the following reasoning: static condensation on stiffness matrix $[K]$ is equivalent to a Gauss elimin-

ation; thus, elimination of the s^{th} equation has the following effect on the j^{th} element in the i^{th} equation:

$$\bar{k}_{ij} = k_{ij} - k_{sj} k_{is}/k_{ss}$$

and for the RHS

$$\bar{P}_i = P_i - P_s k_{is}/k_{ss}$$

It is important to realize that the sequence of eliminations is arbitrary in order. In the super finite element technique the sequence is chosen such that repeating blocks of identical equation formations are eliminated only once. Or in physical terms, constraints in equivalent subregions are released simultaneously.

2.6.2 Reduction of Dynamic Equilibrium Equations

A dynamic analysis using super finite elements can be considered as an extension of the static analysis since the mass matrix is reduced in a parallel fashion; i.e.,

$$[\bar{M}_1] = [T_0]^T [M_0] [T_0] \quad (2.31)$$

thus leading to an extension of equation set (2.30). A simultaneous elimination of internal inertia terms in repeating subregions, or equation blocks, takes place by imposing static deflection shapes as constraints. The m^{th}

level matrices $[\bar{K}_m]$ and $[\bar{M}_m]$ representing stiffness and mass properties of a substructure are assembled into the overall system equation.

Whereas the reduced matrix $[\bar{K}_m]$ stores the exact stiffness properties, mass matrix $[\bar{M}_m]$, derived by energy consistent transformations, represents only approximations of the original inertia properties, unless reduced to massless degrees of freedom only (see Section 2.3). Thus, solutions are always approximate, however, they possess the favourable features of a Ritz solution, namely convergence to the lowest frequencies and mode shapes with a higher convergence rate for lower modes, as well as upper bound solutions if complete and consistent deformation shapes are used in the basic elements.

Internal forces are not known in advance as in a static analysis. Inertia forces which are activated by the natural frequencies of the structure are therefore not taken into account (only those created by static deformation-shapes). Magnitude of the inertia forces inside the sub-region or component, due to dynamic deformation $\{X\}$, is estimated by the product $\{X\}^T \{m\}$, [7]. Deformation vector $\{X\}$ can be expressed as a superposition of normal modes of the component. Vector $\{m\}$ represents internal nodal masses.

Large internal masses and/or large dynamic internal deformations, lead to large values of $\{X\}^T\{m\}$. Thus, errors are introduced into the analysis unless component modes are used. It is therefore vital to limit the size of the super-elements in regions of high mass concentration. Or more generally, the size of the super element should be such that the lowest natural frequencies of the constrained super-element are higher in magnitude than the highest significant natural frequencies of the total structure.

2.7 CONCLUDING REMARKS

Component mode synthesis and static condensation, both based on a Ritz analysis, are two different approaches in dynamic substructuring. It should be mentioned here that the component mode method is quite popular in aerospace work where, for example, one aeroplane wing is represented by one component and obviously normal component modes cannot be neglected in this case. On the other hand, in multi-storey structures - the field of interest in this study - one component such as a panel will not have a significant modal contribution to the total structure. This is one main reason why component modes are not taken into account in the present study. At the same time advantage can be taken of multilevel substructuring, since it is impractical to use multilevel substructuring and component modes in one proce-

dure (Section 2.4). The use of deformation shapes other than the static ones, for the reduction of the mass matrix, was studied but not applied in this work.

Multilevel substructuring is implemented in the next Chapter, in the derivation of a panel element. Emphasis is placed on numerical considerations and on an efficient growth scheme.

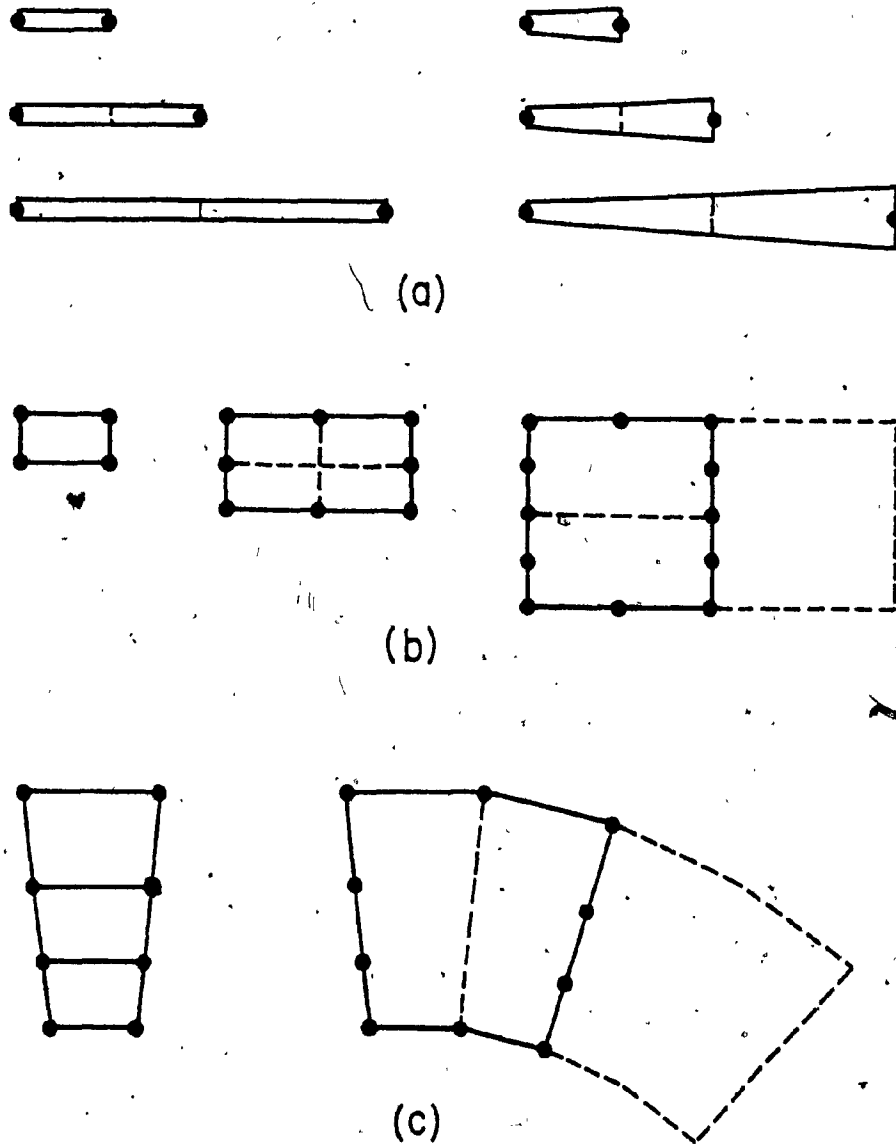


FIG. 2.1 TYPICAL EXAMPLES OF SUPER FINITE ELEMENTS
(a) LINE ELEMENTS, (b) RECTANGULAR ELEMENTS, AND (c) TRAPEZOIDAL ELEMENTS



CHAPTER III
DEVELOPMENT OF A PANEL SUPER ELEMENT

CHAPTER III

DEVELOPMENT OF A PANEL SUPER ELEMENT

3.1 INTRODUCTION

The objective of this Chapter is the derivation of a homogeneous rectangular super finite element which allows an efficient dynamic analysis of shear wall type or box-type multistorey structures. One super element could represent a substructure in a continuous model or an actual physical panel such as a prefabricated panel connected to a structural system in a discrete or continuous manner.

Following the concepts of the previous Chapter, one panel is considered as a substructure where all interior nodes are eliminated, leaving only boundary nodes. Considerations which result in the development of a fourth level super element, in the following called "panel super element" or simply "panel element", are listed below:

- (1) Flexibility in boundary node arrangement is desired, mainly to avoid unnecessary boundary degrees of freedom in the system equation.
- (2) Minimization of eliminated internal nodes and of storage requirements, achieved by multilevel substructuring.

- (3) An efficient and simple reduction algorithm.
- (4) The scheme should allow simple coding procedures as well as practical input.

In terms of computer program development, SUBROUTINE PANEL is created to furnish stiffness, mass and stress-displacement matrices for a panel element or a desired rectangular substructure. This Subroutine is meant to be attached to the element library of a multipurpose program. The present version of SUBROUTINE PANEL, listed in Appendix D, is compatible with SAP4 [24]. The latter is a structural analysis program for linear static and dynamic analysis. Corresponding input instructions for the panel element are included in Appendix C.

Major development steps necessary to establish the Subroutine are discussed in the following Sections. The presentation is best understood by considering one coupling step in a multilevel substructuring procedure, consisting of:

- (a) creation of connectivity arrays,
- (b) assembly into the higher level element, and
- (c) reduction of the matrices to the chosen boundary nodes.

The manner in which connectivity arrays are created depends largely on the "growth pattern" of the super element. This aspect is discussed in Section 3.3, with emphasis on the pattern used in this development. For the assembly, reference is made to standard finite element texts since assembly is straightforward once the connectivity is known. In this study, major consideration is given to the reduction procedure. Whereas in the previous Chapter the theoretical background of a reduction, or static condensation, is formulated, the following Section discusses purely numerical aspects. Two reduction procedures are considered and presented in detail. An in-core solution during the reduction is chosen; thus an efficient storage scheme, discussed in Section 3.5, is an additional requirement in the overall performance of the Subroutine. A procedure for stress recovery adapted to the multilevel substructuring scheme is discussed separately in Section 3.4.

3.2 NUMERICAL CONSIDERATIONS FOR THE REDUCTION PROCEDURE

Since an in-core solution is applied, panel element matrices - namely, stiffness matrix $[K]$, mass matrix $[M]$ as well as stress-displacement matrix $[S]$ - are stored and operated on simultaneously. Load vector $\{P\}$ is not included in the procedure. Consequently, external loads

may be applied only at retained boundary nodes.

In the following discussion of numerical aspects, only matrices $[K]$ and $[M]$ are considered. The reduction procedure itself is presented for one typical transformation step from, level i to level $(i+1)$. Thus, $[K]$ and $[M]$ are assumed to be full. It should be noted that, during the first reduction step, $[K]$ could be banded whereas $[M]$ could be either banded or diagonally lumped.

In order to demonstrate a typical transformation, matrices $[K]$ and $[M]$ (both of order n), representing an assembly from the i^{th} level, are partitioned as follows:

$$[K] = \begin{bmatrix} [K_{11}] & [K_{12}] \\ [K_{21}] & [K_{22}] \end{bmatrix} \quad (3.1a)$$

and

$$[M] = \begin{bmatrix} [M_{11}] & [M_{12}] \\ [M_{21}] & [M_{22}] \end{bmatrix} \quad (3.1b)$$

where subscripts 1 and 2 indicate degrees of freedom to be retained and to be eliminated, respectively. Applying a congruence transformation using matrix $[T]$ from Eq.(2.15), of order n by $(n-r)$, leads to reduced matrices $[\bar{K}]$ and $[\bar{M}]$ of Eqs.(2.16) and (2.17), representing level $(i+1)$. The reduced system is then of order $(n-r)$, where n is the order of the unreduced system, and r represents the

number of degrees of freedom to be eliminated.

A congruence transformation or a reduction procedure implies a numerical algorithm for the evaluation of Eqs. (2.16) and (2.17). In this study two algorithms are compared, which may be termed "block elimination", and "step elimination". In a block elimination a Cholesky decomposition is applied [19], whereas the step elimination is based on an algorithm presented by Wilson [37] and extended in this study to include mass effects.

3.2.1 Block Elimination

To demonstrate the operations necessary in this procedure, reduced stiffness and mass matrices are restated from Eqs. (2.16) and (2.17):

$$[\bar{K}] = [K_{11}] - [K_{12}][K_{22}]^{-1}[K_{21}] \quad (2.16)$$

and

$$[\bar{M}] = [M_{11}] - [M_{12}][K_{22}]^{-1}[K_{21}] - [K_{12}][K_{22}]^{-1}[M_{21}] + [K_{21}][K_{22}]^{-1}[M_{22}][K_{22}]^{-1}[K_{12}] \quad (2.17)$$

Evaluation of the two equations is initiated by Cholesky decomposition of submatrix $[K_{22}]$

$$[K_{22}] = [\tilde{L}][\tilde{L}]^T \quad (3.2)$$

where it should be noted that the formulation

$[\tilde{L}] = [L][D]^{-1/2}$ with $[K_{22}] = [L][D][L]^T$ is slightly more efficient [4]. Matrices $[\tilde{L}]$ and $[L]$ are of a lower triangular form, whereas $[D]$ is diagonal.

Proceeding with a forward substitution

$$[\tilde{L}][R] = [K_{12}] \quad (3.3)$$

yields $[R]$, used for composition of the reduced stiffness matrix

$$[\bar{K}] = [K_{11}] - [R]^T[R] \quad (3.4)$$

which is equivalent to Eq. (2.16). To reduce the mass matrix, $[R]$ is applied in a backward substitution

$$[\tilde{L}]^T[Y] = [R] \quad (3.5)$$

where it can be shown that $[Y] = [K_{22}]^{-1}[K_{21}]$. The reduced mass matrix is then expressed as

$$[\bar{M}] = [M_{11}] - [M_{12}][Y] - [Y]^T([M_{21}] - [M_{22}][Y]) \quad (3.6)$$

a formulation which is equivalent to Eq. (2.17).

3.2.2 Step Elimination

In order to discuss this procedure, the partitioned matrices from Eq. (3.1) are rewritten as

$$[K] = \begin{bmatrix} [K_{11}] & \{g\} \\ \{g\}^T & k_{nn} \end{bmatrix} \quad (3.7a)$$

and

$$[M] = \begin{bmatrix} [M_{11}] & \{h\} \\ \{h\}^T & m_{nn} \end{bmatrix} \quad (3.7b)$$

where $\{g\}$ and $\{h\}$ are vectors of order $(n-1)$ by 1 , corresponding to submatrices $[K_{12}]$ and $[M_{12}]$ of Eq.(3.1). Transformation matrix $[T]$ from Eq. (2.15) is now of order n by $(n-1)$ and is expressed as

$$[T] = \begin{bmatrix} [I] \\ -\frac{\{g\}^T}{k_{nn}} \end{bmatrix} \quad (3.8)$$

Equations (3.7a) and (3.7b) are thus transformed into

$$[\bar{K}] = [K_{11}] - \frac{1}{k_{nn}} \{g\}\{g\}^T \quad (3.9)$$

and

$$\begin{aligned}
 [\bar{M}] = & [M_{11}] - \frac{1}{k_{nn}} \{h\}\{g\}^T - \frac{1}{k_{nn}} \{g\}\{h\}^T + \\
 & + \frac{m_{nn}}{k_{nn}} \{g\}\{g\}^T
 \end{aligned}
 \tag{3.10}$$

reducing $[K]$ and $[M]$ by rank one. Applying this step r times leads to the reduced system which is of order $(n-r)$. A prerequisite for this procedure is a numbering scheme which assigns the highest numbers to the degrees of freedom to be eliminated.

This stepwise elimination procedure is applied in an elegant algorithm, described by Wilson [37] for the reduction of stiffness matrix, stress-displacement matrix and force vector. For this study the mass matrix as expressed by Eq.(3.10) is incorporated into the algorithm. A formulation similar to Eqs. (3.9) and (3.10) was presented by Irons [38] using energy expressions which lead to development of the frontal solution method [39] for reducing the stiffness matrix.

In order to assess block elimination and step elimination in an appropriate manner, a comparison of the two algorithms is included in Section 3.2.3.

3.2.3 Comparison of the Two Procedures

To compare two numerical procedures effectively, it is best to perform an operation count and compute storage requirements [40]. This is demonstrated in Appendix A, while in this Section only the results are discussed. The basic assumption under which the two methods are compared is that both matrices [K] and [M] are full and are reduced simultaneously.

In the operation count, only higher terms of the number of multiplications and divisions are retained according to generally accepted conventions. In Appendix A the following expressions are derived for operation counts:

$$OB = \frac{3}{2} n^2 r - nr^2 - \frac{1}{3} r^3 \quad (3.11)$$

and

$$OS = 2n^2 r - 2nr^2 + \frac{2}{3} r^3 \quad (3.12)$$

where OB and OS are the number of operations for block elimination and step elimination, respectively, and n and r denote total and reduced number of degrees of freedom (see Section 3.2). For comparison, it is convenient to divide both equations by n^3 and to use the substitution: $z = r/n$, where $z < 1$.

Then,

$$\overline{OB} = OB/n^3 = \frac{3}{2} z - z^2 - \frac{1}{3} z^3 \quad (3.13)$$

and

$$\overline{OS} = OS/n^3 = 2z - 2z^2 + \frac{2}{3} z^3 \quad (3.14)$$

Evaluation of these equations for z leads to ratios $\overline{OB}/\overline{OS}$ as a useful criteria for comparison. Variation of z results in the following $\overline{OB}/\overline{OS}$ ratios:

z	$\overline{OB}/\overline{OS}$
0.1	0.77
0.2	0.79
0.3	0.80
0.4	0.80
0.5	0.79
0.6	0.75
0.7	0.69
0.8	0.59
0.9	0.45

This shows that for most practical cases, i.e., for $z = r/n = 0.1$ to 0.7 , a block elimination executes only 70 to 80 percent as many operations as are performed in a step elimination.

Considering storage requirements, the following approximate expressions are derived in Appendix A:

$$\text{Block elimination} : n^2 + (n-r)r$$

$$\text{Step elimination} : n^2$$

The above comparison shows that step elimination is less favorable according to an operation count, but more favorable with regard to storage requirements. Since an in-core reduction scheme is chosen, the storage requirement is considered of higher priority. In addition, for most of the applications in this study the reduction process for a substructure requires only a fraction of the computing time consumed during the solution of the system equations. Thus, application of block elimination would lead to only negligible savings. A step elimination is therefore used in the multilevel reduction scheme. It should be emphasized that a step elimination algorithm is conveniently attached to a computer program, assuming degrees of freedom are labelled as discussed in Section 3.2.2.

A flow chart of the algorithm is attached in Appendix B. Reduction of the stress-displacement matrix, also included in the flow chart, is discussed in Section 3.4. A FORTRAN listing of the step elimination is included in Appendix D (see SUBROUTINE STAR). It is noted that matrices

are stored as one-dimensional arrays, which requires that stiffness and mass matrices be stored in Mode 1, shown in Fig. 3.1(a).

3.3 THE MULTILEVEL GROWTH SCHEME

3.3.1 General Scheme

The general concept of the super finite element technique or multilevel substructuring is discussed in Chapter II, and examples are presented in Fig. 2.1.

Rectangular structural components allow the application of multilevel schemes (Fig. 2.1(b)); however, the growth should be directed so as to allow arbitrary aspect ratios and reasonable flexibility for the retained boundary nodes. Important additional requirements to influence selection of a growth scheme are listed in Section 3.1.

Rectangular growth patterns are possible in two fashions:

- (1) Using double progression, and
- (2) Using single progression, as demonstrated in Table 3.1.

Inspection of the patterns shows that a double progression results in fewer eliminations. However, it is also apparent that a variation of aspect ratio is more difficult and, additionally, more complicated coding procedures are to be expected compared to a single progression.

Thus, a single progression scheme was chosen for the formation of a panel element. In the multilevel scheme described in the following Section, levels from zero to four are included, leading to a doubly symmetric substructure or panel element.

3.3.2 Description of the Suggested Scheme

The applied growth scheme demonstrated in Fig. 3.2 follows the principle of single progression. Hence, the dimension in the horizontal direction (x-direction) remains constant, whereas expansion takes place in the vertical direction (in negative y-direction). The stepwise growth pattern leading to a fourth level panel element can be visualized in three stages:

- (1) Creation of the basic 4-node element (zero level).

- (2) Creation of separate super element formations as shown in Fig. 3.2(a), namely end element, and level one and level two elements. These super elements can assume any of the formations demonstrated in Table 3.1 for single progression. Sequence of creation is indicated in Fig. 3.2(a) by the "growth direction".
- (3) Coupling of the created super elements, which leads to level three (Fig. 3.2(b)) or to level four (Fig. 3.2(c)) in an additional symmetric coupling.

Levels from zero to four are presented below, referring to Figs. 3.3 through 3.7 and to Tables 3.2 and 3.3. The presentation discusses the different possibilities for boundary node arrangements and demonstrates type identifications necessary for input.

3.3.2.1 Level zero

The basic finite element is created. Any type of 4-node, preferably rectangular, finite element is acceptable. Length A and height B for the element, indicated in Fig. 3.3(a), are determined by panel dimensions and by the desired mesh size. Stiffness and mass properties of the 4-node element determine the properties of the panel element.

It should be noted that element mass matrix may be consistent or of a semi-definite nature (diagonal with possible zero terms along the diagonal, see Chapter IV). Considerations necessary for the basic stress-displacement matrix are discussed in Section 3.4.

The sequence of nodal numbering for the 4-node basic element is counter-clockwise, starting with the node on the lower left-hand corner, as shown in Fig. 3.3(a). This numbering convention remains the same for all higher level elements. Figures 3.2(b) and 3.2(c) show the numbering of degrees of freedom for the membrane element and the plate bending element used in this study.

3.3.2.2 Level one

Two types of super elements are created, start element and end element. Both are assembled in a horizontal manner from the basic 4-node finite element as shown in Table 3.2. The panel dimension as well as mesh size in horizontal direction (x-direction), both defined at this level, are expressed by XL, and by NHN the number of horizontal nodal points, respectively (see Table 3.2 and Appendix C).

The start element is basically supplied for the creation of level two elements. For dimensions in vertical direction two possibilities are indicated in Table 3.2, namely B1 or B2, thus allowing a certain flexibility for boundary node arrangement along the vertical side of the

panel element. Three-digit type identification numbers are used for input, i.e., (100)* and (200) when B1 or B2 are applied, respectively. After assembly, start elements are directly carried into level two, or through tape writing into level three.

An end element defines the boundary node arrangement along the upper horizontal side. Six different types (symmetric) are included in the present version of the program, identified by one-digit numbers, ranging from (1) to (6) (see Table 3.2). It is noted that end elements are always of height B1. The matrices for an end element are written on tape, retrieved either in level three or level four.

3.3.2.3 Level two

A single progression using start elements from level one is generated in this level. Two options for level two elements are possible, as apparent from Table 3.3:

- (1) boundary nodes on vertical sides are retained, or
- (2) boundary nodes on vertical sides are eliminated.

* Input identification or the type of super elements is within brackets in Figures, Tables and in the text.

In addition, a level two element may be composed of B1 or B2 start elements.

For input identification the 3-digit number from level one is applied, with the first digit indicating if B1 or B2 is used. The boundary node option is determined by specifying the number of coupling steps with the second or third digit as follows: for eliminated boundary nodes the second location is used (e.g., type (120)) and for retained boundary nodes the third location is used (e.g., type (102)).

For retained boundary nodes at vertical edges, the generation of connectivity arrays can be based on the following observation: the number of boundary nodes increases by $2(2^{\text{NSP}} - 1)$, where NSP gives the number of coupling steps.

Generated level two elements are stored on tape in order to be coupled in level three. Or, alternatively, level two elements are used directly as output elements, or substructures.

3.3.2.4 Level three

Level three is identified by a coupling procedure of level one and level two elements, both retrieved from tape. Each coupling step leaves the two possibilities:

- (1) Boundary nodes are eliminated (NCOPL = 1), or
- (2) Boundary nodes are retained (NCOPL = 2).

Coupling may be demonstrated by the following two examples.

Example 1

Three elements of level one and two, i.e., types (2), (100) and (210), as shown in Fig. 3.4(a) are to be coupled. Two coupling steps are necessary, chosen as $NCOPL(1) = 1$, and $NCOPL(2) = 2$, thus leaving one boundary node in each of the vertical sides. The new level three element shown in Fig. 3.4(b) may be written in short form as (2,100,210/1,2).

Example 2

For the panel element shown in Fig. 3.5, all boundary nodes on the vertical sides are retained. The type of the coupled element is identified as (3,201/2).

A level-three element may be written on tape to be used in level-four, or it could be used for output.

3.3.2.5 Level four

In this level a symmetric coupling is provided for a level-three element or for an end element. A symmetric coupling can be achieved by a rotation about the x-axis, or by an in-plane rotation about the z'-axis, as demonstrated in Fig. 3.6(a). The in-plane rotation was chosen for this procedure.

The two coupling options - namely, boundary nodes, are to be retained or eliminated - are in level four determined by $NCOP4 = 1$ and $NCOP4 = 2$, respectively. In Fig. 3.6 symmetric coupling is demonstrated for a level three element of type (2,110/1). Figures 3.6(b) and 3.6(c) show corresponding level four elements, types (2,110/1,1) and (2,110/1,2), respectively. It is noted that the short form of level three is extended by the value for $NCOP4$.

An example which demonstrates the connectivity arrays necessary for a symmetric coupling is given by Fig. 3.7 and Table 3.4. Connectivity arrays are listed for a membrane case ($ND = 2$) and for a bending case ($ND = 3$) (see Table 3.4). Numbering conventions follow those established in level zero (Fig. 3.3).

3.4 STRESS RECOVERY

In this work a procedure for stress recovery is used, following the multilevel scheme of Section 3.3.2. Thus, a reduced stress-displacement matrix is created, which relates internal stresses to the remaining boundary degrees of freedom. Whereas assembly and reduction of stiffness and mass matrix take place in a closely related manner, separate considerations are necessary for the stress-displacement matrix. A reduction of the stress-displacement matrix is usually performed in finite element analysis when internal nodes in a basic element are eliminated (see also [37]). The same concept is, in the following, adapted to the super finite element technique.

An alternate procedure, normally applied in substructuring, is the complete retrieval of internal displacements and, consequently, any desired stress recovery is possible [7]. In a multilevel scheme this would necessitate a series of back substitutions using Eq. (2.30c). As a result complicated coding procedures are to be expected. For the stress recovery described below a back substitution is therefore not applied. This results, however, in the limitation that displacement output is only available for the retained super nodes. This restriction should be acceptable for most practical applications.

The theoretical aspects which follow are an extension of the concepts presented in Section 2.5, as expressed by Eqs. (2.29) through (2.31).

On element level (level zero) the relationship between stresses and displacements can be stated as

$$\{\sigma\} = [S]\{u\}$$

where $\{\sigma\}$ is the stress vector, $\{u\}$ represents the nodal displacement and $[S]$ is the stress-displacement matrix of order NS by ND, with NS the number of stress components and ND the number of degrees of freedom. The stress-displacement matrix is reassembled and transformed through several levels in an analog manner as demonstrated in Eqs. (2.29) and (2.30) for stiffness and mass matrix, i.e.,

$$\begin{aligned} [\bar{S}_1] &= [S_0][T_0] \\ &\vdots \\ [\bar{S}_m] &= [\bar{S}_{m-1}][\bar{T}_{m-1}] \end{aligned} \quad (3.15)$$

The final stress vector for the subregion, $[\sigma_m]$, is then expressed as

$$[\sigma_m] = [\bar{S}_m]\{\bar{x}_m\} \quad (3.16)$$

where

$\{\bar{x}_m\}$ are displacements of the super nodes.

Equation (3.15) represents a series of coordinate transformations; stresses from Eq. (3.16) are therefore exact to machine precision compared to those which are derived from an equivalent fully assembled finite element model.

To conform the reduction of the stress-displacement matrix with the step elimination algorithm from Section 3.2.2, matrix $[S]$ is stored according to the scheme of Fig. 3.8 and is then partitioned as follows

$$[S] = [[S_1]]\{w\} \quad (3.17)$$

where

$\{w\}$ represents the last column and

$[S_1]$ represents the remaining submatrix of $[S]$.

Applying now the transformation using $[T]$ from Eq. (3.8) results in the reduced stress-displacement matrix

$$[\bar{S}] = [S_1] - \frac{1}{k_{nn}} \{g\}\{w\}^T \quad (3.18)$$

which is compatible with stiffness and mass matrices from Eqs. (3.9) and (3.10). The algorithm requires the eliminated degrees of freedom to be labelled with the highest

numbers in the numbering sequence (see also Section 3.2.2).

The incorporation of Eq. (3.18) into the reduction algorithm is demonstrated by the flow-chart of Appendix B and by SUBROUTINE STAR in Appendix D.

The stress-displacement matrix of the m^{th} level, $[\bar{S}_m]$, is produced in an accumulative way during several reduction steps (see Equation Set (3.15)). Thus, if matrix $[S_0]$ for the basic zero level element represents NS stress components, matrix $[\bar{S}_m]$ for the panel element represents $NS \cdot M$ stress components, where M is the number of basic elements of which the panel element is composed. Locations of stress points, specified in the basic element, determine therefore the distribution of stress points over the whole panel element, as demonstrated in the example of Fig. 3.8. For an assumed situation of plane stress with $NS = 3$ (i.e., $\sigma_x, \sigma_y, \sigma_{xy}$) and two degrees of freedom per node, the order for the stress-displacement matrix in the example presented would be of 24 by 20.

The above example indicates that stress-displacement matrices can be of a large order. On the other hand, the nature of the accumulative procedure does not allow a more selective stress computation which would result in lower order matrices. Core storage is therefore more likely to be exceeded when a stress computation is included in the analysis.

3.5 STORAGE SCHEME

An in-core solution requires that stiffness, mass and stress-displacement matrices as well as those matrices from the previous level used for assembly be kept in high speed storage simultaneously. Computation of starting locations and length of the dynamic arrays, as well as total storage requirement, is therefore necessary at the beginning of the procedure.

The scheme outlined in Table 3.5 indicates 3 in-core storage phases:

- (1) Storage of elements from level zero to level two,
- (2) storage of level-three elements, and
- (3) storage of level-four elements.

Tape storage is used to transfer the arrays from one storage phase to another. The scheme in Table 3.5 gives starting locations for stiffness matrices only (L0 to L5); mass and stress-displacement matrices are stored in a parallel manner, as apparent from the general arrangement.

During all stages the matrices are stored as one-dimensional arrays, stiffness and mass matrices in Mode 1 during reduction phase and in Mode 2 during output phase

(Fig. 3.1), stress-displacement matrices are stored according to Fig. 3.9.

3.6 DISCUSSION AND CONCLUDING REMARKS

A rectangular substructure or panel element is derived where internal nodes are eliminated and which allows flexibility in locations of nodes along the boundaries. A multilevel scheme is developed and a reduction procedure is used which operates efficiently for the scheme presented. The panel element is created in stages from level zero to level four and may be either symmetric (level three) or doubly symmetric (level two and four), as demonstrated in Figs. 3.4 through 3.7 and Table 3.3. Stress recovery is consistent with the multilevel scheme and allows a complete stress computation within the substructure.

Applications in Chapter IV demonstrate the efficient use of the panel element as well as its limitations, in vibration analysis of structures. Examples are also presented for static analysis and stress recovery.

The scheme presented in this Chapter may be considered as the necessary basic development. Extensions to the present version, which may lead to a more general panel element, are possible with little effort.

Suggested additions for the start element (level one) are the following:

- (1) An irregular pattern for the start element allows different spacing of vertical grid lines.
- (2) Allowance of different material properties in the start element could simulate ribs or openings.
- (3) Creation of more than two possible start elements (i.e., introduction of height B3 and B4).

More variations for boundary node arrangement could be achieved by:

- (1) More arbitrary selection for retained nodes in the end element.
- (2) Option of retaining boundary nodes only on one vertical side.

Rotations of the element as a rigid body are independent of the above-mentioned developments and may be included through:

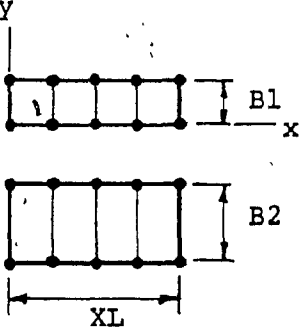
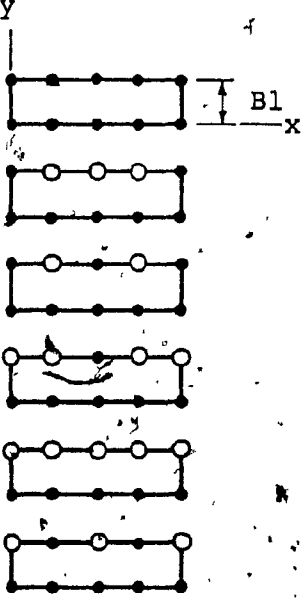
- (1) In-plane rotations
- (2) Out-of-plane rotations, thus allowing a three-dimensional analysis
- (3) Mirror image.

TABLE 3.1 GROWTH PATTERNS

DOUBLE PROGRESSION		SINGLE PROGRESSION
<p>19 Eliminations in 3 Stages</p>	<p>15 Eliminations in 5 Stages</p>	<p>21 Eliminations in 4 Stages</p>

NOTE: Boundary Nodes are not shown.

TABLE 3.2 LEVEL ONE ELEMENTS

IDENTIFICATION (FOR $NHN = 5$)	TYPE
<p>START ELEMENTS</p>	 <p>(100)</p> <p>(200)</p>
<p>END ELEMENTS</p>	 <p>(1) or (100)</p> <p>(2)</p> <p>(3)</p> <p>(4)</p> <p>(5)</p> <p>(6)</p>

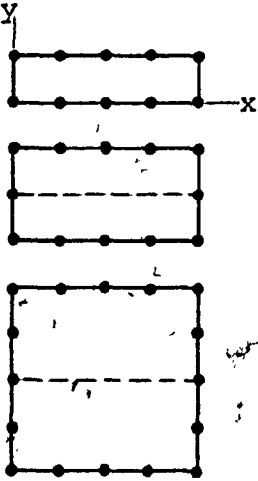
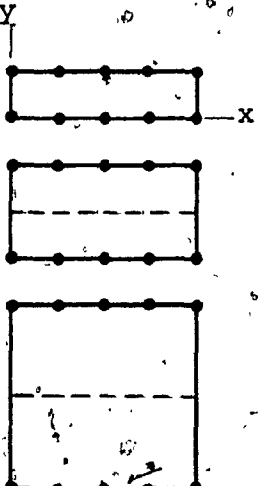
● Retained Nodes

○ Eliminated Nodes

NOTES TO END ELEMENTS:

TYPE (3): Corner and midside nodes are retained ($NHN \geq 5$)TYPE (4): Midside node is retained ($NHN \geq 3$)TYPE (6): Nodes next to corner nodes are retained ($NHN \geq 5$)

TABLE 3.3. LEVEL TWO ELEMENTS

IDENTIFICATION (FOR NHN = 5)	TYPE		REMARKS
	USING B1	USING B2	
 <p>etc.</p>	(100) (101) (102)	(200) (201) (202)	vertical boundary nodes are retained
 <p>etc.</p>	(100) (110) (120)	(200) (210) (220)	vertical boundary nodes are eliminated

• Retained nodes
Eliminated nodes are not shown.

TABLE 3.4 COUPLING IN LEVEL FOUR (EXAMPLE 2)
NUMBERING OF DEGREES OF FREEDOM

Number of Degrees of Freedom	ND = 2		ND = 3	
	Element (1)	Element (2)	Element (1)	Element (2)
1	11	-5	16	7
2	12	-6	17	-8
3	15	-13	18	-9
4	16	-14	22	19
5	13	-15	23	-20
6	14	-16	24	-21
7	5	-11	19	22
8	6	-12	20	-23
9	7	-1	21	-24
10	8	-2	7	16
11	9	-3	8	-17
12	10	-4	9	-18
13			10	1
14			11	-2
15			12	-3
16			13	4
17			14	-5
18			15	-6

TABLE 3.5 DYNAMIC STORAGE ARRAYS FOR MATRICES

GENERAL ARRANGEMENT FOR ARRAYS	LOCATIONS DURING LEVEL ZERO TO LEVEL TWO		LOCATIONS DURING LEVEL THREE	LOCATIONS DURING LEVEL FOUR
<p>LO = 1 KO MO MTOT</p> <p>stiffness matrices mass matrices stress-displacement matrices</p> <p>of same length</p>	Level	Remarks	LO	LO
	L1	zero for B1	KO	KO
	L2	zero for B2	L10	L100
	L3	one start element for B1	L10	L100
	L4	one start element for B2	L10	L100
	L5	two second last coupling step	L10	L100
	KO	one and two end element last step	L10	L100
	LO	KO	L10	L100
	L1	KO	L10	L100
	L2	KO	L10	L100
	L3	KO	L10	L100
	L4	KO	L10	L100
	L5	KO	L10	L100
	KO	KO	L10	L100

1	2	4	7	11
	3	5	8	12
		6	9	13
			10	14
				15

MODE 1

(a)

1	2	3	4	5
	6	7	8	9
		10	11	12
			13	14
				15

MODE 2

(b)

FIG. 3.1 STORAGE SCHEMES FOR STIFFNESS AND MASS MATRIX

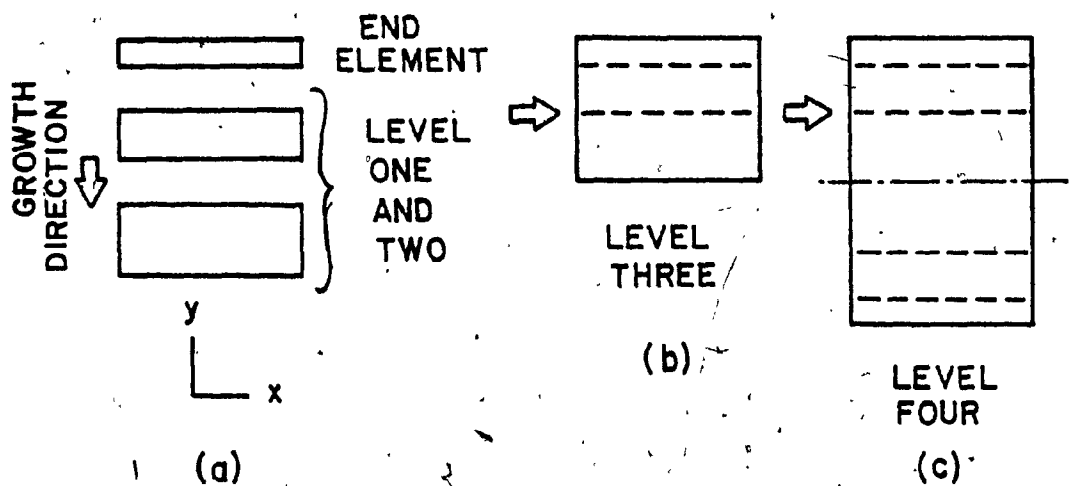


FIG. 3.2 GROWTH SCHEME OF THE LEVEL-FOUR PANEL ELEMENT

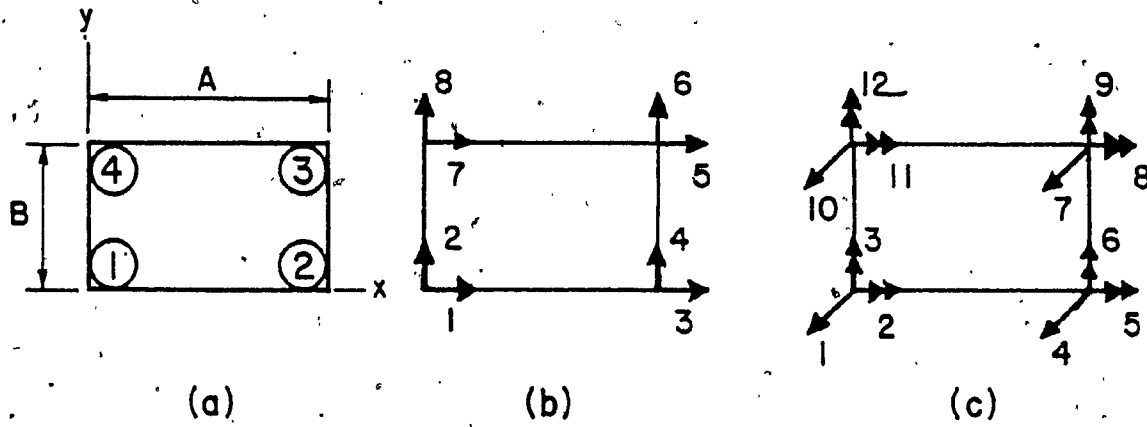


FIG. 3.3 NUMBERING OF 4-NODE ELEMENTS; (a) NODAL NUMBERING, (b) DEGREES OF FREEDOM FOR PLANE STRESS, AND (c) DEGREES OF FREEDOM FOR PLATE BENDING

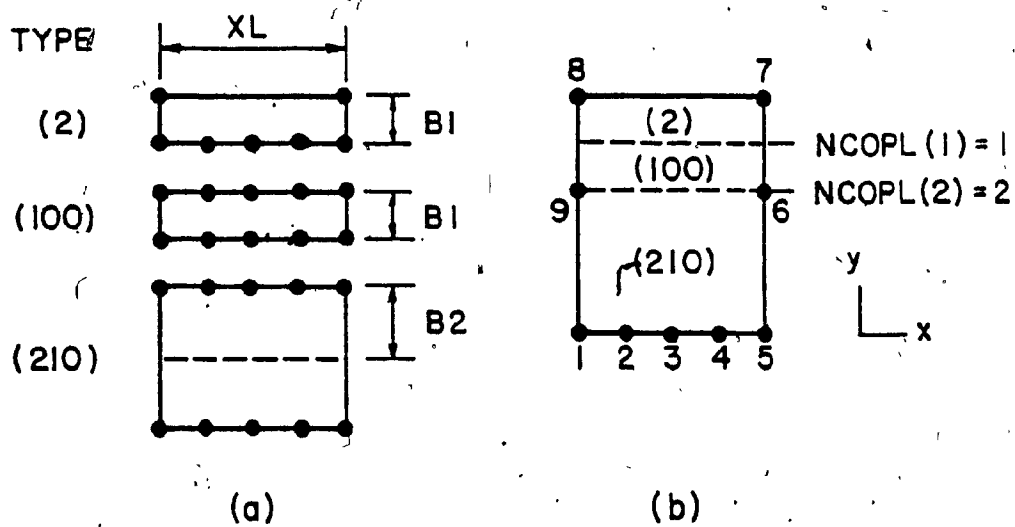


FIG. 3.4 COUPLING IN LEVEL THREE (EXAMPLE 1); (a) LEVEL ONE AND LEVEL TWO ELEMENTS, AND (b) LEVEL THREE ELEMENTS

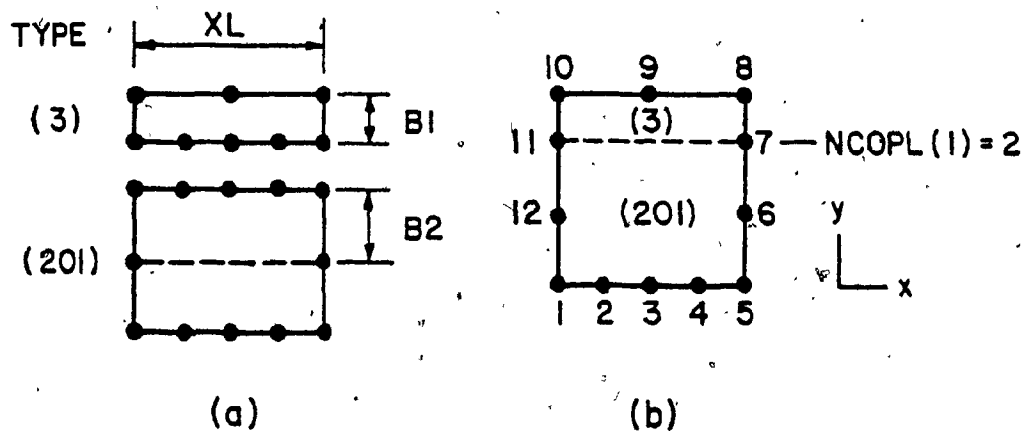


FIG. 3.5 COUPLING IN LEVEL THREE (EXAMPLE 2); (a) LEVEL ONE AND LEVEL TWO ELEMENTS, AND (b) LEVEL THREE ELEMENT

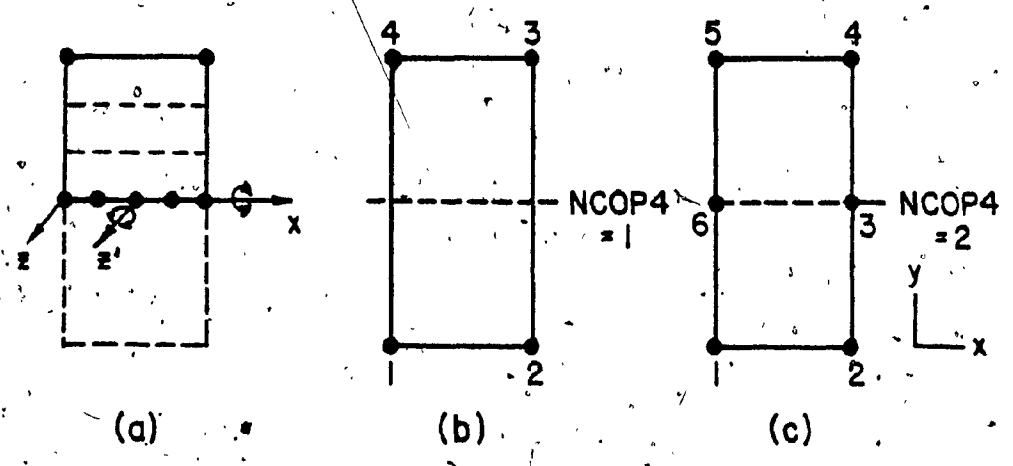


FIG. 3.6 COUPLING IN LEVEL FOUR (EXAMPLE 1)

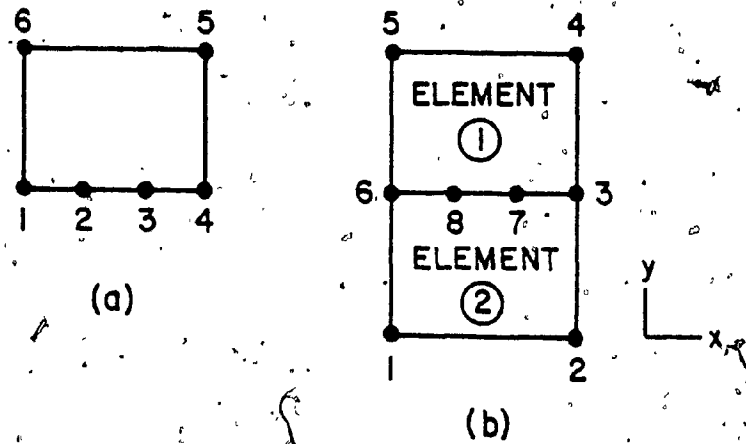


FIG. 3.7 COUPLING IN LEVEL FOUR (EXAMPLE 2); (a) NODAL NUMBERING FOR LEVEL THREE ELEMENT, AND (b) NODAL NUMBERING FOR LEVEL FOUR ELEMENT BEFORE REDUCTION OF INTERNAL NODES

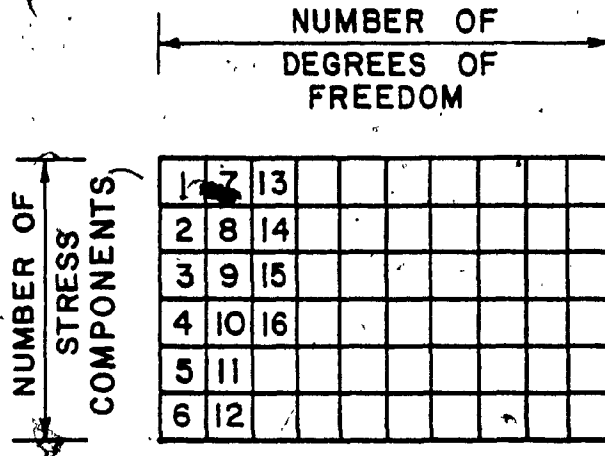
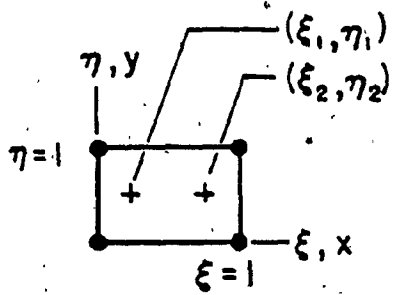
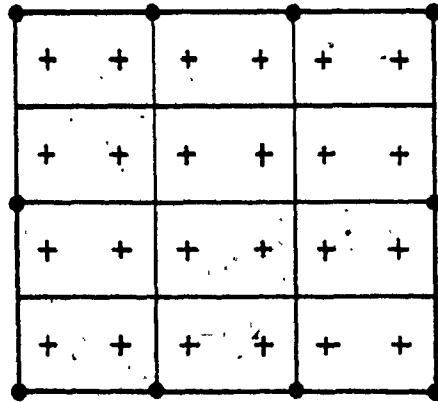


FIG. 3.8 STORAGE SCHEME FOR STRESS-DISPLACEMENT MATRIX



(a)



(b)

FIG. 3.9 EXAMPLE FOR STRESS POINT LOCATIONS; (a) BASIC 4-NODE ELEMENT, AND (b) PANEL ELEMENT

CHAPTER IV

APPLICATIONS USING PANEL ELEMENTS

CHAPTER IV
APPLICATIONS USING PANEL ELEMENTS

4.1 INTRODUCTION

In this Chapter, the panel element described previously and represented by SUBROUTINE PANEL in Appendix D is applied as a substructure option in conjunction with the general purpose program SAP4. For the examples to be presented, full displacement continuity along boundaries between substructures is assumed.

Emphasis is placed on structures subjected to membrane action, which includes free vibrations of shear walls. Static loading is also considered, primarily to demonstrate stress recovery. Solutions of membrane problems are compared to the solutions where a fully assembled finite element model is used.

To demonstrate accuracy and limitations of substructuring techniques in out of plane bending problems, cases of plate bending vibration are studied. Results are compared with analytic solutions.

An important aspect in dynamic analysis of structures is the representation of mass or inertia characteristics. In the previous Chapter mass matrices were assumed to be

consistent and full. However, in practical applications it may be more desirable to employ a lumped mass model in order to reduce computational effort, particularly for large systems. It has been observed that, for many situations in structural engineering, the lumped mass model results in solutions of sufficient accuracy [26]. A lumped mass option is therefore introduced for the panel element.

Mass lumping is conveniently achieved by distributing the total mass of the substructure along the diagonal of the mass matrix according to the diagonal terms of the consistent mass matrix. This procedure is applied easily when only translational degrees of freedom exist, as in plane stress elements or, more generally in all isoparametric finite elements. However, difficulties arise when rotational degrees of freedom are introduced, as may occur in plate bending. The latter is discussed in Section 4.2.

The predominant influence of mass concentration at floor levels in multi-storey structures may override the refined mass representation of an individual panel substructure. This situation is discussed for a plane stress shear wall model in Section 4.3.

For the solution of the free vibration problem the available eigenvalue routines from SAP4 were applied when lumped mass models were used. For consistent mass representa-

tion an in-core solver based on subspace iteration [21] was employed.

4.2 PLATE BENDING VIBRATIONS

In this Section free out of plane vibrations are studied for a single panel substructure which may be considered as one structural element. Since only the lowest modes of an individual panel are likely to contribute to overall structural response, only the first 4 modes are examined. The primary point of interest may be formulated thus: to what degree of accuracy can boundary nodes represent the dynamic properties of a substructure?

The following cases are studied using a non-conforming plate bending element [6]:

- (1) Cantilever plate - investigated for accuracy of frequencies and mode shapes when consistent and lumped mass models are used.
- (2) Square free-free plate - studied for accuracy of frequencies when consistent and lumped mass models are used.
- (3) Cantilever plate - where internal degrees of freedom are introduced by employing panel assemblies.

4.2.1 Single Panel Element as Cantilever Plate

Dimensions, boundary node arrangement and mesh size of the panel element investigated are shown in Fig.4.1(a). Plate thickness is assumed as 0.1 in. (2.54 mm) and material properties are: Young's modulus = 3.0×10^4 ksi (2.07×10^5 MN/m²), Poisson's ratio = 0.3 and mass density = 0.733×10^{-6} kips-sec²/in⁴ (0.00785 kg/cm³). Results, expressed in terms of the four lowest natural frequencies with corresponding mode shapes, are compared to analytic solutions published by Barton [41]. The latter are based on 18-term Rayleigh-Ritz expansions and agree well with experimental results. Influence of the following parameters on accuracy for the free vibration problem are investigated:

- (1) Consistent and lumped mass models using different strategies for lumping of rotary terms.
- (2) Influence of variations in boundary nodes and mesh size.

At the basic element level (zero level) the mass matrix can be introduced either as consistent or as diagonally lumped. The final panel mass matrix is in both cases consistent, and in both cases the computational effort is practically the same. A study of plate bending vibrations showed that it is more desirable to start with a consistent mass matrix, for the following reasons: First, a noticeable improvement in accuracy

is achieved, assuming a consistent mass matrix is also used for the solution of the eigenvalue problem. Secondly, for some deformation shapes applied during elimination, a lumped mass matrix at the basic element level leads to zero rows and columns in the consistent mass matrix of the panel element, thus resulting in numerical instabilities.

The lumping procedure for the consistent mass matrix of the panel element was established from the following considerations. Hinton et al [42] derived expressions of lumped mass terms for certain isoparametric elements, an approach equivalent to scaling the diagonal terms according to the consistent mass matrix when the total element mass is preserved. This method, which may be termed "consistent lumping", is applied in this study for the derivation of a lumped mass matrix for the panel element substructure. Figure 4.2 demonstrates lumping of the translational degrees of freedom for the panel element of Fig. 4.1(a), assuming total panel mass of unity. It is shown that consistent lumping (Fig. 4.2(a)) results in mass terms that are similar to those for lumping according to assumed tributary areas (Fig. 4.2(b)).

For the lumping of rotary terms four possibilities were considered:

Case A:

Zero rotary terms are assumed.

Case B:

The rotary moment of inertia is taken over the thickness of the plate, for unit length, and is then multiplied by the total panel area. This quantity is distributed along the diagonal according to consistent lumping.

Case C:

Rotary inertia terms are used directly from the diagonal of the consistent mass matrix.

Case D:

The mass moment of inertia about the center of gravity of the panel is distributed according to consistent lumping.

