

A Case Study of the Effects  
of Enclosure Elements on  
Energy Consumption in a  
High-rise Office Building

Allen J. Hanley

A Major Technical Report  
in  
the Centre  
for  
Building Studies

Presented in Partial Fulfillment of the Requirements  
for the Degree of Master of Engineering (Building)  
at  
Concordia University  
Montréal, Québec, Canada

December 1983

© Allen J. Hanley, 1983

1

**ABSTRACT**

**A Case Study of the Effects of  
Enclosure Elements on Energy Consumption  
in a high-rise office building**

Allen J. Hanley, Eng.

This report presents a case study of the effects of certain building envelope parameters on the annual energy requirements of a high-rise office building.

The major objective of this study is to evaluate the impact of wall insulation thickness, and the quality of the fenestration on the building energy requirements, so that the planning team can assess the economical feasibility of these various alternatives in the early stages of designing an energy efficient building.

The energy requirements estimates for various composite wall constructions were obtained by using a Commercial Program as made available by Public Works Canada.

The results of the analysis of eleven different composite wall assemblies are tabulated and indicate graphically the variation of energy use that follows increasing thickness of wall insulation and changes in fenestration.

Acknowledgements

The writer would like to acknowledge the direction of the Center for Building Studies who made the budget available for the computer time necessary for this analysis, as well as, the guidance, comments and criticism of Prof. Mal Turaga and other members of the faculty and staff of the Center for Building Studies of Concordia University.

iii  
TABLE OF CONTENTS

	<u>PAGE</u>
TITLE	
ABSTRACT	1
ACKNOWLEDGEMENTS	ii
TABLE OF CONTENTS	iii
TABLE OF SYMBOLS	v
LIST OF FIGURES	iv
LIST OF TABLES	viii
CHAPTER 1. INTRODUCTION	1
1.0 Introduction	2
CHAPTER 2. DESCRIPTION OF THE BUILDING AND OF THE MECHANICAL SYSTEM	8
2.1 General description of the building	9
2.2 Building zoning	9
2.3 Control system and sequence of operation	16
CHAPTER 3. DESCRIPTION OF THE ENERGY REQUIREMENT ESTIMATE PROGRAM	20
3.0 Public works Canada and the ESA	21
3.1 Applications of the ESA in design	22
3.2 The ESA Programs	24

TABLE OF CONTENTS

(CONT'D)

	<u>PAGE</u>
CHAPTER 4.	INPUT DATA USED IN ANALYSIS 29
4.1 Input data	30
4.2 Limitations to the program	44
CHAPTER 5.	SUMMARY AND ANALYSIS OF RESULTS 48
5.1 Summary and discussion of results	49
5.2 Cost analysis	63
CHAPTER 6.	CONCLUSIONS AND RECOMMENDATIONS 71
REFERENCES	74
APPENDIX A-1	Print out of 2 runs of ERE analysis 76
APPENDIX A-2	Glossary to ERE print-outs 98

LIST OF SYMBOLS

Af	floor area (SQ-Ft)
Ag	glazed area (SQ-Ft)
Ar	roof area (SQ-Ft)
Aw	area of curtain wall (SQ-Ft)
BTU	British thermal unit
CFM	cubic foot per minute
Cp	specific heat of air (Btu/lbs. DEG-F)
CD	clear double - glazing
CT	clear triple - glazing.
ERE	energy requirement estimate
'F	degree fahrenheit
ho	BTU/(hr. $F_t^2$ °F)
HVAC	heating, ventilation and air conditioning
kW	kilowatt
MBH	thousand Btu per hour
MPH	miles per hour
RD	reflective double - glazing
S	density of air (lbs/CU-Ft)
SC	shading coefficient of glass (dimensionless)
SHGF	solar heat gain factor (Btu/hr . SQ-Ft)
toa	outdoor air temperature (DEG-F)
tr	room temperature (DEG-F)
ts	supply air temperature (DEG-F)
TD	tinted double - glazing (solar bronze)
Ug	U-value of glazing (Btu / hr . SQ-Ft . DEG-F)
Ur	U-value of roof (Btu/Hr . SQ-Ft . DEG-F)
Uw	U-value of curtain wall (Btu/hr . SQ-Ft . DEG-F)
VAV	variable air volume system

LIST OF FIGURES

- FIGURE 1              Typical floor area  
(arrangement of thermal blocks or zones)
- FIGURE 2              Typical wall section
- FIGURE 3              Variable air volume - air conditioning system schematic
- FIGURE 4              Convector water temperature outdoor and indoor schedule
- FIGURE 5              Typical ventilation riser diagram showing static pressure controllers
- FIGURE "A"            The energy system analysis series
- FIGURE 5.1           Comparative annual energy consumption for various wall assemblies
- FIGURE 5.2           Comparative total annual energy consumption for various wall assemblies
- FIGURE 5.3           Effects of wall insulation thickness on the annual heating energy consumption
- FIGURE 5.4           Effects of glazing variations on annual energy consumption

LIST OF FIGURES

(CONT'D)

**FIGURE 5.5**      Typical monthly load variations  
                          for alternative N°11

**FIGURE 5.6**      Typical monthly zone-by-zone  
                          cooling load variations

LIST OF TABLES

<u>TABLE</u>	<u>TITLE</u>
1	Glazing and wall insulation thickness variations considered in the analysis
2	Average coefficient "U" (Summer and winter) for the composite wall
3	Values of coefficient "U" used
4	Summary of major specifications
5	Sample calculation for infiltration, through the curtain wall for the entire building
6	Energy requirement analysis input data needed
7	Summary of duput data
5.1	Annual energy consumption summary for various wall assemblies
5.2	Comparative annual heating energy and cost for various wall assemblies
5.3	Comparative annual cooling energy and cost for various wall assemblies

LIST OF TABLES

(CONT'D)

<u>TABLE</u>	<u>TITLE</u>
5.4	Comparative electrical energy consumption for various wall assemblies
5.5	Comparative total energy consumption and cost for various wall assemblies
5.6	Heating plant capacity and cost data
5.7	Glass type vs chiller capital costs 1981
5.8	Comparative costs (1981) wall panel assemblies
5.9	Comparative cost data and payback period
5.10	Discounted payback period for the alternative for different energy cost levels

**CHAPTER 1**  
**INTRODUCTION**

1.0 INTRODUCTION

In 1980, an opportunity was presented at the Center for Building Studies to perform an energy analysis for a major office building, that was in the early stages of construction. The HVAC system designs for the building were completed, but there was still time to make modifications to the thickness of the insulation in the curtain walls or to alter the type of fenestration, if these changes could be economically justified.