In Table 4.1 frequencies, derived from both consistent as well as lumped models, are compared with values available in [41] using the following expression:

$$\epsilon_F = \frac{\omega_E - \omega_A}{\omega_E} \times 100 \quad (4.1)$$

where

ω_E and ω_A are frequencies obtained from [41] and from the present study, respectively.

The Table demonstrates good agreement for the consistent model, whereas the use of a lumped model results in large differences for all cases. It is apparent that Cases A to C are very close, thus, the influence of rotary terms lumped according to Cases B and C are of no practical significance. Case D, on the other hand leads to an over-estimation of rotary inertia terms. Therefore, rotary inertia is not included in further investigations (i.e., Case A is used).

Shapes of the four lowest modes are compared in Table 4.2 for consistent and lumped (Case A) models, when maximum amplitudes are normalized to one. Figure 4.3 shows the normalized mode shapes for the symmetric half-plate. As a measure of accuracy the length of the residual ϵ_s (i.e., Euklidean norm) is used, expressed as:

$$\epsilon_s = \|\{\phi_E\} - \{\phi_A\}\|_2 \quad (4.2)$$

where

$\{\phi_E\}$ and $\{\phi_A\}$ represent mode shapes from [41] and from this study, respectively.

Table 4.3 contains the corresponding ϵ_s values for the consistent and the lumped mass models.

A decrease in the number of boundary nodes, as well as different mesh sizes, compared to the panel element of Fig. 4.1(a), is represented by two cases in Table 4.4. As apparent from the values for ϵ_F and ϵ_s , frequencies and mode shapes are not significantly affected by these changes (compared to Tables 4.1 and 4.3).

4.2.2 Single Panel Element as Free-Free Plate

The plate with free-free boundary conditions presented in Fig. 4.1(b) is investigated for frequency accuracy. Plate thickness and material properties have the same values as used in Section 4.2.1.

A free-free body requires the application of eigenvalue shifting [28] in order to eliminate rigid body modes:

$$([K] - \alpha[M])\{\phi\} - \mu[M]\{\phi\} = \{0\} \quad (4.3)$$

The natural frequencies of the plate are then expressed as $\omega_i^2 = \mu_i - \alpha$. For shift value α an approximation of the lowest frequency was used.

Differences in frequencies ϵ_F (see Eq. (4.1)) using [43], are presented in Table 4.5. The values of ϵ_F

are in the range of those for the cantilever plate, Table 4.1. A decrease in accuracy for the consistent model in the third mode is due to the particular nature of the third modal configuration, where the maximum amplitude occurs at the center of the plate.

4.2.3 Assembly of Panel Elements as Cantilever Plate

The investigations of the previous Sections have demonstrated that the use of lumped mass models in plate bending problems results in poor approximations of dynamic properties. To improve accuracy, a panel substructure may be composed of an assembly of panel elements, thereby increasing the number of degrees of freedom and at the same time introducing internal nodes.

The cantilever plate of Fig. 4.1(a) (see Section 4.2.1) is chosen for this study. Table 4.6 demonstrates the assemblies of two, four and eight-panel elements. Differences for frequencies as well as mode shapes, compared to [41], are presented for the consistent and the lumped mass model. As expected, improvements are achieved for both mass models with increasing number and distribution of nodal points over the plate. However, the Table demonstrates also that it is difficult to achieve accurate results in plate bending vibrations when lumped mass models are used. Thus, a consistent mass

model should be applied for substructures whenever plate bending contributes significantly to the overall structural response, and when a distributed inertia characteristic exists.

4.3' STRUCTURAL PROBLEMS IN MEMBRANE ACTION

In this Section applications in the field of structural engineering are presented. Dynamic properties of the total structure are studied, rather than for the substructure only. A single panel element may represent an actual panel or a substructure, where full continuity along the boundaries between substructures is assumed.

In the following examples plane stress elements are used which include incompatible modes, unless mentioned otherwise. The mass matrix at zero element level is represented by a lumped model. For the panel element the mass matrix is derived by consistent lumping as described in Section 4.2.

The applications to follow concentrate primarily on shear wall structures and are:

- (1) Free vibration analysis of a single bay shear wall where floor mass intensity is varied.
- (2) Free vibration analysis of a coupled shear wall.

- (3) Stress recovery, demonstrated for static analysis of a coupled shear wall and a beam bending problem.

4.3.1 Single Bay Shear Wall With Varied Floor Mass Intensity

The 5-storey shear wall shown in Fig. 4.4 is analyzed using the three different panel elements labelled Model A, B and C. Solutions derived for these models are compared to the full finite element assembly of Fig. 4.4(a). The models are studied for homogeneously distributed mass, as well as for mass concentrated at floor levels. Material properties are as follows: Young's modulus = 5.76×10^5 kips/ft² (2.76×10^4 MN/m²), Poisson's ratio = 0.20 and mass density = 0.00482 kips - sec²/ft⁴ (0.0025 kg/cm³). The wall thickness is assumed as 0.667 ft (0.203 m).

Differences in the periods of vibration with respect to the full finite element model, presented for the first 12 modes, are expressed as

$$\epsilon_P = \frac{T_E - T_A}{T_A} \times 100 \quad (4.4)$$

with T_E and T_A representing the periods of the full model and of the models using panel elements, respectively. Values for ϵ_P , derived for homogeneous mass distribution, are shown in Table 4.7. Differences become quite large for higher modes, increasing from Model A to C. Thus, a lumped mass representa-

tion for panel elements may not result in satisfactory solutions for the homogeneous shear wall. Improvement in accuracy could be achieved using a consistent mass model.

Introduction of a concentrated mass at floor levels, however, results in a more realistic situation for practical applications. Figure 4.5 presents differences in frequencies, ϵ_p , for the first 12 modes as a function of floor level mass intensity. Floor mass intensity, or "concentrated mass intensity", is expressed in terms of multiples of the total homogeneous shear wall mass, equally distributed over storey degrees of freedom. It is noted that for a concentrated mass intensity of 10 all periods up to the twelfth mode are approximated within or very close to one per cent, relative to those of the full finite element model. It was also observed that vertical extensional modes (e.g., modes 3 and 6) experience a higher rate of convergence.

Table 4.8 compares the mode shapes of Models A, B and C to those of the full model, for a concentrated mass intensity of 10. The mode shapes are represented by the displacements of five characteristic nodes (left side node at each floor level). Equation (4.2) is applied to derive the length of the residual ϵ_S , with $\{\phi_E\}$ and $\{\phi_A\}$ representing mode shapes of the full model and the model using panel elements, respectively. The magnitudes of ϵ_S demonstrate good approximations for all 12 modes.

The preceding observations indicate that degrees of freedom representing concentrated masses may become predominant in the system equation, thus resulting in close agreement of the lowest eigenpairs for the shear wall models of Fig. 4.4. For concentrated mass intensity less than 1.0, however, good agreement is achieved only for some of the lowest modes. Additionally, the curves in Fig. 4.5 may become discontinuous for higher modes (see for example the dotted lines for modes 9 and 10). This characteristic is due to changes in the sequence of modal configurations.

The application of substructuring techniques does not necessarily lead to savings in execution time, since a large bandwidth may be introduced into the system equation. In addition matrices tend to lose their sparsity due to high connectivity of substructure degrees of freedom. Thus, more iteration steps are necessary (although with fewer degrees of freedom) when subspace iteration is applied for the solution of the eigenvalue problem.

In the preceding examples, approximate execution times expressed as percentages of the full model time were:

- (a) Model A. - 100 per cent
- (b) Model B - 50 per cent
- (c) Model C - 30 per cent.

Therefore, it can be concluded that no savings in computing time can be expected when all boundary nodes of the substructure are retained. However, if some of the boundary nodes are eliminated, considerable savings can be achieved.

The results presented and evaluated in this Section can give representative guidelines for possible savings in computing time, but even more so for the range of accuracy when similar shear wall examples are analyzed using substructuring techniques. For cases with approximately the same number of degrees of freedom, adequately distributed and with concentrated mass intensity of magnitude 10 or higher, satisfactory accuracy should be obtained. If the number of degrees of freedom is increased (i.e., increase in number of storeys or bays), similar accuracy can be expected for lower concentrated mass intensity.

4.3.2 Free Vibration of a Coupled Shear Wall

Figure 4.6 shows a coupled shear wall, represented by a full finite element model and by the corresponding model using panel elements. The following properties are assumed: $E = 5.76 \times 10^5$ kips/ft² (2.76×10^4 MN/m²), $\nu = 0.20$, $\rho = 0.00482$ kips-sec²/ft⁴ (0.0025 kg/cm³), and wall thickness = 1 ft (0.305 m).

The modularity of this structure allows the use of panel elements in an effective way. Figure 4.6(b) shows that three different types of panel elements are created, leading to fairly even distribution of nodal points. Thus, good approximation to the full model can be expected, even for low concentrated mass intensity. It is noted that additional nodes are introduced at the two piers and in connecting beams, to improve mass representation.

In Table 4.9 the first 12 periods for the 2 models are compared. Values for ϵ_p , derived from Eq. (4.4), indicate only small differences between the periods of the two models when mass distribution is homogeneous. Accuracy, however, is still noticeably improved when concentrated floor masses are introduced. For this example, only the weight of floor slabs is considered (slab thickness = 7 in. (0.178 m), span = 24 ft (7.36 m)), leading to a concentrated mass intensity of 3.2.

Execution time for the model using the panel element representation was 85 to 90 per cent that of the full finite element model. The real advantage for using panel elements in this case, however, was demonstrated by a simple and short formulation of the input data.

4.3.3 Static Analysis of a Coupled Shear Wall

A coupled shear wall, similar to the one discussed in the previous Section, is taken from [44]. The full model, as applied in [44], and its panel representation are shown in Figs. 4.7(a) and 4.7(b), respectively. Material properties are: $E = 4.0 \times 10^6$ psi (2.76×10^4 MN/m²), and $\nu = 0.4$. Wall thickness is assumed as 1 ft (0.305 m). Lateral loading is indicated in Fig. 4.7(a) and corresponds to [44]. For this analysis plane stress elements, without incompatible modes, are used.

Distribution of nodal points is not important in a static analysis. Thus, when applying panel elements, only those nodes are retained which are needed for connectivity and symmetry, together with nodes where displacement output is desired. Figure 4.7(b) demonstrates the use of four different panel elements.

Lateral displacements of the left wall are plotted in Fig. 4.8. Results obtained using panel elements are identical to [44]. A stress recovery is performed for the lower left pier. Corresponding to Fig. 3.9(a), two stress points are introduced for the basic element: $\xi_1 = 0.25$, $\eta_1 = 0.5$ and $\xi_2 = 0.75$, $\eta_2 = 0.5$. This arrangement gives full information for normal stress σ_y in the y-direction, inside the basic

4-node element. Figure 4.9 shows the isobars of σ_y . These are in good agreement with [44].

4.3.4 Stress Recovery for Cantilever Beam

A cantilever beam represented by one panel element (Type (140)) and the corresponding full model are shown in Figs. 4.10(b) and 4.10(a), respectively. Material properties are: $E = 3.0 \times 10^4$ ksi (2.07×10^5 MN/m²) and $\nu = 0.3$. Beam width is assumed as 0.5 in. (1.27 cm). The total load of 1 kip (4.45 kN) is applied at nodal points according to a parabolic distribution.

This example demonstrates stress recovery within the substructure. Two stress points are selected in the same manner as in Section 4.3.3; thus, full information is available for normal stresses σ_y and shear stresses σ_{xy} , along $\eta = 0.5$ in each basic element. Magnitudes of σ_y and σ_{xy} are plotted for several sections along the y-axis, as shown in Fig. 4.11. These stresses were compared with an analysis using the full model and one stress point per basic element. The results obtained agreed to machine precision.

4.4 SUMMARY

The panel element derived in Chapter III and attached to the element library of SAP4 was, in the previous Chapter, applied as an effective substructure option. Accuracy achieved and effectiveness with respect to execution time, as well as general applicability, were discussed for examples in plate bending vibrations, plane stress vibrations and in stress recovery. The observations noted are summarized below:

- (a) Free vibration analyses for a cantilever plate and a free-free plate, both consisting of a single panel element (Fig. 4.1), were compared to analytic solutions. Studies of mass representation resulted in the following conclusions:
 - (i) A consistent mass model leads to good approximations of frequencies and associated mode shapes (Fig. 4.3 and Tables 4.1 through 4.6).
 - (ii) Lumped mass models derived by "consistent lumping" (see Section 4.2.1) did not result in satisfactory solutions (Tables 4.1 and 4.4 through 4.5). Different strategies for lumping of rotational degrees of freedom indicated that inclusion of lumped rotary inertia terms

is not necessary. Improvement can be achieved by introducing internal nodes using panel assemblies (Table 4.6).

(b) For free vibration of a one-bay shear wall (Fig.4.4) and a coupled shear wall (Fig. 4.6), comparison with the equivalent full finite element model provided the following conclusions:

- (i) A lumped mass representation for panel element substructures leads to accurate results, provided (1) nodal points are adequately distributed, or (2) concentrated masses are introduced which are of predominant influence relative to panel masses (Fig.4.5).
- (ii) Execution time can be significantly reduced when panel elements are applied (Models B and C of Fig. 4.4). However, if all boundary nodes are retained, no savings can be expected (Model A in Fig.4.4).
- (iii) Use of panel elements as a substructure option results in significant savings in data preparation.

(c) A static analysis of a coupled shear wall (Fig.4.7) and a cantilever beam (Fig.4.10) demonstrated the

effectiveness of the stress recovery scheme
(Figs. 4.9 and 4.11)..

TABLE 4.1 COMPARISON OF FREQUENCIES - CANTILEVER PLATE

Mode	Frequencies (cps)		ϵ_F				
	Ref. [44]	Consistent Mass	Consistent Mass	Lumped Mass CASE CASE CASE CASE A B C D			
1	840	838	0.20	7.7	7.8	9.0	58.0
2	3613	3614	-0.03	34.0	34.0	34.0	62.0
3	5350	5350	-2.3	19.5	20.0	21.0	65.0
4	11930	11930	-1.2	36.6	37.0	38.0	62.0

TABLE 4.2. COMPARISON OF NORMALIZED MODE SHAPES -
CANTILEVER PLATE

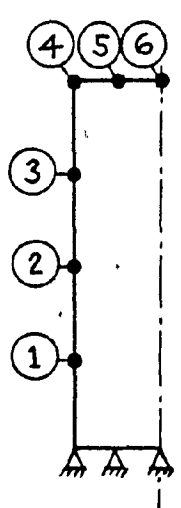
Mode	Node No.	Ref. [41]	Consistent Mass	Lumped Mass CASE A	
1	1	0.089	0.087	0.084	 <p style="text-align: center;">NODE NUMBERING</p>
	2	0.328	0.327	0.321	
	3	0.651	0.649	0.644	
	4	0.997	0.997	0.997	
	5	0.998	0.999	0.999	
	6	1.0	1.0	1.0	
2	1	0.234	0.245	0.264	
	2	0.609	0.600	0.639	
	3	0.845	0.864	0.889	
	4	1.0	1.0	1.0	
	5	0.504	0.505	0.505	
	6	0.0	0.0	0.0	
3	1	0.432	0.439	0.502	
	2	0.758	0.803	1.0	
	3	0.192	0.247	0.490	
	4	-0.952	-0.935	-0.771	
	5	-0.972	-0.982	-0.875	
	6	-1.0	-1.0	-0.917	
4	1	0.592	0.580	0.477	
	2	0.681	0.715	0.532	
	3	-0.133	-0.113	-0.278	
	4	-1.0	-1.0	-1.0	
	5	-0.530	-0.540	-0.539	
	6	0.0	0.0	0.0	

TABLE 4.3 MODE SHAPES COMPARED TO REF. [41] -
CANTILEVER PLATE

Mode	ϵ_s	
	Consistent Mass	Lumped Mass CASE A
1	0.001	0.005
2	0.011	0.026
3	0.018	0.234
4	0.025	0.158

TABLE 4.4 FREQUENCIES AND MODE SHAPES COMPARED TO
REF. [41] - CANTILEVER PLATE AND DIFFERENT
PANEL ELEMENTS

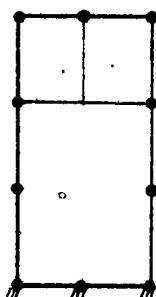
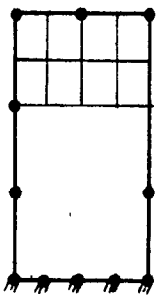
Panel Element Type	Mode	Consistent Mass		Lumped Mass CASE A	
		ϵ_F	ϵ_S	ϵ_F	ϵ_S
 (100, 101/2)	1	0.10	0.002	7.5	0.007
	2	-0.30	0.003	35.0	0.018
	3	-2.8	0.022	19.6	0.239
	4	-0.40	0.020	39.7	0.126
 (3, 100, 110, 110/1, 2, 2)	1	0.20	0.001	8.1	0.006
	2	-0.20	0.004	35.2	0.027
	3	-2.8	0.018	21.7	0.222
	4	-2.7	0.022	38.3	0.147

TABLE 4.5 COMPARISON OF FREQUENCIES - FREE-FREE PLATE

Mode	Frequencies (cps)		ϵ_F	
	Ref. [43]	Consistent Mass	Consistent Mass	Lumped Mass (CASE A)
1 ₀	3257	3302	-1.4	23.1
2	4738	4807	-1.4	38.0
3	5900	6642	-9.1	11.0
4	8500	8585	-1.0	38.0

TABLE 4.6 FREQUENCIES AND MODE SHAPES COMPARED TO REF. [41] - CANTILEVER PLATE AS PANEL ASSEMBLIES

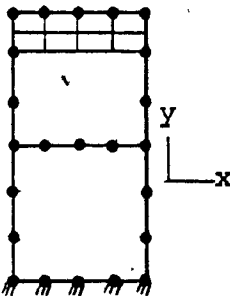
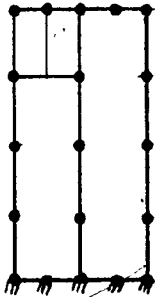
Panel Element Type	Mode	Consistent Mass		Lumped Mass CASE A	
		ϵ_F	ϵ_S	ϵ_F	ϵ_S
 2 x (1,101/1,1)	1	0.3	0.0008	5.4	0.005
	2	0.1	0.005	23.4	0.003
	3	-0.5	0.010	18.2	0.018
	4	-0.2	0.024	25.0	0.039
 2 x (102)	1	0.1	0.001	3.6	0.004
	2	-0.03	0.011	15.6	0.012
	3	-0.7	0.011	10.8	0.116
	4	-0.4	0.021	20.0	0.041

TABLE 4.6 (continued)

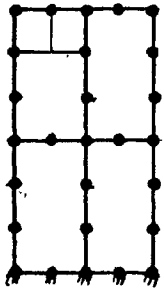
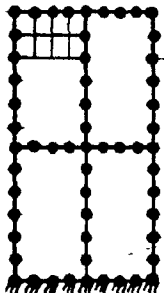
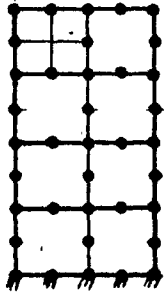
Panel Element Type	Mode	Consistent Mass		Lumped Mass CASE A	
		ϵ_F	ϵ_S	ϵ_F	ϵ_S
 4 x (1,101/2,2)	1 2 3 4	0.2 0.08 -0.4 0.05	0.001 0.004 0.011 0.022	2.1 14.1 7.7 15.6	0.002 0.003 0.063 0.035
 4 x (1,101/2,2)	1 2 3 4	0.3 0.2 0.0 0.1	0.001 0.011 0.008 0.023	2.3 14.9 6.5 15.7	0.002 0.012 0.054 0.053
 8 x (101)	1 2 3 4	0.2 0.1 -0.3 0.2	0.001 0.011 0.011 0.021	1.5 10.4 4.8 9.1	0.002 0.011 0.039 0.026

TABLE 4.7 NATURAL PERIODS COMPARED TO FULL FINITE
ELEMENT MODEL - HOMOGENEOUS SINGLE BAY
SHEAR WALL

Mode	Period (sec)	ϵ_p		
		FULL MODEL	MODEL A	MODEL B
1	0.1455	1.1	1.2	1.6
2	0.0268	3.8	3.8	5.0
3	0.0183	0.2	0.3	0.4
4	0.0111	5.8	6.1	8.1
5	0.0067	8.3	9.1	12.9
6	0.0061	2.1	2.5	3.4
7	0.0047	11.9	13.7	20.6
8	0.0037	5.8	7.0	10.8
9	0.0036	0.7	0.3	8.9
10	0.0030	10.8	17.3	11.9
11	0.0028	15.3	17.7	14.7
12	0.0027	17.3	18.6	16.7

TABLE 4.8 MODE SHAPES COMPARED TO FULL FINITE ELEMENT MODEL - SINGLE BAY SHEAR WALL WITH CONCENTRATED FLOOR MASSES

Mode	ϵ_s		
	MODEL, A	MODEL B	MODEL C
1	0.0001	0.0001	0.0002
2	0.0025	0.0029	0.0043
3	0.0005	0.0004	0.0003
4	0.0042	0.0049	0.0072
5	0.0047	0.0060	0.0096
6	0.0100	0.0083	0.0097
7	0.0059	0.0087	0.0150
8	0.0072	0.0047	0.0059
9	0.0084	0.0038	0.0046
10	0.0690	0.0770	0.0640
11	0.0640	0.0770	0.0790
12	0.0062	0.0160	0.0670

TABLE 4.9 NATURAL PERIODS COMPARED TO FULL FINITE ELEMENT MODEL - COUPLED SHEAR WALL

	Mode	Period (sec)		ϵ_p
		FULL MODEL	USING PANEL ELEMENTS	
Homogeneous Wall	1	0.1895	0.1902	0.50
	2	0.0420	0.0414	1.50
	3	0.0330	0.0329	0.40
	4	0.0314	0.0312	0.84
	5	0.0186	0.0185	1.50
	6	0.0185	0.0181	2.10
	7	0.0174	0.0175	0.56
	8	0.0150	0.0149	0.65
	9	0.0138	0.0136	0.88
	10	0.0130	0.0127	2.20
	11	0.0128	0.0124	2.90
	12	0.0127	0.0123	3.10
Including Floor Masses	1	0.4390	0.4386	0.09
	2	0.0932	0.0935	0.28
	3	0.0734	0.0734	0.14
	4	0.0691	0.0692	0.20
	5	0.0430	0.0430	0.17
	6	0.0400	0.0400	0.08
	7	0.0389	0.0391	0.66
	8	0.0363	0.0364	0.25
	9	0.0340	0.0342	0.44
	10	0.0332	0.0334	0.53
	11	0.0330	0.0332	0.56
	12	0.0329	0.0331	0.59

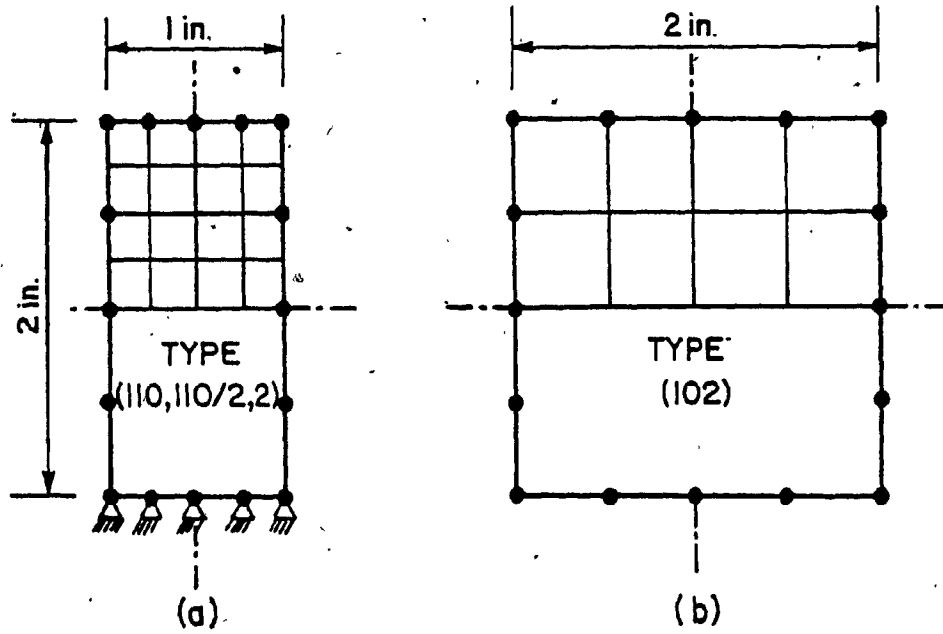


FIG. 4.1 PANEL ELEMENT MODELS FOR FREE VIBRATION IN PLATE BENDING; (a) CANTILEVER PLATE, AND (b) FREE-FREE PLATE

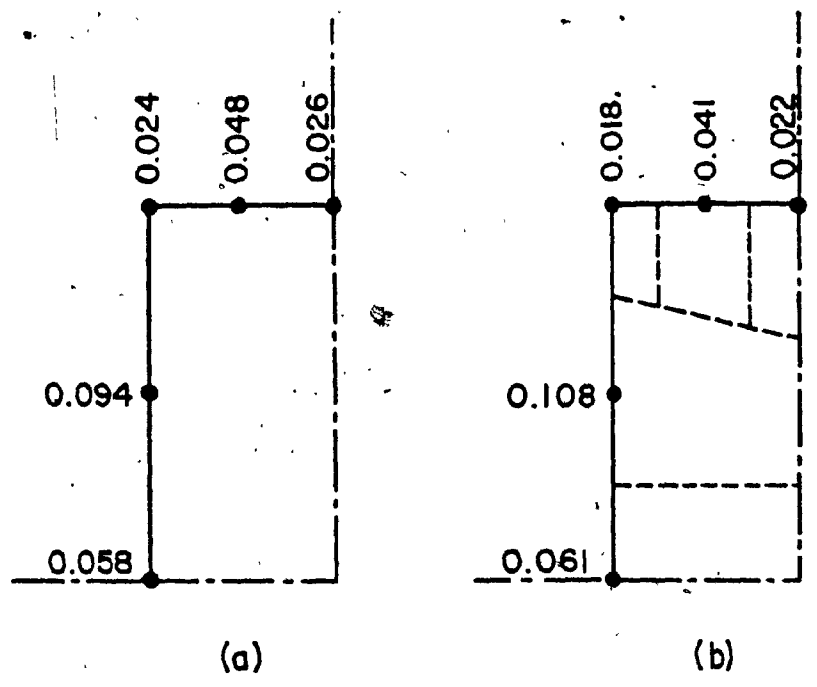
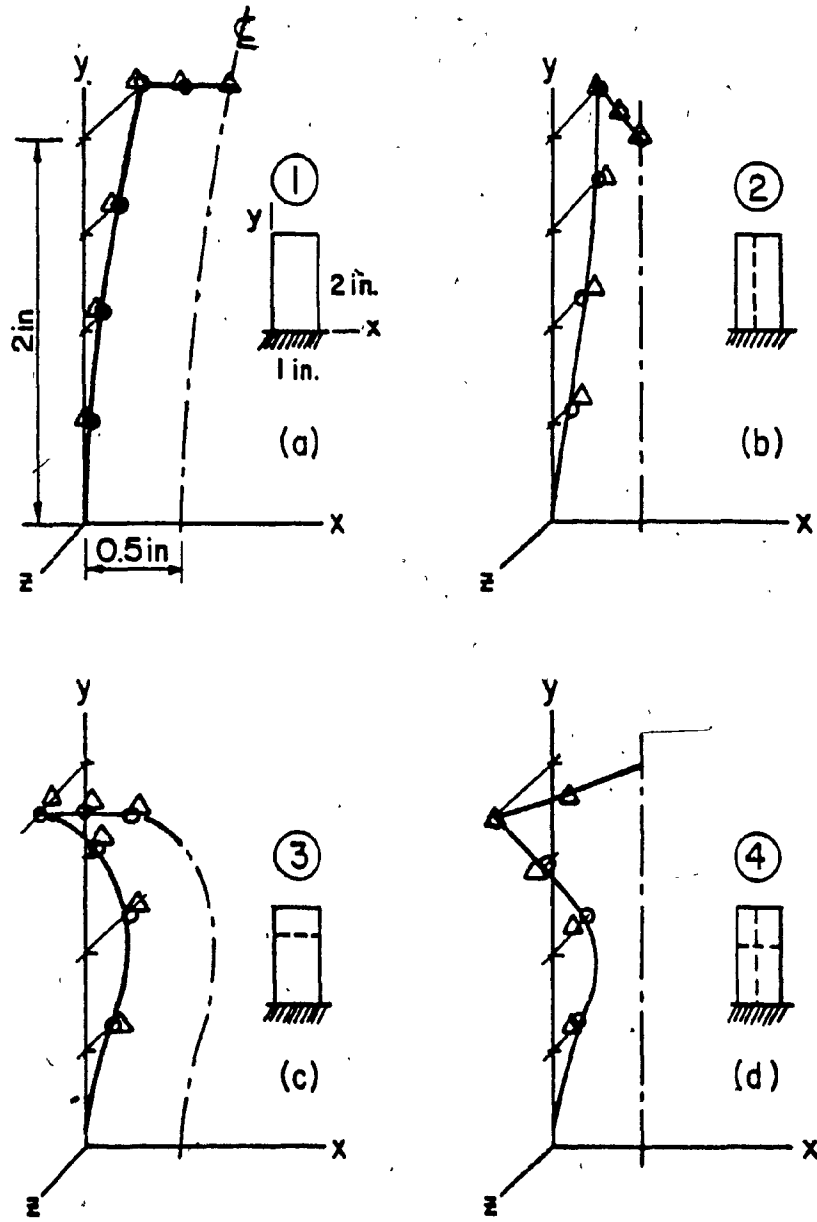


FIG. 4.2 LUMPING OF PANEL MASS IN LATERAL DIRECTION;
(a) CONSISTENT LUMPING, AND (b) LUMPING BY
TRIBUTARY AREAS



MODE NUMBER
 — REF. [41] CONSISTENT MASS LUMPED MASS - - - NODAL LINES

FIG. 4.3 COMPARISON OF MODE SHAPES FOR CANTILEVER PLATE

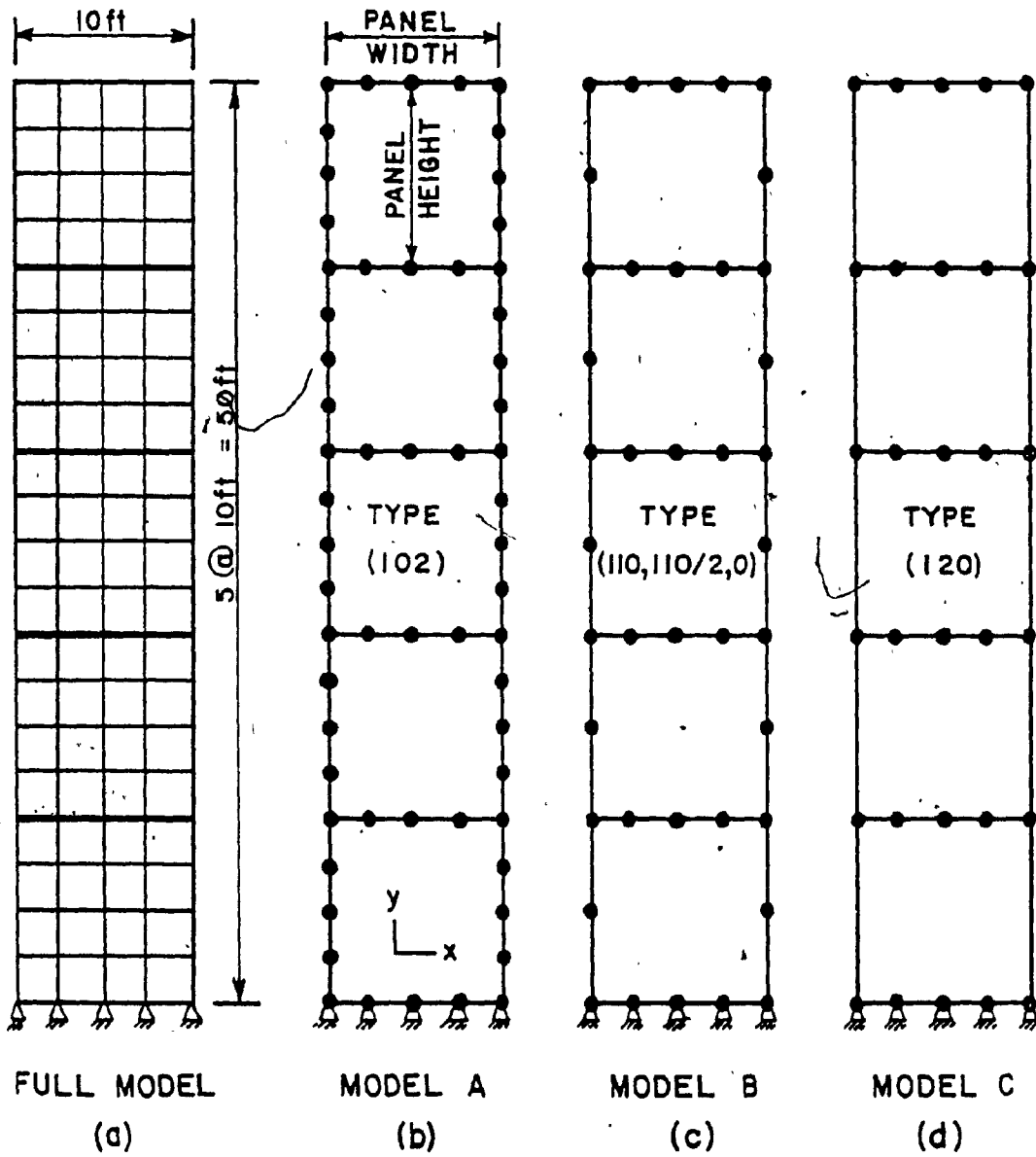


FIG. 4.4 SINGLE BAY SHEAR WALL; (a) FULL FINITE ELEMENT MODEL, AND (b) - (d) USING PANEL ELEMENTS (1 ft = 0.305 m)

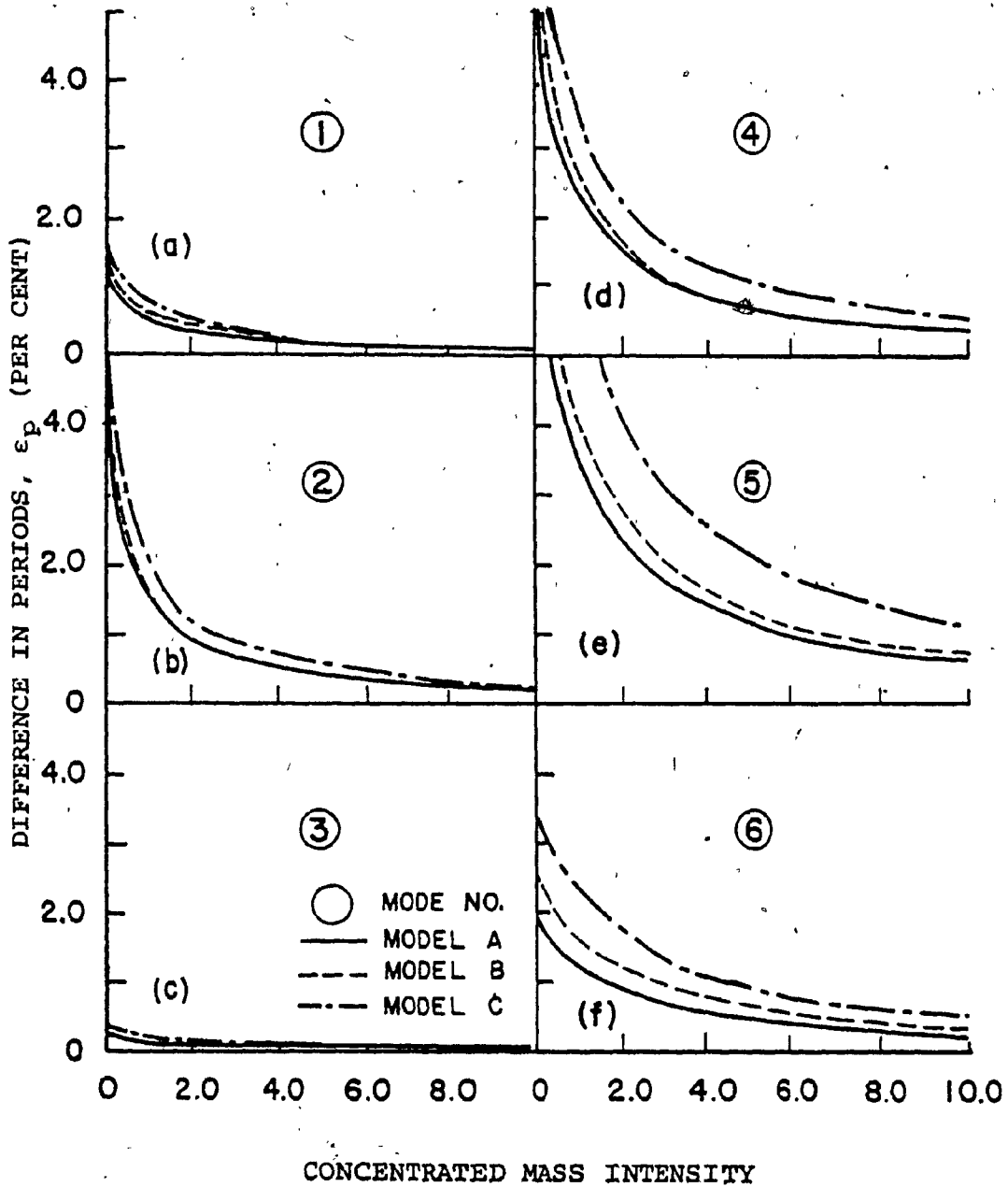


FIG. 4.5 EFFECT OF FLOOR MASS INTENSITY ON DIFFERENCE IN NATURAL PERIODS COMPARED TO THE FULL FINITE ELEMENT MODEL

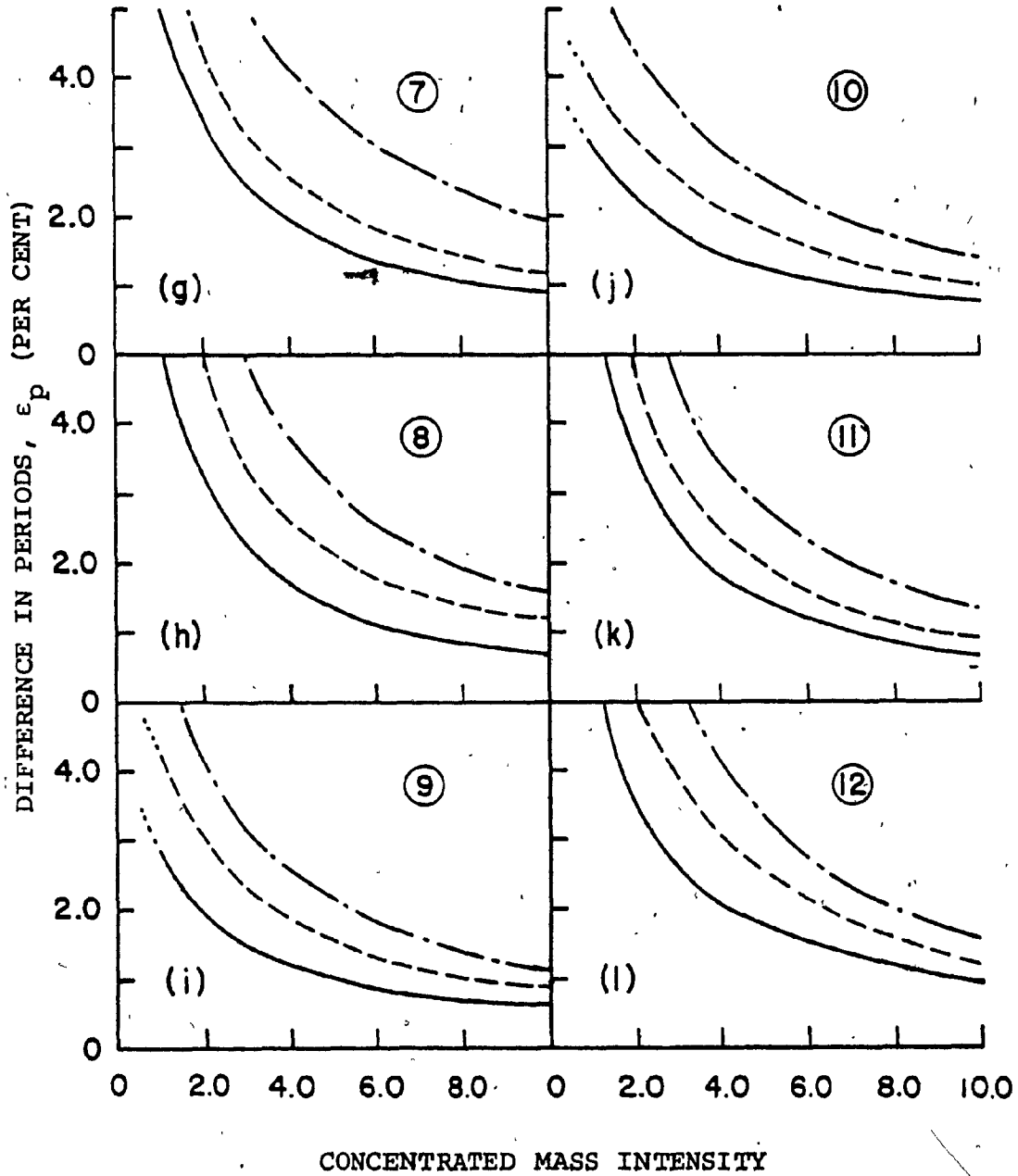


FIGURE 4.5 (continued)

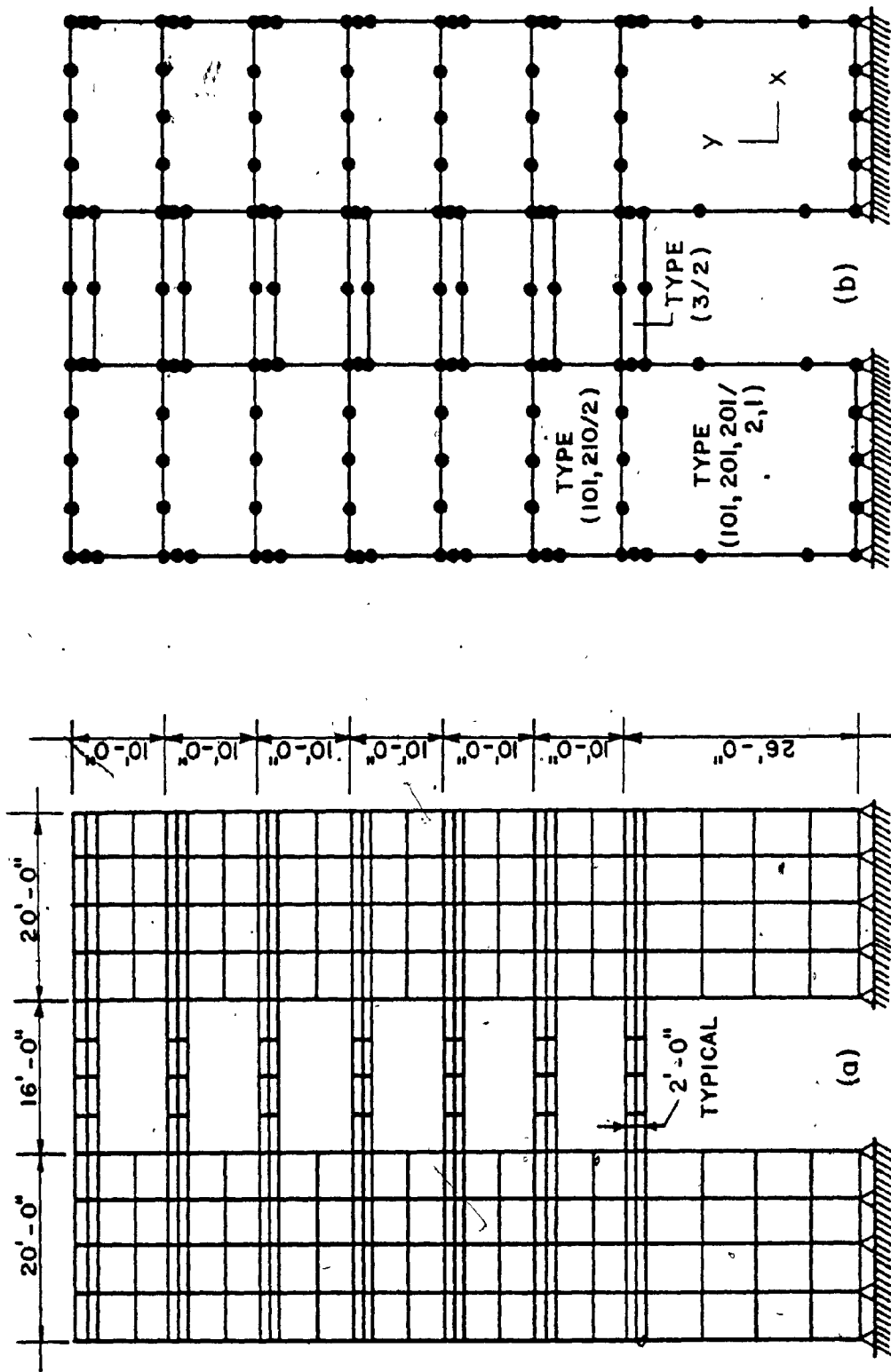


FIG. 4.6 COUPLED SHEAR WALL FOR DYNAMIC ANALYSIS; (a) FULL FINITE ELEMENT MODEL, AND (b) PANEL SUBSTRUCTURE REPRESENTATION (1 ft = 0.305 m)

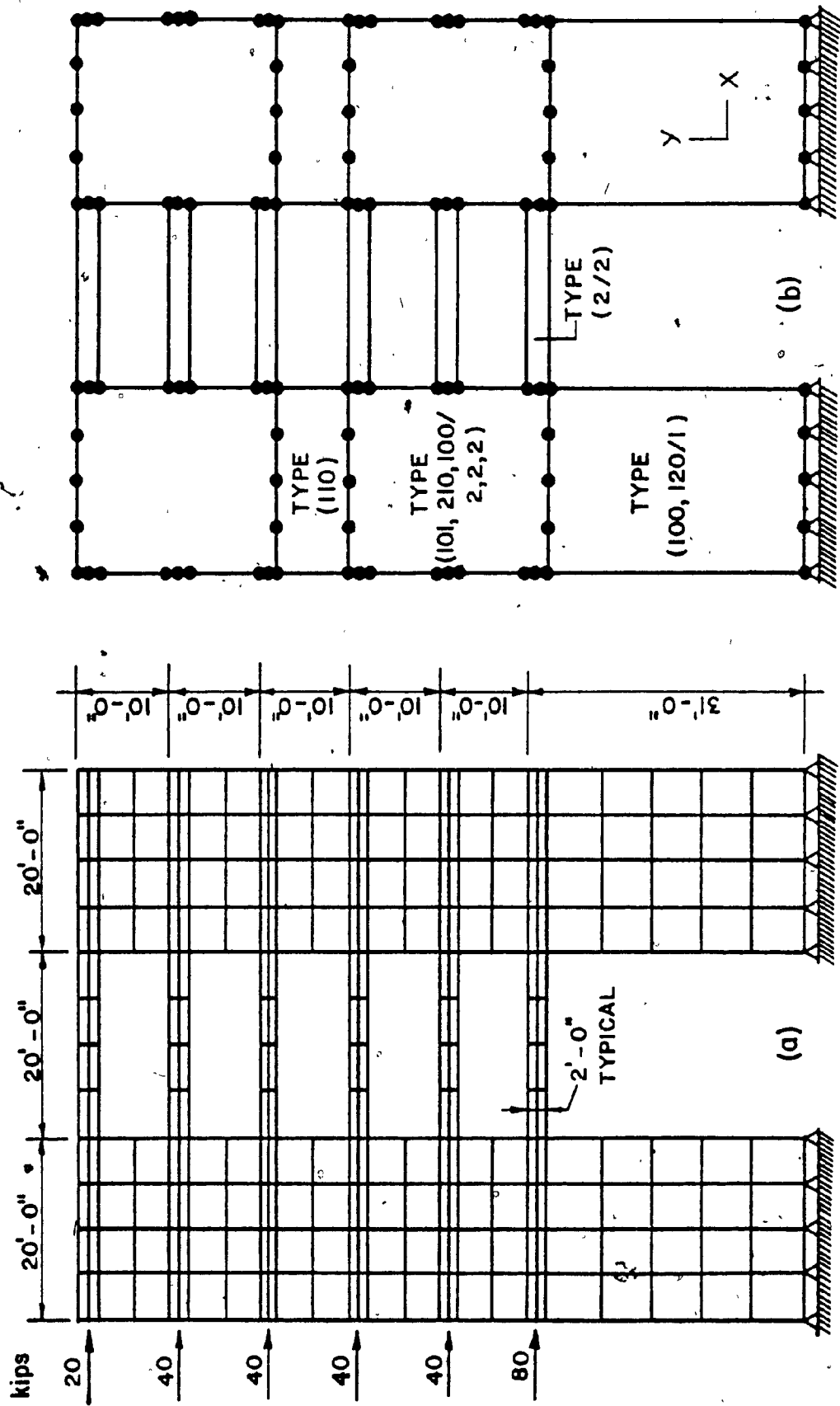


FIG. 4.7 COUPLED SHEAR WALL, FOR STATIC ANALYSIS; (a) FULL FINITE ELEMENT MODEL (FROM [44]), AND (b) PANEL SUBSTRUCTURE REPRESENTATION (1 ft = 0.305 m)

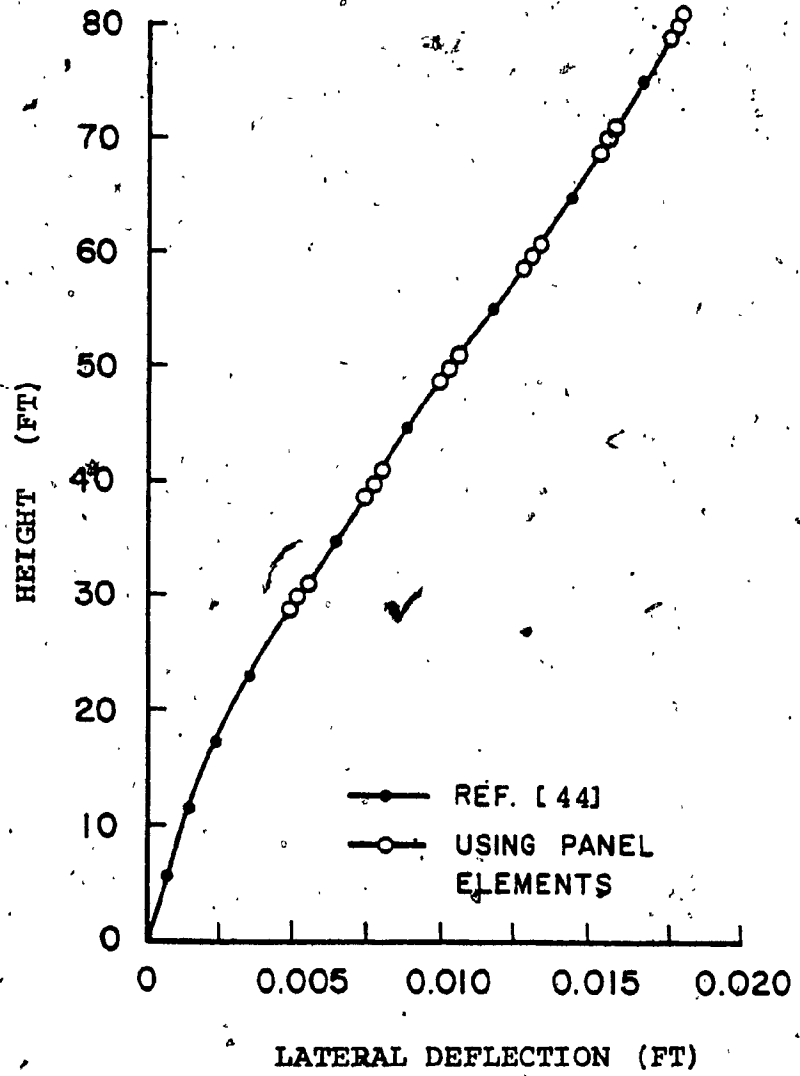


FIG. 4.8 LATERAL DEFLECTION UNDER STATIC LOADING;
LEFT WALL (1 ft = 0.305 m)