The study was therefore, to examine the impact of changes in the various components of the vertical building envelope on annual energy requirements..

The effects of eleven combinations of curtain wall insulation and glazing types, on the annual energy consumption, using a commercially available energy requirement estimate program, were analyzed in this study.

The results of the analysis i.e. the variation of the annual heating, cooling and total energy requirement changes in the enclosure elements were analyzed.

Details of the energy requirement estimation program were presented in Chapter 3.

1.1      Enclosure element variations considered  
in the analysis

The existing building envelop or skin, has a metal curtain wall, with 4" compressed glass fiber insulation, in the non glazed areas. The glazed areas are double glazed with sealed units, with solar bronze outer, and, clear inner panes. The details of the variation were shown in Tables 1, 2 and 3.

1.2      The major findings of the study can be  
summarized as follows:

1- As expected, the heating requirements for the building constituted a major portion of the total annual energy requirements, due to the climatic conditions for this location; consequently, for a given glazing percentage, the wall insulation thickness is the most significant parameter in determining the total energy requirement.

2- Among the glazing types considered, as expected, the clear triple glazing (highest thermal resistance) with 6" - wall insulation (highest thermal resistance) provided the least total

TABLE 1 - GLAZING AND WALL INSULATION VARIATIONS  
CONSIDERED IN THE ANALYSIS

VARIATION NO.	SYMBOL	GLAZING TYPE	GLASS FIBER WALL INSULATION THICKNESS
1	TD	Solar bronze double	4" Base case
2	TD	Solar bronze double	2"
3	TD	Solar bronze double	6"
4	CD	Double clear	4"
5	CT	Triple clear	4"
6	CD	Double clear	2"
7	CT	Triple clear	2"
8	CT	Triple clear	6"
9	RD	Reflective film double	4"
10	RD	Reflective film double	6"
11	CD	Double clear	6"

TABLE 2 - AVERAGE COEFFICIENT "U" SUMMER & WINTER  
FOR COMPOSITE WALL

RUN	Wall Insul.	Glazing	Average			Coefficient "U"			Overall Coefficient "U"			RUN		
			X	%	Wall Coeff.	Winter Glass Coef.f.	%	Summer Glass Coeff.	%	Winter Area	Summer Area			
1	4"	T.D.	.06	.42	.029	.43	.52	.224	.45	.52	.234	.253	.263	1
2	2"	T.D.	.114	"	.055	.43	"	.224	.45	"	.234	.279	.289	2
3	6"	T.D.	.04	"	.019	.43	"	.224	.45	"	.234	.243	.253	3
4	4"	C.D.	.06	"	.029	.49	"	.255	.56	"	.291	.284	.320	4
5	4"	C.T.	.06	"	.029	.31	"	.161	.39	"	.203	.190	.232	5
6	2"	C.D.	.114	"	.055	.49	"	.255	.56	"	.291	.310	.346	6
7	2"	C.D.	.114	"	.055	.31	"	.161	.39	"	.203	.216	.258	7
8	6"	C.T.	.04	"	.019	.31	"	.161	.39	"	.203	.180	.222	8
9	4"	R.D.	.06	"	.029	.43	"	.224	.37	"	.192	.253	.221	9
10	6"	R.D.	.04	"	.019	.43	"	.224	.37	"	.192	.243	.211	10
11	6"	C.D.	.04	"	.019	.49	"	.255	.56	"	.291	.274	.310	11

T.D. = Tinted double glazing - solar bronze

C.D. = Clear double glazing

C.T. = Clear triple glazing

R.D. = Reflective double glazing

R.T. = Reflective triple glazing

U : coefficient expressed in Btu / (hour) (square ft.)  
difference in temperature between the air on the  
two sides.

TABLE 3  
VALUES OF COEFFICIENT "U"

	SC *	U (Winter)	U (Summer)
Double Clear Glazing 2 x $\frac{1}{4}$ " X $\frac{1}{2}$ " Airspace	0.88	0.49	0.56
Triple Clear Glazing	0.71	0.31	0.39
Solar Bronze Outside Clear Inside	0.58	0.43	0.45
Double Glazing			
Double Clear Glazing with bronze film on inside of outer pane	0.38	0.43	0.37
2" Fiberglass Wall Insulation		0.114	
4" "		0.06	
6" "		0.04	

\* SC Shading Coefficient

energy, which is the alternative for the least heating requirement. However the least cooling energy requirement was provided by the alternative N°9 (reflective double glazing with 4" wall insulation), due to the solar heat reflection characteristics of the glazing. Consequently, one can conclude that, for cases where the cooling loads is predominant, the reflective double glazing may be the better choice.

3- Based on the discounted pay back period analysis, for this project, none of the enclosure elements considered are economical, at the energy costs considered, which also indicates that the original design of the building (base case - alternative N°1) is energy effective.

**CHAPTER 2**  
**DESCRIPTION OF THE BUILDING AND**  
**THE MECHANICAL SYSTEM**

2.0 BUILDING AND MECHANICAL SYSTEM DESCRIPTION

2.1 General description of the building.

The building analyzed is a 29 story office building situated in Western Canada and is part of a larger complex comprising a podium 3 stories high, i.e. basement, first and second floors, commercial areas and an adjacent parking garage. The analysis is limited to the 25 office floors only.

2.2 Building zoning

The typical floor plan is shown in Figure 1, with the sub-divisions of the floor area into North, South, East, West and Core Zones, again for purposes of energy requirement estimation.

The core area covers the rentable area in the interior zone but does not include elevator area, wash rooms, duct shafts, staircases as indicated.

This core area, as such, has no outside walls, hence no envelope losses or gains.

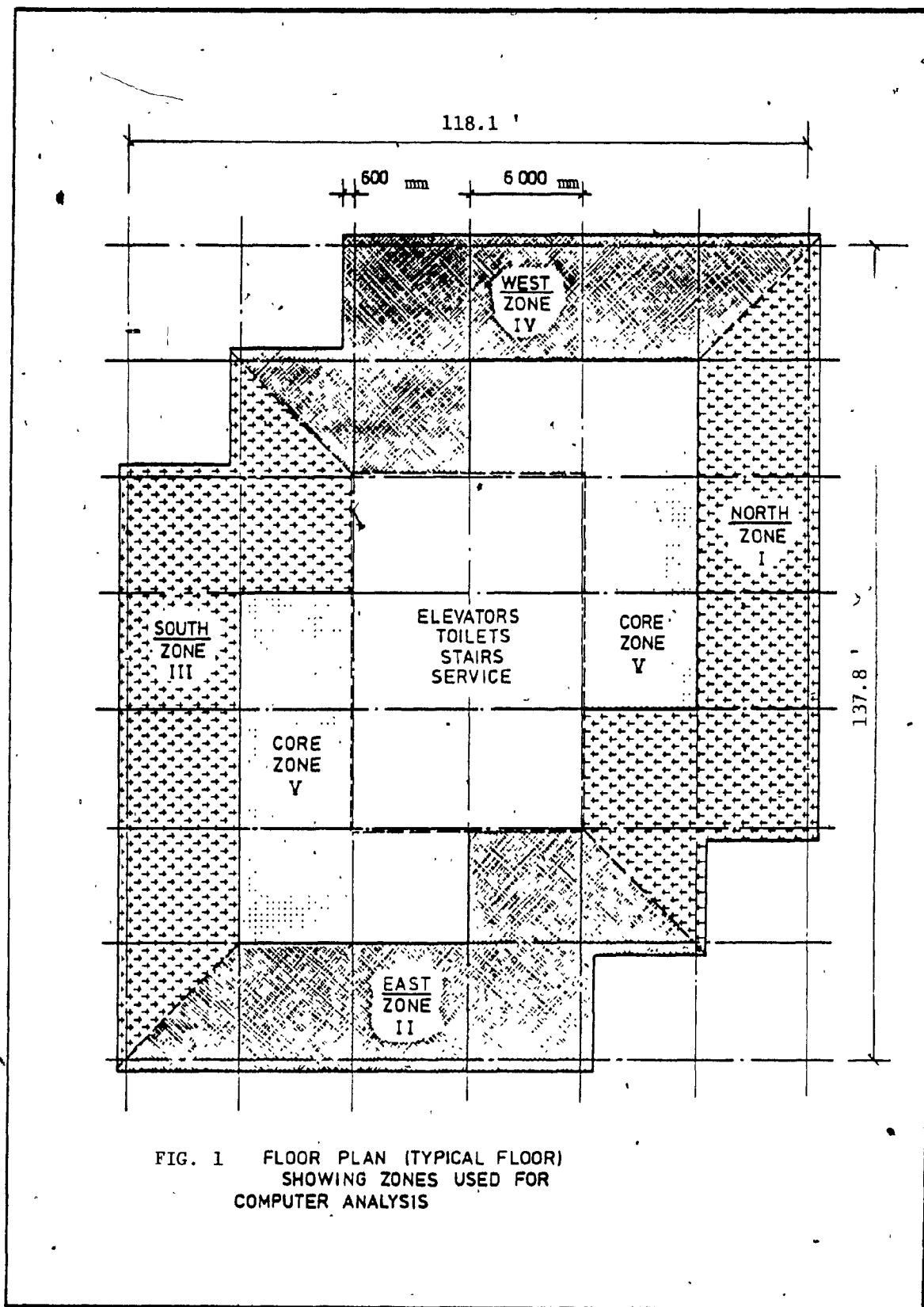


FIG. 1 FLOOR PLAN (TYPICAL FLOOR)  
SHOWING ZONES USED FOR  
COMPUTER ANALYSIS

The building skin or vertical envelope is a metal curtain wall with 4 inches of compressed glass fiber insulation in the non-glazed area. The glazed area consists of double glazed sealed units with bronze solar glass outside and clear glass inside.

The perimeter areas are conventionally one bay or 15 to 20 feet wide to account for the effects of outdoor weather conditions and solar radiation.

A typical section through the wall is shown in Figure 2 and the summary of the major building specifications were shown in Table 4.

The cooling and ventilation is by means of a VAV (variable air volume) system with sprayed coil cooling (1) and economizer cycle as shown in attached Figure 3. Secondary heating is by means of a hot water heated baseboard radiation located on the perimeter of the outer zones. The perimeter heating is controlled by an indoor-outdoor scheduled temperature controller.

---

(1) The supply air is cooled by a coil through which chilled water is circulated. The exterior of the coil is sprayed with water to wash the air, humidify the air and obtain additional evaporative cooling.