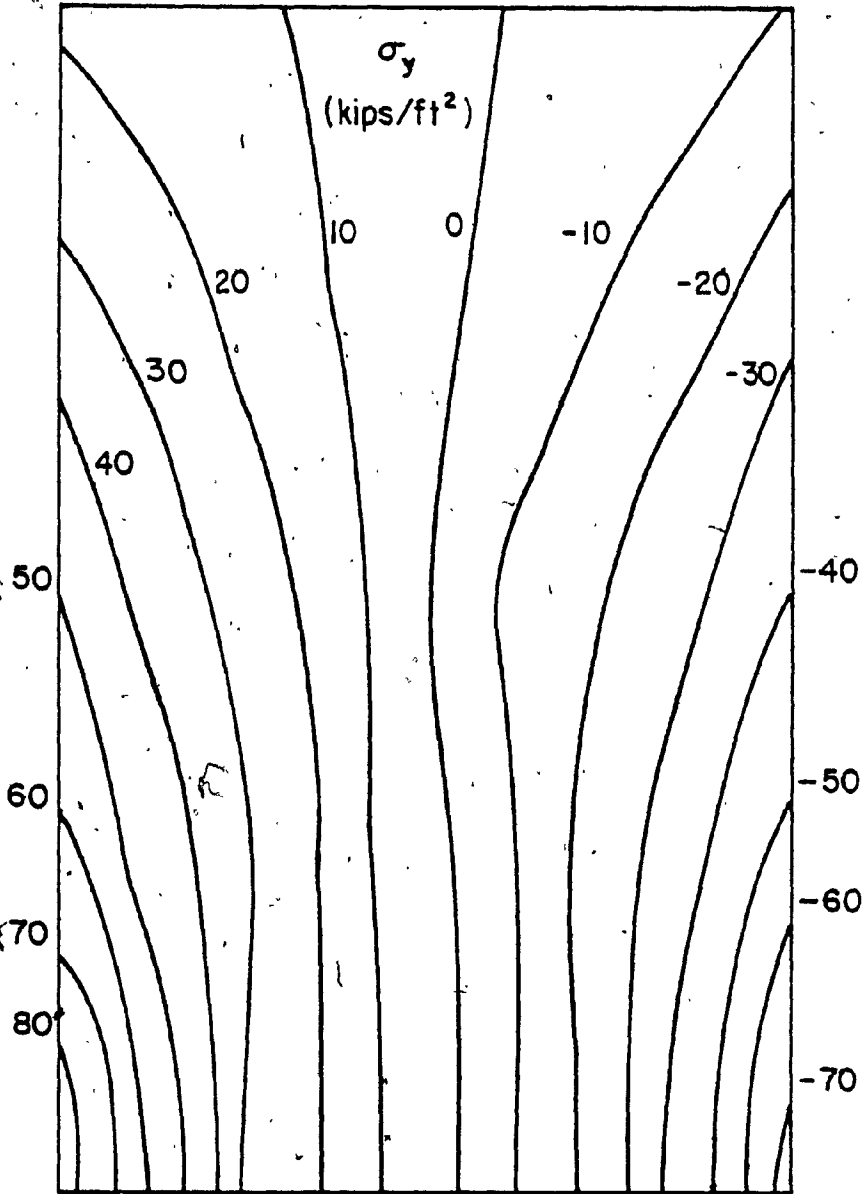


FIG. 4.9 ISOBARS OF σ_y STRESSES DUE TO STATIC
LOADING IN Y LOWER LEFT PIER
(1 kip/ft² = 47.9 kN/m²)

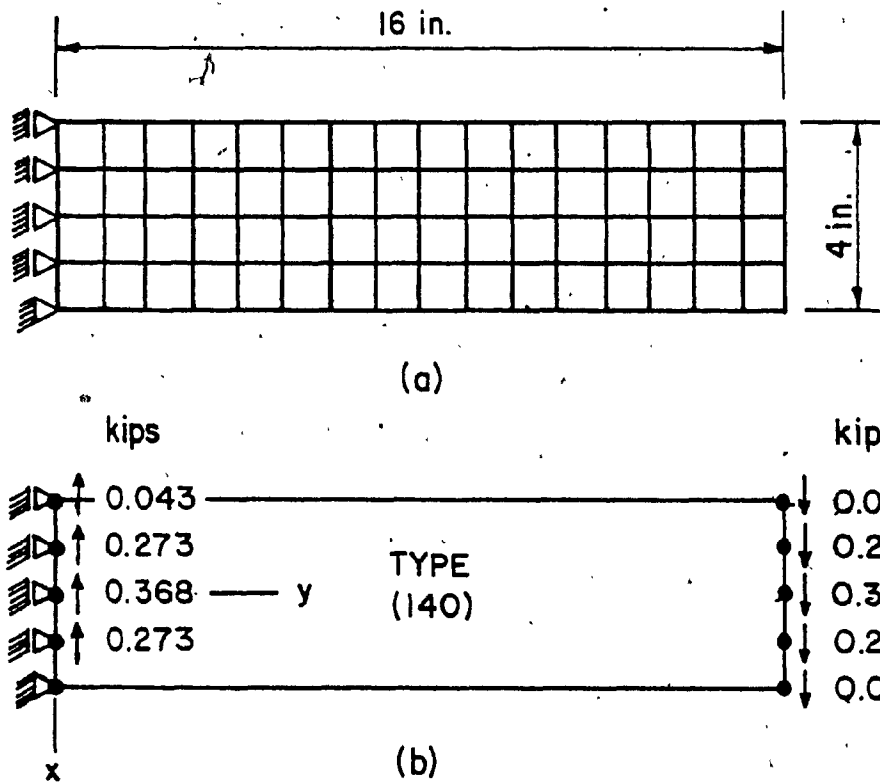


FIG. 4.10 CANTILEVER BEAM UNDER STATIC LOADING; (a) FULL FINITE ELEMENT MODEL, AND (b) PANEL SUBSTRUCTURE (1 in = 2.54 cm, 1 kip = 4.45 kN)

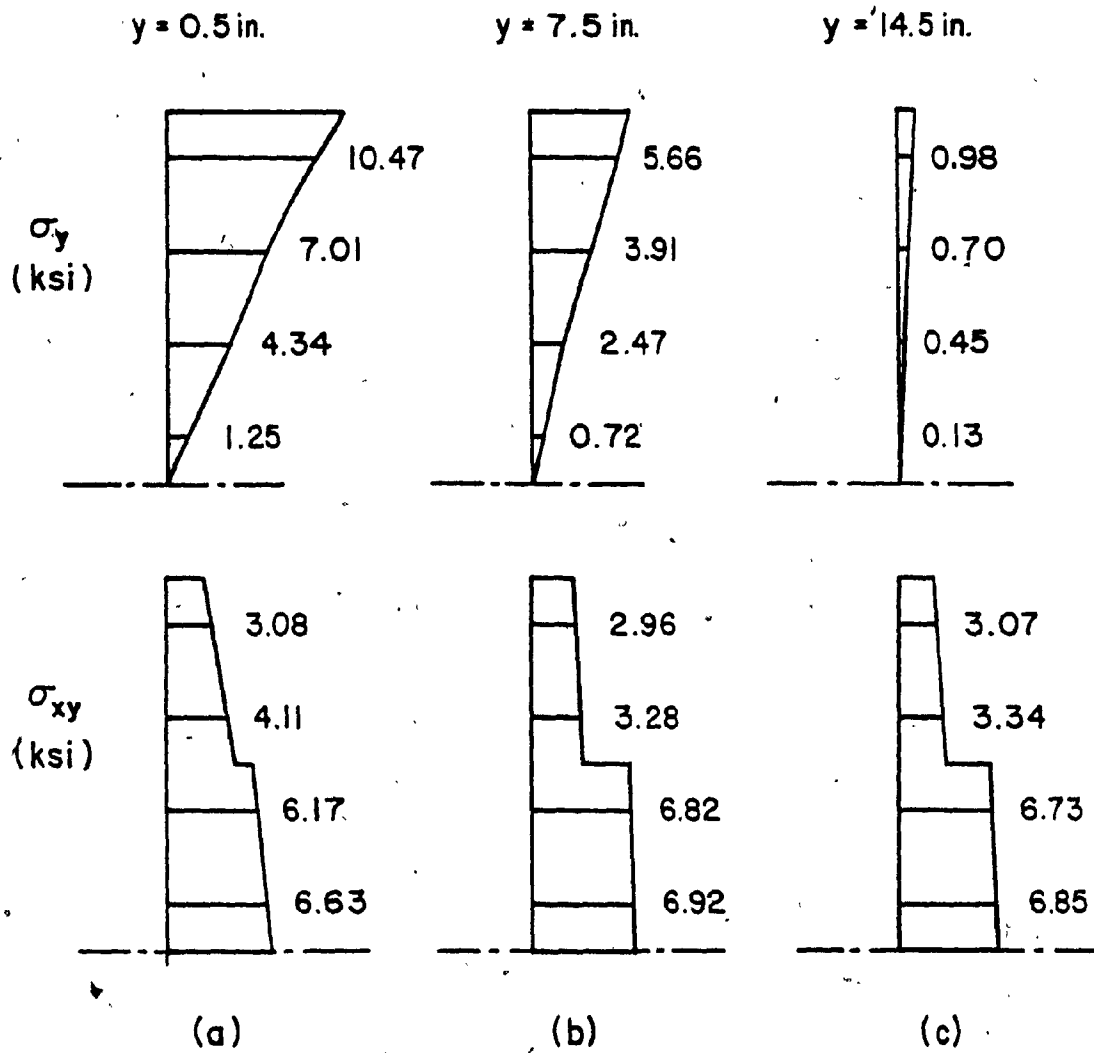


FIG. 4.11 NORMAL STRESS σ AND SHEAR STRESS σ_{xy} AT DIFFERENT CROSS y SECTIONS OF CANTILEVER BEAM (1 in = 2.54 cm, 1 ksi = 6.9 MN/m²).

CHAPTER V

SEISMIC BEHAVIOUR OF A 12-STOREY PRECAST
PANEL SHEAR WALL

CHAPTER V

SEISMIC BEHAVIOUR OF A 12-STOREY PRECAST
PANEL SHEAR WALL5.1 INTRODUCTION

In the first part of this study, a panel element was developed using a super finite element technique. Special considerations which are necessary in dynamic substructuring, such as mass representation and adequate distribution of degrees of freedom over the structure, were discussed in detail. Illustrative examples were presented, for plate bending and shear wall problems, representing applications of panel elements as a substructure option for cases where full continuity exists at the interface of substructures.

The real potential of panel elements in the field of structural engineering lies, however, in applications for pre-fabricated structures or, more precisely, for large-panel structures. Here, panel elements represent prefabricated concrete panels or panels manufactured of other materials, such as corrugated steel. The resulting analysis becomes particularly efficient when panels are connected only at discrete points since a large number of boundary nodes can be eliminated; band width of system matrices, normally increased when applying substructuring techniques, is then significantly decreased.

Precast panel structures are generally visualized as three-dimensional panel assemblies where the main difference from conventional construction lies in the existence of connections at joints between panels, introducing discontinuities into the otherwise monolithic structure. Vertical in-plane panel walls maintain their function as shear walls to resist lateral loads; joint discontinuities along vertical or horizontal lines, however, introduce new aspects into shear wall design and analysis.

If full continuity at horizontal joints exists because of governing vertical loads - a commonly used assumption - individual "panel-columns" act as "elementary cantilevers" interconnected at their vertical interfaces. Alternatively, the shear wall as a whole may be described as a "complex cantilever" and belongs to the category of coupled shear walls.

The literature for precast panel structures involves the area of construction with large prefabricates in a more general sense, which includes also behaviour and design of joints and connections across joints. From the analytical viewpoint, and considering lateral load resistance, it encompasses the field of shear wall design with particular emphasis on coupled shear walls.

Lewicki [45], followed by others [46,47], reviewed procedures for design as well as analysis of fully-panelized and semi-panelized (i.e., with cast-in-place floors) systems. Vertical continuous shear joints attracted most of the attention in the area of connections in panel structures and were studied and reviewed by Lewicki and Pauw [48], as well as by Cholewicki [49] and Olesen [50]. Shemie [51,52] investigated the strength and slip behaviour of discrete connectors for a particular prefabricated system.

The coupled shear wall problem has been solved mostly for uniform cantilevers by assuming a continuous shear and moment resisting lamina, using differential equations or energy methods. Originally proposed by Rosman [53], numerous investigations have been reported for static loading, while more recent developments have concentrated on dynamic analysis [54,55,56,57]. Paulay [58] presented a comprehensive work on shear wall as well as coupled shear wall design for seismic loading. In a recent study by MacLeod [59] an extensive literature review is given on coupled shear wall analysis.

Analytical work on shear walls, with particular attention to panel structures, was performed by Stafford-Smith and Lau [60] who investigated static stability in panel assemblies including three-dimensional cases. Burnett and Rajendra [61] investigated a panelized wall system coupl-

ed by floor slabs and subjected to lateral static loading. Different degrees of fixity in the wall-to-floor connections were assumed with vertical joints either rigid or fully plastic.

Failure sequences were studied by Stafford-Smith and Rahman [62] in order to establish a collapse limit state for precast panel structures, using the finite element method and assuming strength models for joints. Pollner et al [63] analyzed large-panel shear walls with full fixity in horizontal joints, assuming non-linear vertical joint, and lintel beam, characteristics and applying stepwise lateral static loading; a matrix formulation was used in the analysis. A linear dynamic analysis of large precast panel buildings with panels represented by substructures was performed by Frank et al [64]. Horizontal joints, modeled by anisotropic finite elements, were the major parameter in this study.

The present study examines the dynamic behaviour of a large-panel shear wall using response spectrum analysis to account for seismic loading. Horizontal joints are assumed to be rigid (i.e., elementary cantilevers exist) and discrete connectors are located along the vertical joints at floor levels. Elastic behaviour is assumed throughout the structure. The panel structure itself consists of 12 storeys and 3 bays, where panels extend over one storey and one bay. The substructuring technique presented in Chapters III and IV is applied for efficient panel modelling.

The first part of the investigation deals with variations in structural response due to connector shear stiffness, where magnitude changes from rigid to fully flexible while full rigidity in the axial direction is assumed. The effect of non-uniform distribution of connector shear stiffness along the height of the structure is examined; this includes the introduction of monolithic storeys.

In the second part of this investigation the effect of horizontal discontinuities is studied, created by a missing panel or by non-uniform floor mass distribution. Assessment of the importance of connector axial stiffness and a comparison of dynamic analysis to a code static loading concludes the investigation.

Response characteristics are evaluated for overall structural behaviour. However, emphasis is placed on forces incurred in connectors since connections represent the major problem in the design of precast panel structures.

5.2 COMPONENTS OF PANEL STRUCTURES

5.2.1 Overall Structural Functioning

A precast panel structure must fulfill the same structural functions in terms of vertical and horizontal loading as does a conventional fully monolithic structure. Principal differences lie in the more industrialized fabri-

cation and erection procedures, which leads immediately to the problem of joints and connections. The main elements in a panel structure are therefore the panels and the connections or joints between panels. Force transmission across joints can take place in a continuous fashion or through discrete connectors.

An additional consideration, common to conventional monolithic structures as well as to panelized systems but not included in this study, is the interaction between vertical walls and the horizontal slab - or horizontal floor panels - where the slab introduces gravity loads to the walls and performs also the important function of a stiff diaphragm distributing lateral loads to individual resisting walls. The latter are represented in this case by vertical panel assemblies functioning as shear walls.

Stability concepts in panelized systems are generally of a more complex nature than encountered for monolithic structures. Local failure of one panel, e.g., due to failure of connectors at the joints, may lead to partial or overall collapse of the structure, commonly known as progressive collapse.

5.2.2 Panels

In a panel shear wall a single panel or several panels, could be used to extend over the wall width; in the case of a single panel no vertical joints exist. In height, a panel extends normally over one or two storeys. Panel thickness can vary according to building height. Very often, however, panel dimensions are governed by practical considerations such as transportation, for example.

Forces in wall panels are mainly due to membrane action. The magnitude and nature of membrane forces depend largely on the location of the panel in the shear wall and on the type of connections introducing these forces. Special provisions (e.g., in reinforcement) may be necessary when force concentrations exist. Ribs, particularly along panel edges, may improve panel performance and overall strength, as well as the detailing of the connections.

5.2.3 Joints

Properties of the connections across joints influence significantly the overall rigidity of concrete panel structures, for lateral as well as vertical loading. Connection properties affecting response are mainly the shear and axial stiffness. If non-linear effects are taken into account, ductility as expressed by the force-displacement characteristics

becomes important also. Connections can be classified as vertical joints, functioning predominantly as "shear joints", and as horizontal joints, which are predominantly "compression joints", although tension or shear may become critical in particular cases. Depending upon actual details, different connections may be either of the (1) continuous, or (2) discrete type.

In continuous vertical connections, shear stiffness as well as shear strength depend on the degree of reinforcement and the type of joint (e.g., keyed joints) [48,50]. In the axial direction incompressibility is usually assumed. Horizontal, continuously reinforced joints result normally in monolithic behaviour. This leads to the concept of elementary cantilevers in the analysis of panelized structures [46,50].

For discrete connectors, shear and axial stiffnesses are lumped at a point. Large force concentrations can occur with discrete connectors, resulting in localized strength and ductility requirements. Discrete connectors in a vertical joint, subjected mainly to shear forces, could consist of concrete components such as floor slabs or short shear beams, or they could be represented by mechanical connectors. If a mechanical device is applied, not more than two connectors should be used per side, as close as possible to the panel corners, principally for practical reasons [51]. In analysis, this arrangement allows lumping of connectors

from adjacent storeys into one connector at the floor level, provided rigid horizontal joints exist.

Horizontal joints with discrete connectors are generally "drypacked" and a rigid continuity may be preserved due to vertical compressive forces and also to the accompanying frictional resistance (i.e., elementary cantilevers can be assumed in analysis). A drypacked joint should be investigated for tension under lateral loading, since discrete connectors have to accommodate the tension forces. In a tension zone continuity does not exist and non-linear characteristics are introduced into the structure and its analysis.

5.3 DESCRIPTION OF STRUCTURE AND ANALYTIC CONSIDERATIONS

5.3.1 Geometric and Material Properties

The structure selected for study consists of the 12-storey 3-bay concrete panel shear wall shown in Fig.5.1. All panels are one storey in height (i.e., 10 ft (3.05 m)), panel width is 12 ft (3.66 m) and panel thickness is 8 inches (0.203 m). Assumed values for material properties, specified in terms of Young's modulus, Poisson's ratio and mass density, are, respectively: $E = 4,000 \text{ kips/in}^2$ ($2.76 \times 10^7 \text{ kN/m}^2$), $\nu = 0.17$ and $\rho = 0.00482 \text{ kips-sec}^2/\text{ft}^4$ (0.0025 kg/cm^3). Floor masses are based on a 25 ft (7.63 m) width

of tributary floor area. An allowance for a 7-inch (0.178 m) concrete slab, partitions and exterior walls leads to a uniformly distributed dead load of 3.7 kips per ft-width of the wall (54 kN/m), applied at each floor level.* Assumed characteristics and properties for connectors are discussed in Section 5.3.2.

5.3.2 Connector Model and Connector Stiffnesses

The mathematical model of a connector can be described by an idealized two-node spring element with 4 degrees of freedom and zero dimensions in space, as shown in Fig.5.2. The connector element stiffness matrix is then formulated simply as

$$[K]_c = \begin{bmatrix} k_a & 0 & -k_a & 0 \\ 0 & k_v & 0 & -k_v \\ -k_a & 0 & k_a & 0 \\ 0 & -k_v & 0 & k_v \end{bmatrix} \quad (5.1)$$

where k_a and k_v are connector axial and shear stiffness, respectively. This connector element is also described in the literature as the "bond element" [64].

* For simplicity and to allow for possible roof loads (e.g., mechanical equipment) all storeys, including the roof have a lumped mass corresponding to $36 \times 3.7 = 133.2$ kips (593.0 kN).

Computer analysis showed that values of shear stiffness k_v must be varied by an order of magnitude to effect significant changes in response. Analysis also showed that, for the present structure, a value of $k_v = 10^7$ kips/ft (14.6×10^7 kN/in) results in fully rigid conditions. This means that $k_v = 10^7$ kips/ft yields the same results as for a rigid model (i.e., full continuity at connector locations), disregarding minor numerical differences. There was also no noticeable change in the rate of convergence during a subspace iteration for solution of the eigenvalue problem.

It is therefore possible to perform an analysis with values for k_v ranging from 10^7 kips/ft (14.6×10^7 kN/m) to zero and model the given structure with panel-to-panel connection from full fixity at the connecting points to full flexibility between the individual cantilevers. Since the axial stiffness in vertical joints is generally of much less importance than the shear stiffness, k_a was assumed as fully rigid, i.e., $k_a = 10^7$ kips/ft (14.6×10^7 kN/m) in all of the following analyses.

To relate connector stiffness to practical values, two interpretations are possible. First, in relation to a mathematical panel model, Fig. 5.4 shows a half-panel with fixed conditions on one side and possible vertical displacements on the other. A distributed force results in a "panel

stiffness" of $k = 1.25 \times 10^5$ kips/ft (18.2×10^5 kN/m). Another, more practical, approach consists of distributing the concentrated stiffness of the panel connector and computing an equivalent distributed stiffness per unit area of the joint which can be compared to values reported in the literature [48,50] for continuous connections. Table 5.1 shows the relation between panel, distributed joint, and discrete connector stiffnesses.

5.3.3 Summary of Assumptions

Dimensions and material properties of the investigated panel shear wall (see Fig. 5.1) were discussed in Section 5.3.1. The panel mathematical model used in the analysis is derived by applying the substructuring technique described in Chapters III and IV. Connector model and possible connector stiffnesses are presented in Section 5.3.2.

The physical assumptions related to structural behaviour and the approximations made for the finite element analysis are summarized below:

- (1) Each wall, or panel column, acts as an elementary cantilever since full continuity is assumed along horizontal joints. Connections across the vertical joints between individual cantilevers exist only at floor levels, thus leading to a

coupled shear wall effect.

- (2) Panel walls are represented by panels extending over one bay and one storey. The finite element models of panels are shown in Fig. 5.3. Interior nodes are eliminated, whereas vertical boundary nodes may be eliminated or may remain if full continuity is desired (see Sections 5.4.1, and 5.4.2). For the basic four-node element at zero level, plane stress and incompatible displacement modes are assumed. Linear elastic behaviour exists in panel walls.
- (3) Nodal points, with 2 degrees of freedom per node, exist only at connection points between panels or between panels and connectors. At the same time, nodal points are located where high mass concentrations occur, i.e., at floor levels.
- (4) A lumped mass representation is used, expressed by the retained degrees of freedom. Panel masses are defined by the panel lumped mass matrix (see Sections 4.1 and 4.3), whereas floor masses are inserted separately at each floor level.
- (5) The assumed mass lumping, together with the selected number of degrees of freedom, results in solutions of the reduced eigenvalue problem sufficiently

accurate for practical purposes, compared to the full finite element model (see Section 4.3.1).

- (6) Connection between cantilever walls is maintained by discrete connectors located at floor levels and acting as vertical and horizontal springs (Fig. 5.2(a)). Thus, transmission of shear and axial forces is ensured, while moments are not transmitted by connectors. In the axial direction connectors are assumed to be incompressible (i.e., $k_a = 10^7$ kips/ft (14.6×10^7 kN/m)) throughout the investigation. Shear stiffness, however, is varied from rigid to fully flexible (i.e., from $k_v = 10^7$ kips/ft (14.6×10^7 kN/m) to $k_v = 0$). Elastic connector behaviour is assumed.
- (7) Response of the structure due to earthquake excitation is predicted by the response spectrum analysis summarized in Section 5.3.4.

5.3.4 Response Spectrum Analysis

All matrices in this Section represent the reduced system. Nevertheless, notations normally assigned to the full finite element model are used, e.g., $[K]$ instead of $[\bar{K}]$ (see Chapter II). For analytical modelling of the panel

structure the reduced system stiffness matrix $[K]$ is assembled from panel and connector elements, whereas the reduced diagonal lumped mass matrix $[-M-]$ receives contributions directly from floor masses in addition to the panel mass.

Solution of the eigenvalue problem (see Eq. (2.5) or (2.6)) results in frequencies ω and mode shapes $\{\phi\}$, where it is noted that the shapes are expressed by nodal horizontal and vertical displacements in x- and y-directions, respectively. This results in two different types of mode shapes, some predominantly lateral and others predominantly extensional.

Since response is evaluated for lateral ground excitation, the i^{th} modal participation factor γ_i is evaluated for inertia forces acting in x-direction only:

$$\gamma_i = \{\phi\}_i^T [-M-] \{I_x\} \quad (5.2)$$

where

$\{I_x\}$ is a vector selecting masses only in the x-direction with elements either zero or unity.

It is noted that the i^{th} mode shape $\{\phi\}_i$ is orthonormalized with respect to mass matrix $[-M-]$:

$$\{\phi\}_i^T [-M] \{\phi\}_i = 1 \quad (5.3)$$

For known spectral intensity, response is expressed by the following well known relationships [27]. Relative nodal displacements $\{u\}_i$ for the i^{th} mode are

$$\{u\}_i = \{\phi\}_i \gamma_i S_d \quad (5.4)$$

where

S_d represents the spectral displacement as a function of natural periods of vibration.

The i^{th} modal elastic force vector $\{P\}_i$ is conveniently expressed in terms of inertia forces:

$$\{P\}_i = [-M] \{\phi\}_i \gamma_i S_a \quad (5.5)$$

where

S_a is the spectral acceleration with

$$S_a = \omega^2 S_d \quad (5.6)$$

Modal storey shears at any level are obtained by summation of the elastic forces in x-direction from Eq. (5.5). The modal base shear V_i is then expressed as

$$V_i = [1] \{P\}_{ix} \quad (5.7)$$

where

$[1]$ is a row vector of ones

$\{P\}_{ix}$ contains the lateral components (in x-direction) of $\{P\}_i$

Modal base overturning moment M_i is also derived from Eq. (5.5) as

$$M_i = [x]\{P\}_{ix} \quad (5.8)$$

where

$[x]$ is a row vector with elements x_j representing the height of mass j above the ground.

Modal forces in connectors are computed using the displacement response from Eq. (5.4). Axial and shear forces, F_a and F_v , respectively, for the i^{th} mode are then computed as

$$\begin{Bmatrix} F_a \\ F_v \end{Bmatrix}_i = [S]\{u\}_i \quad (5.9)$$

where stress-displacement matrix $[S]$ is derived from connector stiffness matrix $[K]_c$ of Eq. (5.1).

Total response is obtained by combining the first N modal response values in a probabilistic fashion using root-sum-of-squares (RSS). Displacement envelope $\{u\}$, which refers in this Chapter to lateral displacements along the outside of Wall 1 (Fig. 5.1), is derived from modal values as

$$u_{\ell} = \sqrt{u_{\ell_1}^2 + u_{\ell_2}^2 + \dots + u_{\ell_N}^2} \quad (5.10)$$

where

u_{ℓ} signifies the ℓ^{th} element in vector $\{u\}$.

The maximum lateral top displacement D is then expressed as

$$D = u_L \quad (5.11)$$

with L representing the lateral degree of freedom on top of the structure, located on the outside of Wall 1.

Relative vertical joint displacement δ in a connector can be formulated as

$$\delta = \sqrt{(u_{r_1} - u_{s_1})^2 + (u_{r_2} - u_{s_2})^2 + \dots + (u_{r_N} - u_{s_N})^2} \quad (5.12)$$

where

r and s signify the vertical degrees of freedom at nodes i and j (Fig.5.2(b)) for the connector under consideration.

The total response for the elastic force vector $\{P\}$ is then expressed in terms of the ℓ^{th} element P_ℓ as

$$P_\ell = \sqrt{P_{\ell 1}^2 + P_{\ell 2}^2 + \dots + P_{\ell N}^2} \quad (5.13)$$

Horizontal storey shears are derived in a similar manner to that for the modal storey shears. Response for total base shear and overturning moment are expressed as

$$V = \sqrt{V_1^2 + V_2^2 + \dots + V_N^2} \quad (5.14)$$

and

$$M = \sqrt{M_1^2 + M_2^2 + \dots + M_N^2} \quad (5.15)$$

Connector axial and shear forces, formulated as RSS values, are then written as

$$F_a = \sqrt{F_{a1}^2 + F_{a2}^2 + \dots + F_{aN}^2} \quad (5.16)$$

and

$$F_v = \sqrt{F_{v1}^2 + F_{v2}^2 + \dots + F_{vN}^2} \quad (5.17)$$

The dynamic response data reported in this study employs the first 12 modes, i.e., $N = 12$. It was found that approximately 6 to 8 out of the 12 modes are predominantly lateral, thus contributing to lateral response. The alternating predominantly vertical extensional modes were in some cases recognized to appear in as early as the third location

of the modal order.

Modal seismic response values are computed by employing an average spectrum based on the El Centro earthquake of 1940, N-S component, and 5 per cent damping, as given in [66]. All response computations are for this acceleration spectrum normalized to $1 g^1$ peak acceleration. Figure 5.5 shows the El Centro response spectrum as well as that suggested by the 1977 National Building Code of Canada [67] (NBCC-1977), both normalized with respect to peak acceleration.² The NBCC-1977 spectrum is used in Section 5.4.6 where a comparison of dynamic and code static analyses is presented.

5.4 PARAMETRIC INVESTIGATION OF SEISMIC BEHAVIOUR

In the following investigation of seismic behaviour, based on the preceding mathematical representation and response spectrum analysis, overall response of the panel structure is examined with particular emphasis on forces generated in connectors. The analysis was implemented using SAP4 in modified form. The latter involves the introduction of the panel element substructuring, as well as the connector element and computation of the seismic response parameters from Eqs. (5.12) through (5.17). The scheme of the parametric

¹ g = gravitational constant

²Acceleration response according to NBCC-1977 for 5 per cent damping may be expressed as $S_a = 0.433 T^{-1}$, where $S_a \leq 1.0$ when normalized to peak response.

investigation is intended to evaluate the importance of various parameters on expected earthquake behaviour.

The effect of vertical connector shear stiffness is studied for two situations:

- (1) Uniform distribution of connector shear stiffness, and
- (2) Non-uniform shear stiffness distribution over the height of the structure, concentrating on structures with monolithic storeys.

The following cases are studied when constant uniformly distributed connector shear stiffness is assumed:

- (1) Structures with missing panels,
- (2) Asymmetric distribution of floor mass, and
- (3) Connectors with varying axial stiffness.

The monolithic shear wall is used as a convenient reference for evaluation of response. The panel element of Fig. 5.3(a) models monolithic behaviour. Use of the panel element of Fig. 5.3(b), with arrangement shown in Fig. 5.3(d) and employing connector stiffness $k_v = 10^7$ kips/ft (14.6×10^7 kN/m), results, however, in differences noticeable only for higher modes, while basic response parameters remain

practically unchanged. Thus, connectors with stiffness of 10^7 kips/ft (14.6×10^7 kN/m) result in a rigidly connected structure and approximate closely monolithic behaviour. It should be noted that the panel element of Fig. 5.3(b) allows free displacements along the vertical sides, while full continuity is maintained for the element of Fig. 5.3(a). The panel element from Fig. 5.3(c) used in the arrangement of Fig. 5.3(e) is applied to model monolithic storeys (Section 5.4.2.2).

Table 5.2 shows the dynamic properties and maximum response values for the monolithic or homogeneous shear wall, denoted by subscript M and based on the El Centro spectrum of Fig. 5.5 normalized to 1 g. Results, presented and discussed in the following Subsections, are normalized to the corresponding values in Table 5.2.

5.4.1 Effect of Uniform Connector Shear Stiffness

With the magnitude of connector shear stiffness, uniformly distributed along the height of the structure, as the only parameter subjected to variation, the following dynamic properties and seismic response are evaluated for the panel structure when connector shear stiffness k_v varies from rigid (i.e., $k_v = 10^7$ kips/ft (14.6×10^7 kN/m)) to zero:

- (1) Natural periods and modal participation factors for the four lowest lateral modes.

- (2) Base shear, base overturning moment, top displacement and connector maximum relative displacement.
- (3) Envelopes of maximum horizontal storey shears and maximum lateral displacement over the height of the structure.
- (4) Shear and axial forces in connectors - their magnitude and distribution over the height of the structure.
- (5) Magnitude and distribution of stresses in horizontal joints at different floor levels.

5.4.1.1 Overall Structural Response

In Fig. 5.6 lateral natural periods T and corresponding modal participation factors γ are plotted for the four lowest lateral modes as functions of connector stiffness k_v , normalized, respectively, to T_M and γ_M from Table 5.2.

The diagrams show the transition from a monolithic shear wall to three individual cantilevers as connector stiffness k_v decreases. Stiffness smaller than 10^2 kips/ft (14.6×10^2 kN/m) leads to the situation of free vibrating cantilevers linked only by axially stiff springs. Natural periods for the three linked cantilevers with zero shear

coupling are identical to those of a single cantilever.*

Noticeable changes in modal participation factors, for modal order as well as magnitude, within the transition $k_v = 10^6$ to 10^2 kips/ft (14.6×10^6 to 14.6×10^2 kN/m) indicate the different nature of the mode shapes caused by shear coupling (Fig. 5.6(b)).

Total response, based on the first twelve modes and using RSS modal summation, is presented in Figs. 5.7 through 5.12. Figures 5.7(a), 5.7(b) and 5.8(a) show the influence of connector stiffness k_v on base shear V , base overturning moment M and top displacement D , respectively. All values are normalized as indicated.

Figure 5.8(b) shows a diagram for connector relative displacement δ at top and mid-height of the structure, normalized to δ_s , the maximum relative displacement for independent cantilevers (Table 5.2). It should be noted that all response data are influenced by the form of the acceleration response spectrum, particularly since the structure (with $T_M = 0.55$ sec) falls into the short period range.

* This fact was verified by a separate analysis for one cantilever 12 ft (3.66 m) wide and 120 ft (36.6 m) high.

The diagrams of Fig. 5.7 indicate that base shear, and base overturning moment decrease by 35 and 50 per cent, respectively, as connector stiffness decreases from rigid ($k_v = 10^7$ kips/ft (14.6×10^7 kN/m)) to zero. Assuming a panel structure with relatively soft shear connectors, e.g., k_v between 10^4 and 10^3 kips/ft (14.6×10^4 and 14.6×10^3 kN/m) corresponding decreases are still in the magnitude of 20 per cent and 30 per cent. Similarly, Fig. 5.8(a) shows that top deflection is nearly four times as large for zero connector shear stiffness compared to fully monolithic behaviour. For stiffnesses between 10^4 and 10^3 kips/ft, increases are still as large as 80 per cent. Connector relative displacements, zero for a monolithic wall, indicate similar increases shown in Fig. 5.8(b).

Horizontal storey shears, normalized with respect to V_M , are shown in Fig. 5.9(a). Differences in distribution along structure height, demonstrated in Fig. 5.9(b), are more evident for normalization with respect to corresponding base shears. It is apparent that changes in storey shear distribution are noticeable only in the upper part of the structure, provided soft shear connectors are used.

Envelopes of maximum lateral displacements for different connector shear stiffness are presented in Fig. 5.10, with normalization to top deflection D_M of the monolithic wall in Fig. 5.10(a), and to corresponding top deflection D

in Fig. 5.10(b). As for horizontal storey shears, Fig. 5.10(b) shows no significant changes for deflection distributions along the height for varying stiffness k_v . This fact is attributable to the assumption of continuous elementary cantilevers.

On the basis of the above observations, it may be concluded that the distribution of overall structural response parameters - namely storey shears and lateral deflections - is little affected by variation of magnitude of shear stiffness in the vertical joints, provided the panel structure consists of elementary cantilevers.

5.4.1.2 Forces in Connectors

Magnitude and distribution of connector forces are of particular interest in a panel structure. Figure 5.11 shows the effect of connector shear stiffness k_v on connector shear force F_v and on axial force F_a . Shear forces are normalized to F_R from Table 5.2, the maximum connector shear force in the structure with rigid connectors (i.e., $k_v = 10^7$ kips/ft (14.6×10^7 kN/m)), which occurs at the second floor level.

Figure 5.11(a) shows that maximum connector shear force decreases rapidly for decreasing shear stiffness and assumes zero values as k_v approaches zero. However, locations of maximum values shift; this results in more uniform distribution of connector forces for small values of k_v . It

is noted that stiffnesses of magnitude $k_v = 10^4$ and 10^3 kips/ft (14.6×10^4 and 14.6×10^3 kN/m) lead to decreases in connector maximum shear force of as much as 40 per cent and 80 per cent, respectively, compared to a structure with rigid connectors.

Connector axial force F_a is shown in Fig. 5.11(b), expressed as the ratio of corresponding shear force F_v . The magnitude of the axial force is seen to be generally small relative to the shear force. The ratio increases significantly only at the top and bottom of the structure, where corresponding shear forces are small.

It can therefore be concluded that axial connector forces in vertical joints of a panel structure are generally of minor importance if uniform cantilevers exist. Thus, shear force is the major design criterion unless soft connectors are used. Use of the latter, however, implies large deflections (Fig. 5.10).

5.4.1.3 Stresses in Horizontal Joints

The valid use of the concept of elementary cantilevers, where horizontal panel connections are continuous and rigid, requires that seismic forces do not cause excessive opening or slippage in horizontal joints. Hence, the behaviour remains basically elastic. This implies that (1) the seismic axial tension does not exceed the existing axial gravity stress, and

(2) the seismic shear forces do not exceed the frictional joint forces due to gravity. For the second case a coefficient of friction of $\mu = 0.3$ was chosen which represents a conservative value for the present investigation [47]. To examine the above conditions according to the loadings of this study, stresses on horizontal joints are computed at different floor levels. These stresses are obtained in analysis by inserting fully rigid connector elements at nodal points along the horizontal joints.

In Fig. 5.12 distribution of normal force is examined at the base of the structure and at floor levels 3, 6 and 9, for various values of connector shear stiffness k_v . The magnitude of these stresses corresponds to 1 g maximum acceleration for the El Centro spectrum. It is noticed that the maximum stress is not affected significantly by different values of shear stiffness. As expected, the stress distribution changes from the rigid case to independent panel-column resistance as connector shear stiffness varies from 10^7 to 10^2 kips/ft (14.6×10^7 to 14.6×10^2 kN/m).

The maximum normal forces, as well as maximum shear forces for different seismic intensities, are compared to dead load forces in Table 5.3. The Table compares, for each floor level, two situations for seismic loading. Case 1 is computed for a peak spectral acceleration of 0.12 g whereas Case 2 is due to 0.24 g, corresponding to magnitudes suggested in NBCC-1977 for Zones II and III, respectively, when 5 per

cent damping exists.*

The comparison demonstrates that, for Case 1, seismic stresses in horizontal joints remain always smaller than the corresponding dead load stresses. For Case 2, however, it is observed that seismic axial tension exceeds gravity axial tension at the lowest three to four floor levels for $k_v \geq 10^4$ kips/ft (14.6×10^4 kN/m), and at the lowest nine floor levels for $k_v \leq 10^2$ kips/ft (14.6×10^2 kN/m). Also, seismic shear stresses exceed dead load stresses only in the upper two to three storeys and only if $k_v \geq 10^4$ kips/ft (14.6×10^4 kN/m).

Thus, the assumption of elementary cantilevers is valid for the present 12-storey structure for seismic intensity up to Zone II. In areas of seismic intensity corresponding to Zone III, the assumption may be maintained provided horizontal joints are designed for adequate transmission of tensile as well as shear forces.

5.4.1.4 Final Remarks

The preceding results show the transition from rigidly coupled shear walls to independent cantilevers - for all response parameters involved - when vertical shear stiffness is varied from rigid to fully flexible. Figures 5.6 through 5.8 show that

* Peak spectral accelerations 0.12 g and 0.24 g, correspond to the ground accelerations of 0.04 g and 0.08 g, respectively.

a noticeable loss in stiffness of the structure as a whole does not occur unless vertical connector shear stiffness k_v decreases below values between 10^6 and 10^5 kips/ft (14.6×10^6 and 14.6×10^5 kN/m).

This was observed also by Lewicki and Pauw [48] for continuous joints between concrete panels, where a decrease in stiffness from fully rigid to approximately 25 kips/in.³ (1821.0 kN/cm^3), (corresponding to lumped connector stiffness $k_v = 2.82 \times 10^5$ kips/ft (41.2×10^5 kN/m)) produced a decrease in rigidity of the total structure of about 5 per cent. Further decrease in connector shear stiffness, however, results in a significant loss in structural rigidity, as evident in Figs. 5.6, 5.7 and 5.8.

The 12-storey building of this study falls into the short period range, which means an increase in period of vibration - caused by softening of connector stiffness - leads to substantially reduced force attraction, in terms of base shear and overturning moment (Fig. 5.7). A designer can take advantage of this fact by using soft connectors if large deflections, particularly in the joints, can be accommodated. For severe earthquakes the structural damage can then deliberately be attracted to the vertical joints, allowing the panels themselves to remain elastic.

5.4.2 Effect of Irregular Distribution of Connector Shear Stiffness, Including Monolithic Storeys

The effect on structural response of the magnitude of connector shear stiffness in vertical joints, when uniformly distributed, was examined in Section 5.4.1. Variation in the distribution of shear stiffness along the height of the structure may be contemplated in practical applications. This aspect is now studied as a separate parameter. The effect of monolithic or rigid storeys on structural response is considered to be of particular interest; typical examples for structures with monolithic storeys are shown in Fig. 5.13.

In this section, the following variations and discontinuities in shear stiffness distribution are investigated:

- (1) Non-uniform distribution of shear stiffness over the height of the structure,
- (2) Structures with a monolithic top storey, as shown in Fig. 5.13(a), and
- (3) Structures with a monolithic storey at top and at mid-height as shown in Fig. 5.13(b).

The introduction of a transition in stiffness reduces connector peak forces and is investigated for:

- (1) Structures with monolithic top storey (Fig. 5.13(a)),
and
- (2) Structures with monolithic mid-height storey
(Fig. 5.13(c)).

5.4.2.1 Non-Uniform Distribution of Connector Stiffness

The effect of different non-uniform distributions of connector stiffness k_v on connector forces F_v is studied in Fig. 5.14. Situations are examined where k_v varies in a linear or a stepwise manner along the height of the structure. Linear refers, in this case, to the distribution of exponent n , where $k_v = 10^n$ kips/ft (14.6×10^n kN/m). From bottom to top of the structure, stiffness k_v increases (Fig. 5.14(b)) or decreases (Fig. 5.14(a)) over the range from 10^6 to 10^2 kips/ft (14.6×10^6 to 14.6×10^2 kN/m).

For convenience, change of stiffness may be expressed in terms of "stiffness multiple" $m = k_i/k_{i+1}$ or k_{i+1}/k_i with m always greater than 1; k_i and k_{i+1} are connector stiffnesses at floor levels i and $(i+1)$, respectively. Stiffness changes in Fig. 5.14 are of magnitude $m = 10$ for stepwise variation every third floor, and $m = 2.31$ for linear variation at every floor. For the latter stiffness exponent n changes by 0.364 at each floor level.

Figure 5.14 shows that connector forces differ significantly for linear and stepwise variations of connector shear stiffness k_v . It is observed that peak force characteristics are replaced by smooth distributions as the rate of change in stiffness k_v , over the height of the structure is decreased. Thus, it can be concluded that shear connector forces in vertical joints are sensitive to shear stiffness distribution. It is demonstrated that change of stiffness given by multiple $m \approx 2$ produces smooth shear force distributions, whereas $m \approx 10$ leads to the development of high peak forces. Other properties and response parameters, not shown here, do not differ appreciably for linear or stepwise distribution.

5.4.2.2 Monolithic Storeys

Rigid zones, or regions of high stiffness, are generally visualized as being undesirable. Experience of structural damage during earthquakes indicates that large forces are attracted to areas of stiffness concentration. On the other hand, in panelized buildings with discrete connectors it may be advantageous to introduce fully monolithic storeys for reasons of stability. This is particularly true for highrise buildings. Introduction of rigid storeys in the panel structure considered in this study may also be understood as the stiffening of a coupled shear wall, since elementary cantilevers are assumed.

In monolithic storeys no shear deformation takes place at the vertical joint lines. For the analytical representation full panel continuity is therefore required, obtained by the panel arrangement shown in Fig. 5.3(e). The latter is assembled from the panel elements of Fig. 5.3(c). A comparison of the panel arrangement of Fig. 5.3(e) with that of Fig. 5.3(d), however, indicates negligible differences in computed response when $k_v = 10^7$ kips/ft (14.6×10^7 kN/m). Hence, the rigidly connected panel arrangement of Fig. 5.3(d) is used to approximate monolithic behaviour, since it allows forces transmitted through rigid storeys to be expressed in terms of connector forces.

Results which demonstrate the influence of monolithic storeys on fundamental period T , as well as seismic response parameters V , M and D , are presented in Figs. 5.15 through 5.17. Corresponding curves for the uniform structure without monolithic storeys were discussed separately in Section 4.1 (Figs. 5.6 through 5.8). Using the example of a structure with shear stiffness $k_v = 10^3$ kips/ft (14.6×10^3 kN/m) allows the following quantitative evaluation of the effect of introducing the monolithic storeys:

- (1) Stiffness gain of the total structure is measured directly by decrease in fundamental period. Figure 5.15 shows that reduction of T , compared to the

uniform structure, are (a) 20 per cent when a monolithic top storey is introduced, and (b) 45 per cent when monolithic top and mid-height storeys are introduced.

- (2) Higher force attraction, however, is reflected in Figure 5.16 by increases in base shear V in the order of (a) 15 per cent, and (b) 35 per cent, and by corresponding increases in overturning moment M , of (a) 30 per cent, and (b) 55 per cent, when rigid top and rigid top and mid-height storeys are introduced respectively.
- (3) Corresponding decreases for top deflection D , shown in Fig. 5.17, are (a) 30 per cent, and (b) 80 per cent.

The influence of monolithic storeys on envelopes of maximum lateral displacements are shown in Fig. 5.18(a) for shear stiffness $k_v = 10^4$ kips/ft (14.6×10^4 kN/m) and in Fig. 5.18(b) for shear stiffness $k_v = 10^3$ kips/ft (14.6×10^3 kN/m). Differences from the uniform structure become pronounced only for structures with soft shear connectors (i.e., $k_v \leq 10^3$ kips/ft (14.6×10^3 kN/m)). Normalized distribution curves of maximum horizontal storey shears, on the other hand, were found to differ little from those of a uniform structure.

5.4.2.3 Connector Forces in Structures With Monolithic Storeys

Figure 5.19 shows the effect of rigid storeys on connector shear force distribution, when connector stiffness in the non-rigid zones is varied from $k_v = 10^7$ to 10^2 kips/ft (14.6×10^7 to 14.6×10^2 kN/m).

If a monolithic storey exists only at top, Fig. 5.19(a) shows that forces in the lower half of the structure differ little from those for the corresponding uniform structure. However, as expected, a large peak shear force develops in the rigid storey (for the cases presented $10 \leq m \leq 10^5$). A "peak force ratio" may conveniently be introduced to define the ratio between peak force in the rigid storey and the maximum connector shear force in the non-rigid zone. For example, the curves of Fig. 5.19(a) yield peak force ratios of 1.7 and 13.5 for connector shear stiffness $k_v = 10^4$ and 10^3 kips/ft (14.6×10^4 and 14.6×10^3 kN/m), respectively. Ratios increase significantly as k_v decreases.

The structures with monolithic top and mid-height storeys, presented in Fig. 5.19(b), show the predominant influence of a rigid storey at mid-height on the shear force distribution. High peak force ratios of 4.8 and 41.0, for $k_v = 10^4$ and 10^3 kips/ft (14.6×10^4 and 14.6×10^3 kN/m) respectively, lead to the conclusion that a structure with soft connectors relies almost entirely on the strength of a

few monolithic storeys. This is also apparent from the large stiffness increase due to introduction of rigid storeys observed in Figs. 5.15 through 5.17. Importance of axial connector forces, not presented here, remains similar to that for structures with uniform shear stiffness distribution, (see Section 5.4.1).

From the preceding observations, it can be concluded that the introduction of rigid or monolithic storeys increases significantly the stiffness of the structure as a whole. This leads to increased seismic base shear and base overturning moment, and to a corresponding decrease in top deflection (Figs. 5.15 through 5.17). A particular characteristic is the appearance of high peak shear forces, at locations of rigid storeys, of increasing intensity for decreasing connector shear stiffness (Fig. 5.19).

5.4.2.4 Transition in Stiffness

In Figure 5.14 it was demonstrated that irregular peak forces disappear as the rate of change in stiffness k_v decreases. Thus to avoid the large force concentrations of Fig. 5.19 stiffness transitions extending over several storeys are introduced.

For the study which follows connector shear stiffness k_v in the non-rigid zones is assumed as 10^4 kips/ft (14.6×10^4 kN/m). The transition therefore, must accommo-

date a change in stiffness from 10^4 kips/ft to fully rigid conditions (i.e., $k_v = 10^7$ kips/ft (14.6×10^7 kN/m)). For the present panel wall, transitions are considered for two cases: a structure with rigid top storey and one with rigid mid-height storey.

The procedure for selecting an appropriate transition is purely trial and error. The stiffness transitions indicated in Tables 5.4 and 5.5 are expressed in terms of exponent n and stiffness multiple m . A transition extends over either three or four storeys and is "linear" or "smooth". Linear implies a storey-to-storey change in n (as in Fig. 5.14); smooth denotes a more gradual change in connector stiffness k_v as expressed by both n as well as m .

The case of a structure with monolithic top storey is demonstrated in Fig. 5.20. The diagrams show that a linear transition is significantly less effective in reducing force concentration compared to one that is smooth. The latter is seen to lead to decrease of peak force and a more uniform distribution. For a smooth transition over four storeys, peak force decreases by approximately 33 per cent (Fig. 5.20(b)). These curves demonstrate also that a stiffness transition tends to draw peak forces in the direction where maximum shear force would occur in a structure with uniform shear stiffness distribution (see Fig. 5.11(a)).

A structure with monolithic storey at mid-height and smooth stiffness transitions above and below is presented in Fig. 5.21. Peak force is decreased by approximately 30 per cent; however, no major difference is noticed between the transitions over three or four storeys.

The above two examples demonstrate the possibility of avoiding, or at least controlling, peak forces due to the presence of rigid storeys by introducing stiffness transitions. To bridge effectively large stiffness differences the transition should be of the smooth type, beginning slowly ($m = 1.5$ to $m = 2$) after which a high stiffness can be approached rapidly as apparent from Figs. 5.20 and 5.21 together with Table 5.5.

5.4.3 Effect of Missing Panels

Structural discontinuities produced by soft zones where local concentration of flexibility occurs may be introduced into the panelized system by missing panels. Local panel failure during an earthquake could be the cause or, alternatively, architectural requirements at the design stage may dictate such openings. Consequently, the magnitude as well as the nature of force concentrations associated with missing panels requires study.

Considering the structural model of Fig. 5.1, situations are studied where panels are missing in (1) interior

wall 2, and (2) exterior wall 3 between floor levels 2-3, 6-7 and 10-11. Figures 5.22 and 5.23 show the resulting structures. For this study, nearly rigid shear connectors were chosen using $k_v = 10^5$ kips/ft (14.6×10^5 kN/m). It is assumed that connectors at locations where panels are missing maintain full shear and axial stiffness, i.e., $k_v = 10^5$ kips/ft (14.6×10^5 kN/m) and $k_a = 10^7$ kips/ft (14.6×10^7 kN/m). If half stiffnesses are used for these connectors, (i.e., if stiffness multiple $m = 2$ exists), the resulting effect on force distribution is not significant.

The influence of missing panels on fundamental period and on top deflection is summarized in Table 5.6. The data indicate that missing interior panels have little effect on overall structural response, whereas missing exterior panels result in significant softening. For example fundamental period and top deflection both increase by approximately 50 per cent when a panel is removed in wall 3 at level 2-3.

Horizontal storey shears for missing exterior panels, normalized to monolithic base shear V_M , are shown in Fig. 5.24. Except for decreased magnitude, distribution along the height is not particularly sensitive to the introduction of a wall opening. Envelopes of maximum lateral displacements are shown in Fig. 5.25(a) for missing interior panels and in Fig. 5.25(b) for missing exterior panels. Only the latter indicates significant differences compared to the uniform structure.

5.4.3.1 Connector Forces

Figures 5.26 through 5.28 show connector force distributions when panels are missing in the interior bay (Wall 2). It is observed that the discontinuity effect is confined mainly to the vicinity of the opening, with only small increases in shear forces F_v (a maximum of 15 per cent when a panel is removed at level 2-3, Fig. 5.26(a)).

However, a large increase in axial force F_a in the connectors immediately adjacent to the missing panel is to be expected since a concentration of normal stress trajectories takes place next to the opening. Figures 5.26(b), 5.27(b) and 5.28(b) demonstrate this increase which can amount to as much as 20 times the corresponding values for the uniform structure. This maximum occurs when a panel is removed at level 2-3 (Fig. 5.26(b)). The magnitude of peak axial force decreases for panels missing at higher levels (Figs. 5.27(b) and 5.28(b)). The curves for axial-to-shear force ratios indicate similar localized effects, as shown by Figs. 5.26(c), 5.27(c) and 5.28(c).

A large increase in axial force F_a in the connectors immediately adjacent to the missing panel is the most important result, as seen in Figs. 5.26(b), 5.27(b), and 5.28(b). Axial forces increase to as much as 20 times the correspond-

ing values for the uniform structure. This maximum occurs when a panel is removed at level 2-3 (Fig. 5.26(b)). The magnitude of peak axial force decreases for panels missing at higher levels (Figs. 5.27(b) and 5.28(b)). The curves for axial-to-shear force ratios indicate similar localized effects, as shown by Figs. 5.26(c), 5.27(c) and 5.28(c).

Thus, it may be concluded that the redistribution of internal forces, accomplished by large increases of axial connector forces, allows a structure with missing interior panels to maintain the overall response characteristics of the full structure without openings (see Table 5.6, Wall 2).

Figures 5.29 through 5.31 present connector force distributions for panels missing in the exterior bay (Wall 3). Force concentrations are not as localized as for missing centre panels. Also, peak forces tend to assume much larger values. Shear force F_v increases by as much as 50 to 150 per cent compared to the uniform structure, depending on the floor level where panels are missing (Figs. 5.29(a) through 5.31(a)). Axial forces F_a experience extremely large increases, not only immediately adjacent to the missing panel but also well beyond. Peak axial forces reach values up to 40 times those for the corresponding uniform structure (Figs. 5.29(b) through 5.31(b)).

5.4.3.2 Final Remarks

Shear, rather than axial, forces usually govern the design for connectors in vertical joints of panelized structures. Introduction of perforations due to missing panels, however, results in high axial forces. The diagrams in Figs. 5.26(c) through 5.31(c) demonstrate that the ratios between axial and shear force can assume values approaching 2 for certain cases. Connector design must therefore incorporate both shear and axial forces.

The preceding observations indicate that a panel missing in an outer bay results in a major reduction in panel wall overall rigidity, while the corresponding effect from an interior bay is small. It is realized that the softening of one panel shear wall leads, in a complete structure, to a force redistribution to other lateral resisting walls. A three-dimensional analysis is to be carried out in this case.

5.4.4 Asymmetric Floor Mass Distribution

For a panel structure, it is generally assumed that elementary cantilevers possess identical dynamic properties. This assumption is not valid when, for example, non-uniform floor mass distribution exists. To study the effect of the latter, a situation is assumed where 50 per cent of the total

floor mass is assigned to Wall 3, while Walls 1 and 2 carry 25 per cent each. Three cases of uniform connector shear stiffness are considered: $k_v = 10^7$ (14.6×10^7), 10^4 (14.6×10^4) and 10^2 (14.6×10^2) kips/ft (kN/m).