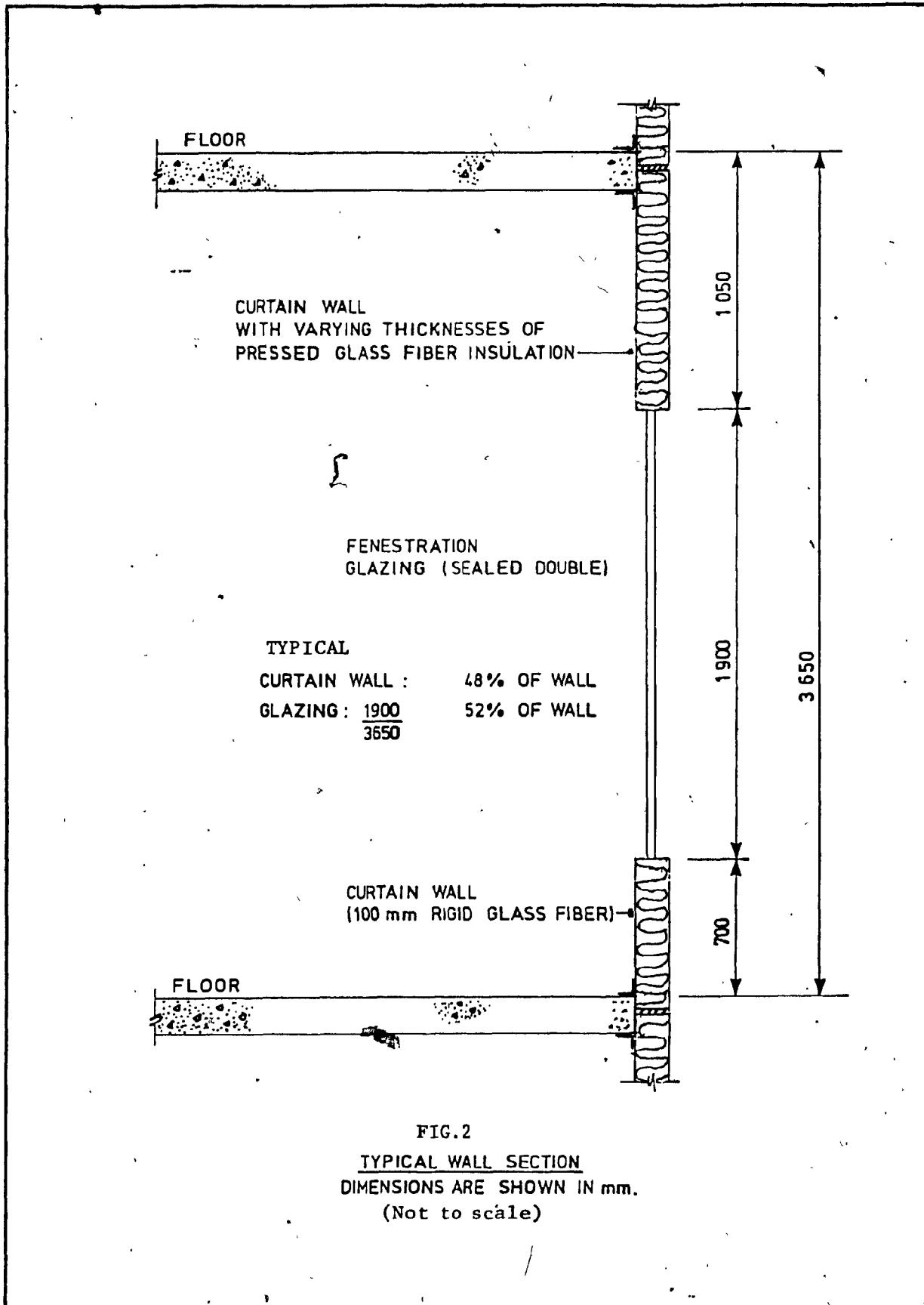


TABLE 4

SUMMARY OF MAJOR SPECIFICATIONS

No. of office floors 25

Total rentable area 322,787 sq. ft. ( $30,000 \text{ m}^2$ )

Typical floor to floor height 11.975 ft. ( 3.65 m )

Glazing .52 % of wall area

Lighting & appliances 3.15 W/sq.ft. ( $33.9 \text{ w/m}^2$ )

HVAC SYSTEM

Total chiller capacity 1302 tons (4578 kW)

Chiller I 44% of total

Chiller II 56% of total

(electrical driven centrifugal chillers)

NO. of boilers 4 (gas fired)

Total capacity 24,000 MBH (5624 kW)

Total air volume flow 92,946 CFM ( 185,450 l/s )

Lighting intensity & type 3 w/ sq. ft. fluorescent

Base case maximum heating or cooling CFM /  $\text{Ft}^2$  : 1.65

H Humidistat  
 T Room Thermostat  
 OT outdoor temperature sensing stat

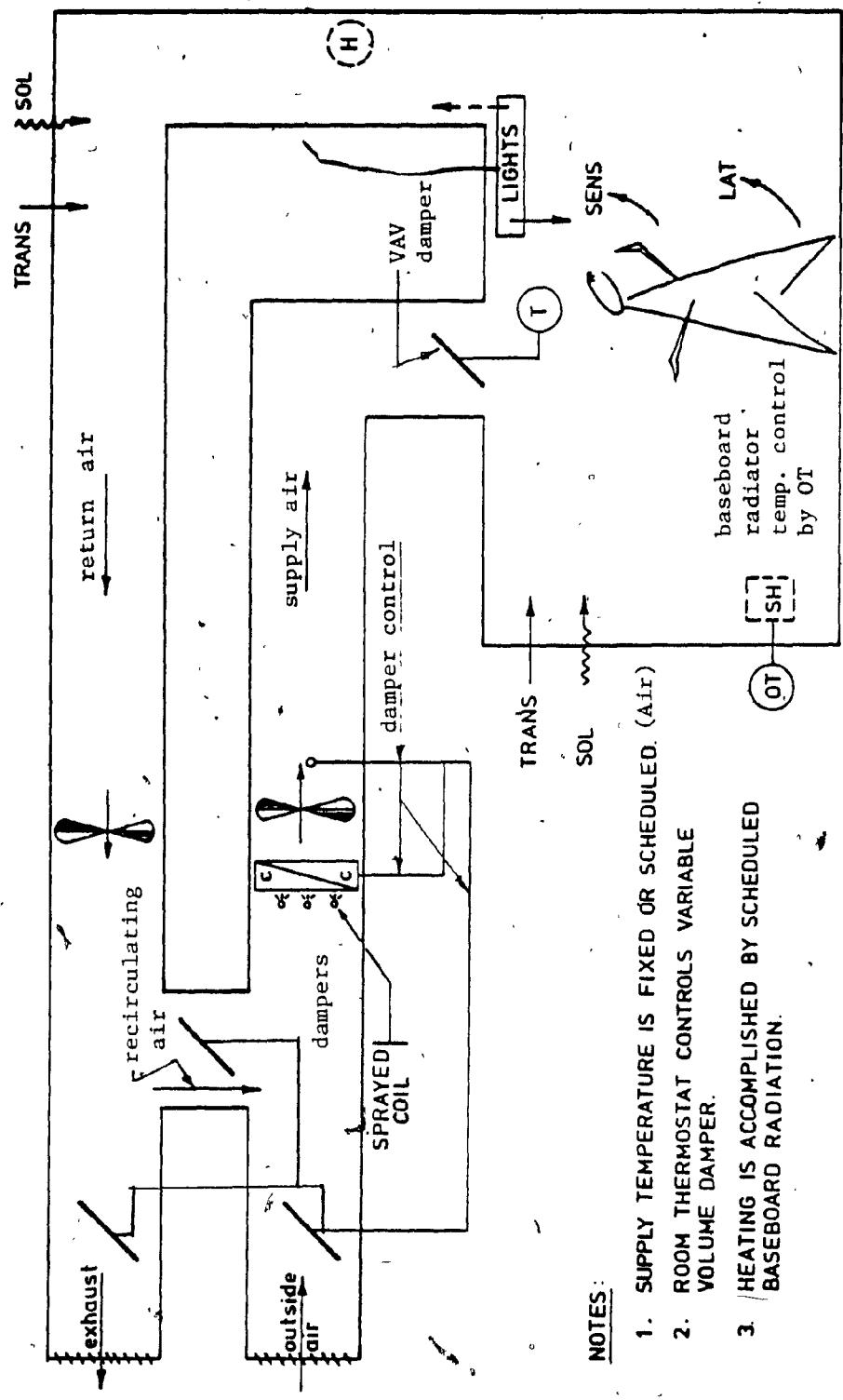


FIG. 3 Variable air volume air conditioning system schematic

The VAV diffusers are controlled by thermostats in the core zone with volume varying from 40 to 100% of design. In the perimeter zones, the VAV (variable air volume) diffusers are used to control the air volume variation. The thermostat controlling the variable air volume also controls the perimeter baseboard radiation in sequence when required.

There are 2 mechanical rooms, the lower on the 3rd floor and the upper on the 29th floor.

Supply and return fans, cooling and spray coils are located at each of these mechanical rooms.

The two electrically driven centrifugal water chillers having a total capacity of 4 578 kW (1302 tons) of refrigerating effect, one for 44% and the other for 56% of the total load, are located in the upper mechanical room. The cooling towers and the 4 gas fired boilers each with a capacity of 1 406 kW (6000 MBH) output, are also located in the upper mechanical room.

Electrical substations and motor control centers are also located in each of these mechanical rooms.

Service water is heated electrically at each floor in the service core adjacent to the wash-rooms.

The VAV diffusers are controlled by thermostats in the core zone with volume varying from 40 to 100% of design. In the perimeter zones, the VAV (variable air volume) diffusers are used to control the air volume variation. The thermostat controlling the variable air volume also controls the perimeter baseboard radiation in sequence when required.

There are 2 mechanical rooms, the lower on the 3rd floor and the upper on the 29th floor.

Supply and return fans, cooling and spray coils are located at each of these mechanical rooms.

The two electrically driven centrifugal water chillers having a total capacity of 4 578 kW (1302 tons) of refrigerating effect, one for 44% and the other for 56% of the total load, are located in the upper mechanical room. The cooling towers and the 4 gas fired boilers each with a capacity of 1 406 kW (6000 MBH) output, are also located in the upper mechanical room.

Electrical substations and motor control centers are also located in each of these mechanical rooms.

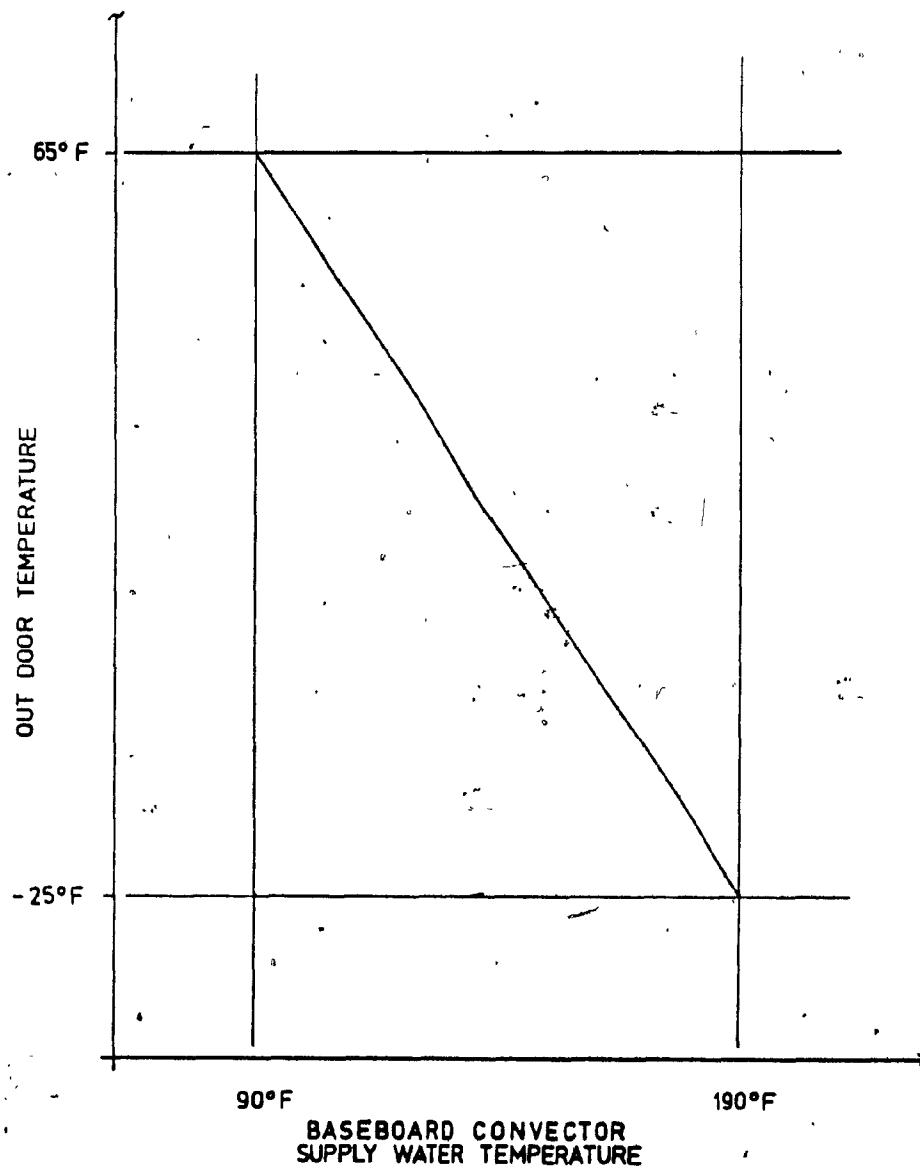
Service water is heated electrically at each floor in the service core adjacent to the wash-rooms.