For all three of the above cases, no appreciable difference is evident for nearly all response values including connector shear forces, compared to the structure with uniform floor mass distribution. Changes reflecting non-uniformly distributed mass occur only in connector axial forces, as apparent from Fig. 5.32 where axial-to-shear force ratios are shown for $k_v = 10^7$ (14.6×10^7) and 10^4 kips/ft (14.6×10^4 kN/m). These ratios indicate a noticeable increase in connector axial force of Joint 2 (Fig. 5.1), adjacent to the higher intensity of floor mass. Axial forces in Joint 1, on the other hand, differ little from those for uniform floor mass distribution.

Thus, changes in axial forces, though generally small in magnitude, allow the panel structure to maintain its response characteristics completely when non-uniform floor mass distribution exists.

The discussions of this Section, as well as those of Section 5.4.3, lead to the conclusion that discontinuities in the horizontal direction within the structure, such as

non-uniform floor mass distribution or a missing panel, result in a redistribution of internal forces, apparent mainly through axial force increase. Thus, the structure as a whole preserves its dynamic properties and response characteristics close to, or in some cases completely identical with, those of the uniform structure (except for panels missing in Wall 3).

5.4.5 Connector Axial Stiffness

In the design of vertical joints and connectors for precast panel shear walls one generally assumes that in the axial direction sufficient strength and adequate stiffness exist. In results previously discussed in this study axial connections, even when fully rigid, do not attract axial forces of significant magnitude (Fig. 5.11(b)), except under special conditions, such as with missing panels, for example. The question then arises: what is the required order of magnitude for axial stiffness sufficient to maintain the same structural response as for axially rigid connectors?

Figure 5.33 shows the influence of connector axial stiffness on fundamental period, when shear stiffness is kept constant at $k_v = 10^7$ and 10^4 kips/ft (14.6×10^7 and 14.6×10^4 kN/m). A transition to independent cantilevers, similar to shear stiffness variation of Fig. 5.6(a), is observed. However, in the present case of varying axial stiffness structural softening begins at much lower stiffness,

i.e.,

$$k_a \approx 10^2 \text{ kips/ft } (14.6 \times 10^2 \text{ kN/m})$$

Also, significant changes in modal configurations take place (not presented here), as is apparent from the reordering of modal participation factors as axial stiffness decreases below 10^2 kips/ft (14.6×10^2 kN/m). This is attributable to shear coupling which continues to exist even when k_a approaches zero.

Distribution of connector shear and axial forces are shown in Fig. 5.34, where shear stiffness is kept constant at $k_v = 10^4$ kips/ft (14.6×10^4 kN/m). The curves of Fig. 5.34(a) demonstrate that no significant change in shear force becomes apparent with variation of axial stiffness k_a . Axial forces, however, are more sensitive to change in connector stiffness k_a , although remaining small in magnitude. Irregular peaks in axial force distribution, at top and bottom of the structure, tend to disappear and smooth distributions develop, decreasing in magnitude from top to bottom, as axial connectors become very flexible (Fig. 5.34(b)).

It appears, therefore, that changes in structural response of only minimal significance occur when axial stiffness varies from fully rigid to very soft, i.e., from 10^7 to 10^2 kips/ft (14.6×10^7 to 14.6×10^2 kN/m). Assuming that adequate axial stiffness is provided (i.e., $k_a > 10^2$ kips/ft

(14.6×10^2 kN/m)), use, in analysis, of axially rigid connectors is justified.

In spite of the constraints imposed by shear coupling, the structure reaches the same fundamental period as independent cantilevers (Fig. 5.33). This result is due to particular kinematic conditions which exist when connector axial stiffness is nearly absent. Large axial displacements allow corresponding vertical displacements and an uncoupled first bending mode can develop. Corresponding shear forces computed from stiffness values satisfy merely numerical conditions and have no practical implications. For periods higher than the fundamental, however, the change in direction of mode shape curvature prevents large axial displacements; hence, shear coupling is activated. Figure 5.33 shows the corresponding increase in the values of the higher periods.

Axial strength as well as axial stiffness for connections across joints are little discussed in the literature on panel structures. Considering the above discussion, further experimental data, particularly for discrete connectors, are necessary to establish a practical range for axial stiffness.

5.4.6 Comparison With Code Static Loading

For economical design of structures, acceptance of non-linear deformation under severe seismic loading is re-

quired. Thus, aseismic codes [67] allow plastic deformation by specifying lateral static loads which result in corresponding computed response only a fraction of that derived in an elastic dynamic analysis. Ductility requirements are consequently established to guarantee structural functioning when plastic deformation takes place. Overall plastic behaviour is often expressed by horizontal storey shear ratios (i.e., dynamic storey shear/design static storey shear) [27,68], representing an estimated average ductility factor, which in this study is referred to as the "force reduction factor".

Panel structures with rigid horizontal joints experience yielding predominantly in the vertical joints, i.e., of vertical shear connectors. However, if flexible shear connectors exist, shear forces and hence the energy absorbed in connectors assume small values, i.e., zero values for the extreme case of independent cantilever action. In such cases panels or the connections across horizontal joints, are expected to participate increasingly in plastic deformation. As shear connector stiffness decreases in magnitude, emphasis on ductility requirements may thus shift from the vertical joints to the panel walls.

In this Section a code equivalent static analysis is compared to a linear dynamic analysis, according to NBCC-1977 specifications, in deriving force reduction factors. The following response parameters are studied:

- (1) Base shear intensity with variation of connector shear stiffness,
- (2) Horizontal storey shear distribution for a rigid panel shear wall ($k_v = 10^7$ kips/ft (14.6×10^7 kN/m)) and for independently acting cantilevers ($k_v = 0$),
- (3) Maximum connector shear force for varying connector shear stiffness, and
- (4) Distribution of connector shear force along the height of the structure.

Base shear V , from dynamic analysis, is shown in Fig. 5.35(a) as a function of shear stiffness k_v . The dynamic base shear is based on the NBCC-1977 spectrum from Fig. 5.5, assuming a ground acceleration of 0.08g which results in 0.24 g peak spectral acceleration when 5 per cent damping exists. Computed values for total base shear are 268 kips and 133 kips for the monolithic shear wall and for independent cantilever action, respectively; intermediate values represent the characteristic transition (see Fig. 5.7(a)).

Code [67] static base shear, also shown in Fig. 5.35(a), is obtained from the expression

$$V = ASKW$$

where

A represents ground acceleration

S is a seismic response factor

K denotes the structural system, and

W is the total weight of the structure.

Importance and foundation factors, also specified in the code, have here been taken as equal to unity. Assumed values are: $A = 0.08g$ and $W = 2030$ kips. Value K is taken as 1.0 since continuity does not exist in vertical joints and also to account for a more ductile system (for precast concrete construction with spliced reinforcement the code recommends $K=1.3$). Seismic response factor, specified as $S = 0.5/\sqrt[3]{T}$ with fundamental period $T = 0.05 H/\sqrt{D}$ (H = building height, D = building width), has values 0.50 and 0.41 for the monolithic wall and independent cantilevers, respectively. Corresponding base shears are $V = 81.0$ kips and $V = 65.0$ kips. The transition between these two values is not available from the code and interpolation following the dynamic analysis curve is employed, which leads to the static curve presented in Fig. 5.35(a).

To express possible plastic action, force reduction factor μ_v may be introduced as the ratio of dynamic to static horizontal storey shear. From the two curves for base shear of Fig. 5.35(a), this factor assumes values ranging from 2.0 to 3.3, as depicted by the curve of Fig. 5.35(b).

Distributions of horizontal storey shears corresponding to $k_v = 10^7$ kips/ft (14.6×10^7 kN/m), (i.e., monolithic shear wall) and to $k_v = 0$ (i.e., independent cantilever

action) are compared in Fig. 5.36, both for dynamic analysis, and for static loading. The code static loading is triangular, with a top force $F_t = 0.004(H/D_s)^2$, (D_s = dimension of the lateral force-resisting system in a direction parallel to the applied forces). For the present case F_t is 4.4 per cent of the base shear. The static and dynamic storey shear distributions result in force reduction factor μ_v varying along the building height. Values ranging from 1.3 to 3.3 are listed in Table 5.7.

Maximum connector shear force, $\max F_v$ is plotted in Fig. 5.37(a) as a function of shear stiffness k_v . Static connector shear forces are computed from a static analysis (i.e., using code triangular loading with top force) for magnitudes of base shear given by the static curve of Fig. 5.35(a). The curve for the force reduction factor for maximum connector shear force, defined as

$$\mu_c = \max F_v(\text{Dynamic}) / \max F_v(\text{Static})$$

is shown in Fig. 5.37(b). For small values of stiffness k_v , μ_c has little meaning since shear forces approach zero (shown as dashed line). The corresponding curve for base shear, also shown in Fig. 37(b), indicates that a redistribution of ductility demand from connectors to walls is expected to take place as shear connectors become increasingly more flexible.

Distributions of connector shear force over the height of the structure are compared in Fig. 5.38 for dynamic analysis and code static loading, with static base shears normalized to corresponding dynamic base shears. Close agreement exists for all cases of connector shear stiffness. Agreement, similar to that indicated in Fig. 5.38, was also observed for non-uniform shear stiffness distributions (not presented).

From the preceding comparison of dynamic and static seismic analyses for the 12-storey panel structure, it may be concluded that force reduction factors (i.e., ductility requirements) fall into the range from 1.3 to 3.3, depending on floor level and on connector stiffness k_v , (Fig. 5.35 and Table 5.7). Plastic deformation is expected to take place mainly in the vertical joints, but it may be transferred to the panel walls themselves when connector shear stiffness is small.

5.4.7 Extension to Non Linear Analysis

The preceding linear dynamic analysis of a panel structure which includes structural discontinuities such as non uniform stiffness distribution and missing panels, is directly applicable only for elastic response. If non linear behaviour is introduced, where local plastification leads to force redistributions, the preceding results can be employed to identify zones of increased ductility and

strength demands, as well as zones where progressive yielding is to be expected. Thus, initial assumptions for a non linear analysis can be deduced.

To initiate a non linear analysis of large panel structures a thorough study of the actual physical behaviour is necessary. A short summary thereof follows mainly References [71,72]. Generally one assumes that the nonlinearities are restricted to the planes of weakness represented by vertical and horizontal joints [73]. As for vertical joints, knowledge of realistic stress displacement relationships for shear and axial action as well as their coupling effect can be sufficient. However, stiffness degradation due to repeated loading may also need to be considered.

In horizontal joints two types of movements predominate: (1) rocking or tilting, and (2) slippage in the joint. The first movement is created by alternating normal tensile forces causing separation or opening at the edges of the horizontal joints. The second movement is caused by horizontal shear forces which overcome the gravitational friction. The coupling of rocking and slippage results in a concentration of shear as well as compressive forces. Of the two actions, slippage gains major importance with respect to energy dissipation.

Finite element procedures to model these rather complex joint characteristics are also suggested in References [71,72]. Four different element types are described: (1) spring connector element for vertical joints, (2) four-node horizontal connection element, and (3), (4) horizontal gap and stop elements. The latter two are to model opening of joints and limited slip due to dowel action, respectively. Since the panels themselves remain elastic they may be represented by substructures; thus, the super finite element technique of the present study may be implemented effectively.

For the solution of the incremental, non linear dynamic equilibrium equation,

$$[M] \Delta \ddot{x} + [C_t] \Delta \dot{x} + [K_t] \Delta x = \Delta P \quad (5.18)$$

a step by step integration has to be applied. The incremental values $\Delta \ddot{x}$, $\Delta \dot{x}$, Δx and ΔP represent acceleration, velocity, displacement and applied force, respectively; $[C_t]$ and $[K_t]$ are tangent damping and stiffness matrices; $[M]$ is the mass matrix. Many procedures have been applied for solution of the equilibrium equation [74,75] which are either explicit [76] or implicit schemes [69]. The two schemes differ in using the equilibrium conditions at the beginning, or at the end of the time step, respectively. The time step to be chosen depends largely on the highest significant frequency. For large panel structures an explicit scheme can be used

effectively [71]; particularly, since repetitive element groups are present and provided mass as well as damping matrices can be applied in diagonal form [28].

5.5 SUMMARY OF RESULTS

A 12-storey 3-bay precast panel structure has been analyzed for lateral seismic loading using response spectrum analysis and the finite element method. Rigid horizontal joints were assumed; at vertical joints flexible connectors are located at floor levels. Assumptions made in the application of the substructuring technique for panel modelling were summarized in Section 5.3.3. Connector model and range of possible connector stiffness were discussed in Section 5.3.2. The results were primarily expressed in terms of

- (1) Overall structural response, and
- (2) Distribution and magnitude of connector forces in vertical joints.

The conclusions from the parametric study in Section 5.4 are summarized below. Although the conclusions are directly applicable only to the 12-storey structure considered, it is believed, that additional studies would indicate a wider application of the results.

(A) The effect on structural response of the magnitude of connector shear stiffness k_v , uniformly distributed along the height of the structure, is summarized as follows:

- (1) Values for dynamic properties and maximum response - for natural period T , modal participation factors γ , base shear V , base overturning moment M and lateral top deflection D - pass through a transition as vertical connector shear stiffness is varied from rigid ($k_v = 10^7$ kips/ft (14.6×10^7 kN/m)) to fully flexible ($k_v = 0$). Values correspond to those of a monolithic wall and of individual cantilevers for rigid and for fully flexible shear connectors, respectively (Figs. 5.6 through 5.8).
- (2) The normalized distribution of maximum response - horizontal storey shear and maximum lateral deflections - along the height of the structure are not significantly affected by change in shear stiffness k_v (Figs. 5.9(b) and 5.10(b)).
- (3) Shear forces in vertical connectors change significantly, in magnitude as well as in distribution along the height, as shear stiffness k_v

is varied (Fig. 5.11(a)). Axial forces, however, are of considerably less importance, since they remain small in magnitude (Fig. 5.11(b)).

- (4) Horizontal joints, assumed as fully continuous in this study, are subjected to net tensile forces in the first three to four storeys in areas of Zone III seismicity (Table 5.3.)

(B) For structures with monolithic storeys the following observations were noted:

- (1) Overall structural stiffness increases significantly when rigid storeys are introduced. These changes are apparent from (a) decreases in fundamental period T , and (b) decreases in top deflection D ; however, larger forces are attracted, reflected by (c) increases in base shear V and base overturning moment M (Figs.) 5.15 through 5.17).
- (2) Shear force distribution in vertical joints is sensitive to the rate of change in connector shear stiffness k_v along the height of the structure (Fig. 5.14). Introduction of monolithic storeys results in large shear force concentra-

tions at the monolithic level, with magnitude depending on connector shear stiffness k_v in the non-rigid zone (Fig. 5.19).

- (3) Stiffness transitions, introduced to control connector peak shear forces, need to be of a "smooth" type (Table 5.4) in order to be effective (Figs. 5.20 and 5.21).

(C) Panels missing at different storey levels in Walls 2 and 3 (Figs. 5.22 and 5.23) affect structural response as follows:

- (1) Removal of an interior panel (in Wall 2) does not alter structural overall behaviour significantly (Table 5.6 and Fig. 5.25(a)); the accompanying local redistribution of internal forces is characterized by large increases of connector axial forces (Figs. 5.26 (b) through 5.28 (b)).
- (2) For missing exterior panels (in Wall 3) extensive change in structural response is encountered (Table 5.5 and Fig. 5.25(b)). Large increases in maximum values and significant redistributions take place for shear and axial forces in vertical connectors (Figs. 5.26 through 5.31).

(D) Asymmetric floor mass distribution, simulating non-identical dynamic properties of the elementary cantilevers results only in a redistribution in connector axial forces (Fig. 5.32); other response parameters remain unchanged.

(E) Variation of connector axial stiffness shows that the magnitude of this parameter is of little importance to structural response, unless relatively small values are involved, i.e., $k_a < 10^2$ kips/ft (14.6×10^2 kN/m), (Figs. 5.33 and 5.34).

(F) A comparison of elastic dynamic analysis with response to code static loading, using NBCC-1977 results in the following observations:

- (1) Force reduction factor μ_v , derived from a comparison of static and dynamic storey shears, ranges from 1.3 to 3.3, decreasing with height along the structure and with connector shear stiffness k_v (Table 5.7 and Fig. 5.35 (b), respectively).
- (2) Force reduction factor μ_c , derived from static and dynamic connector shear forces, ranges from 3.3 to approximately 1 and indicates a transfer of ductility requirement from vertical joints to walls in the panel structure, as connector shear stiffness k_v decreases to small values (Fig. 5.37 (b)).

- (3) The distribution of connector shear forces along the height of the structure obtained from code static loading agrees closely with that from dynamic analysis.

TABLE 5.1 COMPARISON OF CONNECTOR SHEAR STIFFNESS DEFINITION

CONNECTOR STIFFNESS, kips/ft	EQUIVALENT DISTRIBUTED JOINT STIFF- NESS, kips/in ³	<u>CONNECTOR STIFFNESS</u> <u>PANEL STIFFNESS</u>
10 ⁷	868.0	80.0
10 ⁶	86.8	8.0
10 ⁵	8.68	0.8
10 ⁴	0.868	0.08
<p><u>Note:</u> 1 kip/ft = 14.6 kN/m 1 kip/in³ = 2.72 x 10⁵ kN/m</p>		

TABLE 5.2 ABSOLUTE RESPONSE VALUES

VALUES FOR MONOLITHIC WALL		
Lateral Mode No.	Lateral Natural Periods, T_M (sec)	Lateral Modal Participation Factors γ_M
1 (1)*	0.546	6.33
2 (2)	0.112	3.72
3 (4)	0.0497	2.11
4 (6)	0.0315	1.44
Base shear:		$V_M = 798.6$ kips
Base overturning moment:		$M_M = 6.55 \times 10^6$ kips-ft
Top deflection:		$D_M = 0.193$ ft
VALUES FOR WALL WITH RIGID CONNECTORS		
Maximum connector shear force:		$F_R = 283.4$ kips
VALUES FOR SINGLE WALLS		
Relative joint displacement:		$\delta_s = 0.103$ ft at top $\delta_s = 0.077$ ft at mid-height
All response values are for 1 g peak acceleration (El Centro spectrum).		
* Numbering in brackets corresponds to the actual modal order. Note: 1 kip = 4.45 kN; 1kip-ft = 1.36 kN m; 1 ft = 0.305 m		

Table 5.3 FORCES IN HORIZONTAL JOINTS DUE TO VERTICAL DEAD LOAD AND LATERAL SEISMIC LOAD FOR DIFFERENT CONNECTOR SHEAR STIFFNESS, $k_v = 10^n$ kips/ft.

Floor Level	Maximum Forces from Seismic Loads, Kips/ft						Forces from Dead Load kips/ft	
	n = 7		n = 4		n = 2			
	CASE 1	CASE 2	CASE 1	CASE 2	CASE 1	CASE 2		
NORMAL FORCES	11	1.8	3.5	1.1	2.1	1.8	3.5	4.7
	10	2.3	4.5	1.8	3.6	4.7	9.3	9.4
	9	4.3	8.6	3.5	7.0	8.0	16.0	14.1
	8	6.8	13.5	5.6	11.2	11.0	22.0	18.8
	7	8.8	17.6	8.1	16.2	13.9	27.8	23.5
	6	12.5	25.0	10.9	21.7	16.6	33.2	28.2
	5	15.7	31.4	13.9	27.8	19.3	38.5	32.9
	4	19.1	38.1	17.3	34.6	22.5	45.0	37.6
	3	22.6	45.1	21.4	42.7	25.5	51.0	42.3
	2	26.3	52.6	26.3	52.6	30.2	60.3	47.0
	1	30.6	61.1	32.5	65.0	35.3	70.5	51.7
	0	40.5	81.0	42.5	85.0	41.3	82.5	56.4
SHEAR FORCES FOR $\mu = 0.3$	11	1.1	2.1	1.2	2.3	0.6	1.1	1.4
	10	1.6	3.2	1.7	3.3	0.8	1.5	2.8
	9	2.3	4.5	2.0	3.9	1.0	1.9	4.2
	8	2.6	5.2	2.2	4.4	1.0	1.9	5.6
	7	3.0	6.0	2.5	5.0	1.0	1.9	7.0
	6	3.4	6.7	2.7	5.4	1.1	2.1	8.4
	5	3.7	7.4	2.9	5.8	1.2	2.4	9.8
	4	3.9	7.7	3.1	6.2	1.4	2.8	11.3
	3	4.1	8.1	3.2	6.3	1.6	3.2	12.7
	2	4.3	8.5	3.3	6.5	1.8	3.6	14.1
	1	4.5	9.0	3.3	6.5	2.0	3.9	15.5
	0	5.6	11.1	3.6	7.2	2.6	5.2	16.9

NOTE: 1 kip/ft = 14.6 kN/m

TABLE 5.4 LINEAR STIFFNESS TRANSITIONS

Transition Over 3-Storeys		Transition Over 4-Storeys	
Exponent n	Multiples m	Exponent n	Multiples m
7.00	5.62	7.00	3.98
6.25	5.62	6.40	3.98
5.50	5.62	5.80	3.98
4.75	5.62	5.20	3.98
4.00	5.62	4.60	3.98
		4.00	3.98

TABLE 5.5 SMOOTH STIFFNESS TRANSITIONS

Transition Over 3-Storeys		Transition Over 4 Storeys	
Exponent n	Multiples m	Exponent n	Multiples m
7.00	39.8	7.00	25.1
5.40	6.31	5.60	6.31
4.60	2.51	4.80	2.51
4.20	1.58	4.40	1.58
4.00		4.20	1.58
		4.00	

TABLE 5.6 RATIOS FOR FUNDAMENTAL PERIOD AND TOP DEFLECTION FOR STRUCTURE WITH PANELS MISSING AT DIFFERENT FLOOR LEVELS

Floor Level	WALL (2)		WALL (3)	
	Period Ratio T/T_s	Top Deflection Ratio D/D_s	Period Ratio T/T_s	Top Deflection Ratio D/D_s
10-11	1.00	1.00	1.00	1.02
6-7	1.01	1.02	1.11	1.17
2-3	1.02	1.03	1.47	1.58
<p>$T_s = 0.580$ sec; Fundamental period for uniform stiffness distribution with $k_v = 10^5$ kips/ft.</p> <p>$D_s = 0.213$ ft; Top deflection for uniform stiffness distribution with $k_v = 10^5$ kips/ft and 1 g peak acceleration (El Centro spectrum).</p>				
<p><u>Note:</u> 1 ft = 0.305 m 1 kip/ft = 14.6 kN/m</p>				

TABLE 5.7 FORCE REDUCTION FACTORS μ_v ALONG
STRUCTURE HEIGHT

Storey Level	Force Reduction Factor μ_v	
	Monolithic Shear Wall	Single Shear Walls
12	2.2	1.4
11	2.5	1.4
10	2.6	1.3
9	2.6	1.3
8	2.7	1.3
7	2.8	1.5
6	2.8	1.6
5	2.9	1.8
4	3.1	1.9
3	3.1	2.0
2	3.2	2.0
1	3.3	2.0

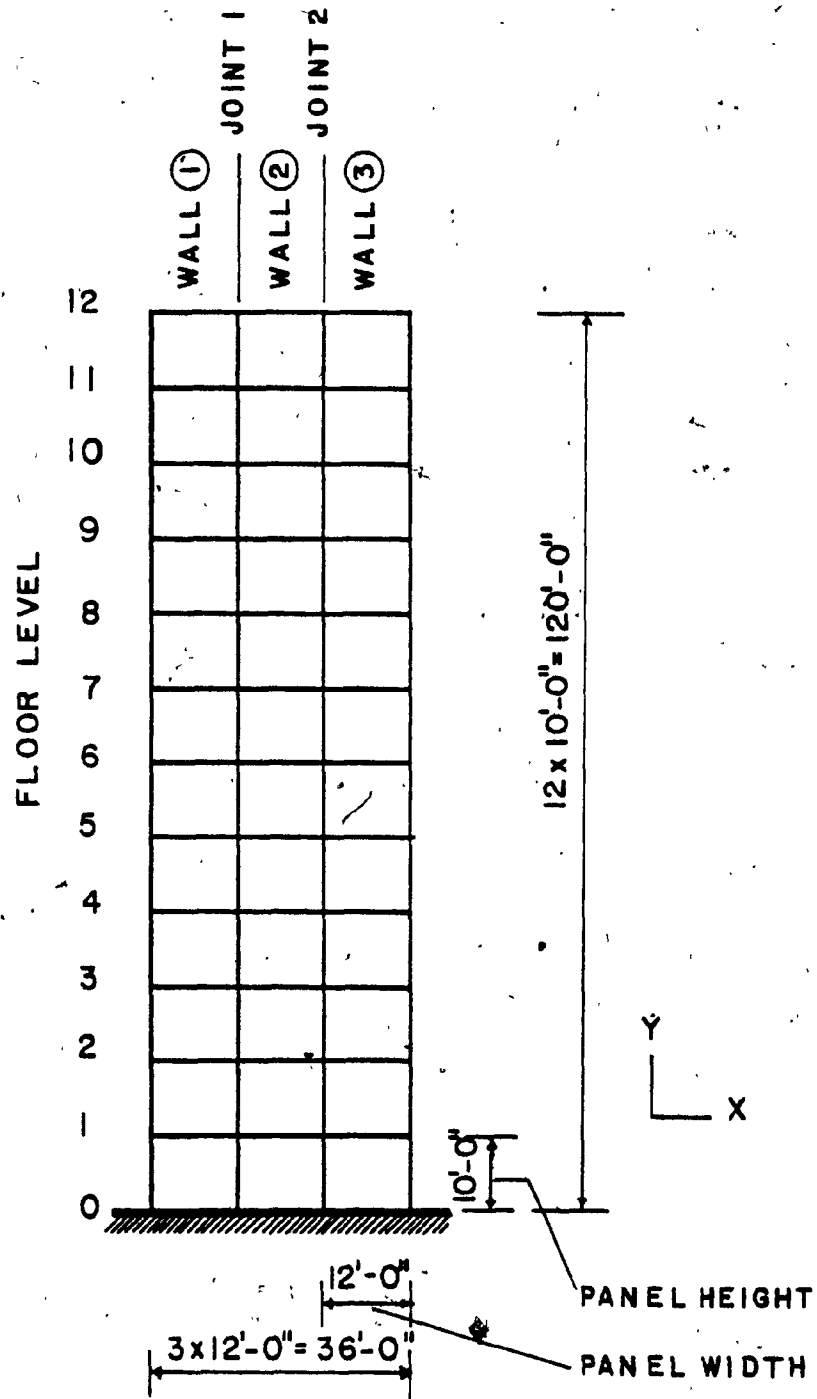


FIG. 5.1 ELEVATION OF PANEL STRUCTURE

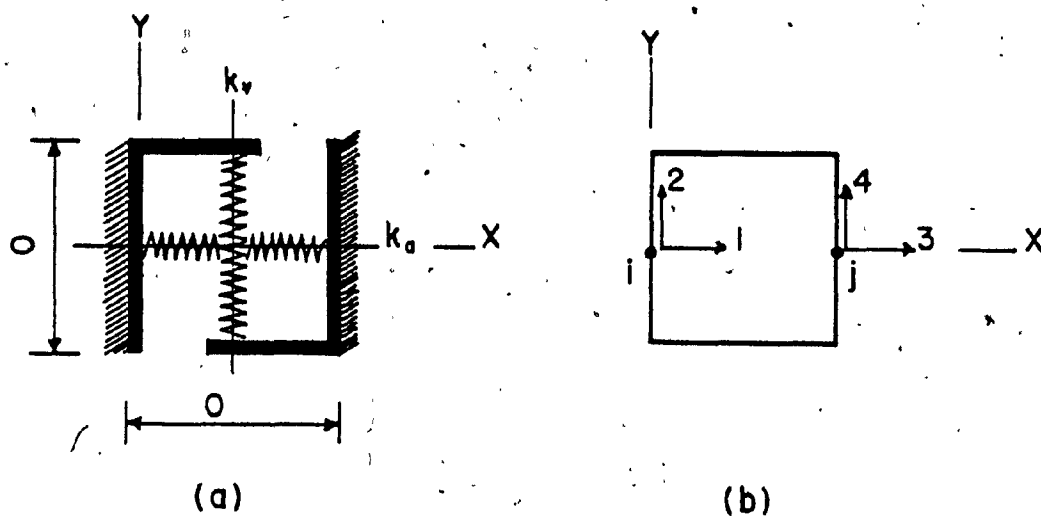
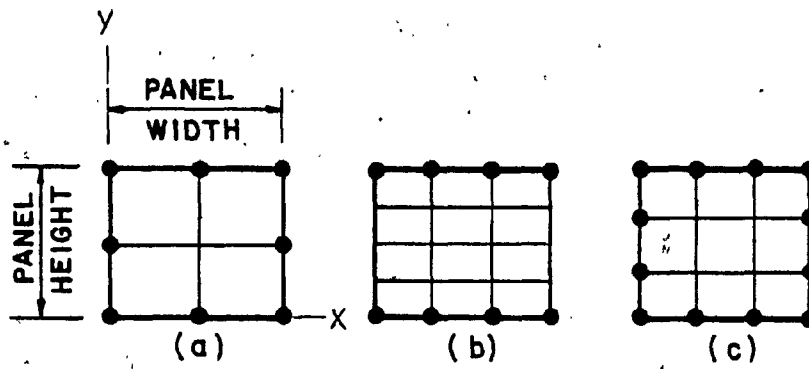
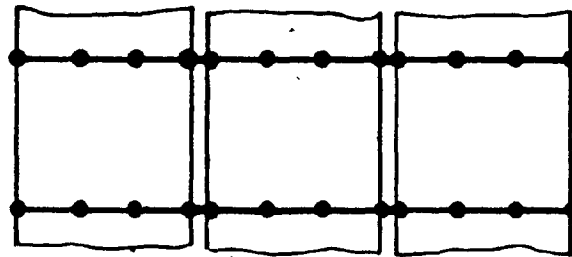


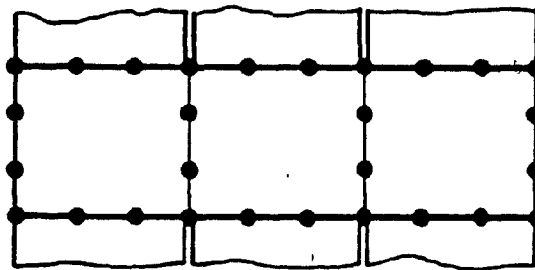
FIG. 5.2 CONNECTOR ELEMENT: (a) IDEALIZATION, AND
(b) DEGREES OF FREEDOM



PANEL ELEMENTS



(d) STOREY WITH CONNECTORS



(e) MONOLITHIC STOREY

FIG. 5.3 PANEL ELEMENTS AND STOREY ARRANGEMENTS ((a) TYPE(101); (b) TYPE(120); (c) TYPE(101,100/2))

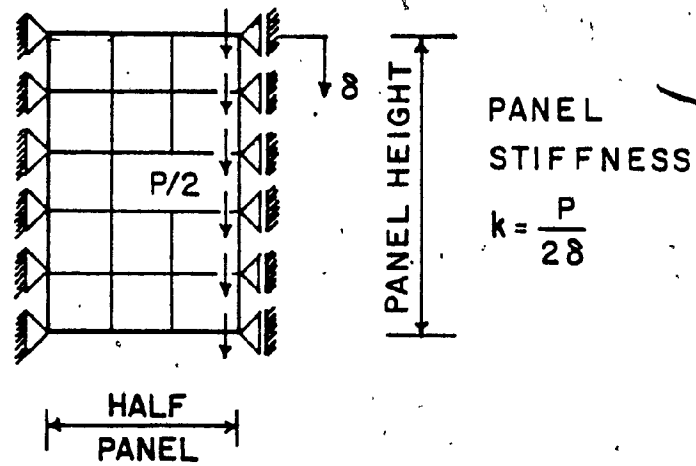


FIG. 5.4 DEFINITION OF PANEL STIFFNESS

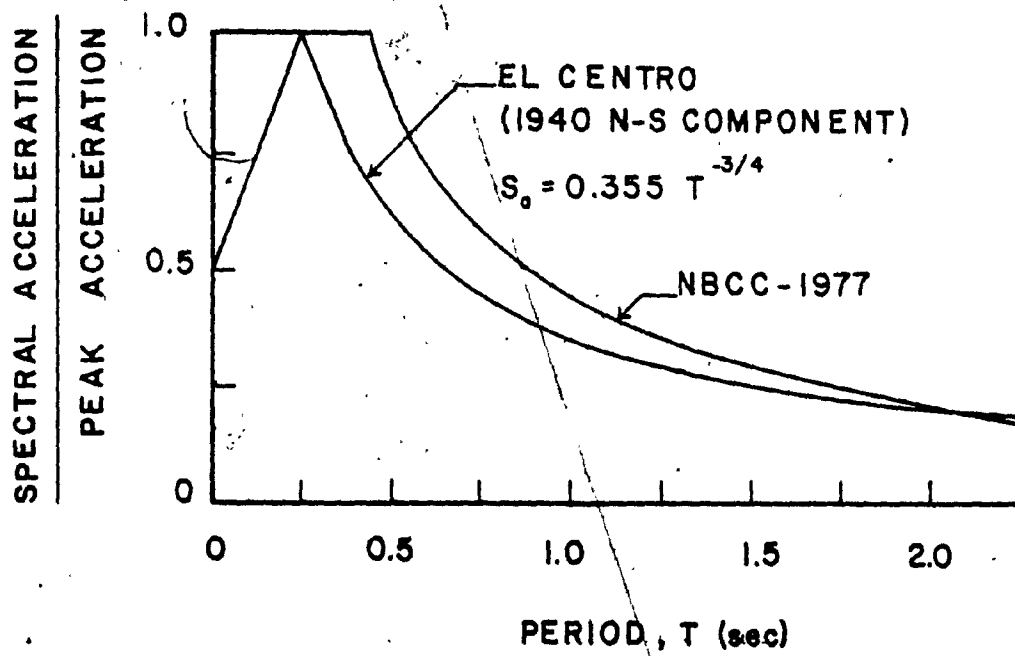


FIG. 5.5 ACCELERATION RESPONSE SPECTRA FOR 5 PER CENT DAMPING

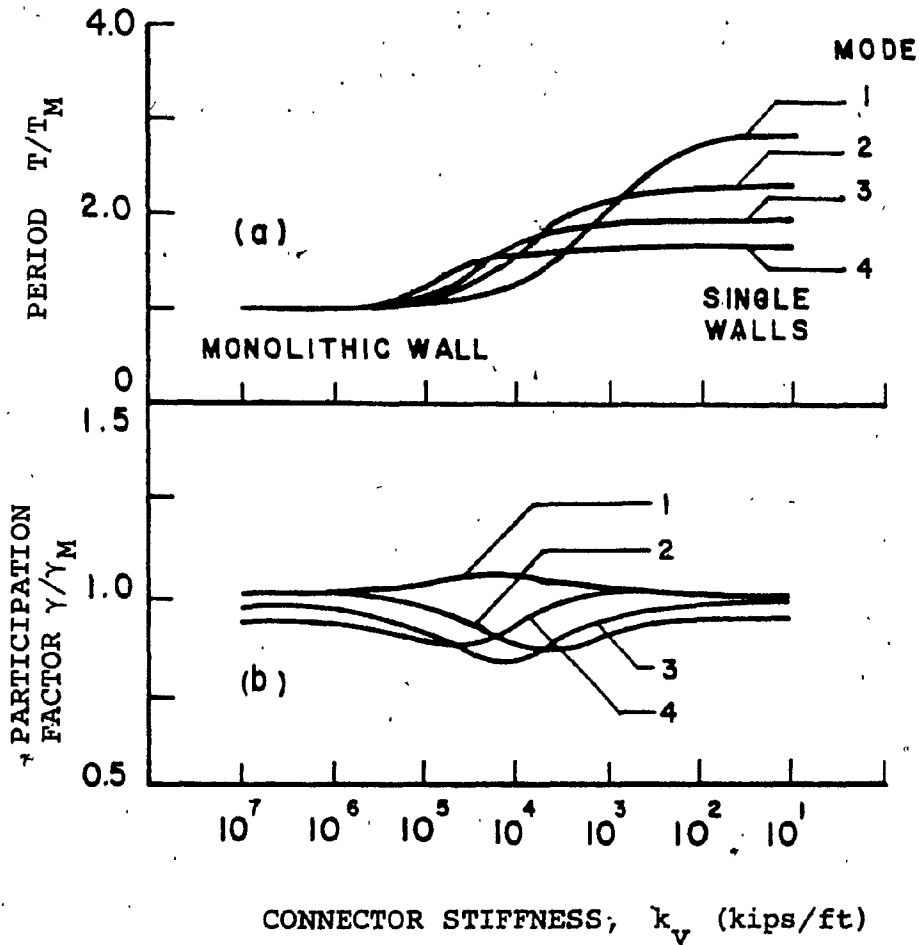


FIG. 5.6 NORMALIZED NATURAL PERIODS AND MODAL PARTICIPATION FACTORS AS FUNCTIONS OF CONNECTOR SHEAR STIFFNESS

(1 kip/ft = 14.6 kN/m)

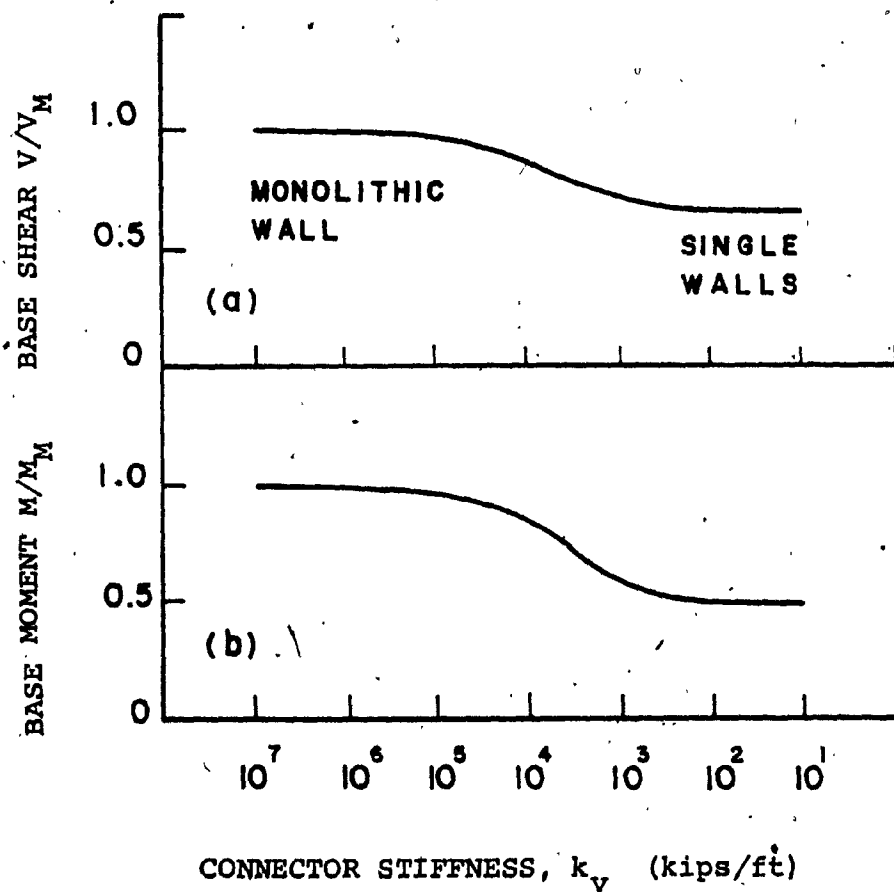


FIG. 5.7 NORMALIZED BASE SHEAR AND OVERTURNING MOMENT AS FUNCTIONS OF CONNECTOR SHEAR STIFFNESS
(1 kip/ft = 14.6 kN/m)

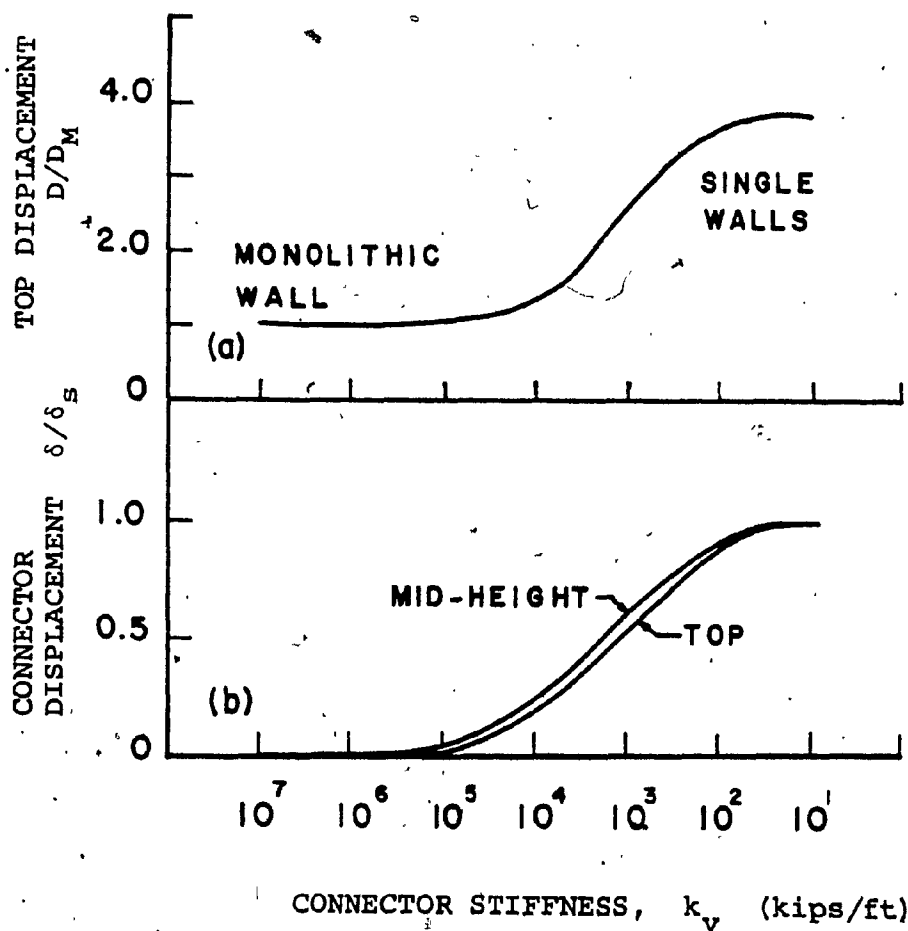


FIG. 5.8 NORMALIZED TOP AND CONNECTOR VERTICAL DISPLACEMENTS AS FUNCTIONS OF CONNECTOR SHEAR STIFFNESS (1 kip/ft = 14.6 kN/m)

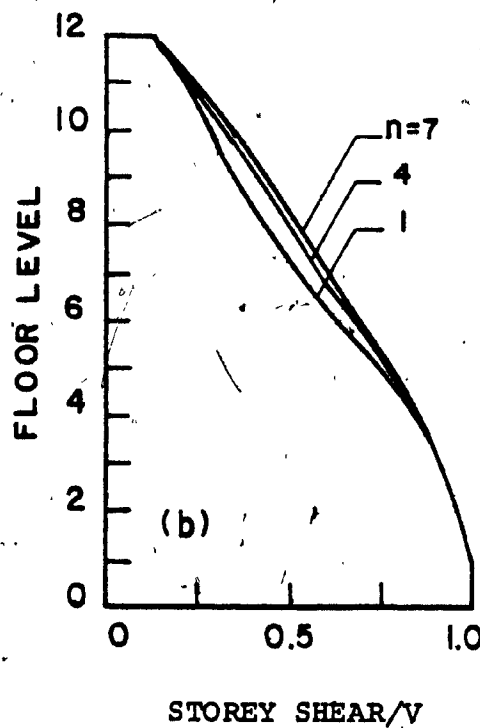
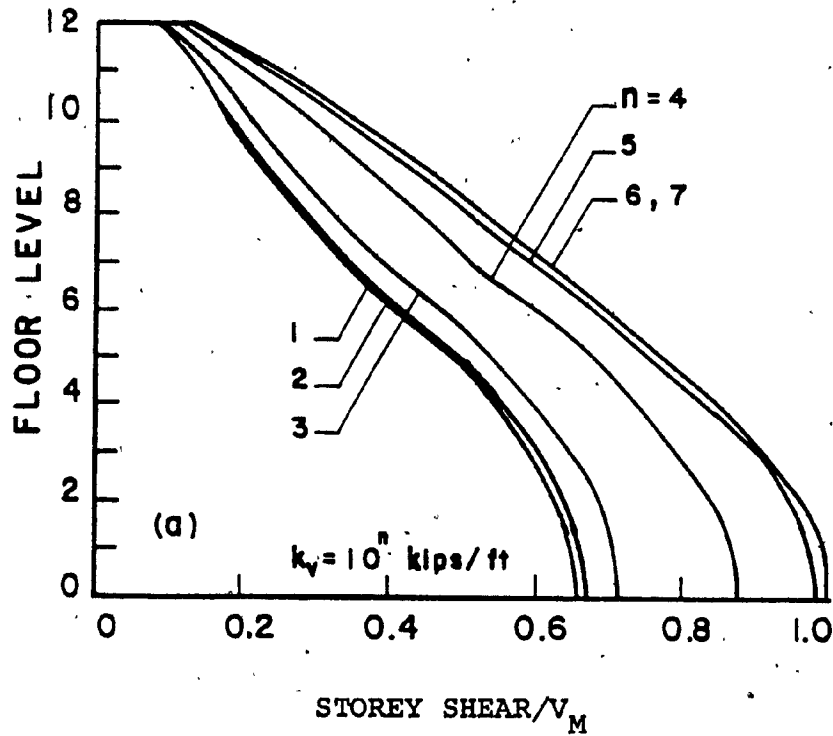


FIG. 5.9 DISTRIBUTION OF HORIZONTAL STOREY SHEARS FOR DIFFERENT CONNECTOR STIFFNESS k_v NORMALIZED TO; (a) BASE SHEAR OF MONOLITHIC WALL, AND (b) ACTUAL BASE SHEAR (1 kip/ft = 14.6 kN/m)

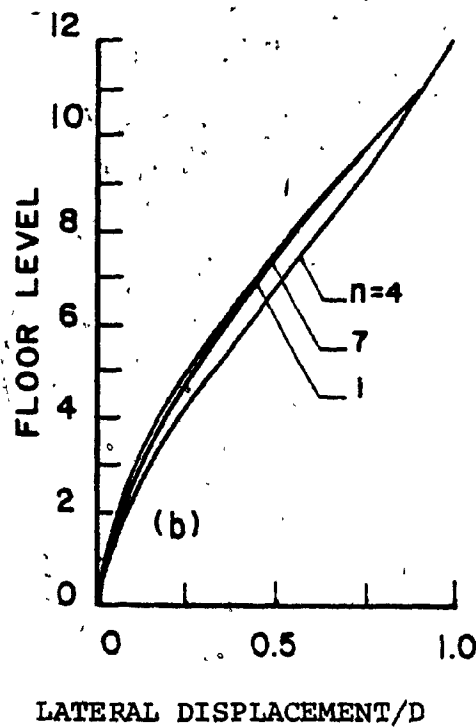
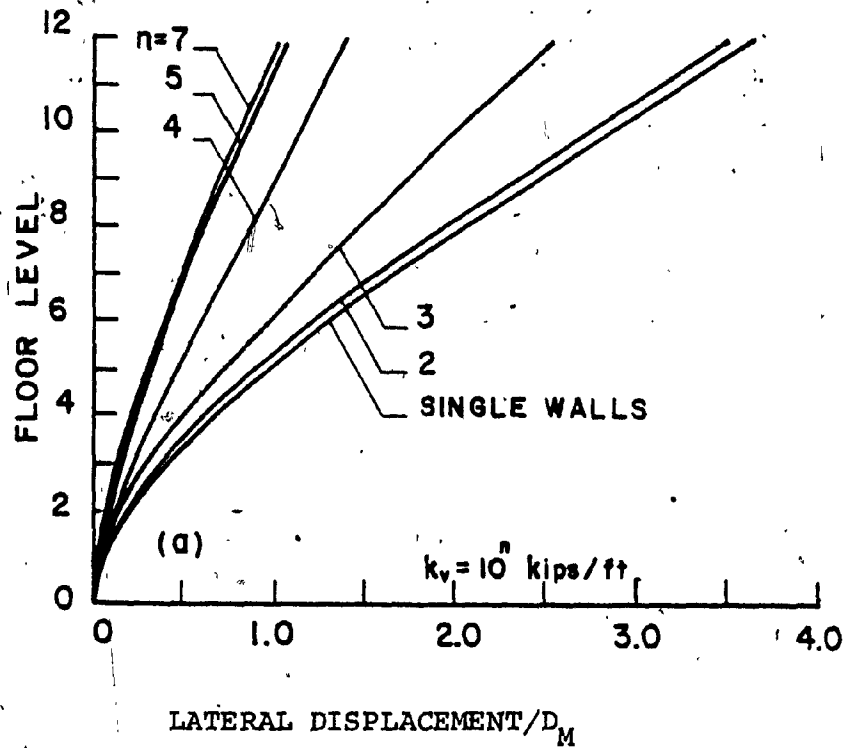


FIG. 5.10 ENVELOPES OF LATERAL DISPLACEMENTS FOR DIFFERENT CONNECTOR STIFFNESSES k_v , NORMALIZED TO; (a) TOP DISPLACEMENT OF MONOLITHIC WALL, AND (b) ACTUAL TOP DISPLACEMENT (1 kip/ft = 14.6 kN/m)

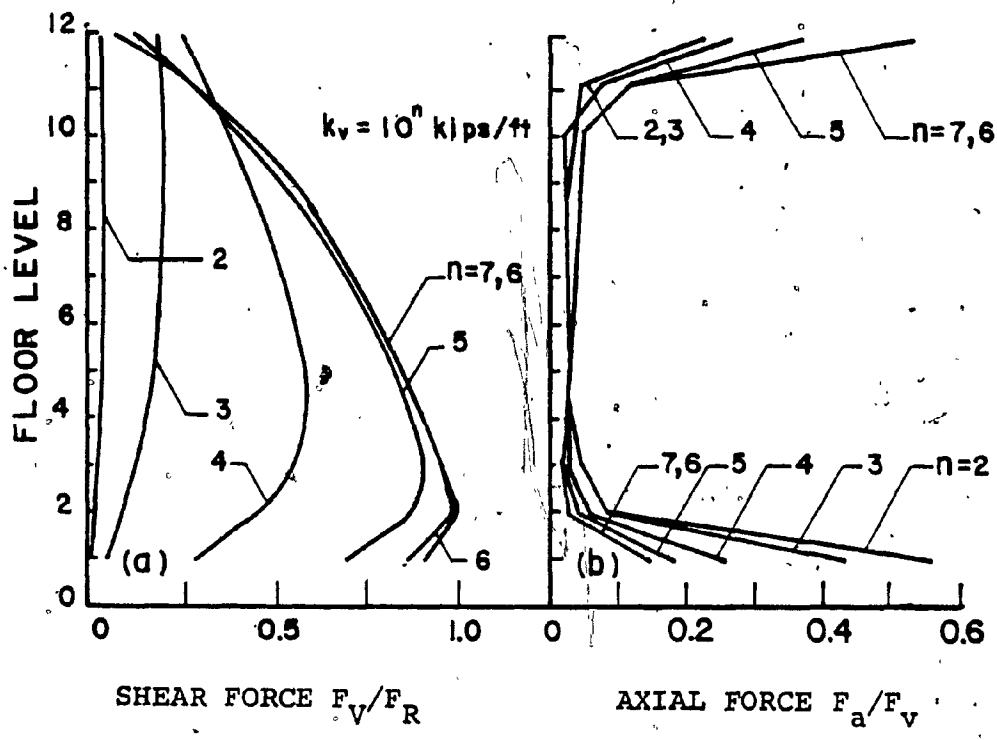


FIG. 5.11 NORMALIZED CONNECTOR SHEAR FORCES AND AXIAL-TO-SHEAR FORCE RATIOS FOR DIFFERENT STIFFNESS k_v
 (1 kip/ft = 14.6 kN/m)

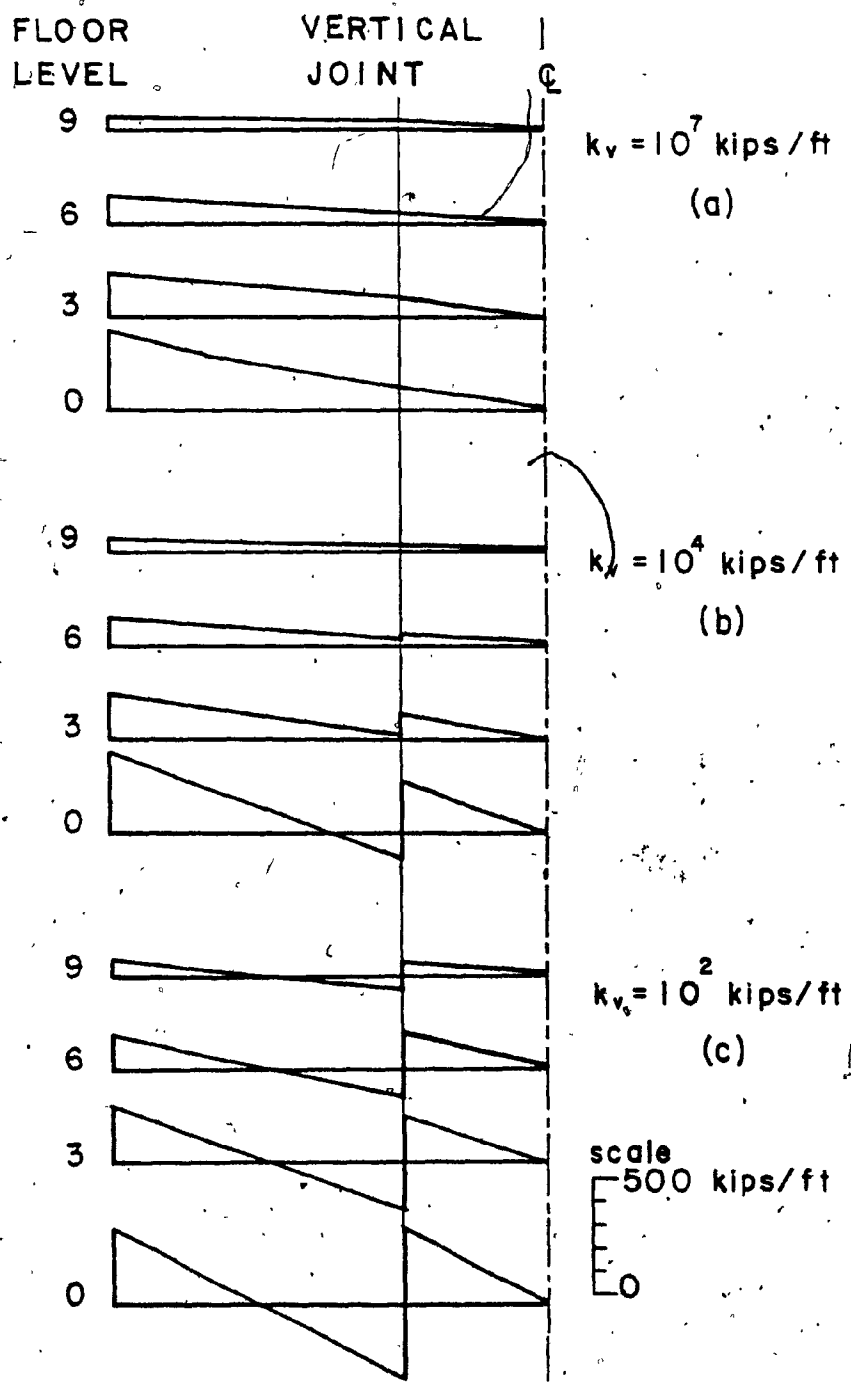


FIG. 5.12 NORMAL FORCES IN HORIZONTAL JOINTS FOR 1 g PEAK ACCELERATION (El Centro Spectrum) AND DIFFERENT CONNECTOR SHEAR STIFFNESS k_v (1 kip/ft = 14.6 kN/m)

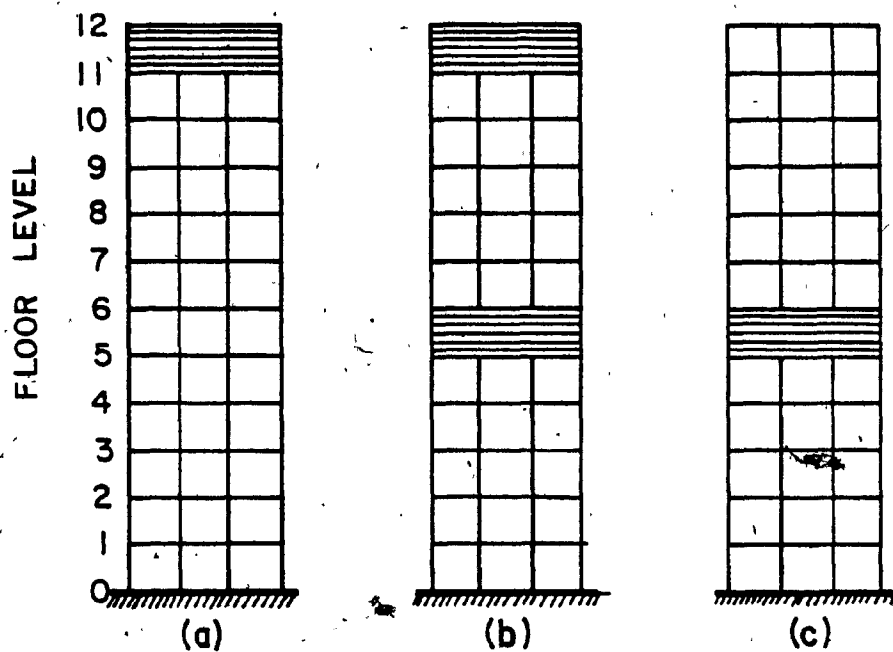


FIG. 5.13 STRUCTURES WITH MONOLITHIC STOREYS:

- (a) MONOLITHIC TOP STOREY
- (b) MONOLITHIC TOP AND MID-HEIGHT STOREY, AND
- (c) MONOLITHIC MID-HEIGHT STOREY

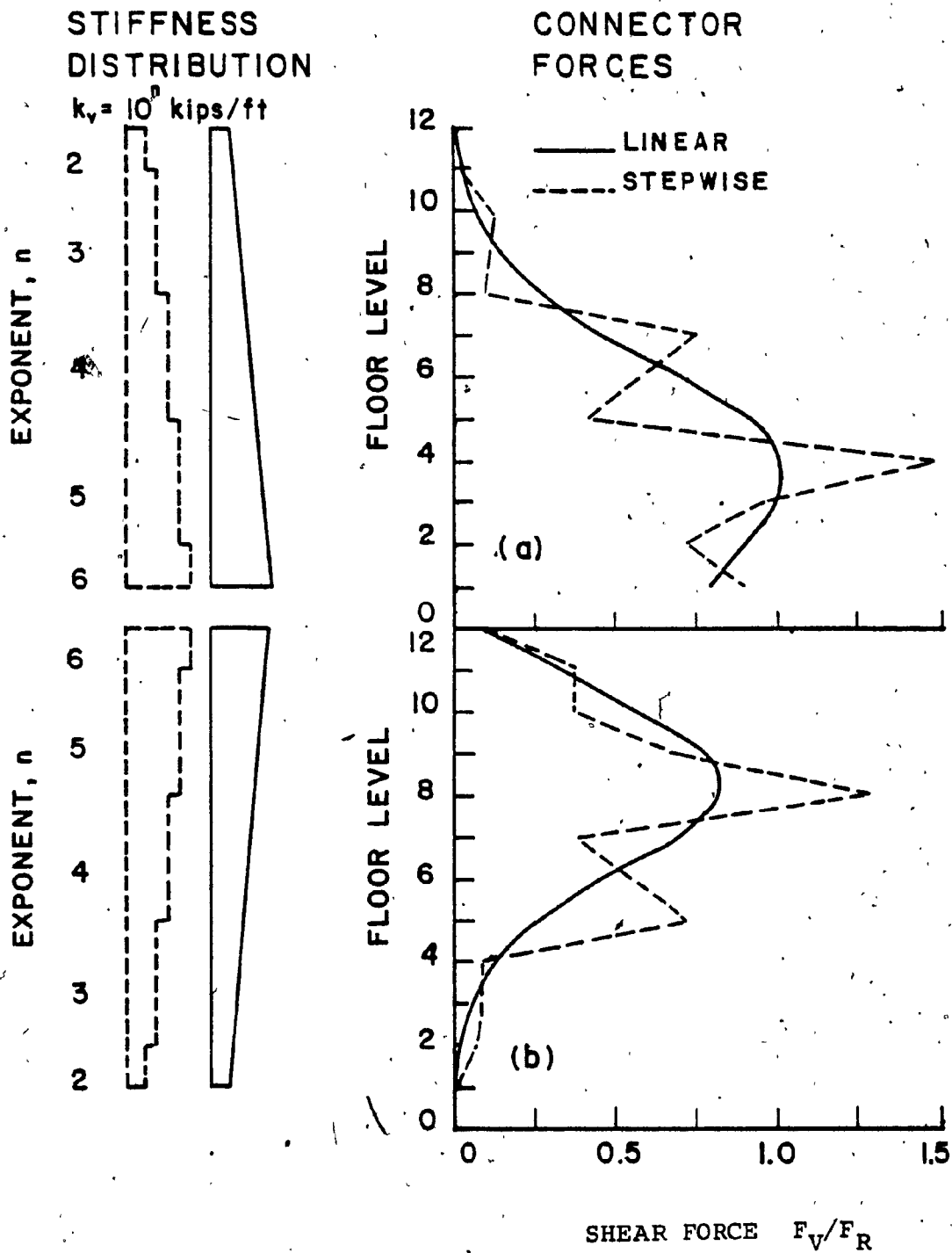


FIG. 5.14 EFFECT OF DISTRIBUTION OF CONNECTOR STIFFNESS k_v ON NORMALIZED CONNECTOR SHEAR FORCES
 (1 kip/ft = 14.6 kN/m)

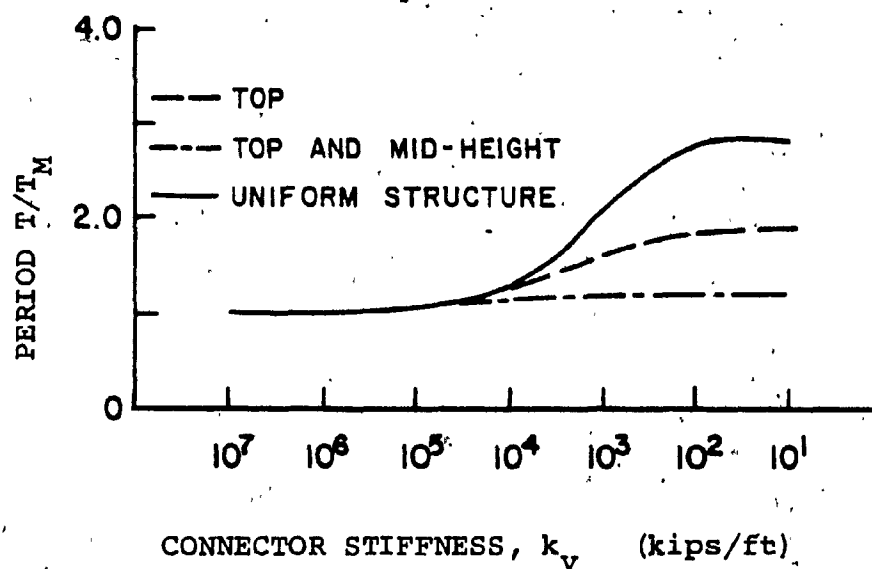


FIG. 5.15 EFFECT OF MONOLITHIC STOREYS AT TOP AND MID-HEIGHT ON NORMALIZED FUNDAMENTAL PERIOD
(1 kip/ft = 14.6 kN/m)

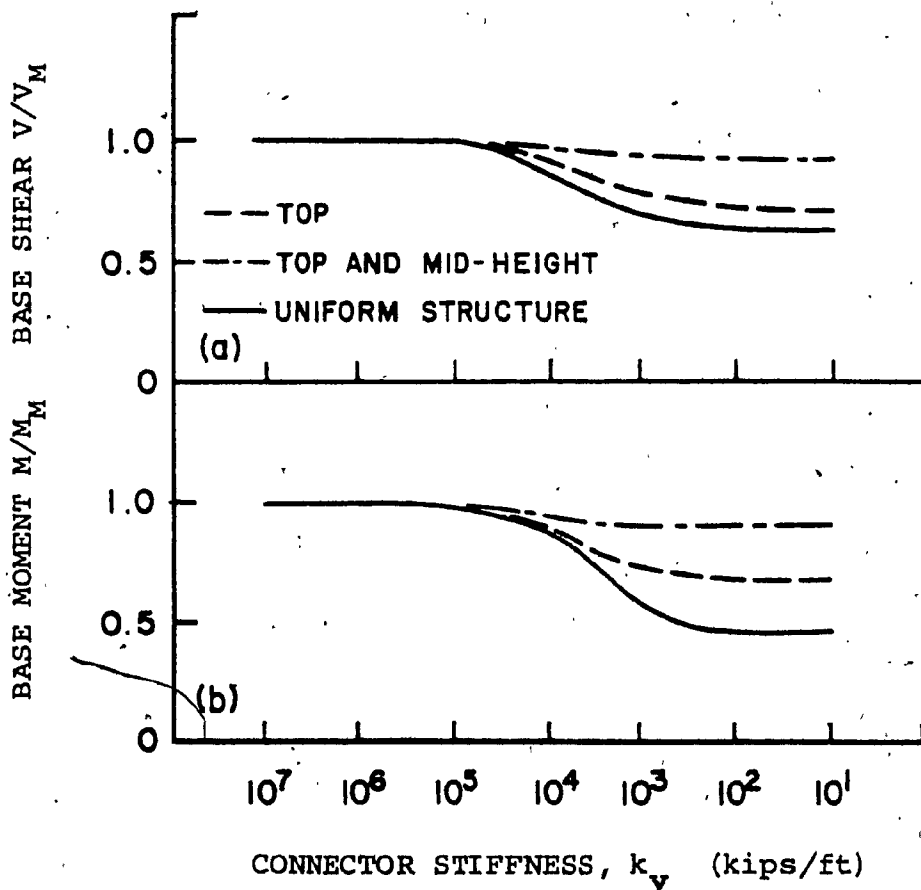


FIG. 5.16 EFFECT OF MONOLITHIC STOREYS AT TOP AND MID-HEIGHT ON NORMALIZED BASE SHEAR AND OVERTURNING MOMENT (1 kip/ft = 14.6 kN/m)

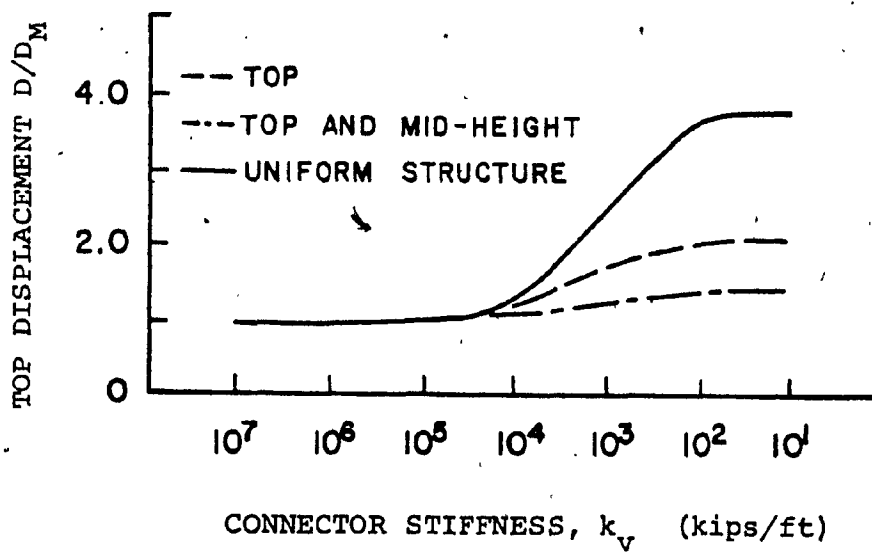


FIG. 5.17 EFFECT OF MONOLITHIC STOREYS AT TOP AND AT TOP AND MID-HEIGHT ON NORMALIZED TOP DEFLECTION (1 kip/ft = 14.6 kN/m)

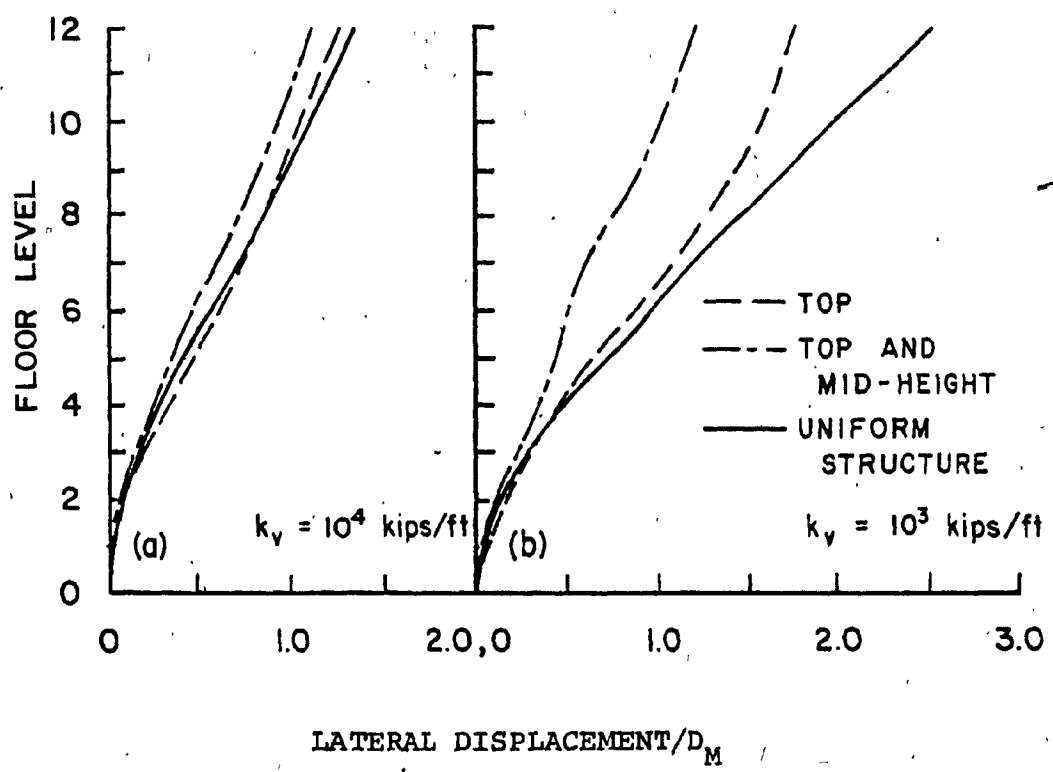


FIG. 5.18 EFFECT OF MONOLITHIC STOREYS AT TOP AND AT TOP AND MID-HEIGHT ON DEFLECTION ENVELOPES (1 kip/ft = 14.6 kN/m)

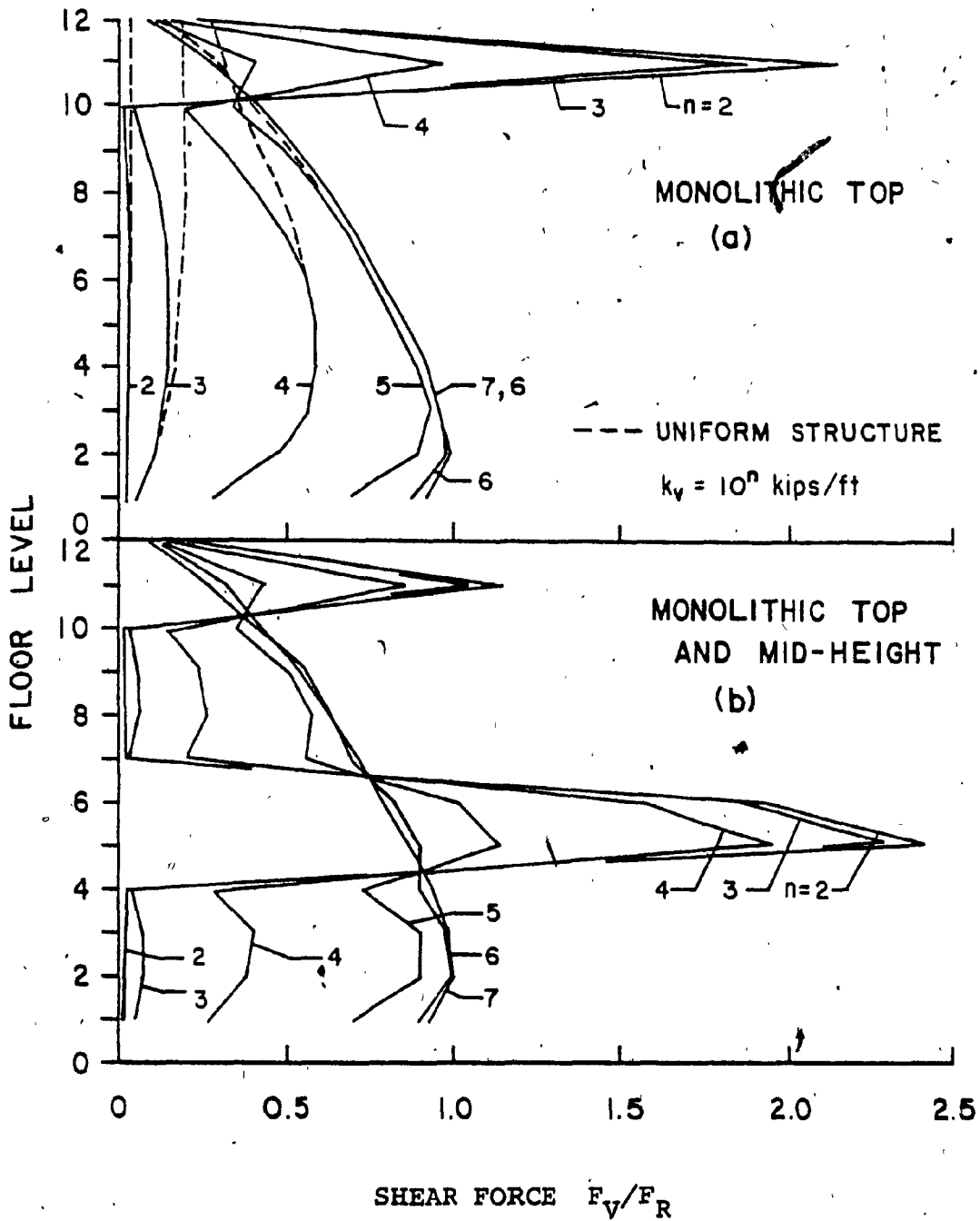


FIG. 5.19 NORMALIZED CONNECTOR SHEAR FORCES IN STRUCTURES WITH MONOLITHIC STOREYS FOR DIFFERENT CONNECTOR STIFFNESS k_v (1 kip/ft = 14.6 kN/m)

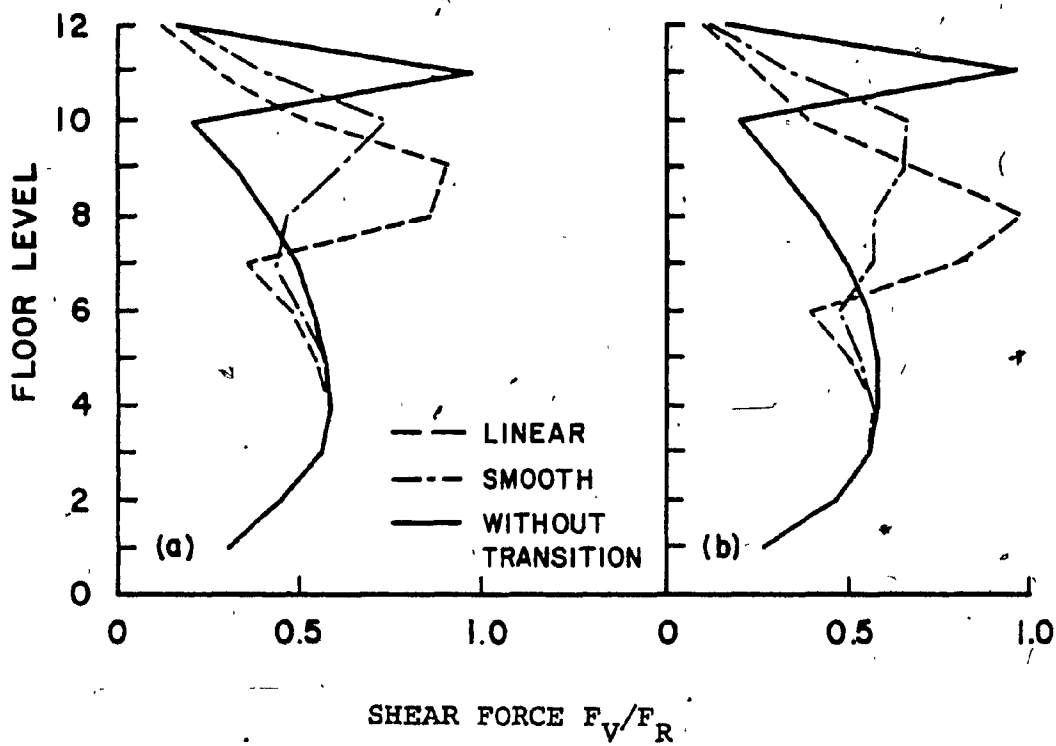


FIG. 5.20 COMPARISON OF LINEAR AND SMOOTH STIFFNESS TRANSITIONS FOR NORMALIZED CONNECTOR SHEAR FORCES IN STRUCTURES WITH MONOLITHIC TOP STOREY; (a) TRANSITION OVER 3 STOREYS, AND (b) TRANSITION OVER 4 STOREYS ($k_v = 10^4$ kips/ft (14.6×10^4 kN/m))

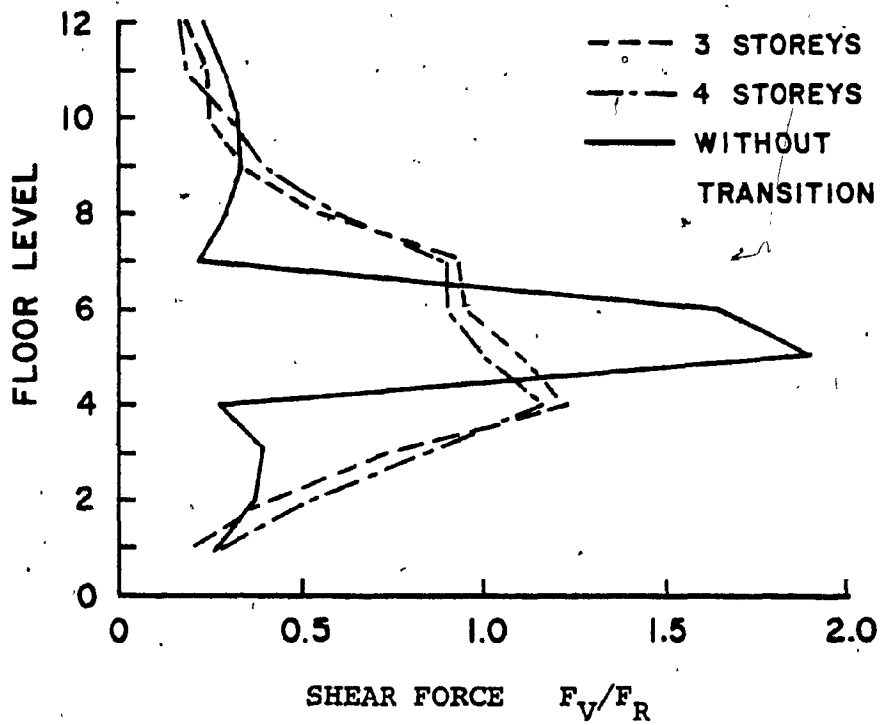


FIG. 5.21 EFFECT OF LENGTH OF SMOOTH STIFFNESS TRANSITION ON NORMALIZED CONNECTOR SHEAR FORCES IN STRUCTURE WITH MONOLITHIC MID-HEIGHT STOREY
 $(k_v = 10^4 \text{ kips/ft } (14.6 \times 10^4 \text{ kN/m}))$

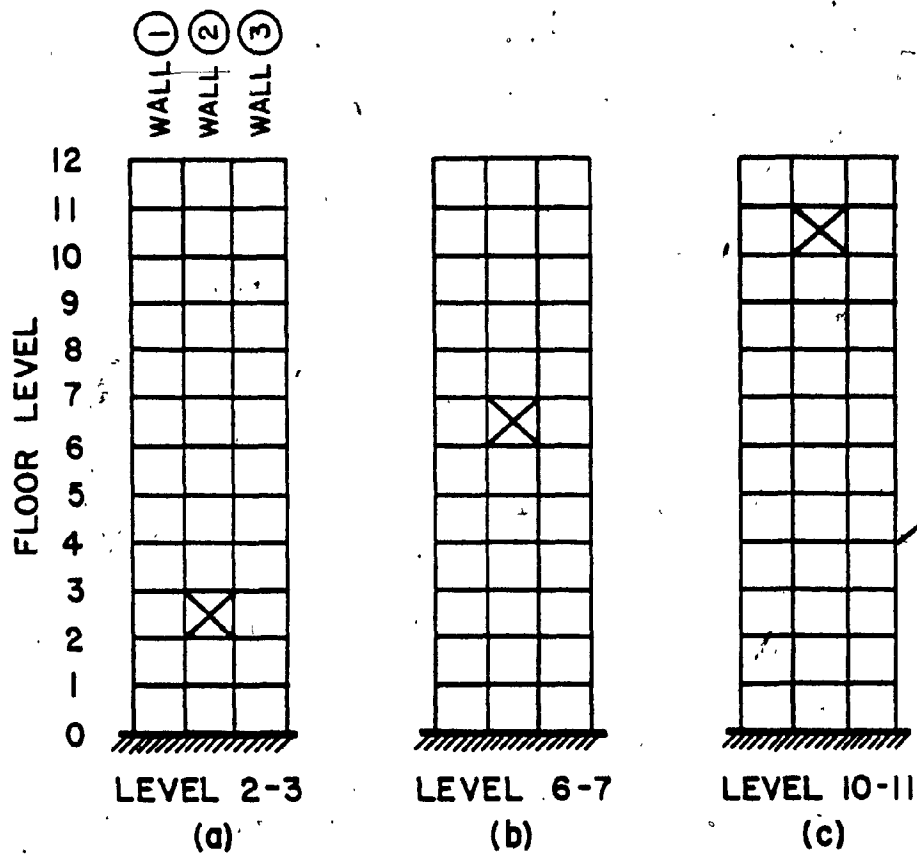


FIG. 5.22 STRUCTURES WITH A PANEL MISSING AT DIFFERENT LEVELS IN WALL 2

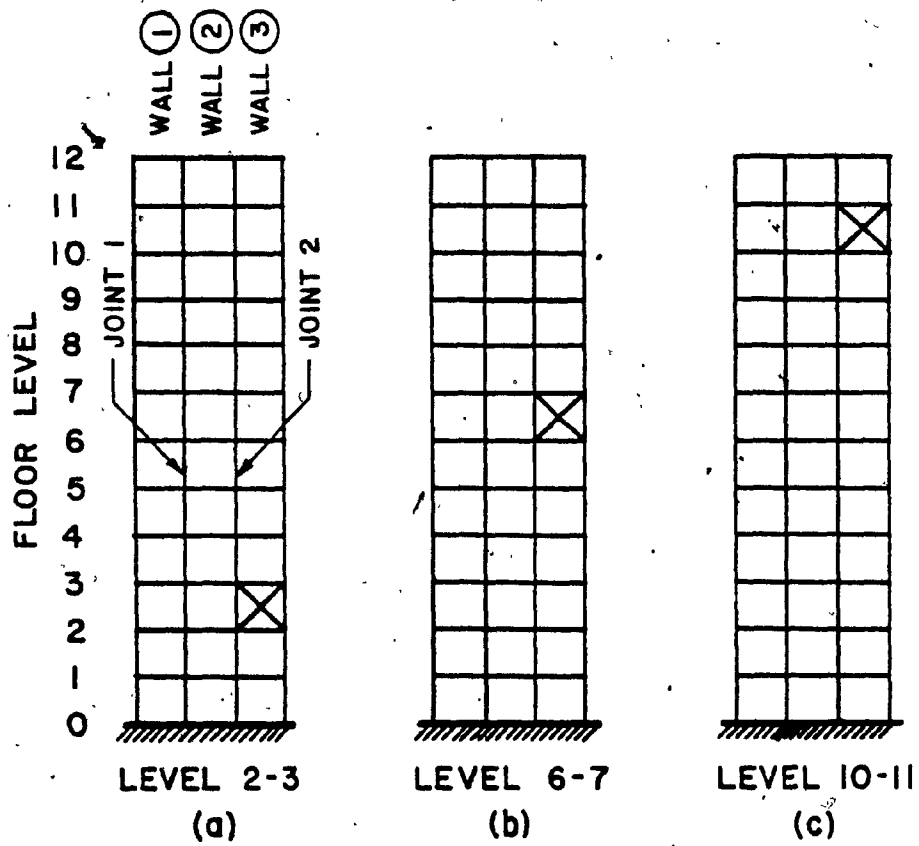


FIG. 5.23 STRUCTURES WITH A PANEL MISSING AT DIFFERENT LEVELS IN WALL 3

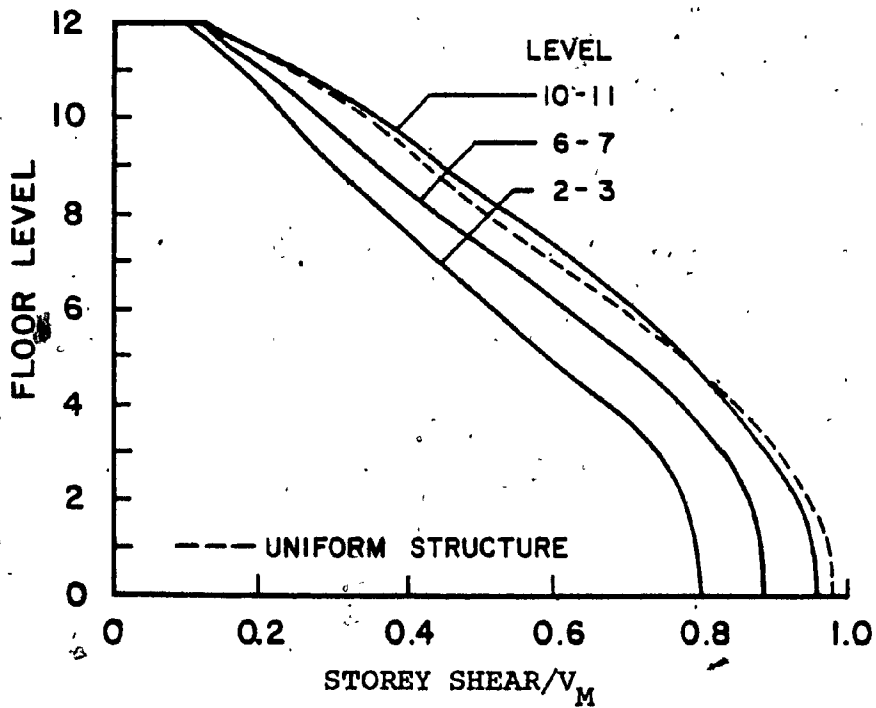


FIG. 5.24 EFFECT OF THE LEVEL OF PANELS MISSING IN WALL 3 ON NORMALIZED HORIZONTAL STOREY SHEARS ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

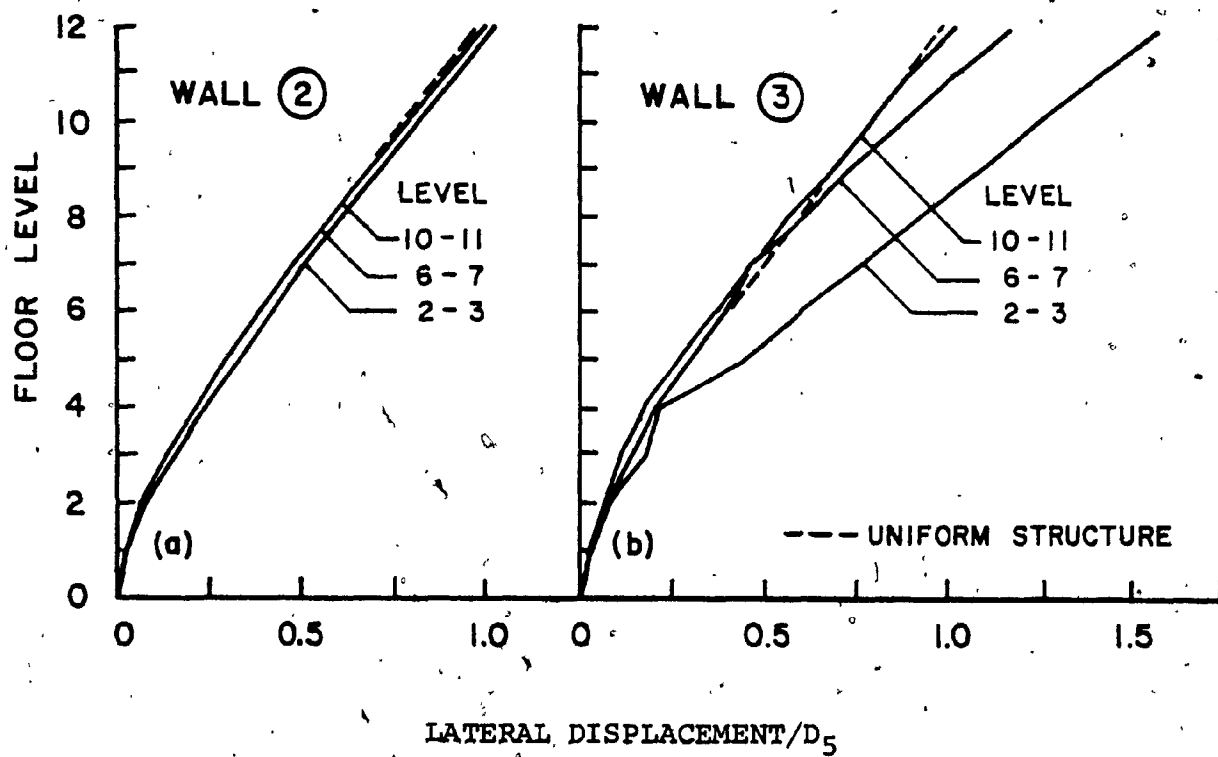


FIG. 5.25 EFFECT OF THE LEVEL OF MISSING PANELS ON NORMALIZED LATERAL DISPLACEMENTS ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

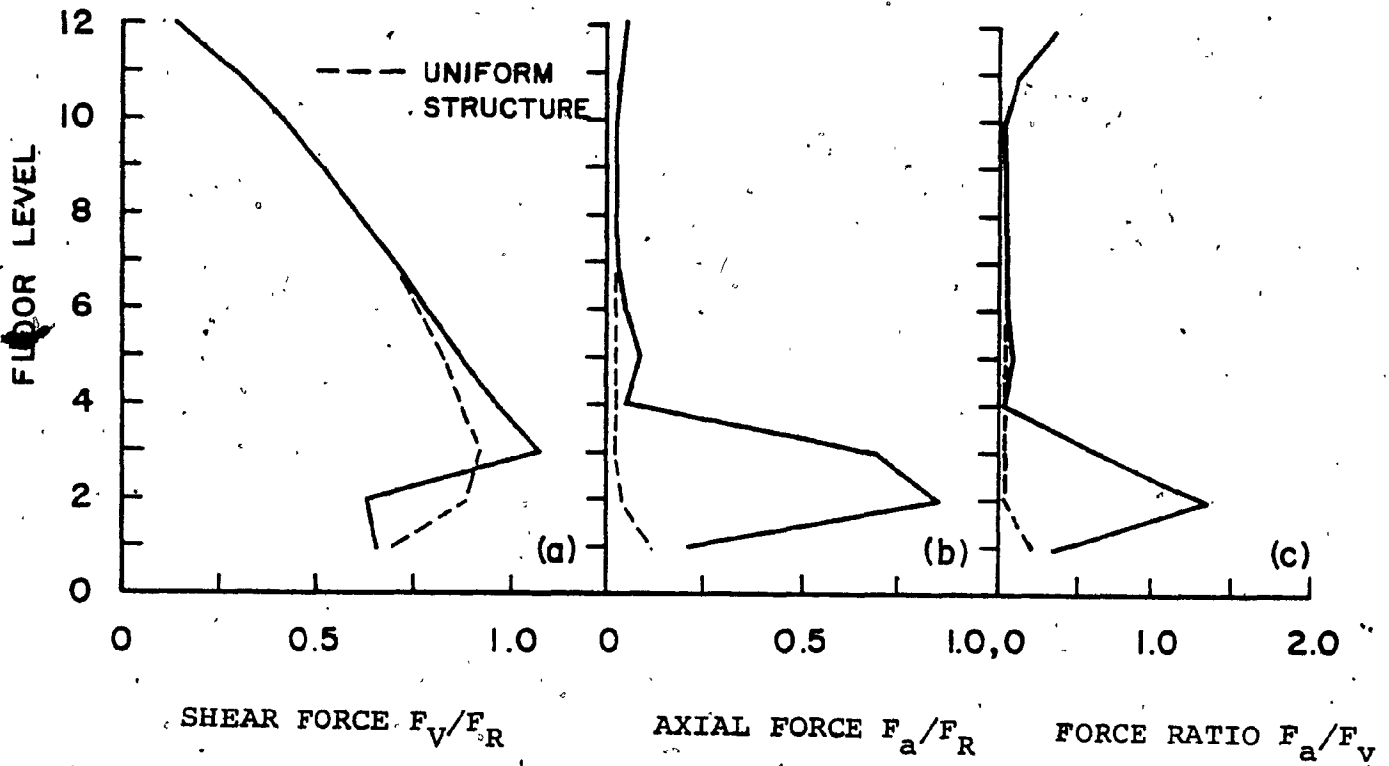


FIG. 5.26 EFFECT OF A PANEL MISSING AT LEVEL 2-3 IN WALL 2 ON NORMALIZED CONNECTOR FORCES ($k_v = 10^5$ kips/ft (14.6 x 10^5 kN/m))

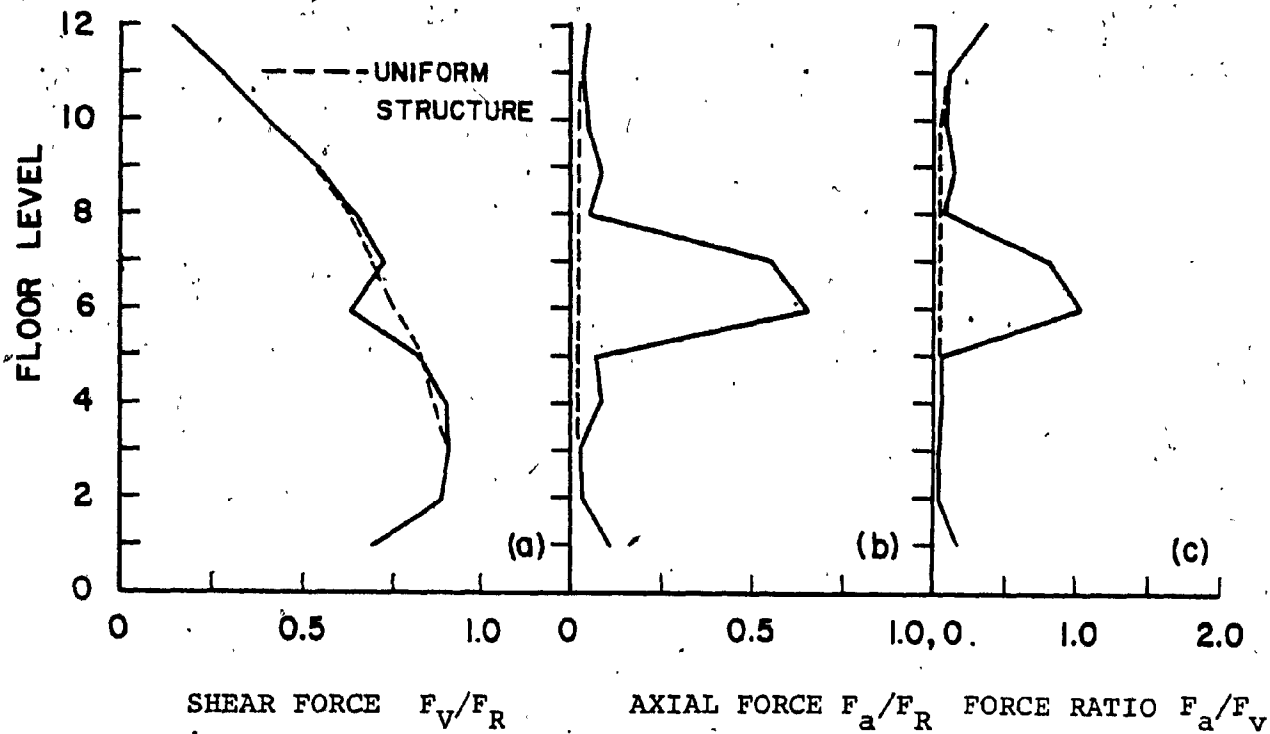


FIG. 5.27 EFFECT OF A PANEL MISSING AT LEVEL 6-7 IN WALL 2 ON NORMALIZED CONNECTOR FORCES ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

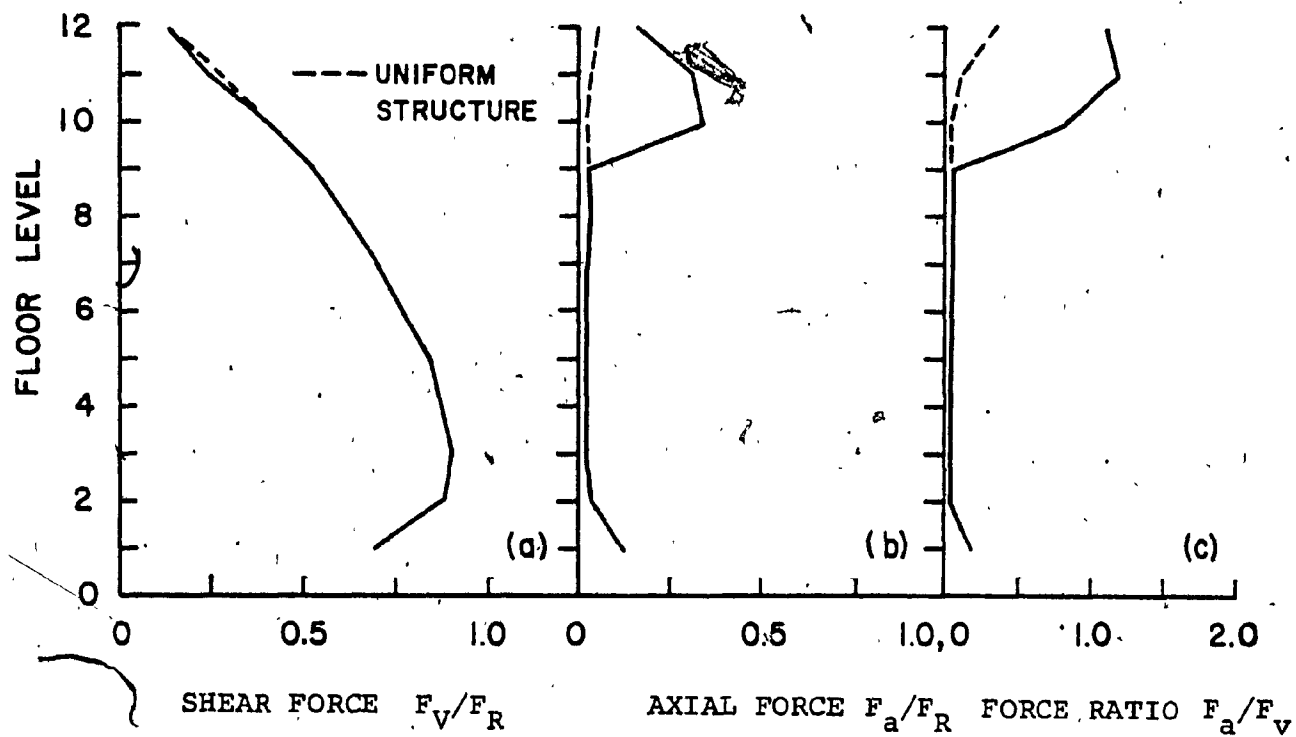


FIG. 5.28 EFFECT OF A PANEL MISSING AT LEVEL 10-11 IN WALL 2 ON NORMALIZED CONNECTOR FORCES ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

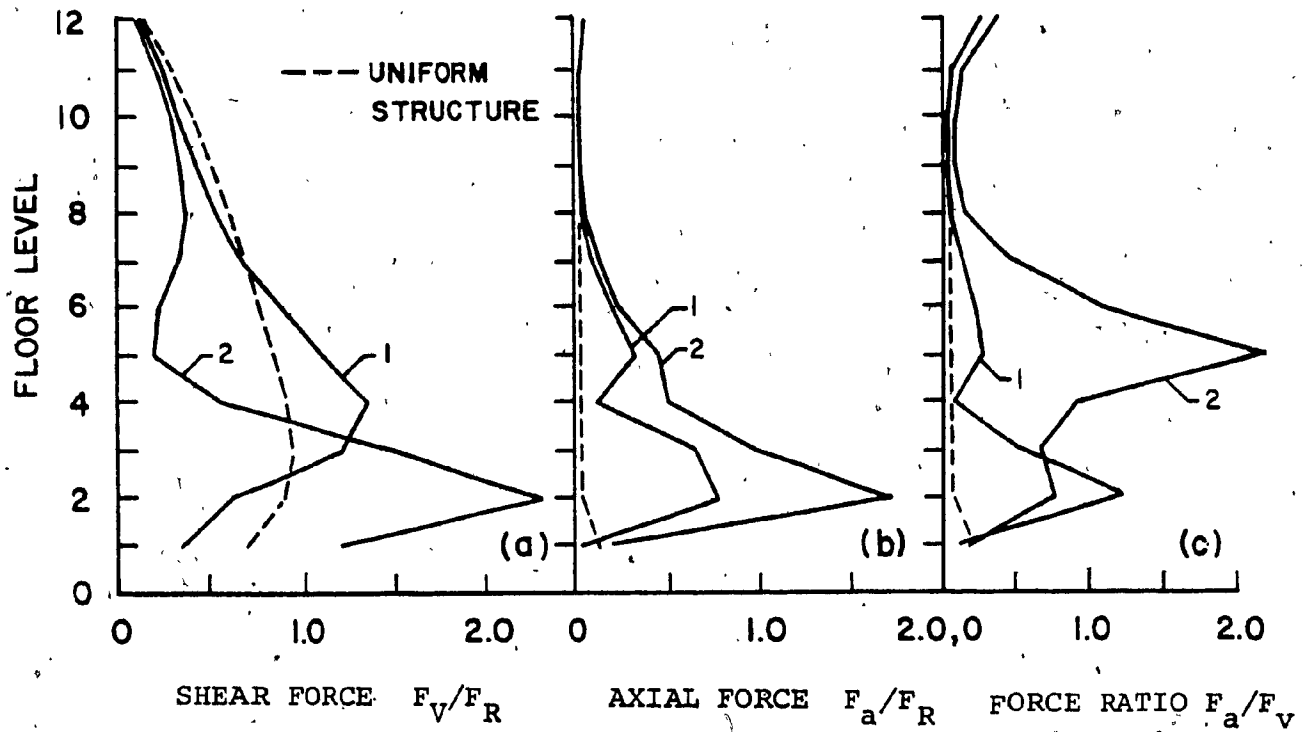


FIG. 5.29 EFFECT OF A PANEL MISSING AT LEVEL 2-3 IN WALL 3 ON NORMALIZED CONNECTOR FORCES ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

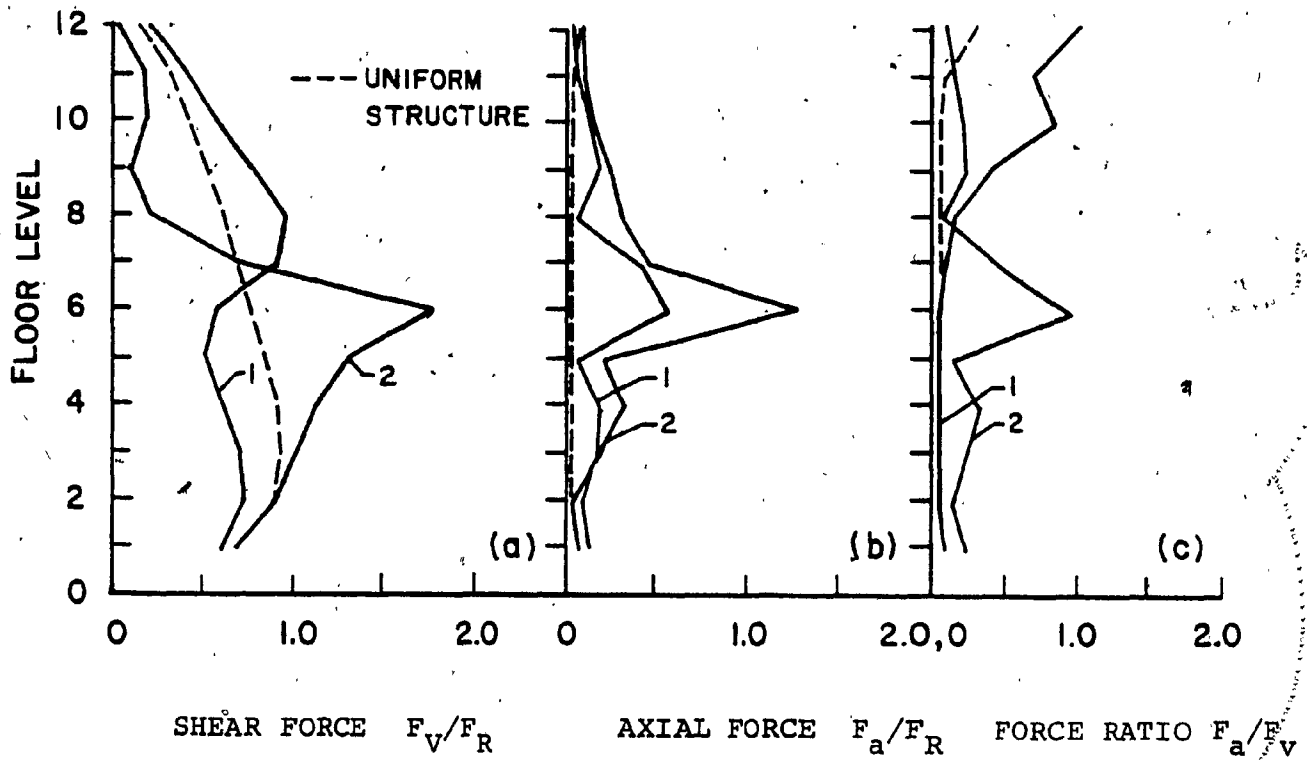


FIG. 5.30 EFFECT OF A PANEL MISSING AT LEVEL 6-7 IN WALL 3 ON NORMALIZED CONNECTOR FORCES ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