### **2.3 Control system, and sequence of operation**

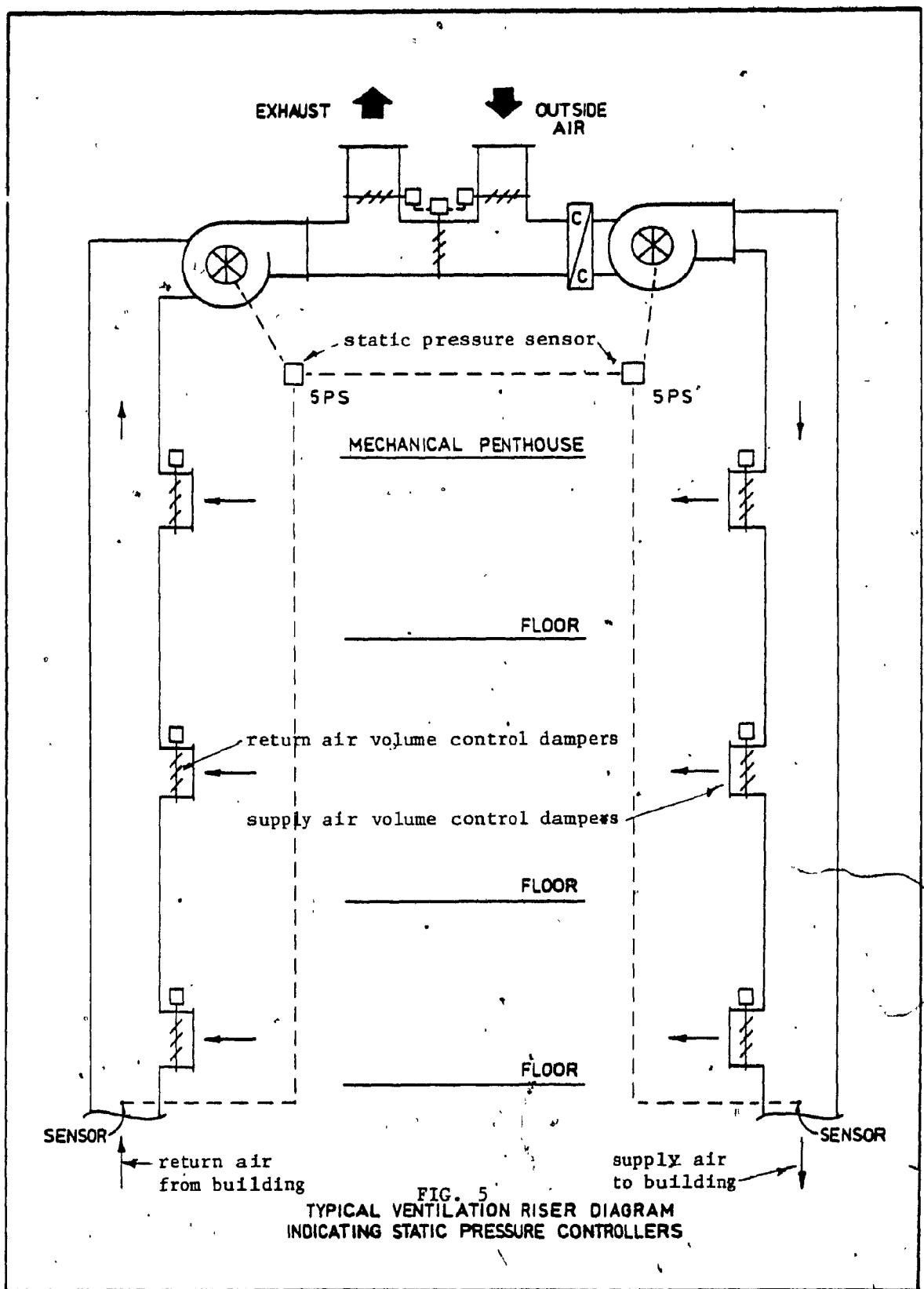
- a) The cooling coils with chillers and cooling towers come into operation when outside air temperature is above 10°C (50°F.) - Below 10°C (50°F.), outside air is mixed with return air to obtain "free cooling".
- b) The supply fan is a draw through type, i.e. the supply air is cooled and then drawn through the supply fan. As the fan and motor are in the air stream, the air picks up the heat from the motor horsepower input and the supply air is reheated by from 2 to 5°F, depending on the mechanical heat available from the fan electrical motors.
- c) The space thermostats control the linear VAV (variable air volume) supply diffusers, and the perimeter heating baseboard radiation in sequence. When the building is unoccupied, the air system can be shut off to conserve energy. In winter, the perimeter baseboard heating compensates for skin transmission losses. The heating water temperature is controlled by an indoor-outdoor temperature control. (see Figure 4).

FIG.4

OUT DOOR/IN DOOR CONTROL.  
WATER TEMP SCHEDULE  
DESIGN ROOM TEMP. = 68°F (20° C)  
(Hot water heating)



- d) Air static pressure controllers are located in the supply and return air ducts, at various levels, to signal to the supply and return air fans, to vary the volume of air circulated as a function of the variation in air quantities demanded by the zones. (see Figure 5). The air demand will vary with variations in the solar load, skin transmission losses or gains, internal load, (people, lights, etc).