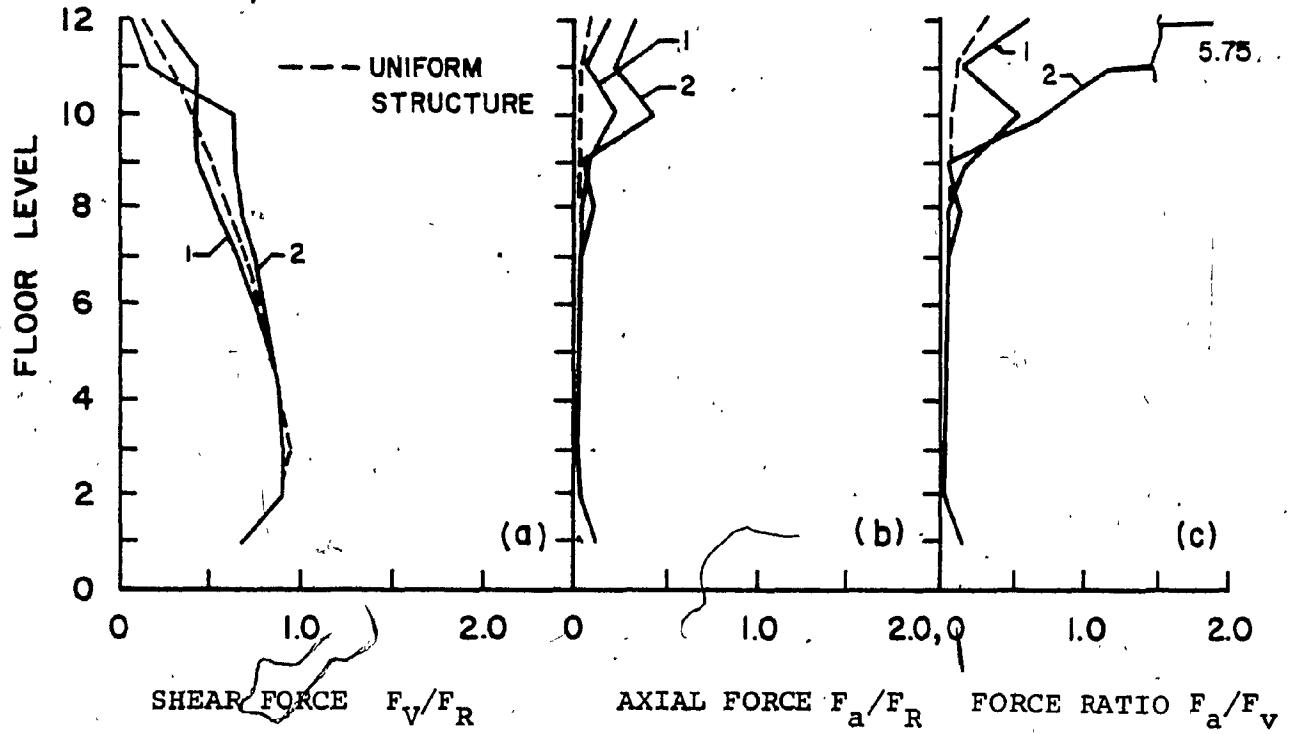


FIG. 5.31 EFFECT OF A PANEL MISSING AT LEVEL 10-11 IN WALL 3 ON NORMALIZED CONNECTOR FORCES ($k_v = 10^5$ kips/ft (14.6×10^5 kN/m))

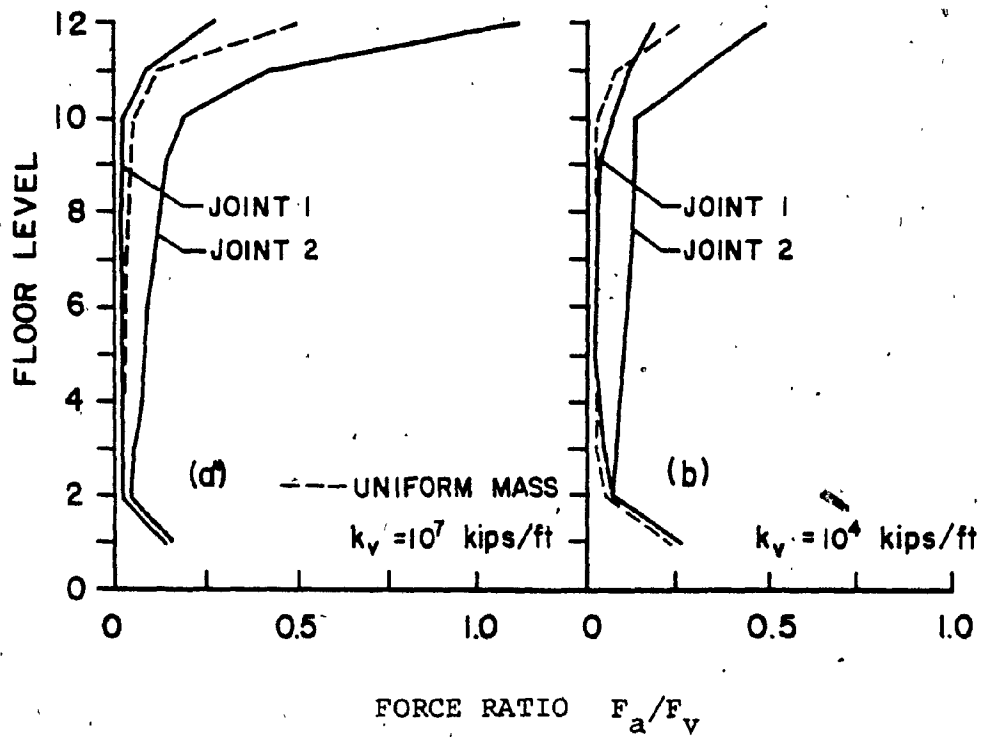


FIG. 5.32 EFFECT OF ASYMMETRIC FLOOR MASS DISTRIBUTION ON CONNECTOR AXIAL TO SHEAR FORCE RATIOS
(1 kip/ft = 14.6 kN/m)

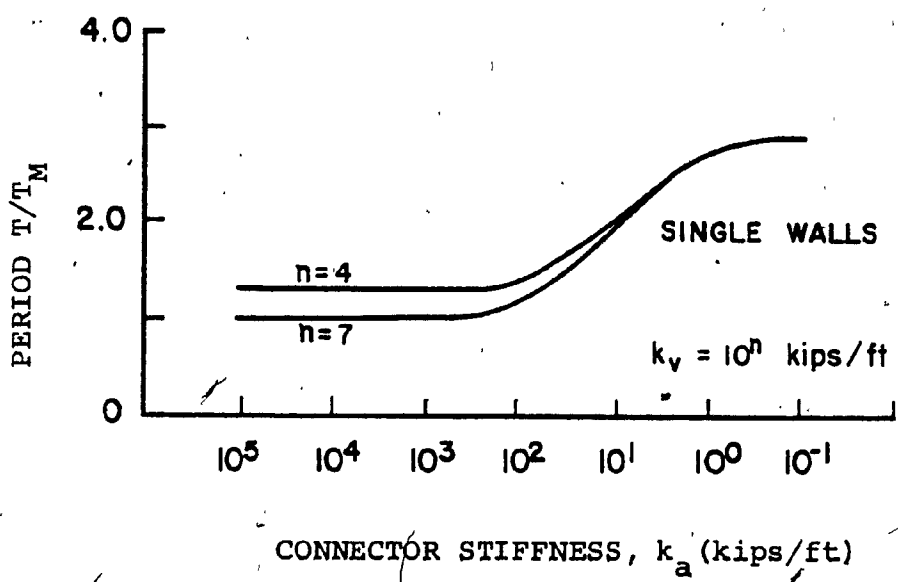


FIG. 5.33 INFLUENCE OF CONNECTOR AXIAL STIFFNESS k_a ON NORMALIZED FUNDAMENTAL PERIOD (1 kip/ft = 14.6 kN/m)

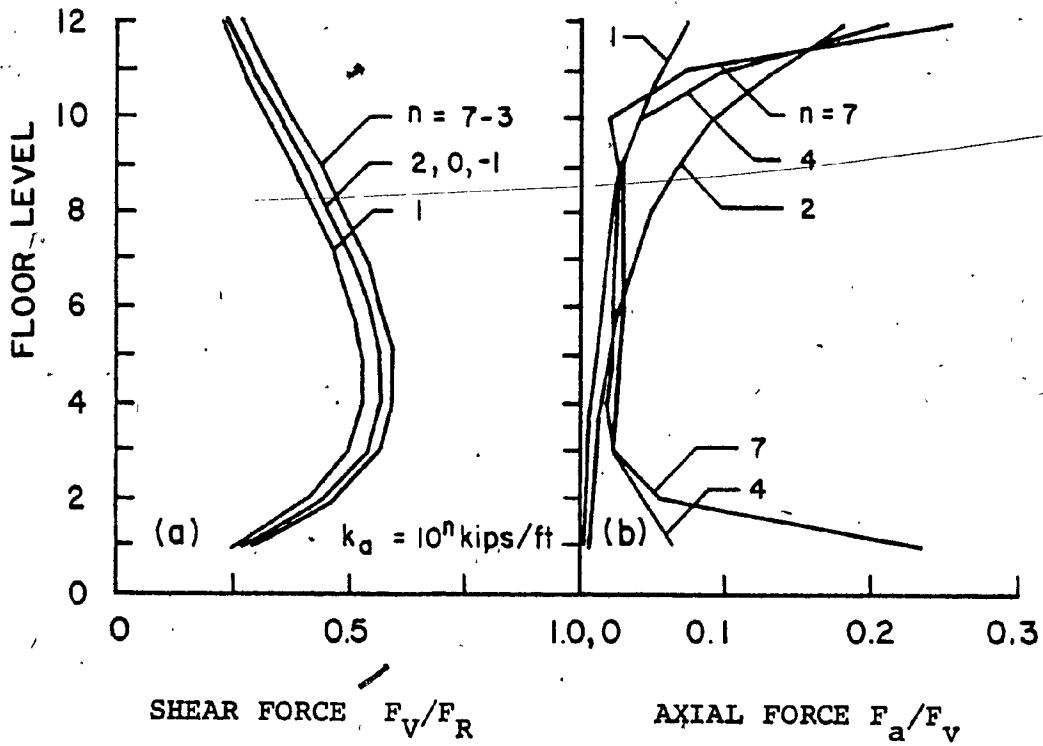


FIG. 5.34 INFLUENCE OF CONNECTOR AXIAL STIFFNESS k_a ON NORMALIZED CONNECTOR SHEAR FORCES AND AXIAL-TO-SHEAR FORCE RATIOS ($k_v = 10^4$ kips/ft (14.6×10^4 kN/m))

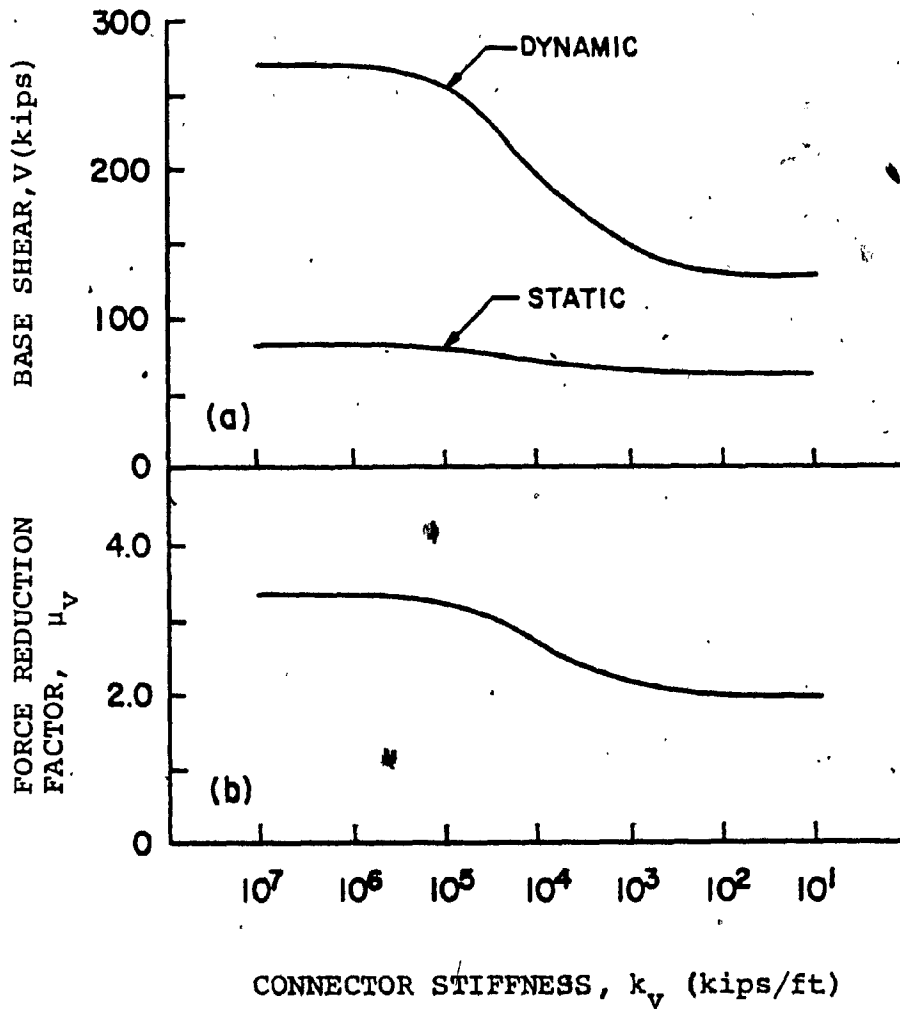


FIG. 5.35 COMPARISON OF BASE SHEARS FOR STATIC AND DYNAMIC ANALYSES (NBCC-1977) AND FORCE REDUCTION FACTOR μ_v (1 kips/ft = 14.6 kN/m, 1 kip = 4.45 kN)

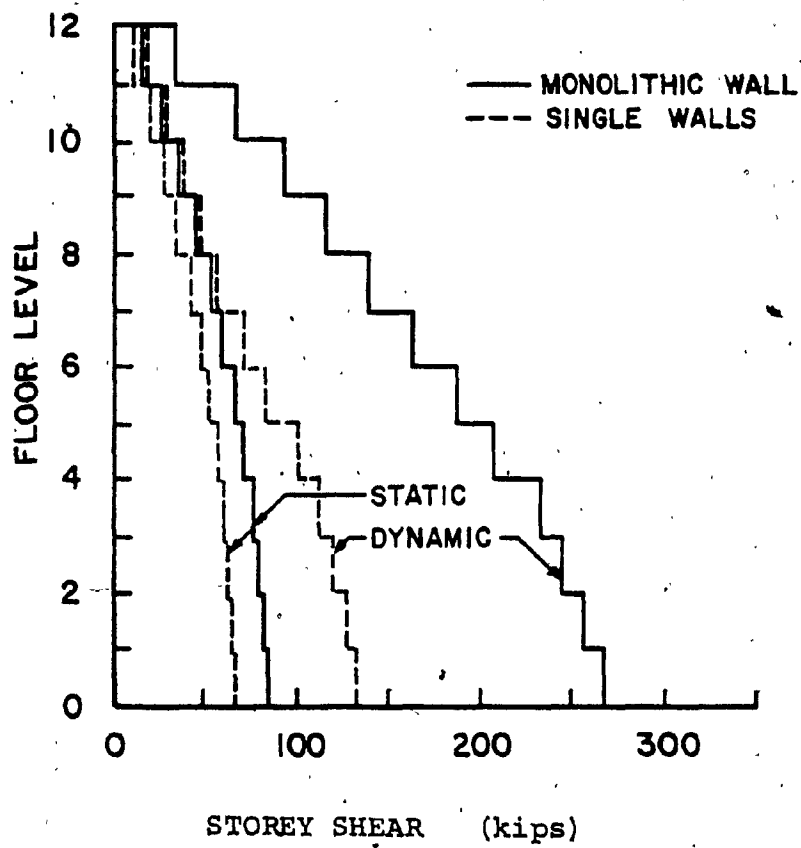


FIG. 5.36 COMPARISON OF HORIZONTAL STOREY SHEARS FOR STATIC AND DYNAMIC ANALYSES (NBCC-1977), (1 kip = 4.45 kN)

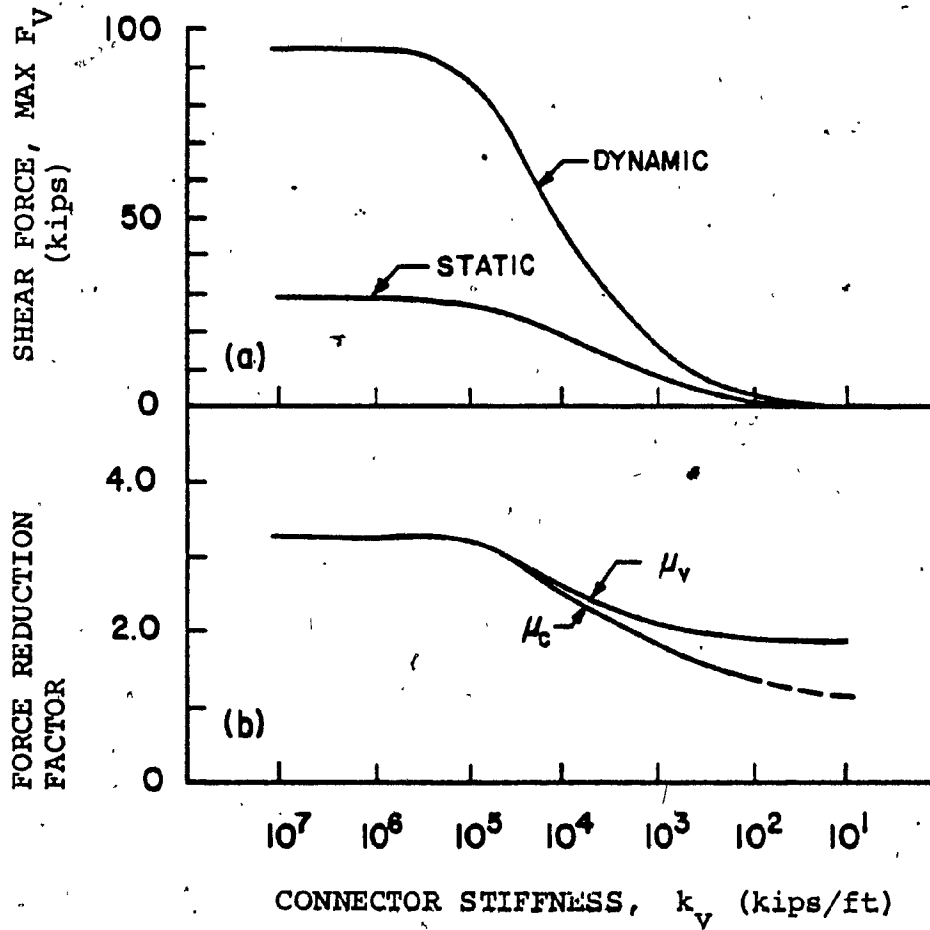


FIG. 5.37 COMPARISON OF MAXIMUM CONNECTOR SHEAR FORCES FOR STATIC AND DYNAMIC ANALYSES (NBCC-1977) AND ASSOCIATED FORCE REDUCTION FACTORS (1 kip/ft = 14.6 kN/m, 1 kip = 4.45 kN)

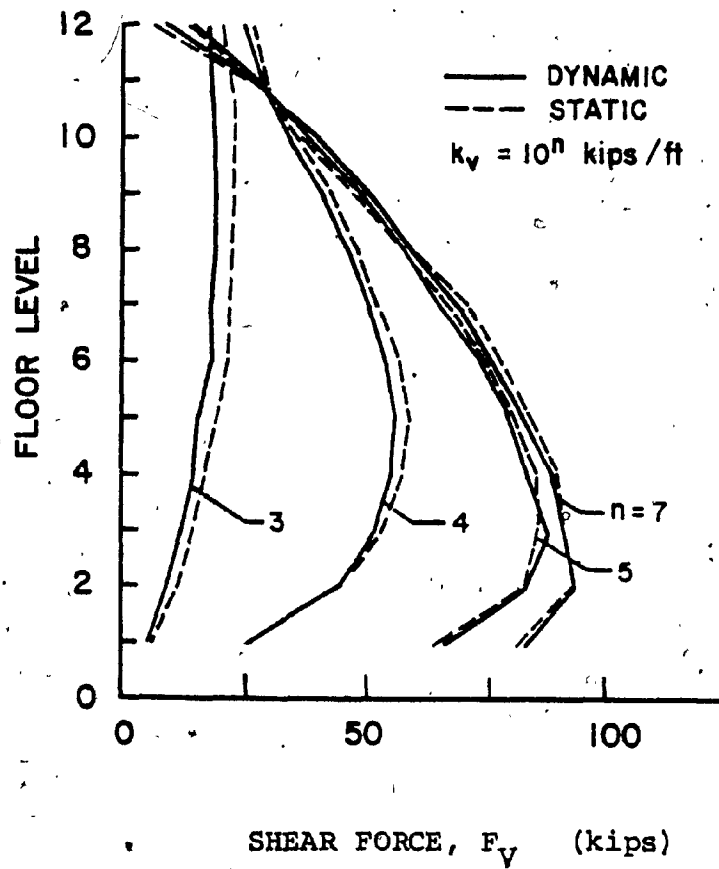


FIG. 5.38 COMPARISON OF CONNECTOR SHEAR FORCE DISTRIBUTIONS FOR DYNAMIC AND NORMALIZED STATIC ANALYSES (1 kip = 4.45 kN)

CHAPTER VI

SUMMARY AND CONCLUDING REMARKS

CHAPTER VI

SUMMARY AND CONCLUDING REMARKS

A substructuring technique based on the finite element method has been developed. This technique results in the creation of a rectangular panel element which may be applied efficiently in linear dynamic analysis, particularly for structural shear wall problems involving large-panel construction. The procedure is formulated in terms of a computer subroutine and the present version may be directly attached to SAP4 [24] as a substructure option.

A review of general theory and various methods applied in dynamic substructuring forms the basis for the present development. The procedure of this study utilizes multilevel substructuring and employs a sequence of static condensations leaving only chosen boundary nodes. Suitable numerical algorithms are investigated and a growth scheme is developed which allows versatile boundary node arrangement as well as simple input instructions. Possible extensions and refinement for the scheme are also discussed. The following panel element matrices, relating to retained boundary nodes can be created:

- (1) stiffness matrix

- (2) mass matrix, which may be consistent or, optionally, diagonally lumped and generated through consistent lumping,
- (3) stress-displacement matrix which relates boundary degrees of freedom to evenly distributed stress points inside the substructure, thus allowing full stress recovery.

Several applications demonstrate the efficient and versatile applicability of the panel element and indicate also expected savings in computing time. The main focus, however, is on solution accuracy for different characteristics and corresponding representation of mass. For plate bending vibrations it is shown that homogeneous inertia properties must be expressed by a consistent substructure mass matrix, in order to model accurately the dynamic properties. However, if concentrated masses are present, a lumped substructure mass matrix is found to provide sufficiently accurate results. This situation is investigated for a 5-storey shear wall where floor mass intensity is varied. Full stress recovery within the substructure (for static loading) is also presented.

The parametric investigation of a 12-storey precast panel structure with discrete connectors along the vertical joints presents an additional and detailed application. Response spectrum analyses for seismic loading, following a parametric scheme, are carried out. Uniform as well as changing stiffness properties along the height are considered and response is evaluated primarily in terms of connector forces. Results which demonstrate the expected magnitudes of response as functions of connector properties, together with estimated ductility requirements, are summarized in Section 5.5. The necessity of a non linear analysis in further investigations as well as the required basic steps is outlined in Section 5.4.7.

For future studies it is recommended to include slip in connections as well as closing and opening of joints in the non linear joint behaviour. It is also suggested to extend the analysis to include different horizontal joint characteristics.

Other aspects concerning layout or geometry of the structure should be considered; namely, introduction of panel heights over two or more storeys, as well as variation of connector locations.

Whereas the present study is limited to a single

12-storey panel wall, future studies should include linked wall assemblies [70], where several panel walls are separated by openings and linked by beams or floor slabs. In addition, structural models should be extended to buildings of 20 and possibly more storeys.

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NOTATION

NOTATION

All symbols are defined in the text. In addition, those used most frequently are listed below, which may include alternative definitions for the same symbol. Notation which relates to input parameters for the computer program are defined in Appendix C.

{a}	eigenvector in Ritz solution
[C]	damping matrix
\bar{C}	reduced damping matrix
c_i	Ritz coordinates
D	top deflection (RSS)
D_M	top deflection (RSS) for monolithic shear wall
E	modulus of elasticity
F_a	connector axial force (RSS)
F_{ai}	connector axial force for i^{th} mode
F_v	connector shear force (RSS)
F_{vi}	connector shear force for i^{th} mode
F_R	maximum connector shear force (RSS) for rigid connectors
{g}	subvector in $\delta [K]$
{h}	subvector in $[M]$
[I]	unity matrix
i	index
j	index

$[K]$	stiffness matrix
$[K_{IJ}]$	submatrix of $[K]$
$[K_0]$	stiffness matrix at zero level
$[\bar{K}]$	reduced stiffness matrix
$[\bar{K}_{IJ}]$	submatrix of $[\bar{K}]$
$[K_i]$	i^{th} reduced stiffness matrix
$[K]_c$	connector stiffness matrix
k_{ij}	element of $[K]$
\bar{k}_{ij}	element of $[\bar{K}]$
k_a	connector axial stiffness
k_v	connector shear stiffness
$[\tilde{L}]$	Cholesky decomposition of $[K]$
$[M]$	mass matrix
$[M]$	diagonal mass matrix
$[M_{IJ}]$	submatrix of $[M]$
$[M_0]$	mass matrix at zero level
$[\bar{M}]$	reduced mass mass matrix
$[\bar{M}_{IJ}]$	submatrix of $[\bar{M}]$
$[\bar{M}_i]$	i^{th} reduced mass matrix
M	base overturning moment (RSS); subscript for monolithic shear wall
M_M	base overturning moment (RSS) for monolithic shear wall.
M_i	base overturning moment for i^{th} mode
m	stiffness multiple
m_{ij}	element in $[M]$

\bar{m}_{ij}	element in $[\bar{M}]$
n	order of unreduced system
OB	operation count in block elimination
OS	operation count in step elimination
$\{P\}$	force vector; elastic force response (RSS)
$\{P\}_i$	elastic force response for i th mode
$\{P\}_{ix}$	elastic force response for i th mode, lateral components
$\{P_0\}$	force vector at zero level
$\{\bar{P}\}$	reduced force vector
$\{\bar{P}\}_i$	i^{th} reduced force vector
P_i	element in $\{P\}$
\bar{P}_i	element in $\{\bar{P}\}$
q	order of reduced eigenvalue problem in Ritz analysis
$[R]$	used in block elimination $[\tilde{L}][R] = [K_{12}]$
RSS	root sum of squares of modal contributions
r	number of degrees of freedom to be eliminated
$[S]$	stress-displacement matrix
$[S_0]$	stress-displacement matrix at zero level
$[\bar{S}]_i$	i^{th} reduced stress-displacement matrix
S_a	acceleration response spectrum
S_d	displacement response spectrum
$[T]$	transformation matrix used for static condensation
$[T(\omega)]$	$[T]$ as function of ω
$[T']$	transformation matrix used for component mode synthesis

- $[T_I]$ submatrix of $[T']$
- $[T_i]$ transformation matrix derived from $[\bar{K}_i]$
- T natural period of vibration; indicates transpose of a matrix
- T_M natural period of vibration for monolithic shear wall
- $\{u\}$ displacement vector; displacement response (RSS)
- $\{u\}_i$ displacement vector for i^{th} mode
- $\{v\}$ solution shapes in Ritz analysis
- V base shear (RSS)
- V_M base shear (RSS) for monolithic shear wall
- V_i base shear for i^{th} mode
- $\{w\}$ subvector of $[S]$
- $\{X\}$ displacement vector
- $\{X_I\}$ subvector of $\{X\}$
- $\{\bar{X}\}$ reduced displacement vector
- $\{\bar{X}_i\}$ i^{th} reduced displacement vector
- $[Y]$ used in block elimination $[\tilde{L}]^T[Y] = [R]$
- $\{y\}$ displacement vector for reduced equation of motion
- z used in operation count $z = r/n$
- γ modal participation factor
- γ_M modal participation factor for monolithic shear wall
- γ_i modal participation factor for i^{th} mode

δ	relative vertical joint displacement (RSS)
δ_s	relative joint displacement (RSS) for single walls
ϵ	measure for accuracy of frequencies and mode shapes
η_i	vertical coordinate for stress point i
μ_v	force reduction factor for horizontal storey shear and base shear
μ_c	force reduction factor for connector shear force
ν	Poisson's ratio
ξ_i	horizontal coordinate for stress point i
ρ	Rayleigh's quotient; mass density
$\{\sigma\}$	stress vector
$\{\sigma_i\}$	stress vector for i^{th} level
$\sigma_x, \sigma_y, \sigma_{xy}$	stresses in plane stress analysis
$\{\phi\}$	mode shape vector
$\{\phi_I\}$	mode shape vector associated with internal degrees of freedom
$\{\phi\}_i$	i^{th} mode shape vector
$\{\psi\}_i$	i^{th} assumed Ritz shape
ω	natural frequency
ω_i	i^{th} natural frequency

APPENDIX A

APPENDIX A

OPERATION COUNT AND STORAGE REQUIREMENTS
FOR BLOCK AND STEP ELIMINATION

The conditions under which the operation count is performed are described in Section 3.2.3. Since symmetric matrices are involved, operations take place only in the upper triangle. Operation counts and storage requirements are presented separately for block and step elimination.

A1.1 BLOCK ELIMINATION

Block elimination consists of a series of matrix operations resulting in the following equations of Section 3.2.1:

$$[\bar{K}] = [K_{11}] - [R]^T[R] \quad (3.4)$$

$$[\bar{M}] = [M_{11}] - [M_{12}][Y] - [Y]^T([M_{21}] - [M_{22}][Y]) \quad (3.6)$$

The two equations, evaluated in Table A.1, on page 237 lead to a total number of operations given by

$$OB = \frac{3}{2} n^2 - nr^2 - \frac{1}{3} r^3 + \text{terms of lower order} \quad (3.11)$$

and is obtained from summation of column 3. Corresponding storage requirements listed in Column 4 are $(n^2 + r(n-r))$.

A1.2 STEP ELIMINATION

The equations for one step elimination, derived in Section 3.2.2, are:

$$[\bar{K}] = [K_{11}] - \frac{1}{k_{nn}} \{g\}\{g\}^T \quad (3.9)$$

and

$$[\bar{M}] = [M_{11}] - \frac{1}{k_{nn}} \{h\}\{g\}^T - \frac{1}{k_{nn}} \{g\}\{h\}^T + \frac{m_{nn}}{k_{nn}^2} \{g\}\{g\}^T \quad (3.10)$$

An operation count for stiffness matrix $[K]$ reduced during r steps is performed as follows:

Step	Operation Count
1 st	$\frac{(n-1)n}{2} \approx \frac{n^2}{2}$
2 nd	$\frac{(n-2)(n-1)}{2} \approx \frac{(n-1)^2}{2}$
⋮	⋮
r th	$\frac{(n-r)n-r+1}{2} \approx \frac{(n-r+1)^2}{2}$
TOTAL	$N = \frac{1}{2} \sum_{i=n-r+1}^n i^2$

or

$$N = \frac{1}{2} \sum_{i=1}^n i^2 - \frac{1}{2} \sum_{i=1}^{n-r} i^2 \quad (\text{A.1})$$

An evaluation of the series leads to

$$N = \frac{1}{12} (n(n+1)(2n+1)) - \frac{1}{12} (s(s+1)(2s+1)) \quad (\text{A.2})$$

with

$$s = n-r$$

and

$$N = \frac{1}{2} n^2 r - \frac{1}{2} nr^2 + \frac{1}{6} r^3 + \text{terms of lower order} \quad (\text{A.3})$$

It can be seen from Eq. (3.10) that the same number of operations is required three more times for the reduction of the mass matrix giving a total

$$OS = 4N$$

or

$$OS = 2n^2 r - 2nr^2 + \frac{2}{3} r^3 + \text{terms of lower order} \quad (\text{3.12})$$

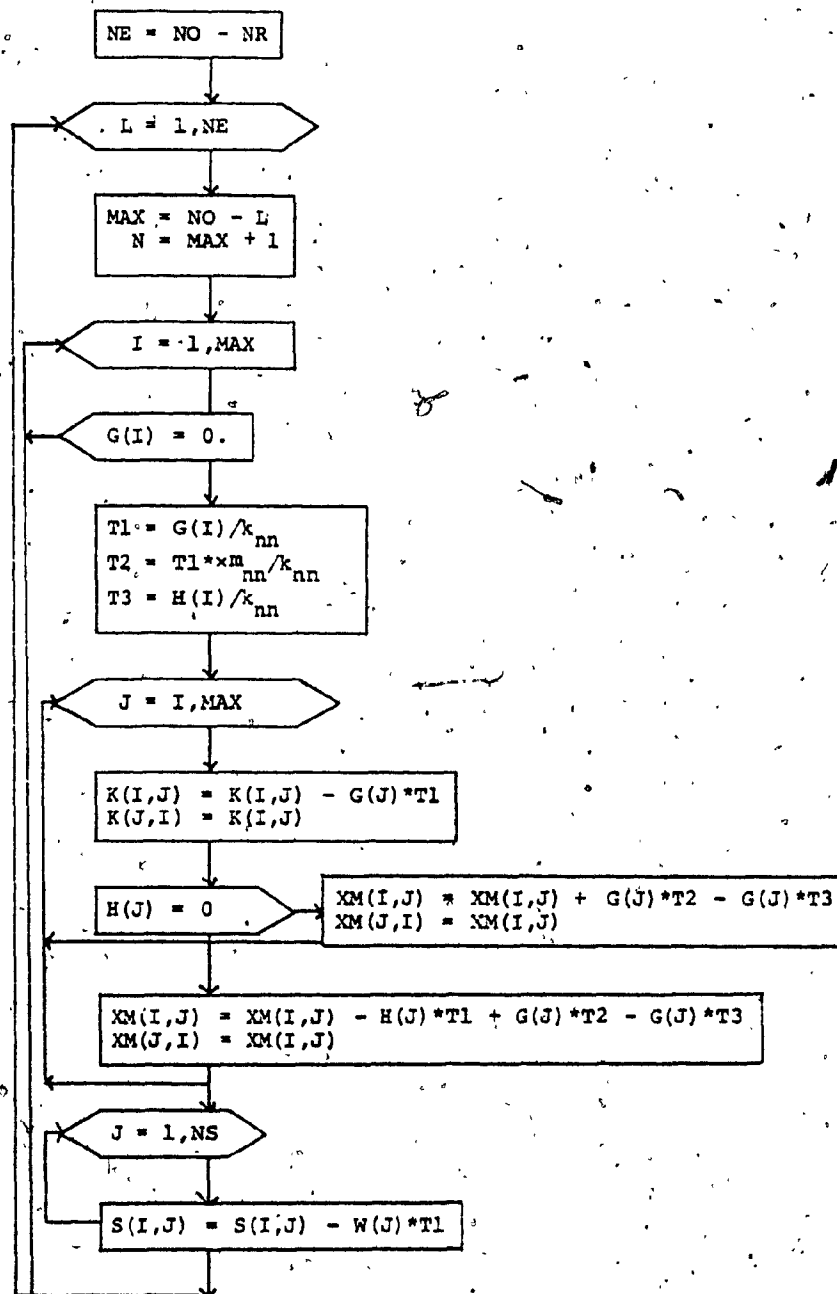
Considering storage requirements, it is obvious that only [K] and [M] need to be kept in core, leading to a total of approximately n^2 storage locations.

TABLE A.1 OPERATION COUNT AND STORAGE REQUIREMENT
FOR BLOCK ELIMINATIONS

Matrix Operation	Remarks	Operation Count	Storage Requirement
$[K_{22}] = [\tilde{L}][\tilde{L}]^T$		$\frac{r^3}{6}$	$[K], [\tilde{L}]$
$[\tilde{L}][R] = [K_{21}]$	FS* for $[R]$	$\frac{r(r-1)(n-r)}{2}$	$[K], [\tilde{L}]$
$[R]^T[R]$	Symmetric	$\frac{r(r-n)^2}{2}$	$[K]$
$[\tilde{L}]^T[Y] = [R]$	BS for $[Y]$	$\frac{r(r-1)(n-r)}{2}$	$[K]$
$[Z_1] = [M_{22}][Y]$		$r^2(n-r)$	$[K], [M], [Z_1]$
$[Z_2] = [M_{21}] - [Z_1]$			$[K], [M], [Z_2]$
$[Y^T][Z_2]$	Symmetric	$\frac{r(r-n)^2}{2}$	$[K], [M], [Z_2]$
$[M_{12}][Y]$	Symmetric	$\frac{r(n-r)^2}{2}$	$[K], [M]$
TOTAL		OB	$\underbrace{[K], [M], [Z_2]}_{n^2 + r(n-r)}$
<p>* FS Forward substitution BS Backward substitution</p>			

APPENDIX B

APPENDIX B

FLOWCHART FOR REDUCTION ALGORITHM USING STEP
ELIMINATION

B1.1 SYMBOLS FOR VARIABLES AND MATRICES

NO	Order of original matrices
NR	Order of reduced matrices
K()	Stiffness matrix
G()	Subvector in stiffness matrix
k_{nn}	Last diagonal element in stiffness matrix
XM()	Mass matrix
H()	Subvector in mass matrix
xm_{nn}	Last diagonal element in mass matrix
S()	Stress-displacement matrix
W()	Sub-vector in stress-displacement matrix

B1.2 STORAGE SCHEME FOR MATRICES

The algorithm requires that degrees of freedom to be eliminated are labelled with the highest numbers in the numbering sequence.

APPENDIX C

APPENDIX C

INSTRUCTIONS FOR INPUT AND DESCRIPTION OF
PROGRAM ARRANGEMENT TO CONFORM WITH SAP4

This Appendix contains a description as well as input instructions of additions to the structural analysis program SAP4 [24]. Some sections of this Appendix are to be inserted into the SAP4 manual at corresponding locations:

- (1) Section C1.1, "Panel Element Description" corresponds to Section 4.9 in the SAP4 manual.
- (2) Section C1.2, "Connector Element Description" corresponds to Section 4.10.
- (3) Section C1.3, "Extension of Response Spectrum Analysis" is attached to Section 7.3.
- (4) Section C2.1, "Input Instructions for Panel Element" is inserted as pages from IV.10.1 to IV.10.5.
- (5) Section C2.2, "Input Instructions for Connector Element" is inserted as pages from IV.11.1 to IV.11.2.
- (6) Section C2.3, "Additional Input Instructions for Response Spectrum Analysis" is to be inserted after item c. "spectrum data" on page VII.24.

Input examples as well as the overall program arrangement are presented in Sections C3.1 and C4.1, respectively.

C1.1 PANEL ELEMENT DESCRIPTION

A panel element is considered as a rectangular sub-structure with all interior nodes eliminated, leaving only boundary nodes.

The applied "four level" growth scheme demonstrated in Fig. 3.2 follows a single progression. Hence, the dimension in the horizontal direction (x-direction) remains constant, whereas expansion takes place in the vertical direction (in negative y-direction). The stepwise growth pattern can be visualized in three stages:

- (1) Creation of the basic 4-node element (zero level).
- (2) Creation of separate super element formations as shown in Fig. 3.2(a), namely end element, and level one and level two elements. These super elements can assume any of the formations demonstrated in Table 3.1 for single progression. Sequence of creation is indicated in Fig. 3.2(a) by the "growth direction".
- (3) Coupling of the created super elements which leads to level three (Fig. 3.2(b)), or to level four (Fig. 3.2(c)) in an additional symmetric coupling.

A "consistent lumped" element mass matrix is derived from the full consistent mass matrix by distributing the total panel mass according to diagonal terms of the consistent mass matrix. A stress recovery throughout the panel element is pos-

sible by specifying stress point locations in the zero level element (see Fig. 3.9).

C1.2 CONNECTOR ELEMENT DESCRIPTION

The connector element model can be described as an idealized two-node spring element with 4 degrees of freedom and zero dimensions in space, as shown in Fig. 5.2. The connector element stiffness matrix is then formulated by specifying connector axial and shear stiffnesses.

CL.3 EXTENSION OF RESPONSE SPECTRUM ANALYSIS

For an analysis of multistorey buildings, horizontal storey forces as well as base shears and base overturning moments, can be determined in an additional computation. Storey forces for the r th mode are herein expressed as

$$\{P\}_r = [M] \{\phi\}_r \gamma_r S_a$$

The mass assigned to an individual storey is summed from nodal masses of the corresponding nodal points prescribed during input phase. The modal base shear is computed from force vector $\{P\}_r$ as,

$$V_r = [1] \{P\}_{rx}$$

where $[1]$ is a row vector of ones and $\{P\}_{rx}$ contains the lateral components of $\{P\}_r$. The modal base overturning moment is derived accordingly:

$$M_r = [x] \{P\}_{rx}$$

where elements in row vector $[x]$ specify the height of storey masses above ground.

It is noted that the present program version is limited to two-dimensional cases, where x - and y -directions are defined as horizontal and vertical, respectively.

C2.1 INPUT INSTRUCTIONS FOR PANEL ELEMENT

TYPE 10 - PANEL ELEMENT

A rectangular homogeneous panel element with interior nodes eliminated and optional boundary node arrangement is created. The element is developed in a one-dimensional growth fashion during a four-level elimination procedure (Fig. 3.2).

Additional instructions for input which has to precede that for panel elements are as follows:

- (1) Coordinates of nodal points relating only to panel elements do not have to be specified (i.e., $x = y = z = 0.0$ for those points; see "NODAL POINT DATA" page III.1 in SAP4 manual). Boundary conditions of panel elements are defined in x-y plane.
- (2) If zero level elements (basic 4-node elements) are created by SAP4, the following considerations are necessary: (a) Additional, fully constrained nodes (in y-z plane) have to be introduced in order to specify dimensions of those elements ("NODAL POINT DATA", page III.1); (b) Zero level element data have to precede directly panel element data. For element type and sequence of creation see NOTE (3), page 251.

1. Control Card (10 15)

notes	columns	variable	entry
	5		Enter the number "9"
	6 - 10	NUME	Number of panel elements
	11 - 15	NUMAT	Number of different material property cards
(1)	16 - 20	IMAS	EQ. 2 for "consistent lumped" mass (EQ. 1 for consistent mass)
(2)	21 - 25	ND	Number of degrees of freedom per node
(3)	26 - 30	IS4	Number of different 4-node zero level elements created in SAP4

NOTES/

- (1) Only a lumped mass model is accepted in SAP4 during assembly and solution. For IMAS = 1, Subroutine PANEL creates and prints a consistent panel element mass matrix; then exit follows.
- (2) At present available for rectangular 4-node elements:
 - ND = 2 Plane stress elements are created by PANEL (IS4 = 0) or by SAP4 (IS4 ≠ 0).
 - ND = 3 Nonconforming plate bending elements are created by PANEL.
- (3) At present, element "TYPE 4" for plane stress or plane strain can be created by SAP4 and retrieved by PANEL. The sequence of creation has to follow the sequence of created panel elements. Each group of identical panel elements require either one (with B1) or two (with B1 and B2) start elements. Hence, one or two zero level elements are to be defined according to the creation sequence of start elements (for definition of B1 and B2, see Section 3.b, Card 1, page 253). See also NOTE (2) on page 250.

2. Material Property Cards
(I5,3F10.0,F15.0)

For each different set of properties one card is required; total = NUMAT.

notes	columns	variable	entry
	5	I	Number of property card
	6 - 15	E(I)	Young's modulus
	16 - 25	POI(I)	Poisson's ratio
	26 - 35	THI(I)	Panel thickness
	36 - 50	RO(I)	Mass density per unit volume

3. Panel Element Cards

a. nodal numbering

(I6I5,/,I0I5,/,I6I5,/,I0I5)

notes	columns	variable	entry
	5	M	Number of panel element
	6 - 10	NNOD	Number of nodal points
	11 - 15	IE(1)	Nodal numbers, arranged
	16 - 20	IE(2)	counterclockwise (see
	⋮	⋮	Figs. 3.4 through 3.7)
	⋮	IE(NNOD)	
(1)	cont'd in (15)	IGEN	Total number of identical panels; for nodal generation

NOTES/

- (1) If IGEN ≠ 0, the next nodal numbering card is expected to give M, NNOD, with array IE(1), ..., IE(NNOD) expressing the incremental differences of nodal points of two consecutive panel elements. The remaining (IGEN-2) nodal numbering cards need only M for input. For an identical panel element which is off sequence in the nodal generation, the nodal numbering has to be specified after which a new generation may follow, for which no further properties need be specified.

b. geometric properties

Card 1 (I2,I3,3I5,4F10.0,I5)

notes	columns	variable	entry
	2	IRPT	EQ.1; present panel element is identical to the previous one, no further input is necessary for this panel element EQ.2; same as above, in addition no stresses are computed EQ.0; default, normal continuation
	5	MATTYP	Material property card
	6 - 10	NON	Number of horizontal nodes
	11 - 15	NLV	Number of level one and level two elements
	16 - 20	NCOP4	Type of level four coupling (see Fig. 3.6); EQ.0. skips level four
	21 - 30	XL	Length of panel element in x-direction
	31 - 40	YL	Length of panel element in y-direction
	41 - 50	B1	Width of first start element
	51 - 60	B2	Width of second start element; if not required, B2 = 0
	61 - 65	NSPT	Number of stress points in basic 4-node element (at present only for static analysis)

Card 2 (16I5)

notes	columns	variable	entry
	5	LEV(1)	Type identifications for level one and level two elements, see Tables 3.2 and 3.3; total = NLV
	6 - 10	LEV(2)	
	:	:	
	:	LEV(NLV)	

Card 3 (16I5)

notes	columns	variable	entry
	5	NCOPL(1)	Types of coupling (level 3), see Figs. 3.4 and 3.5. If no coupling required, skip this card
	6 - 10	NCOPL(2)	
	:	:	
	:	NCOPL(NLV+1)	

c. location of stress points in basic
4-node element (I5,2F10.0)

One card is necessary for each stress point, total = NSPT. If NSPT = 0, skip this card.

notes	columns	variables	entry
	5	NPT	Number of stress point
6 - 15		XSI(NPT)	} Natural coordinates for stress points, see Fig. 3.9
16 - 25		ETA(NPT)	

d. printing options

If not printing is required, leave one blank card.

notes	columns	variables	entry
	5	IPLOC	EQ.1; prints starting locations of dynamic arrays; otherwise EQ.0
6 - 10		IPLSTOP	Same as IPLOC, but exit follows
11 - 15		IPMAT	EQ.1; prints stiffness and mass matrix

C2.2 INPUT INSTRUCTIONS FOR CONNECTOR ELEMENT

TYPE 11 - CONNECTOR ELEMENT

A two-node connector or link element with two degrees of freedom per node - representing axial and shear deformations - is created.

1. Control Card (3I5)

notes	columns	variables	entry
	4 - 5		Enter number "10"
	6 - 10	NUME	Number of connector-elements
	11 - 15	NUMAT	Number of different material properties

2. Material Property Cards (I5,2F10.0)

For each different set of properties one card is required.

notes	columns	variables	entry
	5	MTY	Number of property card
	6 - 15	SA(MTY)	Axial stiffness
	16 - 25	SS(MTY)	Shear stiffness

3. Numbering and Orientation Card (7I5)

notes	columns	variables	entry
	5	MI	Number of element, not necessarily in numbering order
	6 - 10	NI	Node I
	11 - 15	NJ	Node J
	16 - 20	MTY	Number of property card
	21 - 25	IVH	Orientation IVH.EQ.1 for vertical joint IVH.EQ.2 for horizontal joint
	26 - 30	NS	Force components to be calculated NS.EQ.0 if no forces are to be calculated NS.EQ.2, axial and shear forces are calculated

(continued)

notes	columns	variables	entry
(1)	31 - 35	IGEN	Node generation IGEN.EQ.0, no generation, skip card number 4

4. Numbering Card for Node Generation (3I5)

notes	columns	variables	entry
	5	M11	Last element in generation series
	6 - 10	NI1	Node NI of element M11
	11 - 15	NJ1	Node NJ of element M11

NOTES/

- (1) If IGEN-NE.0 a numbering series for the next elements $(MI + IGEN)$, $(MI + 2 \times IGEN)$ etc. is generated. Card No. 4 is necessary to indicate the last element of the series. The value $(M11 - MI)$ has to be a multiple of IGEN. Nodal points NI and NJ are spaced at equal intervals. For each new series to be created a new pair of cards 3 and 4 is necessary.

C2.3 ADDITIONAL INPUT INSTRUCTIONS FOR RESPONSE SPECTRUM ANALYSIS

These data which allow the computation of storey shears, base shear and overturning moment for multistorey structures are to be inserted after item c. "spectrum data" on page VII.24.

d. number of storeys and storey height
(15, F10.0)

notes	columns	variable	entry
	1 - 5	NSY	Total number of storeys
	6 - 15	SHT	Storey height

3. storey mass information (2I5)

For each storey one card is required (total = NSY), starting with the top storey.

notes	columns	variable	entry
(1)	1 - 5	ISY	Number of storey, in descending sequence from top (i.e., for top storey ISY = NSY)
	6 - 10	NJT	Number of nodal points assigned to the current storey. This includes nodes between floor levels. No nodes can be left out.

NOTES/

- (1) The computational scheme of this Section requires that the global nodal numbering of the structure is in increasing order from bottom to top.

C3.1 INPUT EXAMPLES

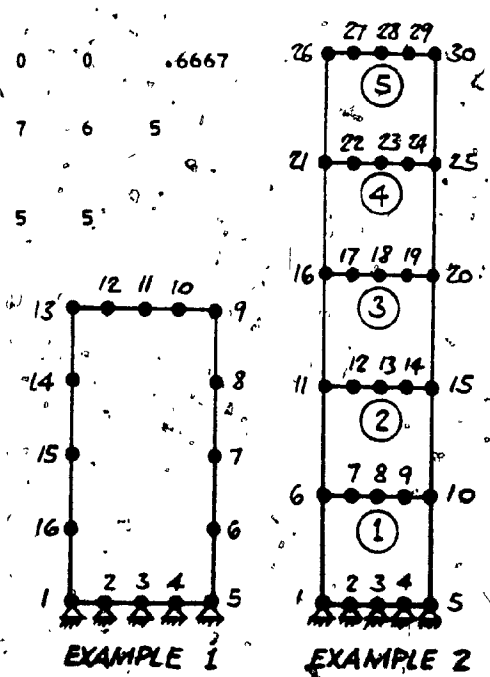
INPUT EXAMPLE 1 - PANEL CANTILEVER (FIG.4.1A)

16	1	0	5	1													
1	1	1	1	1	1	1	1	0	0	0	0	0	0	0	0	0	1
5	1	1	1	1	1	1	1	0	0	0	0	0	0	0	0	0	1
6	1	1	0	0	0	1	0	0	0	0	0	0	0	0	0	0	1
16	1	1	0	0	0	1	0	0	0	0	0	0	0	0	0	0	1
9	1	1	2	3													
1	30000.00		.30	.10	.000000733028												
1	16	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16
1	5	2	2	1.0000	2.0000	.2500											
110	110																
2																	
1	0	1															

INPUT EXAMPLE 2 - SHEAR WALL WITHOUT FLOOR MASSES (FIG.4.4D)

34	2	12	1	0													
1	1	1	1	1	1	1	0	0	0	0	0	0	0	0	0	0	1
2	1	1	1	1	1	1	0	0	0	0	0	0	0	0	0	0	1
5	1	1	1	1	1	1	0	0	0	0	0	0	0	0	0	0	1
6	0	0	1	1	1	1	0	0	0	0	0	0	0	0	0	0	1
30	0	0	1	1	1	1	0	0	0	0	0	0	0	0	0	0	1
31	1	1	1	1	1	1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
32	1	1	1	1	1	1	0.0	2.5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
33	1	1	1	1	1	1	0.0	2.5	2.5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
34	1	1	1	1	1	1	0.0	0.0	0.0	2.5	2.5	0.0	0.0	0.0	0.0	0.0	
4	1	1	2	00													
1			.00481778														
	576000.00	576000.00	576000.00	.20	.20	.20	240000.00										

0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
1	31	32	33	34	1	0	0	0	0	0	0	.6667					
9	5	1	2	2	1	0.6667	.00481778										
1	10	1	2	3	4	5	10	9	8	7	6	5					
11	5	1	0	10.000	10.000	2.50											
120	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
2	10	5	5	5	5	5	5	5	5	5	5	5	5	5	5	5	5
1																	
3																	
1																	
4																	
5																	
1																	
0	1																
0																	
0																	
0																	



C4.1 OVERALL SCHEME OF CHANGES ON SAP4

Alterations to SAP4 which are described in this Appendix consist of (i) exchange of Subroutines, (ii) insertion of new Subroutines and (iii) modification on existing Subroutines. The following files were created for this purpose:

- (1) ELM9, ELN9 and PANEL, relating to the panel element (where ELM9 and ELN9 replace ELMT9 and ELT9 in SAP4),
- (2) ELM10, ELN10 and CONN, relating to the connector element (where ELM10 and ELN10 replace ELMT10 and ELT10 in SAP4),
- (3) FORCES, relating to additions in response spectrum analysis, and
- (4) MOS to perform modifications on existing SAP4 Subroutines.

The file manipulations necessary on SAP4 are indicated by the scheme on the two following pages. Listings of the files ELM9, ELN9 (containing the new files of ELMT9 and ELT9) as well as PANEL are given in Appendix D.

File Manipulations on SAP4

REC	NAME	REC	NAME
1	OVLZER	56	QDCOS
2	SABA C	57	TDCOS
3	INPUTJ	58	TRFPRD
4	ELTYPE	59	SLST
5	TRUSS	60	SLCCT
6	BEAM	61	LSTSTR
7	PLANE	62	LCTMOM
8	THREED	63	OVL6
9	SHELL	64	ELT7
10	BOUND	65	CLAMP
11	SOL21	66	OVL7
12	ELM19 ELM9 R	67	ELT8
13	ELM10 ELM10 R	68	THOFE
14	ELM11	69	INP21
15	PIPE	70	CROSS2
16	INL	71	VECTR2
17	ERROR	72	SSLAW
18	ADDSTF	73	STR21
19	PRINTD	74	FACEPR
20	STRESS	75	DER30S
21	CALBAN	76	FNCT
22	STRSC	77	OVL8
23	PRIST	78	ELT9 ELM9 R
24	STJ016	79	OVL9
25	OVL1	80	ELT10 ELM10 R
26	ELT1	81	OVL10
27	RUSS	82	ELT11
28	OVL2	83	OVL11
29	ELT2	84	ELT12
30	TEAM	85	PIPEK
31	NEW8M	86	PIPEK2
32	SLAVE	87	PIPEK3
33	OVL3	88	SELECT
34	ELT3A4	89	TANGBC
35	PLNAX	90	BENDDC
36	ELAW	91	TANGKS
37	POSINV	92	PINVER
38	QUAD	93	BENDKS
39	FORMB	94	OVL12
40	VECTOR	95	SOLE0
41	CROSS	96	SESOL
42	DOT	97	OVL13
43	OVL4	98	SOLE10
44	ELT5	99	MODES
45	BRICK8	100	SLOCK
46	DERIV	101	SECNTD
47	LOAD	102	BANDET
48	LOSTR	103	MULT
49	OVL5	104	SSPCB
50	ELT6	105	DECOMP
51	TPLATE	106	INVECT
52	STRETR	107	REDBAK
53	CSTSTR	108	EIGSOL
54	LCT9ST	109	JACOBI
55	QTSHEL	110	SHECK

PANEL #
Assoc. S/Rs A
CONN A

File Manipulations on SAP4 (continued)

REC	NAME	
111	OVL14	
112	DYNRES	
113	HISTRY	
114	EMID	
115	LOAD1	
116	GMTN	
117	LOAD2	
118	RESPON	
119	DISPLR	
120	DISPLY	
121	ELOUTH	
122	PLOT	
123	STRSD1	
124	OVL15	
125	DYNSPE	
126	RESPEC	M
127	EMIDR	
128	SPECTR	M
129	SD	
130	STRESR	M
131	ELOUTH	FORCES A
132	OVL16	
133	SBSOL	
134	STEP	
135	ADDMAS	
136	PLOAD	
137	EMIOS	
138	GROUND	
139	INDLY	
140	INTHIS	
141	LOADV	
142	INOUT	
143	SOLSTP	
144	TRIFAC	
145	REDVK	
146	SDSPY	
147	SPLY	
148	FLOUTS	
149	OPL	
150	• EOF •	

Symbols:

- C ... Changes as indicated in Appendix D on page
R ... Replace indicated SAP4 Subroutines
A ... Insert indicated Subroutines
M ... Modifications as given in file MOS

APPENDIX D
FORTRAN LISTING OF SUBROUTINE PANEL

APPENDIX D

FORTRAN LISTING OF SUBROUTINE PANEL

The following Subroutine is an extension of the element library of the multipurpose program SAP4. A "panel element" is created applying the scheme as outlined in Chapter III.

Most of the important variables used are defined in Appendix C. For matrices and arrays the following symbols are employed:

S ()	stiffness matrices
B ()	mass matrices
F ()	stress-displacement matrices
LM ()	connectivity arrays

Subroutines to create element stiffness, mass and stress-displacement matrices on zero level are not included in this Listing (see SUBROUTINE ESTIF, ELMA and ESTRES).

The Overlay containing SUBROUTINE PANEL as well as SUBROUTINE ELM9 (on page 265) are to be inserted at indicated locations in SAP4. In addition, the following changes in the main program SAP4 are necessary:

- (1) Addition of Tape 12 and Tape 13
- (2) Extension of Common Block
/ELPAR/ by integer MTOTEM
- (3) Specify, MTOTEM = 4045.

```

OVERLAY (SSAP,10,0)
PROGRAM ELI9
VARIABLE MODE SUPER FINITE ELEMENT OVERLAY
CUMRDN A(1)
CUMRDN 7EH/AA(1)
COMMON /ELPAR/ NPAR(14),VUMNP,MBAND,NELTYP,N1,NZ,
1 N3,N4,N5,NTOT,NEO,MTOTEM
CUMRDN /JUNK/ DUM(100),N6,N7,N8,N9,N10
1 CALL PANEL(AA(N1),AA(N2),AA(N3),AA(N4),AA(N5),AA(N6),
1 AA(N7),AA(N8),AA(N9),NUMNP)
RETURN
END

SUBROUTINE PANEL(IU,X,Y,Z,T,E,PUI,MT,THL,NUMNP)
1
2 C SUBROUTINE TO CREATE A RECTANGULAR SUPER FINITE
3 C ELEMENT WITH VARIABLE BOUNDARY MODE ARRANGEMENT
4 C THIS SUBROUTINE IS COMPATIBLE WITH "S.A.P.4"
5 C
6 C PROGRAMMED BY H.P. HOUTTELMAIER
7 C CONCORDIA UNIV MONTREAL 1978
8 C
9 C DIMENSION IO(NUMNP,1),X(1),Y(1),Z(1),T(1),E(1),PUI(1)
10 C DIMENSION W(1),THI(1),LN(400),LEI(30),LEZ(30)
11 C DIMENSION XS(12,12),XP(12,4),XL(12),XSS(78)
12 C COMMON /ELPAR/ NPAR(14),VUMNP,MBAND,NELTYP,N1,NZ,N3,
13 N4,N5,NTOT,NEO,MTOTEM
14 C COMMON /EXTRA/ MDEX
15 C COMMON /JUNK/ LEVI(15),NCOPL(15),NLEVI(15),LE(30),
16 N4,N5,NTOT,NEO,MTOTEM
17 C XSI(10),ETA(10)
18 C EQUIVALENCE(LM13),XS(1,1)),(LM(159),XP(1,1))
19 C EQUIVALENCE(LM207),AM(1)),(LM220),XSS(1))
20 C NTAP=12
21 C NUME=MPAR(2)
22 C NJMAT=MPAR(3)
23 C NU=MPAR(5)
24 C NJMB=0
25 C NCOUNT=-10
26 C WRITE(6,1006) NUME,NUMAT,MPAR(4),ND,IS4
27 C IF(ND.EQ.3) GO TO 140
28 C IF(NU.EQ.1) GO TO 140
29 C IF(MJMB.EQ.1) GO TO 140
30 C IF(LE(154.EQ.0) GO TO 140
31 C IF(LE(154.EQ.0) GO TO 140
32 C
33 C GET BASIC ELEMENT FROM TAPE 7 (ELEMENT LIBRARY OF SAP4)
34 C AND WRITE ON TAPE 13
35 C
36 C REWIND 2
37 C NL=IND 13
38 C GO 9 IN=1,154
39 C READ(2) LRA,NDA,(LN(N1),I=1,NDX),C(XS(I,J),J=1,NDX),
1 I=1,NDX), (EXP(I,J),J=1,NDX),J=1,4), (LM(I),I=1,NDX)
REARRANGE PLANE STRESS ELEMENT (LL14)
11=0
UU 3 I=1,7,2
11=1,1
LM(1)=11
3 LM(1)=11+1
UU 7 I=1,NDX
11=LM(I)
JJ=LM(J)
IF(LE(LJ,J)) GO TO 102
11=LM(J)
JJ=LM(J)
102 CONTINUE
11=LM(I)
JJ=LM(I)
11=(J-1)*J/2+1
11=J-1
LRA=ENDX+11*NDX/2
WRITE(13) LRA,(XSS(I),I=1,LRA)
6 CONTINUE
REWIND 13
140 CONTINUE
UU 10 I=1,NJMAT
READ(5,1001) NI,6(N1),PUL(N1),THI(N1),WIC(N1)
WRITE(6,1007)
10 WRITE(6,1007) NI,PUL(N1),THI(N1),WIC(N1)
130 REWIND REAP
1MAS=MPAR(4)
NUMB=NUMB+1
READ(5,1002) M,NNDU,(LE(I),I=1,NNDU),LGEH
CHECK IF NODE GENERATION REQUIRED
IF(IGEN.NE.0) GO TO 50
GU TO 52
50 NCOUNT=NUMB+1*LEN-1
NNDU1=NNDU
NUMB1=NUMB+1
UU 31 I=1,NNDU1
11(1)=11(1)
GU TO 70
52 CONTINUE
IF(NU.EQ.1) GO TO 55
IF(NU.EQ.1) GO TO 60
55 CONTINUE
NNDU=NNDU1
GU TO 70
60 CONTINUE
UU 32 I=1,NNDU1
11(1)=11(1)
11(1)=11(1)+1E2(1)
32 11(1)=11(1)
GU TO 70
60 CONTINUE
UU 34 I=1,NNDU1
11(1)=11(1)+1E2(1)
11(1)=11(1)
34 CONTINUE

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PANE 100 IF (HUMB.E.V.NCOUNT) NCOUNT=0
PANE 101 70 CONTINUE
PANE 102 HEAD(5,1005) MATTP,MHM,NLVI,NCUP4,XL,YL,XY1,XY2,NSPI
PANE 103 IF (MATTP.EQ.2000) NS=0
PANE 104 IF (MATTP.GE.1000) GO TO 125
PANE 105 WRITE(6,1008)
PANE 106 WRIT(6,1003) MATTP,MHM,NLVI,NCUP4,XL,YL,XY1,XY2,NSPI
PANE 107 NCUP,NLVI-1
PANE 108 READ(7,1002) (LEVI(I),I=1,NLVI)
PANE 109 WRITE(6,1009)
PANE 110 WRITE(6,1002) (LEVI(I),I=1,NLVI)
PANE 111 IF (NCUP.NE.0) HEAD(5,1002) (NCUP(I),I=1,NCUP)
PANE 112 WRIT(6,1010)
PANE 113 IF (NCUP.NE.0) WRIT(6,1002) NCUP, (NCUP(I),I=1,NCUP)
PANE 114 IF (IND.EQ.2) NS=3*NSPI
PANE 115 IF (IND.EQ.3) NS=0
PANE 116 IF (NS.EQ.0) GO TO 85
PANE 117 C
PANE 118 C NUMBER OF STRESS COMPONENTS AND STRESS POINT LOCATIONS
PANE 119 C
PANE 120 WRITE(6,1012)
PANE 121 DO 3 I=1,NSPT
PANE 122 READ(7,1001) NPNT,AS(1:NPNT),ETA(NPNT)
PANE 123 WRITE(6,1011) NPNT,AS(1:NPNT),ETA(NPNT)
PANE 124 85 CONTINUE
PANE 125 C
PANE 126 C INSTRUCTIONS FOR PRINTING
PANE 127 C
PANE 128 READ(5,1002) IPLOC,IPLOCSTOP,IPMAT,NC,ISH
PANE 129 WRITE(6,1013)
PANE 130 WRITE(6,1002) IPLOC,IPLOCSTOP,IPMAT,NC,ISH
PANE 131 WRITE(6,1014)
PANE 132 C
PANE 133 C CALCULATE STORAGE ALLOCATIONS FOR DYNAMIC ARRAYS
PANE 134 C
PANE 135 CALL ALLOC(IPLOC,IPLOCSTOP,ND,MHM,NLVI,
PANE 136 1 LEVI,NLEVI,NCUP,NCUP4,
PANE 137 1 LU,L1,L2,L3,L4,L5,L10,L20,L100,L200,
PANE 138 KU,K1,K2,K3,K4,K5,K10,K20,K100,K200,
PANE 139 3 MD,M1,M2,M3,M4,M5,M10,M20,M100,M200,MS,
PANE 140 4 MEND,HALF,HALES,MHM)
PANE 141 C
PANE 142 C CHECK STORAGE LENGTH OF COMMON AT(MTUT) , AND OF
PANE 143 C COMMON /ER/ QUOT(UTER) IN ADJUST
PANE 144 C
PANE 145 MUF=NR00*ND
PANE 146 MUK=MUF*(INDF+1)/2
PANE 147 LMO1=2+6*MUF*MUK
PANE 148 IF (LUTUTEN.GE.LRUI) 30 TO 87
PANE 149 MXX=LMO1-MUTOTEN
PANE 150 WRITE (6,918) MXX,LRUI
PANE 151 STOP
PANE 152 87 CONTINUE
PANE 153 IF (MHM.GT.MTUT) GO TO 90
PANE 154 GO TO 95
PANE 155 90 CONTINUE
PANE 156 MHI=MHM-MTUT
PANE 157 WRITE(6,915) MHI
PANE 158 STOP
PANE 159 95 CONTINUE
PANE 160 DO 11 I=1,MIUT
PANE 161 11 A(I)=0.0
PANE 162 DO 12 I=1,600
PANE 163 12 ZMI(I)=0
PANE 164 AX=AL/INIM-1
PANE 165 IF (MODEX.EQ.1) GO TO 125
PANE 166 C
PANE 167 C CKAIFE ELEMENT STIFFNESS MASS AND STRESS-DISPLACEMENT
PANE 168 C MATRICES (ZERO LEVEL)
PANE 169 C
PANE 170 MP=MATTP
PANE 171 C
PANE 172 CALL LVL(IND,AV,BY1,BY2,ALU),ALLI,AI(KU),A(K1),
PANE 173 1 A(KU),A(K1),A(K2),A(K3),A(K4),
PANE 174 1 TH(IMP),E(IMP),PUI(IMP),A(I(MP)),IS4)
PANE 175 C
PANE 176 C CREATE 1ST LEVEL ELEMENTS, I.E., START AND END ELEMENT
PANE 177 C (END ELEMENT IS STORED ON TAPE NTAP)
PANE 178 C
PANE 179 CALL LVL(NTAP,ND,NS,MHM,LM(1),LM(101),LEVI,BY2,
PANE 180 1 A(LD),A(L1),A(L2),A(L3),A(L5),
PANE 181 1 A(KU),A(K1),A(K2),A(K3),A(K4),
PANE 182 1 A(MD),A(M1),A(M2),A(M3),A(M5))
PANE 183 C
PANE 184 C CHECK IF OTHER LEVELS ARE REQ'D
PANE 185 C
PANE 186 4F(NLVI.GT.1) GO TO 96
PANE 187 IF (LEVI(1).LT.100) 30 TO 97
PANE 188 NSTEP=LLVI(1)-100
PANE 189 IF (NSTEP.EQ.0,AND,NCUP4.EQ.0) GO TO 98
PANE 190 GO TO 96
PANE 191 97 IF (NCUP4.EQ.0) GO TO 99
PANE 192 GO TO 96
PANE 193 98 N=2*NNH*ND
PANE 194 LN=L2
PANE 195 KR=K2
PANE 196 GO TO 105
PANE 197 99 N=(NEND+NNH)*ND
PANE 198 LN=L5
PANE 199 K4=K5
PANE 200 GO TO 105
PANE 201 96 CONTINUE
PANE 202 C
PANE 203 C CREATE 2ND LEVEL ELEMENTS USING START ELEMENTS
PANE 204 C
PANE 205 CALL LVL2(NTAP,ND,NS,MHM,NLVI,LEVI,NCUP,NE=2,NSO2,
PANE 206 1 LM(1),LM(101),LM(201),A(L2),A(L3),A(L4),A(L5),
PANE 207 A(K2),A(K3),A(K4),A(K5),
PANE 208 2 A(M2),A(M3),A(M4),A(M5))
PANE 209 M=NE*2
PANE 210 NSO=NSO2
PANE 211 LN=L5
PANE 212 K4=K5
PANE 213 MH=M5
PANE 214 IF (NCUP4.EQ.0) GO TO 130
PANE 215 C
PANE 216 C CREATE 3RD LEVEL ELEMENT USING 1ST AND 2ND LEVEL ELMTS.
PANE 217 C
PANE 218 CALL LVL3(NTAP,ND,NS,MHM,LEVI,NCUP,NCUP4,NSO3,LM(1),
PANE 219 1 LM(101),LM(201),A(L1),A(KU),A(K1),A(K10),A(M1),A(M10))

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PANE 220 M=HALF*ND
PANE 221 NSU=NSU3
PANE 222 MN=ND
PANE 223 LN=1
PANE 224 KN=KO
PANE 225 100 CONTINUE
PANE 226 IF(NCOP4.EQ.0) GO TO 105
PANE 227 C
PANE 228 C LEVEL4=CAPLATES A SYMMETRIC SUPER ELEMENT
PANE 229 C USING THE 3RD LEVEL ELEMENT
PANE 230 C
PANE 231 CALL LVL4(ENTAP,ND,NS,NIN,MHALF,MHALF,MEND,NCOP,NCOP4,
PANE 232 1 MHA,NSU4,LM(1),LM(10),LM(20),A(L0),A(L100),
PANE 233 2 A(K0),A(K100),A(MU),A(M100))
PANE 234 N=NL44
PANE 235 NSU=NSU4
PANE 236 MN=M100
PANE 237 LN=L100
PANE 238 KN=K100
PANE 239 105 CONTINUE
PANE 240 C
PANE 241 C REARRANGE MATRICES (FROM MODE 1 TO MODE 2 )
PANE 242 C
PANE 243 CALL STORE2(ENTAP,N,200,LM,A(LN))
PANE 244 IF(ENTAP.EQ.1) CALL WRITE2(NG,ISH,M,A(LN))
PANE 245 C
PANE 246 C CHECK IF MASS MATRIX LUMPED OR CONSISTENT
PANE 247 C
PANE 248 IF(MAS.NE.1) GO TO 110
PANE 249 CALL STORE2(ENTAP,N,200,LM,A(KN))
PANE 250 IF(ENTAP.EQ.1) CALL WRITE2(NG,ISH,M,A(KN))
PANE 251 RETURN
PANE 252 110 CALL LUMPDI(MAS,ND,M,AL,FL,TH(IMP),d(TMP),A(KN),LN)
PANE 253 KAN=KAN-1
PANE 254 IF(ENTAP.EQ.1) WRITE(6,1004) (A(I),I=1,KN,KAN)
PANE 255 WRITE(6,1015)
PANE 256 120 CONTINUE
PANE 257 125 CONTINUE
PANE 258 WRITE(6,1002) M,NNDD,(IE(1),I=1,NNDD),IGEN
PANE 259 I(ND,EQ.3) IIA=2
PANE 260 I(ND,EQ.2) IIA=0
PANE 261 DU 30 I=1,NNDD
PANE 262 IEL=IE(I)
PANE 263 DU 30 L=1,NO
PANE 264 IJ=(I-1)*ND+L
PANE 265 I2=IIA*L
PANE 266 30 IEL(I)=D(I,I)=I2)
PANE 267 NON=NON+ND
PANE 268 IF(MON.NE.N) WRITE(6,116) NON,M
PANE 269 IF(MON.NE.N.AND.MODEX.PG.0) STOP
PANE 270 I(MODLA.EQ.1) GO TO 130
PANE 271 C
PANE 272 C CALCULATE BANDWIDTH AND WRITE STIFFNESS AND MASS
PANE 273 C MATRIX ON TAPE 2
PANE 274 C
PANE 275 CALL CALPAN(BANDU,KN,A(KN),A(LN),ND)
PANE 276 NSTU=NON+NSU
PANE 277 WRITE(1) M,ND,NS,NJ,ISTU
PANE 278 I(INS.EQ.0) GO TO 136
PANE 279 K1=M1

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MN2=RN+NSTU-1
WRITE(1) (LM(I),I=1,ND),I(A(1),I=1,MN1,MN2)
IF(ENTAP.NE.1) GO TO 130
IF(MATTYPE.EQ.1000) GO TO 136
WRITE(6,920)
WRITE(6,1002) NON,NSU
DU 42 L=1,ND
11=NR*(L-1)*NSU
12=NR*LN*NSU-1
WRITE(6,1004) (A(I),I=1,12)
WRITE(6,922)
PANE 294 42 CONTINUE
PANE 295 136 CONTINUE
PANE 296 IF(NUMB.NE.MUNE) GO TO 340
PANE 297 C
PANE 298 1003 FURMAT(15,3F10.0,F15.0)
PANE 299 1005 FURMAT(415,4F10.0,F415)
PANE 300 1006 FURMAT(23H NUMBER OF ELEMENTS // 16 /
PANE 301 23H MASS FURMAT(100 // 16 /
PANE 302 23H EG,1, CONSISTENT // 16 /
PANE 303 23H EG,2, LUMPED // 16 /
PANE 304 23H O.D.F. PER NODE // 16 /
PANE 305 23H NUMBER OF BASIC ELEMENTS // 16 /
PANE 306 23H CREATED BY SAP4 // 16 /
PANE 307 1007 FURMAT(36H R A T E R I A L - P R O P L I E S //
PANE 308 1 2X,3H NM,6X,6H FIM),5X,
PANE 309 7H NUC(N),5X,6H TH(IN),4X,6H MASS(MN),71
PANE 310 32H G E U M - P M U P E R T I E S //
PANE 311 1008 FURMAT(11,32H G E U M - P M U P E R T I E S //
PANE 312 1 56H HTP,5H NM,5H NLV,5H NCDP,4H BY,3X,5H MSP),71
PANE 313 7X,3H YL,7X,4H YI,6,6,4H BY,3X,5H MSP),71
PANE 314 1009 FURMAT(11,18H LEVEL,1,1,1,1,1,1)
PANE 315 1010 FURMAT(11,26H NCDP NCDP ((I),I=1,NCUP /)
PANE 316 1011 FURMAT(11,4E13.6)
PANE 317 1012.FURMAT(11,14H STRESSPOINT,1,1,1,1,1,1)
PANE 318 1013 FURMAT(11,11,37H P R I N T - I N S T R U C T I O N S //
PANE 319 1 10H ASI(INSP),4X,91 ETAINSP),71
PANE 320 1014 FURMAT(11,1,28H CONTROL-INFORMATIONS START,1,1)
PANE 321 1 6H IPLOC,6H JSTUP,6H IPMAT,71
PANE 322 1015 FURMAT(26H CONTROL-INFORMATIONS END,1,1,1,1,1,1)
PANE 323 1 19H ELEMENT-RUMBLEING,71
PANE 324 2 5H M,66H NNDD,24H ((L(I),I=1,NNDD)),36N,71
PANE 325 1002 IUPMAT(261,1,1,2015)
PANE 326 1003 FURMAT(415,4F10.0,F415)
PANE 327 1004 FURMAT(11,12E11.4)
PANE 328 915 FURMAT(11,27H INCREASE (PAGE (HIT)) BY 16)
PANE 329 916 FURMAT(11,10H CHECK HOLE,1,1,1,1,1,1)
PANE 330 91E FURMAT(20H INCREASE MINUM BY 16, 10H ...LEFL=16)
PANE 331 920 FURMAT(11,11)
PANE 332 922 FURMAT(11)
PANE 333 RETURN
PANE 334 END

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ALLO 1 SUBROUTINE ALLU(L10,LL5,ML,NHR,MLV1,
ALLO 2 1 LVL,NLVL1,MLUP,NCUPL,1,0P4,

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1  LTHALF1,LT2,LT3,LT4,LT5,LT10,LT20,LT100,LT200,
2  K10,K11,K12,K13,K14,K15,K10,K120,K1100,K1200,
3  M10,M11,M12,M13,M14,M15,M10,M120,M1100,M1200,
4  NS,NEHD,NHALF,NHALF5,NHM)
5  DIMENSION L1(1),NCUPL(1),NLEVI(1)
6  WRITE(6,910)
7  L1=L2*4
8  L3=L4*2*NHN
9  L5=L6*1
10 L11=LT2-L12=LT3-L14=LT4=LT5=LT10=LT20=LT100=LT200=1
11 K10=K11=K12=K13=K14=K15=K16=1
12 NCUPL=NHALF=NHALF5=NHM=NHM5=0
13 M10=M20=M30=M40=M50=M60=1
14 DU TO 1-1,NLVI
15 NLEVI(1)=0
16
17
18
19
20
21 C
22 L1,LT2 ARE STORAGE-END-LOCATIONS FOR ARRAYS
23 S(1),S(2) IN LEVEL AND LEVEL
24
25 C
26 L1,LT2 ARE STORAGE-END-LOCATIONS FOR ARRAYS
27 S(1),S(2) IN LEVEL AND LEVEL
28
29
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ALLU 03 IF (NEM5=61,M5) M5=NEM5
ALLU 04 IF (NEM6=61,M6) M6=NEM6
ALLU 05 IF (NCUP=EQ,0) GU TO 22
ALLU 06 M20=M20+NEM6
ALLU 07 IF (L1=EQ,(NLV1-1)) M10=M20
ALLU 08 MDIF=M20-M10
ALLU 09 IF (MDIF=GT,M10) M10=MDIF
ALLU 10 22 CONTINUE
ALLU 11 C
ALLU 12 IF (MCUP4=EQ,0) GO TO 65
ALLU 13 M100=M20
ALLU 14 IF (NCUP=EQ,0) M100=1
ALLU 15 IF (L5=V1(1),GT,100) M100=M6
ALLU 16 M200=2*M100
ALLU 17 65 CONTINUE
ALLU 18 L1=L1*N0
ALLU 19 L2=L2*N0
ALLU 20 L3=L3*HD
ALLU 21 L4=L4*HD
ALLU 22 L5=L5*HD
ALLU 23 L6=L6*ND
ALLU 24 L11=LTU*(L1+1)*L1/2
ALLU 25 L12=LT1*(L2+1)*L2/2
ALLU 26 IF (ICT=EQ,0) L12=L11+1
ALLU 27 L13=L12*(L3+1)*L3/2
ALLU 28 L14=L13*(L4+1)*L4/2
ALLU 29 IF (ICT=EQ,0) L14=L13+1
ALLU 30 L15=L14*(L5+1)*L5/2
ALLU 31 L16=L15*(L6+1)*L6/2
ALLU 32 C
ALLU 33 L110,L120 ARE STORAGE-LOCATIONS FOR ARRAYS
ALLU 34 S(1),S(2) IN LEVEL
ALLU 35 NMS=L16
ALLU 36 IF (NCUP=EQ,0) GO TO 130
ALLU 37 L10=1
ALLU 38 L20=NLEVI(1)
ALLU 39 IF (NCUP=EQ,1) GO TO 100
ALLU 40 GO TO 110
ALLU 41 100 L10=NLEVI(1)
ALLU 42 L20=L10+NLEVI(2)-NHN
ALLU 43 NHALF=L20-NHN
ALLU 44 IF (NCUPL(1)=EQ,2) NHALF=NHALF*2
ALLU 45 GO TO 120
ALLU 46 110 CONTINUE
ALLU 47 DU 25 1-1,NCUP
ALLU 48 11+1
ALLU 49 L20=L20+NLEVI(1)-2*NHN
ALLU 50 IF (NCUPL(1)=EQ,2) L20=L20+2
ALLU 51 NHALF=L20
ALLU 52 IF (L1=EQ,(NCUP-1)) L10=L20
ALLU 53 25 CONTINUE
ALLU 54 IF (NCUPL(MCUP)=EQ,1) L20=L20+NHN
ALLU 55 IF (NCUPL(MCUP)=EQ,2) L20=L20+NHN-2
ALLU 56 120 CONTINUE
ALLU 57 DU 27 1-1,NLVI
ALLU 58 L10=L10*ND
ALLU 59 L20=L20*ND
ALLU 60
ALLU 61
ALLU 62

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ALLU 183 195 CONTINUE
ALLU 184 145 CONTINUE
ALLU 185 IF(ILL.EQ.1.DM.ILS.EJ.1) GO TO 167
ALLU 186 GO TO 160
ALLU 187 165 CONTINUE
ALLU 188 C
ALLU 189 WRITE(6,900) (NLEVI(I),I=1,4,VL1)
ALLU 190 WRITE(6,900) (L2,L3,L4,L5,L6,L10,L20,L100,L700)
ALLU 191 WRITE(6,900) (L10,L11,L12,L13,L14,L15,L16)
ALLU 192 WRITE(6,900) (L100,L120,L1100,L1200)
ALLU 193 WRITE(6,900) (KT0,KT1,KT2,KT3,KT4,KT5,KT6)
ALLU 194 WRITE(6,900) (KT10,KT100,KT1000,MT200)
ALLU 195 WRITE(6,900) (MEND,MFIRST,MHALF)
ALLU 196 WRITE(6,900) MM
ALLU 197 WRITE(6,900) M1,M2,M3,M4,M5,M6,M10,M20,M100,M200
ALLU 198 WRITE(6,900) (MT0,MT1,MT2,MT3,MT4,MT5,MT6)
ALLU 199 WRITE(6,900) (MT10,MT100,MT1000,MT200)
ALLU 200 WRITE(6,900) MHALFS
ALLU 201 WRITE(6,900) MMMS
ALLU 202 CC
ALLU 203 160 CONTINUE
ALLU 204 WRITE(6,915)
ALLU 205 IF(ILS.EQ.1) STOP
ALLU 206 IF(INS.NE.0) MMH=MMMS
ALLU 207 900 FORMAT(12I6)
ALLU 208 910 FORMAT(10H IN ALLUC)
ALLU 209 915 FORMAT(10H END ALLUC)
ALLU 210 RETURN
ALLU 211 END

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CALP 1 1 SUSPOUTINE CALP(MBAND,LM,AM,S,NUN)
CALP 2 C
CALP 3 DIMENSION LM(1),AM(1),S(1)
CALP 4 COMMON /EXTRA/ MODER,MTB
CALP 5 MIN=100000
CALP 6 MAX=0
CALP 7 DO 800 L=1,NUN
CALP 8 IF(FLML(EQ.0)) GO TO 800
CALP 9 IF (LM(L).GT.MAX) MAX=LM(L)
CALP 10 IF (LM(L).LT.MIN) MIN=LM(L)
CALP 11 800 CONTINUE
CALP 12 MDIF=MAX-MIN+1
CALP 13 IF (MODIF.GT.MBAND) MBAND=MDIF
CALP 14 IF(MODER.EQ.1) GO TO 910
CALP 15 MMH=MDN*(NUN+1)/2
CALP 16 LK0=L0+MDN*MDK
CALP 17 MDN1=MDN+1
CALP 18 MDH5=MDN+5
CALP 19 DO 10 I=MDN1,MDH5
CALP 20 10 XN(I)=0.0
CALP 21 C
CALP 22 WRITE(2) (LK0,NUN,(LM(I),I=1,NUN),(S(I)),I=1,MDK)
CALP 23 1 (XN(I),I=1,MDN1,MDN5),(XN(I),I=1,MDN)
CALP 24 RETURN
CALP 25 C
CALP 26 110 WRITE (1) NUN,NS,(LM(1),I=1,MDN)
CALP 27 RETURN
CALP 28 END

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L110=(L10*11)+L10/2+1
L120=L110+(L20*11)+L20/2
NFIRST=NLEVI(1)
MHALFS=M20*(MHN-1)+45
IF(LT20.GT.MMH) MMH=L120
130 CONTINUE
IF(INCUP%.EQ.0) GO TO 140
S(1)=5*(1) IN LEVEL4
L110,LT20 ARE STORAGE LOCATIONS FOR ARRAYS
L110,LT20 ARE STORAGE LOCATIONS FOR ARRAYS
S(1)=5*(1) IN LEVEL4
IF(INCUP.EQ.0) MHALF=MEND+MMH
IF(INCUP.EQ.0) MHALFS=(MHN-1)*NS
L100=NHALF*ND
L200=2*L100+MHN*ND
L1100=(L100+1)*L100/2+1
L1200=L1100+(L200+1)*L200/2
IF(L1200.GT.MMH) MMH=L1200
KI VALUES GIVE CORRESPONDING LOCATIONS FOR MASS MATRIX
140 CONTINUE
KT0=MMH
KI1=KT0+L11
KI2=KT0+L12
KI3=KT0+L13
KI4=KT0+L14
KI5=KT0+L15
KI6=KT0+L16
KI10=KT0+L110
KI20=KT0+L120
KI100=KT0+L1100
KI200=KT0+L1200
MMH=2*MMH
150 CONTINUE
MT0=MT1+MT2+MT3+MT4+MT5+MT6+1
MT10=MT20+MT100+MT200+1
IF(INS.EQ.0) GO TO 165
MT0=MMH
MT1=MT0+NS*1
MT2=MT1+NS*12
IF(FACT.LQ.0) MT3=MT1+1
MT3=MT2+NS*(MHN-1)+3
MT4=MT3+NS*(MHN-1)+4
IF(FACT.GQ.0) MT4=MT3+1
MT5=MT4+NS*NS*(MHN-1)+5
MT6=MT5+M*NS*(MHN-1)+6
MMH=MT6
IF(INCUP.EQ.0) GO TO 150
MT0=MT0+MT10+MT20+NS*(MHN-1)+13
MT20=MT10+MT20+NS*(MHN-1)+120
MMH=MT20
150 CONTINUE
11 INCP%.EQ.0) GO TO 125
MT100=MT0+MT100+NS*(MHN-1)+1100
MT200=MT100+MT200+NS*(MHN-1)+1200
MMH=MT200

```

```

MT0=MT1+MT2+MT3+MT4+MT5+MT6+1
MT10=MT20+MT100+MT200+1
IF(INS.EQ.0) GO TO 165
MT0=MMH
MT1=MT0+NS*1
MT2=MT1+NS*12
IF(FACT.LQ.0) MT3=MT1+1
MT3=MT2+NS*(MHN-1)+3
MT4=MT3+NS*(MHN-1)+4
IF(FACT.GQ.0) MT4=MT3+1
MT5=MT4+NS*NS*(MHN-1)+5
MT6=MT5+M*NS*(MHN-1)+6
MMH=MT6
IF(INCUP.EQ.0) GO TO 150
MT0=MT0+MT10+MT20+NS*(MHN-1)+13
MT20=MT10+MT20+NS*(MHN-1)+120
MMH=MT20
150 CONTINUE
11 INCP%.EQ.0) GO TO 125
MT100=MT0+MT100+NS*(MHN-1)+1100
MT200=MT100+MT200+NS*(MHN-1)+1200
MMH=MT200

```

```

STAR 19 T2=I1*B(L0B)/S(L0H)
STAR 20 T3=B(L0A)/S(L0B)
STAR 21 DU 30 J=I*MAX
STAR 22 LUC=LUI(J)-J+I
STAR 23 LUD=MAX+J
STAR 24 S(L0C) = S(L0C)-S(L0D)+FI
STAR 25 I(L0(L0D),E0,0,0) GO TU 100
STAR 26 B(L0C)-B(L0D)-B(L0D)+I1+S(L0D)+T2-S(L0D)+T3
STAR 27 GO TU 30
STAR 28 100 CONTINUE
STAR 29 B(L0C)+B(L0C)+S(L0D)+T2-S(L0D)+T3
STAR 30 CONTINUE
STAR 31 IF(MS.E0,0) GO TU 20
STAR 32 DU 35 J=I*MS
STAR 33 L0B=(I-1)*MS+J
STAR 34 L0F=(J-1)*MS+J
STAR 35 F(L0E)=F(L0E)+F(L0F)+T1
STAR 36 35 CONTINUE
STAR 37 20 CONTINUE
STAR 38 10 CONTINUE
STAR 39 RETURN
STAR 40 END

```

```

SUBROUTINE LVALIND(A,BY1,BY2,S1,S2,BL,BZ,
1 F1,F2,XS1,ETA,NS,TE,P,PA,LS4)
DIMENSION S(11),S2(11),B(11),B2(11)
DIMENSION F(11),F2(11),XS(11),ETA(11)
DIMENSION ST(12,12),LUC(12)
DATA LUC/1,2,3,10,11,12,7,8,9,4,5,6/
WRITE(6,9011)
BY=BY1
IC=0
N4=N4*NU
NF=(N4+1)*N4/2
N5=N4*NS
100 CONTINUE
WRITE(6,904) F,P,PA,AB,RT

```

```

CREATION OF ELEMENT STIFFNESS MATRIX
IF(I1S4.E0.0) CALL ESTIFIND,P,E,P,A,B,F2,A,S1,F1A1
IF(I1S4.NE.0) READ(13) NTA,S2(11),I1,I1X)
CREATION OF ELEMENT MASS MATRIX
CALL ELMAIND,I,M,A,BY1,B2)
CREATION OF ELEMENT STRESS-DISPLACEMENT MATRIX
CALL ESTRESIND,N4,NS,E,P,A,B,F2,A,S1,F1A1
IF(FC.E0.1) GO TU 140
IF(PTZ.NL.0,0) GO TU 12C
DU 27 I=1,MT
S(11)=S2(11)
S2(11)=0.0
N(11)=B2(11)
25 B2(11)=0.0

```

```

SUBROUTINE ASSEMBL(N,S,SS,B,BB,LMS,NS,NSS,F,FF)
ASSEMBLY ROUTINE
DIMENSION L(M),S(11),S2(11),B(11),B2(11)
DIMENSION LMS(11),F(11),FF(11)
DU 10 I=1,N
DU 20 J=1,M
Q=1.0
IF(LM(1).LT.0) Q=-Q
IF(LM(J).LT.0) Q=-Q
IF(ABS(LM(1)) .GT. ABS(LM(J))) GO TU 200
IP = ABS(LM(1))
IC = ABS(LM(J))
LUC = (IC-1)*J/2 + IR
200 IF = ABS(LM(J))
210 INL=(IC-1)*J/2 + IR
LUD=(J-1)*J/2 + I
SS(INEW) = SS(INEW) + S(LUD)*Q
BB(INEW) = BB(INEW) + B(LUD)*Q
20 CONTINUE
IF(MS.E0,0) GO TU 10
Q=1.0
ASSE 29 ASSE 29
ASSE 26 ASSE 26
ASSE 27 IR=ABS(LM(1))
ASSE 28 DU 25 J=1,MS
ASSE 29 IC=LMS(J)
ASSE 30 INL=(IR-1)*MS+IC
ASSE 31 IOLD=(I-1)*MS+J
ASSE 32 F(FINEW)=FF(INEW)+F(IOLD)*Q
ASSE 33 25 CONTINUE
ASSE 34 10 CONTINUE
ASSE 35 FMO
ASSE 36 FMO

```

```

SUBROUTINE STAR10,NU,MP,S,B,NS,FF)
REDUCTION ALGORITHM
DIMENSION S(11),B(11),F(11),L(11)
DU 5 J=1,NO
L(J)=L(J)+1+J/2
NE=NO-NP
DU 10 M=1,NE
MAX=NU-M
M=MAX+1
MAX=L(M)
V=L(M)
DU 20 I=1,MAX
LUA=MAX+1
LUB=NM
IF(S(LUA).E0,0,0) GO TU 20
I1 = S(LUA)/S(LUB)

```

```

LVLU 1 SUBROUTINE LVALIND(A,BY1,BY2,S1,S2,BL,BZ,
LVLU 2 1 F1,F2,XS1,ETA,NS,TE,P,PA,LS4)
LVLU 3 C
LVLU 4 DIMENSION S(11),S2(11),B(11),B2(11)
LVLU 5 DIMENSION F(11),F2(11),XS(11),ETA(11)
LVLU 6 DIMENSION ST(12,12),LUC(12)
LVLU 7 DATA LUC/1,2,3,10,11,12,7,8,9,4,5,6/
LVLU 8 WRITE(6,9011)
LVLU 9 BY=BY1
LVLU 10 IC=0
LVLU 11 N4=N4*NU
LVLU 12 NF=(N4+1)*N4/2
LVLU 13 N5=N4*NS
LVLU 14 100 CONTINUE
LVLU 15 WRITE(6,904) F,P,PA,AB,RT
LVLU 16 C
LVLU 17 C CREATION OF ELEMENT STIFFNESS MATRIX
LVLU 18 C
LVLU 19 IF(I1S4.E0.0) CALL ESTIFIND,P,E,P,A,B,F2,A,S1,F1A1
LVLU 20 IF(I1S4.NE.0) READ(13) NTA,S2(11),I1,I1X)
LVLU 21 C
LVLU 22 C CREATION OF ELEMENT MASS MATRIX
LVLU 23 C
LVLU 24 CALL ELMAIND,I,M,A,BY1,B2)
LVLU 25 C
LVLU 26 C CREATION OF ELEMENT STRESS-DISPLACEMENT MATRIX
LVLU 27 C
LVLU 28 CALL ESTRESIND,N4,NS,E,P,A,B,F2,A,S1,F1A1
LVLU 29 IF(FC.E0.1) GO TU 140
LVLU 30 IF(PTZ.NL.0,0) GO TU 12C
LVLU 31 DU 27 I=1,MT
LVLU 32 S(11)=S2(11)
LVLU 33 S2(11)=0.0
LVLU 34 N(11)=B2(11)
LVLU 35 25 B2(11)=0.0

```

```

LVL0 36 IF(NS.EQ.0) GO TO 60
LVL0 37 DO 30 I=1,NS
LVL0 38 F1(I)=F2(I)
LVL0 39 30 F2(I)=0.0
LVL0 40 60 CONTINUE
LVL0 41 GO TO 140
LVL0 42 120 CONTINUE
LVL0 43 DO 35 I=1,NI
LVL0 44 S1(I)=S2(I)
LVL0 45 S2(I)=0.0
LVL0 46 U1(I)=B2(I)
LVL0 47 LVL0 48 BY=BY2
LVL0 49 IC=1
LVL0 50 IF(NS.EQ.0) GO TO 70
LVL0 51 DO 36 I=1,NIS
LVL0 52 F1(I)=F2(I)
LVL0 53 36 F2(I)=0.0
LVL0 54 70 CONTINUE
LVL0 55 GO TO 100
LVL0 56 140 CONTINUE
LVL0 57 WRITE(6,902)
LVL0 58 901 FORMAT(11H IN LEVEL0 )
LVL0 59 902 FORMAT(11H END LEVEL0)
LVL0 60 904 FORMAT(12E11.4)
LVL0 61 906 FORMAT(12I5)
LVL0 62 RETURN
LVL0 63 END

LVL1 1 SUBROUTINE LVLINTAP,ND,NS,MH,M,LMS,LEVI,BY2,
LVL1 2 S1,S2,S51,S52,STEM,B1,B2,BB1,BB2,BTEM,
LVL1 3 F1,F2,FF1,FF2,FF2)
LVL1 4 C
LVL1 5 DIMENSION S1(1),S2(1),S51(1),S52(1),STEM(1)
LVL1 6 DIMENSION B1(1),B2(1),BB1(1),BB2(1),BTEM(1)
LVL1 7 DIMENSION F1(1),F2(1),FF1(1),FF2(1),FF2(1)
LVL1 8 DIMENSION LEVI(1),MH(1),LMS(1)
LVL1 9 WRITE(6,902)
LVL1 10 C
LVL1 11 C CREATES START ELEMENT FOR BY1, OR FOR BY1 AND BY2
LVL1 12 C
LVL1 13 CALL STELIND,NS,MH,M,LMS,LEVI,S1,S2,S51,S52,
LVL1 14 B1,B2,BB1,BB2,F1,F2,FF1,FF2)
LVL1 15 C
LVL1 16 IF(LEVI(1).GE.100) GO TO 100
LVL1 17 C
LVL1 18 C CREATES END ELEMENT AS SPECIFIED BY LEVI(1) USING BY1
LVL1 19 C
LVL1 20 CALL ENLINTAP,ND,NS,MH,M,LMS,LEVI(1),
LVL1 21 S1,S51,STEM,B1,BB1,BTEM,F1,FF1,FF2)
LVL1 22 C
LVL1 23 100 CONTINUE
LVL1 24 WRITE(6,904)
LVL1 25 902 FORMAT(11H IN LEVEL1)
LVL1 26 904 FORMAT(11H END LEVEL1)
LVL1 27 RETURN
LVL1 28 END

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```

SUBROUTINE STELIND,NS,MH,M,LMS,LEVI,S1,S2,S51,S52,
1 B1,B2,BB1,BB2,F1,F2,FF1,FF2)
2 C
3 DIMENSION LM(1),S1(1),S2(1),S51(1),S52(1)
4 DIMENSION B1(1),B2(1),BB1(1),BB2(1)
5 DIMENSION LMS(1),F1(1),F2(1),FF1(1),FF2(1)
6 WRITE(6,950)
7 ND2 =2*ND
8 ND4 =4*ND
9 M =MH-1
10 NS=NS*M
11 UU 100 MEL =1,M
12 C
13 C CONNECTIVITY ARRAY FOR MYP=1
14 C
15 CALL ENLIND,LM,M,ND,ND2)
16 IF(NS.EQ.0) GO TO 115
17 DO 10 I=1,NS
18 11=(MEL-1)*NS+I
19 LMS(I)=I
20 CONTINUE
21 10 CONTINUE
22 WRITE(6,900) (LMS(I),I=1,NS)
23 115 CONTINUE
24 CALL ASSEMBLM,ND4,S1,S51,B1,BB1,LMS,NS,NS5,F1,FF1)
25 100 CONTINUE
26 IF(NS.EQ.0) GO TO 150
27 DO 10 I=1,M
28 CALL ENLIND,LM,M,ND,ND2)
29 IF(NS.EQ.0) GO TO 120
30 DO 15 I=1,NS
31 11=(MEL-1)*NS+I
32 LMS(I)=I
33 CONTINUE
34 15 CONTINUE
35 120 CONTINUE
36 CALL ASSEMBLM,ND4,S2,S52,B2,BB2,LMS,NS,NS5,F2,FF2)
37 110 CONTINUE
38 WRITE(6,952)
39 950 FORMAT(12H IN STARTEL)
40 952 FORMAT(12H END STARTEL)
41 RETURN
42 END

```

```

STEL 1
STEL 2
STEL 3 C
STEL 4
STEL 5
STEL 6
STEL 7
STEL 8
STEL 9
STEL 10
STEL 11
STEL 12
STEL 13 C
STEL 14 C
STEL 15 C
STEL 16
STEL 17
STEL 18
STEL 19
STEL 20
STEL 21
STEL 22
STEL 23
STEL 24
STEL 25
STEL 26
STEL 27
STEL 28
STEL 29
STEL 30
STEL 31
STEL 32
STEL 33
STEL 34
STEL 35
STEL 36
STEL 37
STEL 38
STEL 39
STEL 40
STEL 41
STEL 42

ENL 1
ENL 2
ENL 3 C
ENL 4
ENL 5
ENL 6
ENL 7
ENL 8
ENL 9
ENL 10
ENL 11
ENL 12
ENL 13

```

```

SUBROUTINE LVLINTAP,ND,NS,MH,M,LMS,LEVI,BY2,
1 S1,S2,S51,S52,STEM,B1,B2,BB1,BB2,BTEM,
2 F1,F2,FF1,FF2,FF2)
3 C
4 DIMENSION S1(1),S2(1),S51(1),S52(1)
5 DIMENSION B1(1),B2(1),BB1(1),BB2(1)
6 DIMENSION F1(1),F2(1),FF1(1),FF2(1)
7 DIMENSION LEVI(1),MH(1),LMS(1)
8 WRITE(6,902)
9 100 CONTINUE
10 WRITE(6,904)
11 902 FORMAT(11H IN LEVEL1)
12 904 FORMAT(11H END LEVEL1)
13 RETURN
14 END

SUBROUTINE ENLINTAP,ND,NS,MH,M,LMS,LEVI,
1 S1,S51,STEM,B1,BB1,BTEM,F1,FF1,FF2)
2 C
3 DIMENSION S1(1),S51(1),S52(1)
4 DIMENSION B1(1),BB1(1),BTEM(1)
5 DIMENSION F1(1),FF1(1),FF2(1)
6 DIMENSION LM(1),LMS(1)
7 WRITE(6,950)
8 ND2=ND*2
9 ND4=ND*4
10 MH=MH-1
11 NS=NS*M
12 UU 100 MEL =1,M
13 C
14 C CONNECTIVITY ARRAY FOR MYP=1
15 C
16 CALL ENLIND,LM,M,ND,ND2)
17 IF(NS.EQ.0) GO TO 115
18 DO 10 I=1,NS
19 11=(MEL-1)*NS+I
20 LMS(I)=I
21 CONTINUE
22 10 CONTINUE
23 WRITE(6,900) (LMS(I),I=1,NS)
24 115 CONTINUE
25 CALL ASSEMBLM,ND4,S1,S51,B1,BB1,LMS,NS,NS5,F1,FF1)
26 100 CONTINUE
27 IF(NS.EQ.0) GO TO 150
28 DO 10 I=1,M
29 CALL ENLIND,LM,M,ND,ND2)
30 IF(NS.EQ.0) GO TO 120
31 DO 15 I=1,NS
32 11=(MEL-1)*NS+I
33 LMS(I)=I
34 CONTINUE
35 15 CONTINUE
36 CALL ASSEMBLM,ND4,S2,S52,B2,BB2,LMS,NS,NS5,F2,FF2)
37 110 CONTINUE
38 WRITE(6,952)
39 950 FORMAT(12H IN STARTEL)
40 952 FORMAT(12H END STARTEL)
41 RETURN
42 END

```

```

ENL 1
ENL 2
ENL 3 C
ENL 4
ENL 5
ENL 6
ENL 7
ENL 8
ENL 9
ENL 10
ENL 11
ENL 12
ENL 13

```

```

ENEL 14 ENEL 14 01=ND2*HNH
ENEL 15 ENEL 15 01=INI*NI/2
ENEL 16 ENEL 16 00 5 I=I*NT
ENEL 17 ENEL 17 S*ND(1)=0.0
ENEL 18 ENEL 18 BEND(1)=0.0
ENEL 19 ENEL 19 5 CONTINUE
ENEL 20 C ENEL 20 C
ENEL 21 C ENEL 21 C F U R M F N D E L E M E N T
ENEL 22 C ENEL 22 C GO TO (110,120,130,140,110,150) M T Y P
ENEL 23 ENEL 23
ENEL 24 C ENEL 24 C T Y P E I U R 5
ENEL 25 C ENEL 25 C
ENEL 26 C ENEL 26 C
ENEL 27 ENEL 27 F I O C O N T I N U E
ENEL 28 ENEL 28 01=2*ND*HNH
ENEL 29 ENEL 29 01=INI*NI/2
ENEL 30 ENEL 30 NST=NS*NI
ENEL 31 ENEL 31 00 10 I=I*NT
ENEL 32 ENEL 32 SEND(1)=SSI(1)
ENEL 33 ENEL 33 BEND(1)=BBI(1)
ENEL 34 ENEL 34 10 CONTINUE
ENEL 35 ENEL 35 N U D E S = 2 * H N H
ENEL 36 ENEL 36 I F ( N S . E Q . 0 ) G O T O 112
ENEL 37 ENEL 37 00 15 I=I*NT
ENEL 38 ENEL 38 F L U I D ( 1 ) = F F I ( 1 )
ENEL 39 ENEL 39 112 CONTINUE
ENEL 40 ENEL 40 I F ( M T Y P . E Q . 1 ) G O T O 115
ENEL 41 ENEL 41 N U D E S = N H N
ENEL 42 ENEL 42 N R L D = N H N * N D
ENEL 43 ENEL 43 50 T O 225
ENEL 44 C ENEL 44 C T Y P E 2
ENEL 45 C ENEL 45 C
ENEL 46 C ENEL 46 C 120 00 200 N E L = 1 * M
ENEL 47 ENEL 47
ENEL 48 ENEL 48
ENEL 49 ENEL 49 C A L L E M D 2 ( N E L , L M , M , N D , N D 2 , N D 3 , N D 4 , N U D E S )
ENEL 50 ENEL 50 I F ( N S . E Q . 0 ) G O T O 130
ENEL 51 ENEL 51 00 20 I=I*NS
ENEL 52 ENEL 52 I I = ( N E L - 1 ) * N S + 1
ENEL 53 ENEL 53 L M S ( I ) = I I
ENEL 54 ENEL 54 20 C O N T I N U E
ENEL 55 ENEL 55 W R I T E ( 6 , ' 0 0 0 ' ) ( L M S ( I ) , I = 1 , N S )
ENEL 56 ENEL 56 135 C O N T I N U E
ENEL 57 ENEL 57 C A L L A S S E M ( L M , N D 4 , S I , S E N D , B I , B E N D , L M S , N S , N S S , F I , F E N D )
ENEL 58 ENEL 58
ENEL 59 ENEL 59 200 C O N T I N U E
ENEL 60 ENEL 60 N R L D = ( N H N * 2 ) * N D
ENEL 61 ENEL 61 00 T O 225
ENEL 62 C ENEL 62 C T Y P E 3
ENEL 63 C ENEL 63 C
ENEL 64 C ENEL 64 C
ENEL 65 ENEL 65 130 I F ( N H N . G T . 4 ) G O T O 102
ENEL 66 ENEL 66 W R I T E ( 6 , ' 9 0 0 ' )
ENEL 67 ENEL 67 S T O P
ENEL 68 ENEL 68
ENEL 69 ENEL 69 102 C O N T I N U E
ENEL 70 ENEL 70 00 220 N E L = 1 * M
ENEL 71 ENEL 71 C A L L E M D 2 ( N E L , L M , M , N D , N D 2 , N D 3 , N D 4 , N U D E S )
ENEL 72 ENEL 72 I F ( N S . E Q . 0 ) G O T O 139
ENEL 73 ENEL 73 00 25 I=I*NS

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ENEL 74 ENEL 74 I I = ( N E L - 1 ) * N S + 1
ENEL 75 ENEL 75 L M S ( I ) = I I
ENEL 76 ENEL 76 25 C O N T I N U E
ENEL 77 ENEL 77 W R I T E ( 6 , ' 9 0 0 ' ) ( L M S ( I ) , I = 1 , N S )
ENEL 78 ENEL 78 138 C O N T I N U E
ENEL 79 ENEL 79 C A L L A S S E M ( L M , N D 4 , S I , S E N D , B I , B E N D , L M S , N S , N S S , F I , F E N D )
ENEL 80 C ENEL 80 C
ENEL 81 ENEL 81 220 C O N T I N U E
ENEL 82 ENEL 82 N R L D = ( N H N * 3 ) * N D
ENEL 83 ENEL 83 00 T O 225
ENEL 84 C ENEL 84 C T Y P E 4
ENEL 85 C ENEL 85 C
ENEL 86 C ENEL 86 C
ENEL 87 ENEL 87 140 00 222 N E L = 1 * M
ENEL 88 C ENEL 88 C
ENEL 89 ENEL 89 C A L L E M D 4 ( N E L , L M , M , N D , N D 2 , N D 3 , N D 4 , N U D E S )
ENEL 90 ENEL 90 I F ( N S . E Q . 0 ) G O T O 142
ENEL 91 ENEL 91 00 30 I=I*NS
ENEL 92 ENEL 92 I I = ( N E L - 1 ) * N S + 1
ENEL 93 ENEL 93 L M S ( I ) = I I
ENEL 94 ENEL 94 30 C O N T I N U E
ENEL 95 ENEL 95 W R I T E ( 6 , ' 9 0 0 ' ) ( L M S ( I ) , I = 1 , N S )
ENEL 96 ENEL 96 142 C O N T I N U E
ENEL 97 ENEL 97 C A L L A S S E M ( L M , N D 4 , S I , S E N D , B I , B E N D , L M S , N S , N S S , F I , F E N D )
ENEL 98 C ENEL 98 C
ENEL 99 ENEL 99 222 C O N T I N U E
ENEL 100 ENEL 100 N R E D = N U D E S * N D
ENEL 101 ENEL 101 00 T O 225
ENEL 102 C ENEL 102 C T Y P E 6
ENEL 103 C ENEL 103 C
ENEL 104 C ENEL 104 C
ENEL 105 ENEL 105 150 I F ( N H N . G T . 3 ) G O T O 103
ENEL 106 ENEL 106 W R I T E ( 6 , ' 9 7 0 ' )
ENEL 107 ENEL 107 S T O P
ENEL 108 ENEL 108 103 C O N T I N U E
ENEL 109 ENEL 109 00 223 N E L = 1 * M
ENEL 110 ENEL 110 C A L L E M D 2 ( N E L , L M , M , N D , N D 2 , N D 3 , N D 4 , N U D E S )
ENEL 111 ENEL 111 I F ( N S . E Q . 0 ) G O T O 144
ENEL 112 ENEL 112 00 35 I=I*NS
ENEL 113 ENEL 113 I I = ( N E L - 1 ) * N S + 1
ENEL 114 ENEL 114 L M S ( I ) = I I
ENEL 115 ENEL 115 35 C O N T I N U E
ENEL 116 ENEL 116 144 C O N T I N U E
ENEL 117 ENEL 117 W R I T E ( 6 , ' 9 0 0 ' ) ( L M S ( I ) , I = 1 , N D 4 )
ENEL 118 ENEL 118 C A L L A S S E M ( L M , N D 4 , S I , S E N D , B I , B E N D , L M S , N S , N S S , F I , F E N D )
ENEL 119 ENEL 119 223 C O N T I N U E
ENEL 120 ENEL 120 225 C O N T I N U E
ENEL 121 ENEL 121 N D T O T = 2 * N H N * N D
ENEL 122 ENEL 122
ENEL 123 C ENEL 123 C
ENEL 124 ENEL 124 C A L L S T A P ( L M , N D T O T , N R L D , S F N D , B E N D , N S S , F I , F E N D )
ENEL 125 ENEL 125 C O N T I N U E
ENEL 126 ENEL 126 I C U D = N U D E S * N D
ENEL 127 ENEL 127 N T = ( N D T O T ) * H U D / 2
ENEL 128 ENEL 128 N S T = N S * M * N D
ENEL 129 C ENEL 129 C
ENEL 130 ENEL 130 W R I T E ( I N T A P I R T , H U D )
ENEL 131 ENEL 131 W R I T E ( I N T A P I R S E N D ( I ) , I = 1 , N I )
ENEL 132 ENEL 132 W R I T E ( I N T A P I R B E N D ( I ) , I = 1 , N I )
ENEL 133 ENEL 133 I I ( 45 . E Q . 0 ) G O T O 145

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LVL3 11 C
LVL3 12 C
LVL3 13 C
LVL3 14 C
LVL3 15 C
LVL3 16
LVL3 17
LVL3 18
LVL3 19 C
LVL3 20 C
LVL3 21 C
LVL3 22
LVL3 23
LVL3 24
LVL3 25
LVL3 26
LVL3 27
LVL3 28
LVL3 29
LVL3 30
LVL3 31
LVL3 32 C
LVL3 33
LVL3 34
LVL3 35
LVL3 36
LVL3 37
LVL3 38
LVL3 39
LVL3 40
LVL3 41
LVL3 42
LVL3 43
LVL3 44
LVL3 45
LVL3 46
LVL3 47
LVL3 48 C
LVL3 49
LVL3 50 C
LVL3 51 C
LVL3 52 C
LVL3 53 C
LVL3 54
LVL3 55
LVL3 56
LVL3 57
LVL3 58
LVL3 59
LVL3 60
LVL3 61
LVL3 62
LVL3 63
LVL3 64 C
LVL3 65
LVL3 66
LVL3 67 C
LVL3 68
LVL3 69
LVL3 70