CHAPTER 3  
DESCRIPTION OF THE ENERGY REQUIREMENT  
ESTIMATION PROGRAM

## (Energy Systems Analysis)

In 1971 Public Works Canada instituted a program to identify methods whereby the energy consumption of public buildings could be reduced. During this study, it was demonstrated that computer simulation is the most realistic and useful tool for the analysis of energy consumption in buildings. Owing to the complexity of both building structures and HVAC systems, and to the large variability in space loads from people, lights, transmission, solar, and equipment, computer simulation is indispensable to predict the energy requirements of the space and of the system servicing that space.

The potential for energy and cost savings via building energy analysis is very significant. The National Energy Board has predicted that before 1980 the energy consumption in commercial buildings alone will exceed

$1.5 \times 10^{15}$  BTU annually at a cost of some  $\$3 \times 10^9$  based on \$2. per  $10^6$  BTU. Numerous examples of building energy analysis with the ESA Series and other programs both in Canada and the U.S.A. have shown that 10% - 20% energy waste may not be an unrealistic figure. Thus in Canada, savings

on the order of \$0.5 billion annually are possible via the minimization of energy waste from commercial buildings alone.

### 3.1 Application of ESA (Energy Systems Analysis programs) in Design

The Energy System Analysis Series is a library of computer programs developed by Ross F. Meriwether and Associates, Inc. for hour-by-hour calculation of the annual energy consumption in buildings including the simulation of various types of air-side systems and mechanical plants; for applying local utility rate schedules to these demands and consumptions; and for combining these costs with other owning and operating costs for year-by-year cashflow projections. Each major step in the complete energy system analysis is handled by a different program, thereby permitting the Engineer to evaluate the results of one part before finalizing inputs and proceeding with the next part. As a design tool, the programs permit the Engineer to predict the effects on energy performance of: different air-side system types, various control temperatures, airflow quantities, operation schedules, heat recovery or economizer cycles, various equipment types and accessory combinations. The impact on owning and operating costs by competing energy sources, alternate utility rates, and a variety of economic factors, may be determined.

An important distinction exists between design point load programs and energy consumption programs. The Energy System Analysis Series (ESA) is designed to calculate monthly and annual energy requirements and costs, not design point heating and cooling loads. These programs begin with design point loads for the overall building, or major building sections, and distribute them throughout a full year cycle of the building's operation. If the Engineer is already using a load program, it is usually a simple matter to use a summation of the zone load components from that program as input for the building energy requirements program. The ESA programs are intended to supplement rather than replace existing load programs or calculation techniques. The programs offer the consulting Engineer an excellent means to evaluate his design and to modify it according to improvements learned by noting how energy requirements for his system respond to varying load and weather conditions.

Weather data is, of course, necessary for an energy analysis. Hourly weather data that is used in the energy requirements calculations is obtained from Environment Canada and consists of 8,760 hourly values of dry bulb temperature, dew point temperature, and cloud cover for some typical year. Programs have been developed for selecting and preparing this weather data for use in the Energy

Systems Analysis Series. Since the weather data is usually selected and ordered only once for a given city, Public Works Canada would normally process the weather data on behalf of the user. If a project is located in a city for which suitable weather data already exists, the user can access the data without delay. Solar radiation tables, as published in the ASHRAE Handbook of Fundamentals, are available with the weather data which is supplied to the users. One may, alternatively, create one's own solar heat gain factors with the aid of the Solar Table Generating Program as outlined below.

### 3.2 The ESA programs

The Energy System Analysis (ESA) series contains the following six programs:

- Energy Requirements Estimate (ERE)
- Total Coincident Requirement (TCR)
- Equipment Energy Consumption (EEC)
- Monthly Utility Costs (MUC)
- Economic Comparison of Systems (ECS)
- Solar Table Generating Program (SL2)

Figure A shows the typical flow of information via these programs in the process of performing an energy analysis. A brief resume of each program is given below. Complete descriptions of each program, including input/output reference instructions, are contained in subsequent chapters.

Energy Requirements Estimate (ERE) Program - calculates hourly thermal and electrical loads for the space under consideration. These loads are used in simulating the hour-by-hour operation of the air-side system in maintaining the desired space conditions. This simulation provides the hourly and annual building energy requirements. Typically a building is partitioned into "thermal blocks", or thermodynamically uniform zones, and one ERE simulation, or run, is performed for each thermal block. This block may be unique relative to other blocks in the building, or it may be one typical of many, such as a single unit in an apartment building.

Total Coincident Requirement (TCR) Program - sums the results from the ERE program for the individual thermal blocks so that a complete picture of the hour-by-hour thermal and electrical energy requirements is presented. In addition, total loads with actual diversity are indicated. As shown in Figure A, the ERE outputs are stored on tape in preparation for input to the TCR program.