```

CREATE CONNECTIVITY FOR THE 140 ELEMENTS
 CREATE CONNECTIVITY ARRAY FOR THE TWO SUP. ELEM.
 CONNECTED AT THE PRESENT COUPLING STEP
 IF(I1-EQ.0) GO TO 110
 GO TO 120
 110 CONTINUE
 READ END ELEMENT (UP FIRST ELEMENT FOR I1=1) FROM TAPE
 HEAD(NTAP) NT1=NI
 READ(NTAP) (S(I1),I=1,NT1)
 READ(NTAP) (B(I1),I=1,NT1)
 LAST=NI/ND
 IF(4S-EQ.0) GO TO 115
 READ(NTAP) NST1=NS1
 READ(NTAP) (F(I1),I=1,NST1)
 LASTS=NS1
 115 CONTINUE
 120 CONTINUE
 CALL CONC3(11,ND,NS,NHH,NDTOT,NEX1,LAST,NEAT,
 NDOTS,LASTS,LEVI,HL(PL,LM1,LM2)
 130
 DO 10 I=1,NDTOT
 DO 10 J=1,NDTOT
 TO=I+J-1
 S3(1)=0.0
 S3(1)=0.0
 10 J(1)=0.0
 IF(NS-EQ.0) GO TO 125
 DO 12 I=1,NST1
 12 LMS(I)=I
 NTS=NDTOT*NDTOT
 DO 13 I=1,NTS
 13 F3(I)=0.0
 125 CONTINUE
 CALL ASSEM(LM1,LAST,S,S3,B,B3,LM5,LAST5,NDOT5,F,F3)
 READ (I1) SUP. ELEM. FROM TAPE NTAP (TU BE COUPLED
 IN THE PRESENT LOOP.)
 READ(NTAP) NT2=NS2
 READ(NTAP) (S(I1),I=1,NT2)
 READ(NTAP) (B(I1),I=1,NT2)
 IF(NS-EQ.0) GO TO 130
 READ(NTAP) NST2=NS2
 READ(NTAP) (F(I1),I=1,NST2)
 NEATS=NS2
 DO 14 I=1,NEATS
 14 LMS(I)=I+LASTS
 130 CONTINUE
 CALL ASSEM(LM2,NEX1,S,S3,B,B3,LM5,NLXIS,NDOT5,F,F3)
 NTS=NDOT5*NDOT5
 CALL STAP(LM1,NDOT,NFAT,J,B3,NDOT5,F3)
 IL=0
 DO 15 I=1,NEXT

```

LVL2 53
LVL2 54
LVL2 55
LVL2 56
LVL2 57
LVL2 58
LVL2 59
LVL2 60
LVL2 61
LVL2 62
LVL2 63
LVL2 64
LVL2 65
LVL2 66
LVL2 67
LVL2 68
LVL2 69
LVL2 70
LVL2 71
LVL2 72
LVL2 73
LVL2 74
LVL2 75
LVL2 76
LVL2 77
LVL2 78
LVL2 79
LVL2 80
LVL2 81
LVL2 82
LVL2 83
LVL2 84
LVL2 85
LVL2 86
LVL2 87
LVL2 88
LVL2 89
LVL2 90
LVL2 91
LVL2 92
LVL2 93
LVL2 94
LVL2 95
LVL2 96
LVL2 97
LVL2 98
LVL2 99
LVL3 1
LVL3 2
LVL3 3
LVL3 4
LVL3 5
LVL3 6
LVL3 7
LVL3 8
LVL3 9
LVL3 10

```

36 LMS(I)=I+NS1
 120 CONTINUE
 CALL ASSEM(LM2,N1,S1,S2,B1,B2,LM5,NS1,NS2,F1,F2)
 CALL START(LM1,NS2,NS2,S2,B2,NS2,F2)
 IF(I1-EQ.NSTEP) GO TO 10
 10
 DO 40 I=1,N2
 DO 40 J=1,N2
 10-10-1
 S(1)=S2(I1)
 50 B(1)=B2(I1)
 IF(NS-EQ.0) GO TO 125
 DO 45 I=1,NST2
 45 F(1)=F2(I1)
 125 CONTINUE
 10 CONTINUE
 N12=(N2+1)*N2/2
 170 CONTINUE
 WRITE(NTAP) NT2=N2
 WRITE(NTAP) (S2(I1),I=1,NT2)
 WRITE(NTAP) (B2(I1),I=1,NT2)
 IF(NS-EQ.0) GO TO 174
 WRITE(NTAP) NST2=NS2
 WRITE(NTAP) (F2(I1),I=1,NST2)
 174 CONTINUE
 IF(NCUP-EQ.0) GO TO 176
 DO 60 I=1,NT2
 S2(1)=0.0
 S2(1)=0.0
 60 B2(1)=0.0
 IF(NS-EQ.0) GO TO 176
 DO 65 I=1,NST2
 65 F2(1)=0.0
 176 CONTINUE
 NEW=N2
 NSU=NS2
 B(1)=16.956
 904 FORMAT(1X,12E11.4)
 906 FORMAT(7)
 920 FORMAT(1X,E11.4)
 900 FORMAT(12I5)
 902 FORMAT(30I3)
 954 FORMAT(12H END LEVEL22)
 956 FORMAT(12H END LEVEL22)
 RETURN
 END
 SUBROUTINE LVL3(NTAP,ND,NS,NHH,NDTOT,NEX1,NCOP,NCOPL,NDOT5,
 1 LM1,LAST,NS,S,S3,B,B3,F,F3)
 DIMENSION S(1),S3(1),B(1),B3(1)
 DIMENSION LMS(1),F(1),F3(1)
 WRITE(6,900)
 N1=NST1+N2=NST2
 LASTS=NEATS=0
 PENDING NTAP
 DO 50 I=1,NCOPL

```

LVL3 71      DO 15 J=1,NEX1
LVL3 72      10=10*1
LVL3 73      S(I1)=S3(I1)
LVL3 74      15 B(I1)=B3(I1)
LVL3 75      IFMS=EQ.O) GO TO 135
LVL3 76      N5=NEX1*NDTOT
LVL3 77      DO 20 I=1,N5
LVL3 78      20 F(I)=F3(I)
LVL3 79      135 CONTINUE-
LVL3 80      LAST = NEX1/ND
LVL3 81      LASTS=NDTOT
LVL3 82      50 CONTINUE
LVL3 83      WRITE(6,902)
LVL3 84      900 FORMAT(11H IN LEVEL3)
LVL3 85      902 FORMAT(11H END LEVEL3)
LVL3 86      RETURN
LVL3 87      END

```

```

LVL4 41      -MES=MS1/MS
LVL4 42      11=0
LVL4 43      DO 30 I=1,MES
LVL4 44      DO 30 L=1,MS
LVL4 45      11=11+1
LVL4 46      30 LMS(I)=MS1*(MES-1)+MS+L
LVL4 47      120 CONTINUE
LVL4 48      C
LVL4 49      CALL ASSE(LM2,NHALF,S5,S6,B8,B4,LMS,MS1,MS51,F,F4)
LVL4 50      CALL STAK(LM1,NDTOT,NEW,S4,B8,MS51,F4)
LVL4 51      NST=NE*MS1
LVL4 52      WRITE(6,904)
LVL4 53      902 FORMAT(11H IN LEVEL4)
LVL4 54      904 FORMAT(11H END LEVEL4)
LVL4 55      RETURN
LVL4 56      END

```

```

LVL3 71      DO 15 J=1,NEX1
LVL3 72      10=10*1
LVL3 73      S(I1)=S3(I1)
LVL3 74      15 B(I1)=B3(I1)
LVL3 75      IFMS=EQ.O) GO TO 135
LVL3 76      N5=NEX1*NDTOT
LVL3 77      DO 20 I=1,N5
LVL3 78      20 F(I)=F3(I)
LVL3 79      135 CONTINUE-
LVL3 80      LAST = NEX1/ND
LVL3 81      LASTS=NDTOT
LVL3 82      50 CONTINUE
LVL3 83      WRITE(6,902)
LVL3 84      900 FORMAT(11H IN LEVEL3)
LVL3 85      902 FORMAT(11H END LEVEL3)
LVL3 86      RETURN
LVL3 87      END

```

```

SUBROUTINE END1(NEL,LM,M,ND,ND2)
DIMENSION LM(1)
DO 15 I=1,ND2
LM(I) = (NEL-1)*ND+I
15 LM(ND2+1) = (2*M-NEL+1)*ND+1
RETURN
END

```

```

SUBROUTINE END2(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
DIMENSION LM(ND4)
IF(NEL.NE.1) GO TO 500
DO 25 I=1,ND
LM(I) = I
25 LM(ND3+1) = (M+2)*ND+I
GO TO 500

```

```

SUBROUTINE LVL4(INTAP,ND,NS,NHM,NHALF,NHALFS,MEND,NCOP,
1 NCUP4,NEW,MS51,LM1,LM2,LMS,S5,S6,B8,B4,F,F4)
DIMENSION LM(1),LM2(1),S(1),S4(1),B(1),B4(1)
WRITE(6,902)
MS1=MS51=0
IF ONLY TWO ENDELEMENTS ARE SYMMETRIC
COUPLED READ FROM INTAP
IF (NCUP.NE.0) GO TO 100
REWIND INTAP
READ(INTAP) NET,NE
READ(INTAP) (S(I),I=1,NET)
READ(INTAP) (B(I),I=1,NET)
NHALF=NE/ND
IF (MS.EQ.0) GO TO 100
READ(INTAP) NST,NS1
READ(INTAP) (F(I),I=1,NS1)
NHALFS=NS1
100 CONTINUE
CALL CONC4(ND,NHM,NHALF,MEND,NDTOT,NEW,NCUP4,LM1,LM2)
NT=(NDTOT+1)*ND*101/2
DO 10 I=1,NT
54(I)=0.0
10 84(I)=0.0
IF (MS.EQ.0) GO TO 110
MS1=N51*2
MS51=NDTOT*MS51
DO 15 I=1,NS1
F4(I)=0.0
15 F4(I)=0.0
DO 20 I=1,NS1
LM5(I)=I
20 LM5(I)=1
110 CONTINUE
CALL ASSE(LM1,NHALF,S5,S6,B8,B4,LMS,MS1,MS51,F,F4)
IF (MS.EQ.0) GO TO 120

```

```

SUBROUTINE END1(NEL,LM,M,ND,ND2)
DIMENSION LM(1)
DO 15 I=1,ND2
LM(I) = (NEL-1)*ND+I
15 LM(ND2+1) = (2*M-NEL+1)*ND+1
RETURN
END

```

```

SUBROUTINE END2(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
DIMENSION LM(ND4)
IF(NEL.NE.1) GO TO 500
DO 25 I=1,ND
LM(I) = I
25 LM(ND3+1) = (M+2)*ND+I
GO TO 500

```

```

SUBROUTINE LVL4(INTAP,ND,NS,NHM,NHALF,NHALFS,MEND,NCOP,
1 NCUP4,NEW,MS51,LM1,LM2,LMS,S5,S6,B8,B4,F,F4)
DIMENSION LM(1),LM2(1),S(1),S4(1),B(1),B4(1)
WRITE(6,902)
MS1=MS51=0
IF ONLY TWO ENDELEMENTS ARE SYMMETRIC
COUPLED READ FROM INTAP
IF (NCUP.NE.0) GO TO 100
REWIND INTAP
READ(INTAP) NET,NE
READ(INTAP) (S(I),I=1,NET)
READ(INTAP) (B(I),I=1,NET)
NHALF=NE/ND
IF (MS.EQ.0) GO TO 100
READ(INTAP) NST,NS1
READ(INTAP) (F(I),I=1,NS1)
NHALFS=NS1
100 CONTINUE
CALL CONC4(ND,NHM,NHALF,MEND,NDTOT,NEW,NCUP4,LM1,LM2)
NT=(NDTOT+1)*ND*101/2
DO 10 I=1,NT
54(I)=0.0
10 84(I)=0.0
IF (MS.EQ.0) GO TO 110
MS1=N51*2
MS51=NDTOT*MS51
DO 15 I=1,NS1
F4(I)=0.0
15 F4(I)=0.0
DO 20 I=1,NS1
LM5(I)=I
20 LM5(I)=1
110 CONTINUE
CALL ASSE(LM1,NHALF,S5,S6,B8,B4,LMS,MS1,MS51,F,F4)
IF (MS.EQ.0) GO TO 120

```

```

SUBROUTINE END1(NEL,LM,M,ND,ND2)
DIMENSION LM(1)
DO 15 I=1,ND2
LM(I) = (NEL-1)*ND+I
15 LM(ND2+1) = (2*M-NEL+1)*ND+1
RETURN
END

```

```

SUBROUTINE END2(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
DIMENSION LM(ND4)
IF(NEL.NE.1) GO TO 500
DO 25 I=1,ND
LM(I) = I
25 LM(ND3+1) = (M+2)*ND+I
GO TO 500

```

```

SUBROUTINE LVL4(INTAP,ND,NS,NHM,NHALF,NHALFS,MEND,NCOP,
1 NCUP4,NEW,MS51,LM1,LM2,LMS,S5,S6,B8,B4,F,F4)
DIMENSION LM(1),LM2(1),S(1),S4(1),B(1),B4(1)
WRITE(6,902)
MS1=MS51=0
IF ONLY TWO ENDELEMENTS ARE SYMMETRIC
COUPLED READ FROM INTAP
IF (NCUP.NE.0) GO TO 100
REWIND INTAP
READ(INTAP) NET,NE
READ(INTAP) (S(I),I=1,NET)
READ(INTAP) (B(I),I=1,NET)
NHALF=NE/ND
IF (MS.EQ.0) GO TO 100
READ(INTAP) NST,NS1
READ(INTAP) (F(I),I=1,NS1)
NHALFS=NS1
100 CONTINUE
CALL CONC4(ND,NHM,NHALF,MEND,NDTOT,NEW,NCUP4,LM1,LM2)
NT=(NDTOT+1)*ND*101/2
DO 10 I=1,NT
54(I)=0.0
10 84(I)=0.0
IF (MS.EQ.0) GO TO 110
MS1=N51*2
MS51=NDTOT*MS51
DO 15 I=1,NS1
F4(I)=0.0
15 F4(I)=0.0
DO 20 I=1,NS1
LM5(I)=I
20 LM5(I)=1
110 CONTINUE
CALL ASSE(LM1,NHALF,S5,S6,B8,B4,LMS,MS1,MS51,F,F4)
IF (MS.EQ.0) GO TO 120

```

```

SUBROUTINE END1(NEL,LM,M,ND,ND2)
DIMENSION LM(1)
DO 15 I=1,ND2
LM(I) = (NEL-1)*ND+I
15 LM(ND2+1) = (2*M-NEL+1)*ND+1
RETURN
END

```

```

SUBROUTINE END2(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
DIMENSION LM(ND4)
IF(NEL.NE.1) GO TO 500
DO 25 I=1,ND
LM(I) = I
25 LM(ND3+1) = (M+2)*ND+I
GO TO 500

```

```

SUBROUTINE LVL4(INTAP,ND,NS,NHM,NHALF,NHALFS,MEND,NCOP,
1 NCUP4,NEW,MS51,LM1,LM2,LMS,S5,S6,B8,B4,F,F4)
DIMENSION LM(1),LM2(1),S(1),S4(1),B(1),B4(1)
WRITE(6,902)
MS1=MS51=0
IF ONLY TWO ENDELEMENTS ARE SYMMETRIC
COUPLED READ FROM INTAP
IF (NCUP.NE.0) GO TO 100
REWIND INTAP
READ(INTAP) NET,NE
READ(INTAP) (S(I),I=1,NET)
READ(INTAP) (B(I),I=1,NET)
NHALF=NE/ND
IF (MS.EQ.0) GO TO 100
READ(INTAP) NST,NS1
READ(INTAP) (F(I),I=1,NS1)
NHALFS=NS1
100 CONTINUE
CALL CONC4(ND,NHM,NHALF,MEND,NDTOT,NEW,NCUP4,LM1,LM2)
NT=(NDTOT+1)*ND*101/2
DO 10 I=1,NT
54(I)=0.0
10 84(I)=0.0
IF (MS.EQ.0) GO TO 110
MS1=N51*2
MS51=NDTOT*MS51
DO 15 I=1,NS1
F4(I)=0.0
15 F4(I)=0.0
DO 20 I=1,NS1
LM5(I)=I
20 LM5(I)=1
110 CONTINUE
CALL ASSE(LM1,NHALF,S5,S6,B8,B4,LMS,MS1,MS51,F,F4)
IF (MS.EQ.0) GO TO 120

```

```

SUBROUTINE END1(NEL,LM,M,ND,ND2)
DIMENSION LM(1)
DO 15 I=1,ND2
LM(I) = (NEL-1)*ND+I
15 LM(ND2+1) = (2*M-NEL+1)*ND+1
RETURN
END

```

```

SUBROUTINE END2(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
DIMENSION LM(ND4)
IF(NEL.NE.1) GO TO 500
DO 25 I=1,ND
LM(I) = I
25 LM(ND3+1) = (M+2)*ND+I
GO TO 500

```

```

SUBROUTINE LVL4(INTAP,ND,NS,NHM,NHALF,NHALFS,MEND,NCOP,
1 NCUP4,NEW,MS51,LM1,LM2,LMS,S5,S6,B8,B4,F,F4)
DIMENSION LM(1),LM2(1),S(1),S4(1),B(1),B4(1)
WRITE(6,902)
MS1=MS51=0
IF ONLY TWO ENDELEMENTS ARE SYMMETRIC
COUPLED READ FROM INTAP
IF (NCUP.NE.0) GO TO 100
REWIND INTAP
READ(INTAP) NET,NE
READ(INTAP) (S(I),I=1,NET)
READ(INTAP) (B(I),I=1,NET)
NHALF=NE/ND
IF (MS.EQ.0) GO TO 100
READ(INTAP) NST,NS1
READ(INTAP) (F(I),I=1,NS1)
NHALFS=NS1
100 CONTINUE
CALL CONC4(ND,NHM,NHALF,MEND,NDTOT,NEW,NCUP4,LM1,LM2)
NT=(NDTOT+1)*ND*101/2
DO 10 I=1,NT
54(I)=0.0
10 84(I)=0.0
IF (MS.EQ.0) GO TO 110
MS1=N51*2
MS51=NDTOT*MS51
DO 15 I=1,NS1
F4(I)=0.0
15 F4(I)=0.0
DO 20 I=1,NS1
LM5(I)=I
20 LM5(I)=1
110 CONTINUE
CALL ASSE(LM1,NHALF,S5,S6,B8,B4,LMS,MS1,MS51,F,F4)
IF (MS.EQ.0) GO TO 120

```

```

END4 6 R2=(M/2)*1.0
END4 7 IF(L1-LO.PZ) GO TO 800
END4 8 WRITE(6,910)
END4 9 STOP
END4 10 800 CONTINUE
END4 11 IF(INEL.GE.(M/2)) GO TO 700
END4 12 DO 48 I=1,NDZ
END4 13 LM(I)=(NEL-1)*ND+I
END4 14 46 LM(NDZ+1)=(2*M-NEL+1)*ND+I
END4 15 GO TO 790
END4 16 C
END4 17 700 IF(NEL.NE.(M/2)) GO TO 710
END4 18 DO 52 I=1,ND
END4 19 LM(I)=(NEL-1)*ND+I
END4 20 LM(ND+1)=NEL*ND+I
END4 21 LM(NDZ+1)=(M+1)*ND+I
END4 22 52 LM(NDZ+1)=(1.5*M+2)*ND+I
END4 23 GO TO 790
END4 24 C
END4 25 710 IF(NEL.NE.(M/2+1)) GO TO 720
END4 26 DO 56 I=1,ND
END4 27 LM(I)=(NEL-1)*ND+I
END4 28 LM(ND+1)=NEL*ND+I
END4 29 LM(NDZ+1)=(1.5*M+1)*ND+I
END4 30 56 LM(NDZ+1)=(M+1)*ND+I
END4 31 GO TO 790
END4 32 C
END4 33 720 DO 58 I=1,NDZ
END4 34 LM(I)=(NEL-1)*ND+I
END4 35 58 LM(NDZ+1)=(2*M-NEL+2)*ND+I
END4 36 790 CONTINUE
END4 37 NNODES=M+1+I
END4 38 910 FORMAT(12H NNN NOT ODD )
END4 39 RETURN
END4 40 END

SUBROUTINE EN06(NEL,LM,M,NDZ,ND3,ND4,NODES)
DIMENSION LM(ND4)
M1=1
M2=2
M3=M-1
M4=M
IF(MLL.EQ.M1) GO TO 10
IF(MLE.EQ.M2) GO TO 20
IF(MEL.EQ.M3) GO TO 30
IF(MEL.FU.M4) GO TO 40
DO 12 I=1,ND
LM(I)=1
LM(ND+1)=ND+1
LM(NDZ+1)=(M+1)*ND+1
CONTINUE
GO TO 100
DO 22 I=1,ND
LM(I)=ND+I
END06 1
END06 2 C
END06 3 DIMENSION LM(ND4)
END06 4 M1=1
END06 5 M2=2
END06 6 M3=M-1
END06 7 M4=M
END06 8 IF(MLL.EQ.M1) GO TO 10
END06 9 IF(MLE.EQ.M2) GO TO 20
END06 10 IF(MEL.EQ.M3) GO TO 30
END06 11 IF(MEL.FU.M4) GO TO 40
END06 12 GO TO 50
END06 13 DO 12 I=1,ND
END06 14 LM(I)=1
END06 15 LM(ND+1)=ND+1
END06 16 LM(NDZ+1)=(M+1)*ND+1
END06 17 LM(NDZ+1)=(2*M+1)*ND+1
END06 18 CONTINUE
END06 19 GO TO 100
END06 20 C
END06 21 DO 22 I=1,ND
END06 22 LM(I)=ND+I
END06 23

```

```

END3 1 SUBROUTINE EN03(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
END3 2 DIMENSION LM(ND4)
END3 3 IF(NEL.NE.1) GO TO 500
END3 4 DO 34 I=1,ND
END3 5 LM(I)=1
END3 6 LM(ND+1)=ND+1
END3 7 LM(NDZ+1)=(2*M+1)*ND+1
END3 8 34 LM(NDZ+1)=(M+1)*ND+1
END3 9 GO TO 690
END3 10 C
END3 11 500 IF(NEL.GE.(M/2)) GO TO 610
END3 12 DO 36 I=1,NDZ
END3 13 LM(I)=(NEL-1)*ND+I
END3 14 LM(ND+1)=NEL*ND+I
END3 15 36 LM(NDZ+1)=(2*M-NEL+2)*ND+1
END3 16 GO TO 690
END3 17 C
END3 18 610 IF(NEL.NE.(M/2)) GO TO 620
END3 19 DO 38 I=1,ND
END3 20 LM(I)=(NEL-1)*ND+I
END3 21 LM(ND+1)=(NEL)*ND+I
END3 22 LM(NDZ+1)=(M+2)*ND+I
END3 23 38 LM(NDZ+1)=(1.5*M+3)*ND+I
END3 24 GO TO 690
END3 25 C
END3 26 620 IF(NEL.NE.(M/2+1)) GO TO 630
END3 27 DO 40 I=1,ND
END3 28 LM(I)=(NEL-1)*ND+I
END3 29 LM(ND+1)=(NEL)*ND+I
END3 30 LM(NDZ+1)=(1.5*M+2)*ND+I
END3 31 40 LM(NDZ+1)=(M+2)*ND+I
END3 32 GO TO 690
END3 33 C
END3 34 630 IF(NEL.EQ.M) GO TO 640
END3 35 DO 42 I=1,NDZ
END3 36 LM(I)=(NEL-1)*ND+I
END3 37 42 LM(NDZ+1)=(2*M-NEL+3)*ND+I
END3 38 GO TO 690
END3 39 C
END3 40 640 DO 44 I=1,ND
END3 41 LM(I)=(NEL-1)*ND+I
END3 42 LM(ND+1)=(NEL)*ND+I
END3 43 LM(NDZ+1)=(NEL+1)*ND+I
END3 44 44 LM(NDZ+1)=(M+4)*ND+I
END3 45 690 CONTINUE
END3 46 NNODES=M+1+3
END3 47 WRITE(6,903)
END3 48 903 FORMAT(7H TYPE 3)
END3 49 RETURN
END3 50 END

SUBROUTINE EN04(NEL,LM,M,ND,ND2,ND3,ND4,NODES)
DIMENSION LM(ND4)
M1=M/2
M2=M/2

```

```

NUS2 6
NUS2 7
NUS2 8
NUS2 9
NUS2 10
NUS2 11
NUS2 12
NUS2 13
NUS2 14
NUS2 15
NUS2 16
NUS2 17
NUS2 18
NUS2 19
NUS2 20
NUS2 21
NUS2 22
NUS2 23
NUS2 24
NUS2 25

```

```

M2=NI
N3=3*ND*NHN
DU 10 I=1,M
LMI(I)=N+1
LM2(I)=I
LM2(N+1)=N2+1
10 CONTINUE
IC=0
NHN3=N3
DU 20 ICOUNT=1,NHN
DU 30 I=1,ND
IC=IC+1
LMI(IC)=NHN3-NH+1
30 CONTINUE
NHN3=NHN3-ND
20 CONTINUE
WRITE(6,950)
950 FORMAT(11H END NUS10Z)
RETURN
END

```

```

SUBROUTINE SINOZ(ND,NHN,I1,LM1,LM2,N1,N2,N3)
DIMENSION LMI(1),LM2(1)
M=ND*NHN
N2=2*ND*NHN
N3=3*ND*NHN
NM=M-2*ND
IS=(2*(I1-1)-1)*ND
N1=N2+2*IS
N2=N2+4*IS
N3=N3+4*IS

```

```

SIN2 1 C
SIN2 2 C
SIN2 3
SIN2 4
SIN2 5
SIN2 6
SIN2 7
SIN2 8
SIN2 9
SIN2 10
SIN2 11
SIN2 12 C
SIN2 13 C
SIN2 14 C
SIN2 15 C
SIN2 16 C
SIN2 17 C
SIN2 18 C
SIN2 19
SIN2 20
SIN2 21
SIN2 22
SIN2 23
SIN2 24
SIN2 25
SIN2 26
SIN2 27
SIN2 28
SIN2 29
SIN2 30
SIN2 31
SIN2 32
SIN2 33
SIN2 34
SIN2 35
SIN2 36
SIN2 37

```

```

IS=DOF OF HALF THE SJUL NUDES OF PREVIOUSLY
ASSEMBLED ELEMENT (NOT INCL. CORNER NUDES)
N1 NU OF DOF OF PREVIOUS ASSEM. SUP ELEM.
N2 NU OF DOF OF SUP ELEM ASSEM. IN PRESENT LOOP
N3 SAME AS N2, BUT INCL. INTERIOR DOF.
IF(I1.EQ.1) GO TO 100
DU 10 I=1,ND
LMI(I)=LM2(N2+IS-ND+1)+N2+3*IS+ND+1
LMI(N-ND+1)=LM2(N+1)+N+IS+1
10 CONTINUE
DU 20 I=1,N
LMI(N+1)=N+2*IS+ND+1
LM2(I)=LM2(I)
20 CONTINUE
DU 30 I=1,NM
LM2(N+1)=N+1+NM+1
30 CONTINUE
N1=NHN-2
IC=ND
NHN3=NHN
DU 33 ICOUNT=1,NHN
DU 36 I=1,ND
IC=IC+1
LMI(IC)=NHN3-ND+1

```

```

SUBROUTINE CONC2IC,ND,NHN,I1,LM1,LM2,N1,N2,N3)
DIMENSION LMI(1),LM2(1)
IF(IC.EQ.1)GO TO 120
SIFNUDES ARE NOT ELIMINATED
CALL SINOZ(ND,NHN,I1,LM1,LM2,N1,N2,N3)
GO TO 130
120 CONTINUE
SIFNUDES ARE ELIMINATED
CALL NUS12(ND,NHN,LM1,LM2,N1,N2,N3)
130 CONTINUE
RETURN
END

```

```

SUBROUTINE NUS12(ND,NHN,LM1,LM2,N1,N2,N3)
DIMENSION LMI(1),LM2(1)
M=ND*NHN
N1=2*ND*NHN

```



```

CON4 14 C
CON4 15
CON4 16
CON4 17
CON4 18
CON4 19
CON4 20
CON4 21
CON4 22
CON4 23
CON4 24
CON4 25
CON4 26
CON4 27
200 CALL SIND6 (ND, NHM, NHMF, HFN9, NDUT, NE, L1, LM2)
300 CONTINUE
    NHMF=NHMF+ND
    DU 10 I=1, NHMF
    10 LM2(I)=LM2(I)
    IF (ND.EQ.2) GO TO 400
    DU 20 I=1, NHMF, 3
    20 LM2(I)=LM2(I)
    IF (ND.EQ.3) GO TO 400
    400 CONTINUE
    NEND=NEND+ND
    RETURN
    END

```

```

SUBROUTINE NUS14 (ND, NHR, NHMF, NHRU, NDUT, NE, L1, LM2)
DIMENSION LM1(3), LM2(1)
NHR=NHR+ND
NE=(2*NHALF-2*NHR)*ND
NDUT=NE*NHD
NINT=NHR
NEND=NHD
NEND=NEND+ND
NSID=(NHMF+ND-NHD)/2
NNTD=NDUT
DU 15 I=1, NINT
    DU 20 I=1, ND
        10 I=I+1
    20 LM1(I)=NNTD-ND+I
    NNTD=NNTD-ND
    NH=(NHMF-NHR)*ND
    DU 30 I=1, NH
        30 LM1(NHD+I)=NEND+NSID+I
    DU 40 I=1, NHD
        40 LM2(I)=NH+I
    DL 50 I=1, NSID
        50 LM2(NHD+I)=NE-NSID+I
    NH=NH+ND*NSID
    DU 60 I=1, NH
        60 LM2(NHD+NSID+I)=I
    RETURN
    END

```

```

SUBROUTINE SIN4 (ND, NHR, NHMF, NHRU, NDUT, NE, L1, LM2)
DIMENSION LM1(1), LM2(1)
NHR=NHR+ND
NE=(2*NHALF-2*NHR)*ND
NDUT=NE*(NHR-2)*ND
NINT=NHR-2
NEND=(NHR-2)*ND
NEND=NEND+ND
NSID=(NHMF+ND-NHD)/2

```

```

SIN3 1 C
SIN3 2 C
SIN3 3 C
SIN3 4 C
SIN3 5 C
SIN3 6 C
SIN3 7 C
SIN3 8
SIN3 9
SIN3 10
SIN3 11
SIN3 12
SIN3 13
SIN3 14
SIN3 15
SIN3 16
SIN3 17
SIN3 18
SIN3 19
SIN3 20
SIN3 21
SIN3 22
SIN3 23
SIN3 24
SIN3 25
SIN3 26
SIN3 27 C
SIN3 28 C
SIN3 29 C
SIN3 30
SIN3 31
SIN3 32
SIN3 33
SIN3 34
SIN3 35
SIN3 36
SIN3 37
SIN3 38
SIN3 39
SIN3 40
SIN3 41
SUBROUTINE SIN03 (ND, NHM, NDUT, NEXT, LAST, NEXT, LM1, LM2)
DIMENSION LM1(1), LM2(1)
COUPLING TYPE 2
CONNECTIVITY ARRAY FOR PREVIOUS ELEMENT
    HINT=(NHM-2)*ND
    NEXT=NDUT-NINT
    IS2=NEXT/2-NHM*ND
    DU 55 I=1, ND
        55 LM1(I)=NEXT-IS2-ND+I
    10=ND
    NNTD=NDUT
    NHR2=NHM-2
    DU 60 I=1, NHR2
        60 I=I+1
    LM1(I)=NNTD-ND+I
    62 CONTINUE
    NNTD=NNTD-ND
    60 CONTINUE
    IR=LAST-NINT-ND
    DU 65 I=1, IR
        65 LM1(IR+I)=NEXT+I
    65 LM1(IR)=NEXT/2+I

```

```

CONNECTIVITY ARRAY FOR NEXT SUP. ELEM.
    IR=NEXT/2+ND
    DU 70 I=1, IR
        70 LM2(I)=I
    DU 75 I=1, NINT
        75 LM2(IR+I)=NEXT+I
    15=IR-NHM*ND
    IF (IS.EQ.0) RETURN
    DU 80 I=1, IS
        80 LM2(IR+I)=NEXT-IS+I
    RETURN
    END

```

```

SUBROUTINE CON4 (ND, NHR, NHMF, NHRU, NDUT)
1 NEW=NCOP4, LM1, LM2)
DIMENSION LM1(1), LM2(1)
IF (NCOP4.FO.1) GO TO 100
IF (NCOP4.FO.2) GO TO 200
COUPLING TYPE 1
100 CALL NUS14 (ND, NHM, NHMF, NHRU, NDUT, NL, LM1, LM2)
GO TO 300
COUPLING TYPE 2

```

```

FORM, COL, BY COL, FOR NUMBERING
SEE FIG. 3-3
RETURN
END

SUBROUTINE ESTRLS(NDU,NU,NSL,PA,AB,FA,AST,ETA)
DIMENSION FMS(N4),ASTI(1),ETA(1)
ELEMENT STRESS-DISPL. MATRIX IS CREATED
ND = NUMBER OF DEGREES OF FREEDOM PER NODE
VS = NUMBER OF TOTAL STRESS COMPONENTS IN ELEMENT
E = YOUNG'S MODULUS
P = POISSON'S RATIO
A = ELEMENT WIDTH
B = ELEMENT HEIGHT
F(1) = STRESS-DISPL. MATRIX STORED AS IN FIG. 3-B
ETA(1) = COORDINATE IN X-DIRECTION
RETURN
END

```

ELMA 13 C
ELMA 14 C
FLMA 15
ELMA 16

ESTR 1
ESTR 2 C
ESTR 3
ESTR 4 C
ESTR 5 C
ESTR 6 C
ESTR 7 C
ESTR 8 C
ESTR 9 C
ESTR 10 C
ESTR 11 C
ESTR 12 C
ESTR 13 C
ESTR 14 C
ESTR 15 C
ESTR 16
ESTR 17

```

SUBROUTINE ESTIF(T,P,AB,S)
DIMENSION S(1)
ELEMENT STIFFNESS MATRIX IS CREATED
ND = NUMBER OF DEGREES OF FREEDOM PER NODE
T = ELEMENT THICKNESS
E = YOUNG'S MODULUS
P = POISSON'S RATIO
A = ELEMENT WIDTH
B = ELEMENT HEIGHT
S(1) = STIFFNESS MATRIX STORED IN UPPER TRIANG.
FORM, COL, BY COL, FOR NUMBERING
SEE FIG. 3-3
RETURN
END

```

ESTI 1
ESTI 2 C
ESTI 3
ESTI 4 C
ESTI 5 C
ESTI 6 C
ESTI 7 C
ESTI 8 C
ESTI 9 C
ESTI 10 C
ESTI 11 C
ESTI 12 C
ESTI 13 C
ESTI 14 C
ESTI 15 C
ESTI 16
ESTI 17

```

SUBROUTINE ELMA(NDU,T,AP,AB,ST)
DIMENSION M(1)
ELEMENT MASS MATRIX IS CREATED
ND = NUMBER OF DEGREES OF FREEDOM PER NODE
TU = MASS DENSITY PER UNIT
A = ELEMENT WIDTH
B = ELEMENT HEIGHT
M(1) = MASS MATRIX STORED IN UPPER TRIANG.

```

ELMA 1
ELMA 2 C
ELMA 3
ELMA 4 C
ELMA 5 C
ELMA 6 C
ELMA 7 C
ELMA 8 C
ELMA 9 C
ELMA 10 C
ELMA 11 C
ELMA 12 C

```

SUBROUTINE COMP(LIMAS,ROUN,AL,TL,THI,HT,UB,DT)
DIMENSION B(1),B(11)
GO TO (101,102),LIMAS
101 WRITE(6,1001)
STOP
102 CONTINUE
RO=WT
DO 10 I=1,N
11=(I+1)*I/2
10 B(I)=B(11)
IF(RO,ME-3) GO TO 100
UPDATES CONSISTENT LUMPED MASS FROM CONSISTENT FULL
MASS FOR PLATE BENDING (ROT, DUF ASSUMED AS ZERO)
DO 15 I=1,N*J
TU=TO*B(11)
15 CONTINUE
HA=RO*THI*AL*YL
DO 20 I=1,N*J
IX=I*J
1X=1*J
1Y=1*J
B(11)=B(11)+HA/TU
B(11)=0.0
B(11)=0.0
20 CONTINUE
100 CONTINUE
11=(ROUN-2) GO TO 110

```

LUMD 1
LUMD 2 C
LUMD 3
LUMD 4
LUMD 5
LUMD 6
LUMD 7
LUMD 8
LUMD 9
LUMD 10
LUMD 11
LUMD 12
LUMD 13
LUMD 14 C
LUMD 15 C
LUMD 16 C
LUMD 17 C
LUMD 18
LUMD 19
LUMD 20
LUMD 21
LUMD 22
LUMD 23
LUMD 24
LUMD 25
LUMD 26
LUMD 27
LUMD 28
LUMD 29
LUMD 30 C
LUMD 31 C
LUMD 32 C
LUMD 33

```

SUBROUTINE ELMA(NDU,T,AP,AB,ST)
DIMENSION M(1)
ELEMENT MASS MATRIX IS CREATED
ND = NUMBER OF DEGREES OF FREEDOM PER NODE
TU = MASS DENSITY PER UNIT
A = ELEMENT WIDTH
B = ELEMENT HEIGHT
M(1) = MASS MATRIX STORED IN UPPER TRIANG.

```

ELMA 1
ELMA 2 C
ELMA 3
ELMA 4 C
ELMA 5 C
ELMA 6 C
ELMA 7 C
ELMA 8 C
ELMA 9 C
ELMA 10 C
ELMA 11 C
ELMA 12 C

```

SUBROUTINE ELMA(NDU,T,AP,AB,ST)
DIMENSION M(1)
ELEMENT MASS MATRIX IS CREATED
ND = NUMBER OF DEGREES OF FREEDOM PER NODE
TU = MASS DENSITY PER UNIT
A = ELEMENT WIDTH
B = ELEMENT HEIGHT
M(1) = MASS MATRIX STORED IN UPPER TRIANG.

```

ELMA 1
ELMA 2 C
ELMA 3
ELMA 4 C
ELMA 5 C
ELMA 6 C
ELMA 7 C
ELMA 8 C
ELMA 9 C
ELMA 10 C
ELMA 11 C
ELMA 12 C

```

25 1-1,NZ
1A=1
1Y=1+1
1X=10X+BT(1A)
1Y=10Y+BT(1Y)
25 CONTINUE
1A=10+111+XL*Y
10 30 1-1,NZ
1A=1
1Y=1+1
8(11)=BT(11)*RA/10X
8(11)=BT(11)*BA/10Y
30 CONTINUE
110 CONTINUE
1001 FORMAT( 30H CONSISTENT N. M. NOT INCLUDED)
RETURN
END

```

```

LUMD 34
LUMD 35
LUMD 36
LUMD 37
LUMD 38
LUMD 39
LUMD 40
LUMD 41
LUMD 42
LUMD 43
LUMD 44
LUMD 45
LUMD 46
LUMD 47
LUMD 48
LUMD 49
LUMD 50

```

```

SUBROUTINE STUREZ(MA,NALV,V,A)
DIMENSION V(LV),A(1)
REMIND NA
NI=(N+1)*N/2
IF(NE.LV) LV=NT
NF=NT/LV
Q=1.0
QNF=Q/NF
UNT=Q*NT
ULV=Q*LV
URF=Q*NT/ULV
LVLAST=NT-NF*LV
NFI=NF+1
IF(QNF.LV.QINF) NFI=NF
IF(QNF.EQ.QINF) LVLAST=LV
NI=NZ=0
NI=(1+1)*LV*1
N2=1*LV
IF(1.EQ.NF) N2=(1+1)*LV+LVLAST
WRITE(MA) (A(J),J=N2,NZ)
10 CONTINUE
PELIND NA
1000
IR=0
1A=0
10 20 1-1,N
11-1-1
LUC=(1+1)*1/2+(1+1)*1
1-1*1
10 CONTINUE
WRITE(MA) (A(J),J=N2,NZ)
1001 FORMAT( 30H CONSISTENT N. M. NOT INCLUDED)
RETURN
END

```

```

STUR 1
STUR 2
STUR 3
STUR 4
STUR 5
STUR 6
STUR 7
STUR 8
STUR 9
STUR 10
STUR 11
STUR 12
STUR 13
STUR 14
STUR 15
STUR 16
STUR 17
STUR 18
STUR 19
STUR 20
STUR 21
STUR 22
STUR 23
STUR 24
STUR 25
STUR 26
STUR 27
STUR 28
STUR 29
STUR 30
STUR 31
STUR 32
STUR 33
STUR 34
STUR 35
STUR 36
STUR 37
STUR 38
STUR 39
STUR 40

```

```

SUBROUTINE STUREZ(MA,NALV,V,A)
DIMENSION V(LV),A(1)
REMIND NA
NI=(N+1)*N/2
IF(NE.LV) LV=NT
NF=NT/LV
Q=1.0
QNF=Q/NF
UNT=Q*NT
ULV=Q*LV
URF=Q*NT/ULV
LVLAST=NT-NF*LV
NFI=NF+1
IF(QNF.LV.QINF) NFI=NF
IF(QNF.EQ.QINF) LVLAST=LV
NI=NZ=0
NI=(1+1)*LV*1
N2=1*LV
IF(1.EQ.NF) N2=(1+1)*LV+LVLAST
WRITE(MA) (A(J),J=N2,NZ)
10 CONTINUE
PELIND NA
1000
IR=0
1A=0
10 20 1-1,N
11-1-1
LUC=(1+1)*1/2+(1+1)*1
1-1*1
10 CONTINUE
WRITE(MA) (A(J),J=N2,NZ)
1001 FORMAT( 30H CONSISTENT N. M. NOT INCLUDED)
RETURN
END

```

```

STUR 41
STUR 42
STUR 43
STUR 44
STUR 45
STUR 46
STUR 47
STUR 48
STUR 49
STUR 50
STUR 51
STUR 52
STUR 53

```

```

SUBROUTINE WRITEZ(MN2,ISH,HA)
MN2 ...NUMBER OF COLUMNS TO BE PRINTED
UN ONE PAGE,BY DEFAULT = 12
ISH ...EQ.0 NU ZERO KJ45 PRINTED ULLUW (MZN,NR?)
N ...ORDER OF MATRIX
V ...WORKING VECTOR (MAX. 12 COLUMNS)
DIMENSION V(12),A(1)
IF(MN2.FU.0) MN2=12
NT=(M+1)*N/2
Q=1.0
NB=M/2
JINH=NB*U
UN=H+Q
MN2=MN2+Q
UNB=H/2*Q
NB=NB+1
IF(QINH.EQ.UNB) NB=NB-1
WRITE(6,VZ0)
00 10 K=1,NB
J1=MN2*(K-1)+1
J2=MN2*K
N2=MN2
NV=N
IF(ISH.EQ.0) NN=J2
IF(K.EQ.NB) NN=N
IF(J2.GT.N) N2=N-J1+1
00 20 1-1,MN
00 30 J=1,M2
11-1-1
LUC=(1+1)*1/2+(1+1)*N-111
IF(J2.LT.1) GU TO 100
IF(1.GT.J1.AND.1.LT.J2) GU TO 110
IF(J1.EE.1) GU TO 120
GU TO 100
110 N1=J1+1
00 40 J=N1,M2
40 V(J)=A(LUC-N1+1+J)
GU TO 100
120 CONTINUE
00 50 J=1,M2
50 V(J)=A(LUC+J1-1+J)
100 CONTINUE
WRITE(6,VZ0)
WRITE(6,VZ0)
10 CONTINUE
000 FORMAT(12E11.4)
420 FORMAT(1H)
K1=UPH
K1=UPH
END

```

```

WRITE 1
WRITE 2
WRITE 3
WRITE 4
WRITE 5
WRITE 6
WRITE 7
WRITE 8
WRITE 9
WRITE 10
WRITE 11
WRITE 12
WRITE 13
WRITE 14
WRITE 15
WRITE 16
WRITE 17
WRITE 18
WRITE 19
WRITE 20
WRITE 21
WRITE 22
WRITE 23
WRITE 24
WRITE 25
WRITE 26
WRITE 27
WRITE 28
WRITE 29
WRITE 30
WRITE 31
WRITE 32
WRITE 33
WRITE 34
WRITE 35
WRITE 36
WRITE 37
WRITE 38
WRITE 39
WRITE 40
WRITE 41
WRITE 42
WRITE 43
WRITE 44
WRITE 45
WRITE 46
WRITE 47
WRITE 48
WRITE 49
WRITE 50
WRITE 51
WRITE 52
WRITE 53

```



```

ELM9 1 SUBROUTINE ELM9
ELM9 2 C
ELM9 3 C VARIABLE NODE PANEL ELEMENT
ELM9 4 C
ELM9 5 COMMON A(1)
ELM9 6 COMMON /ELPAR/ NPAK(14),NUMNP,MBAND,MELTYP,N1,N2,N3,
ELM9 7 N4,N5,MIDT,NEO,MIDTEM
ELM9 8 LT,LH,DUM(98),N6,N7,N8,N9,N10,LM(100)
ELM9 9 COMMON /FTRK/ MODER,MTR
ELM9 10 COMMON /EM/ AA(1)
ELM9 11 WRITE(6,902)
ELM9 12 NPAK=12
ELM9 13 IF INPAR(1)=0,0) GO TO 500
ELM9 14 WRITE(6,910)
ELM9 15 WRITE(6,920)
ELM9 16 N6=N5+NUMNP
ELM9 17 R1=N6/NPAK(3)
ELM9 18 RH=N7/NPAK(3)
ELM9 19 R9=N8/NPAK(3)
ELM9 20 N10=N9/NPAK(3)
ELM9 21 C
ELM9 22 C CHECK STORAGE LENGTH OF COMMON /EM/ AAR(10)EM)
ELM9 23 C
ELM9 24 I1=EMTUTEM-6E,N10) GO TO 110
ELM9 25 MAX=N10-MIDTEM
ELM9 26 WRITE(6,925)MAX,N10
ELM9 27 STOP
ELM9 28 110 CONTINUE
ELM9 29 DO 10 I=1,N10
ELM9 30 10 A(I)=A(I)
ELM9 31 C
ELM9 32 CALL OVERLAY(4HSSAP,8,0,0HPCALL)
ELM9 33 DO 20 I=1,N10
ELM9 34 20 A(I)=A(I)
ELM9 35 GO TO 850
ELM9 36 C
ELM9 37 C STRESS CALCULATION FOR STATIC LOADING
ELM9 38 C
ELM9 39 500 CONTINUE
ELM9 40 REWIND NPAK
ELM9 41 IL=0
ELM9 42 DO 30 L=1,LH
ELM9 43 IL=IL+1
ELM9 44 I1=N3*(IL-1)+NEU
ELM9 45 I2=N3*IL+NEG-1
ELM9 46 WRITE(MUL) (A(I),I=1,12)
ELM9 47 CONTINUE
ELM9 48 NUME=NPAK(2)
ELM9 49 DO 800 MM=1,NUME
ELM9 50 REWIND NPAK
ELM9 51 READ(I) NUMB,NUMP,N5,NSS,NST
ELM9 52 IEMS=EQ,0) GO TO 800
ELM9 53 READ(I) (LM(I),I=1,HRUN),(ALL),I=1,NST)
ELM9 54 WRITE(6,940) NUMP,N5,NSS,NST
ELM9 55 WRITE(6,940) (LM(I),I=1,HRUN)
ELM9 56 C
ELM9 57 C A( I TO NST) STORES NUM STRESS/USPL. -MATRIX
ELM9 58 C
ELM9 59 WRITE(6,910)
ELM9 60 WRITE(6,932) NUMB

```

```

WRITE(6,934)
DO 700 L=1,LH
REWRITE(1) (A(I),I=1,NEJ)
WRITE(6,935) (A(I),I=1,NEJ)
AA( 1 STORES NUM DISPLACEMENTS FOR LOADCASE L
A(LU) TO LU2) STORES NEW CALCULATED STRESSL)
DO 32 I=1,NSS
I1=NST+1
32 A(I1)=0.0
LOI=NST+1
LU2=NST+NSS
DO 720 K=1,NSS
DO 720 I=1,HRUN
IS=LOI+K-1
ISD=(I-1)*NSS+K
10=LM(I)
PREID=EQ,0) GO TO 720
A(I1)=A(I1)+A(ISD)*AA(MU)
720 CONTINUE
NEL=HSS/MS
WRITE(6,912)
DO 35 K=1,NEL
I1=(K-1)*NS+LOI
I2=K*NS+LOI-1
WRITE(6,942) K,L,(A(I),I=1,12)
35 CONTINUE
WRITE(6,913)
700 CONTINUE
800 CONTINUE
850 CONTINUE
WRITE(6,930)
902 FORMAT(10H IN ELM9)
910 FORMAT(1H)
912 FORMAT(17)
913 FORMAT(17)
925 FORMAT(2E11.4)
930 FORMAT(39H STRESS F U R P A N E L E M E N T (J))
932 FORMAT(18H ELEMENT LOAD,7)
934 FORMAT(18H ELEMENT I M H N U M B E R (C A S E ,7)
942 FORMAT(16,5X,16,2X,9E11.4,7,17X,9E11.4,7,17X,9E11.4)
920 FORMAT(36H E L E M E N T - T Y P E F L N H I 9)
23H P A N E L E M E N T (17)
930 FORMAT(10H END ELM9)
950 FORMAT(5,6E11.4)
955 FORMAT(20H INCREASE NUMBER AT (6,9H) ...N10, 10)
RETURN
END

```