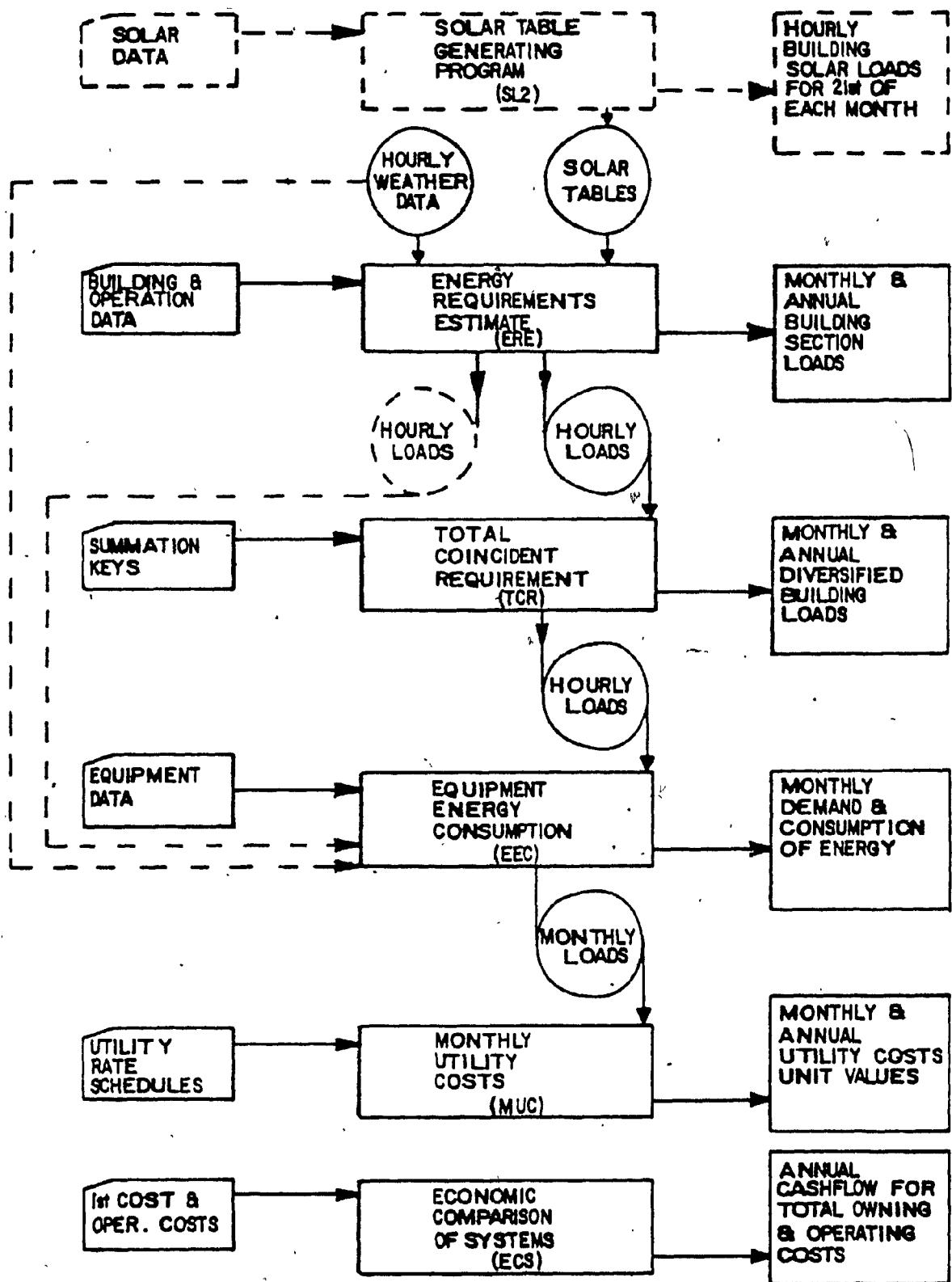


FIG. A

THE ENERGY SYSTEM ANALYSIS SERIES  
26

Equipment Energy Consumption (EEC) Program - utilizes the output data from the ERE or TCR programs in combination with the capacity and part load performance of each piece of equipment supplying energy to the air-side system. The program determines the monthly and annual energy peaks and consumption for energy input to various systems and accessories being evaluated.

Monthly Utility Costs (MUC) Program - calculates the monthly and annual energy costs for each type of service (gas, electric, steam or hot water, etc.). This calculation utilizes the results from the EEC program and the local utility rate schedules.

Economic Comparison of Systems (ECS) Program - uses the energy costs determined in MUC plus other operating costs, such as maintenance and operating labor, and combines these costs with initial investment and associated owning cost factors such as taxes, insurance, and depreciation, to find the annual cashflow each year for up to thirty years.

Solar Table Generating Program (SL2) - allows the user to generate his own solar heat gain tables for use as input to the ERE program. Items which are considered include, location, ground reflectivity, facting directions (to within one degree), glass characteristics, and lagging.

In this analysis, the ERE program only was utilized.

**CHAPTER 4**  
**INPUT DATA USED IN ANALYSIS**

#### 4.0 INPUT DATA USED IN THE ANALYSIS

4.1 To carry out this case study on the effects of enclosure elements on the energy requirements of a high-rise office building, the following data and/or assumptions were used in preparing the input cards:

4.1.1 Lighting level 3 watts per sq. ft. of floor area.

4.1.2 Electrical input for electrical appliances 0.15 watt/sq. ft.

4.1.3 Net leasable office space for 25 typical floors

- North thermal block or zone: 64,906 sq. ft.
- South thermal block or zone: 64,906 sq. ft.
- East thermal block or zone: 54,250 sq. ft.
- West thermal block or zone: 54,250 sq. ft.
- Core thermal block or zone: 84,475 sq. ft.

TOTAL: 322,787 sq. ft.

4.1.4 Total perimeter envelope or (wall + window) area:

ZONE	NORTH	SOUTH	EAST	WEST	TOTAL (sq. ft.)
North	37,128	-----	6,483	-----	43,611
South	-----	37,128	-----	6,483	43,611
East	6,482	-----	31,234	-----	37,716
West	-----	6,482	-----	31,234	<u>37,716</u>
TOTAL SKIN:					162,654

These areas with 48% insulated curtain wall and 52% glazing were used for heat loss and heat gain calculations.

4.1.5 The reference year weather data is the "typical" year, taken as an average over many years with all anomalies or unusual occurrences corrected or smoothed out.

4.1.6 The weather data used for the dynamic analysis was the Canadian Government weather tape for Edmonton for a typical year.

Solar gains are actual ASHRAE solar gains on an hourly basis for a typical year with all necessary corrections incorporated.

Holiday schedule - it was assumed that the building would not be occupied on week-ends and holidays i.e. New Year's, Good Friday, Easter Sunday, Memorial Day, Canada Day, Bank Holiday, Labor Day, Thanksgiving, Christmas.

#### 4.1.7 Weeks-ends and holidays operation

- Lighting level was assumed at 5% over 24 hours or 0.15 watts per sq. Ft.
- HVAC schedule 0-24 hours fans and cooling off, heat on perimeter on an outdoor-indoor control, heat off at 65°F. outside temperature and above.
- HVAC systems are completely shut off.
- Perimeter heating comes into operation to maintain minimum unoccupied indoor temperature of 18°C (65°F.) controlled by an outdoor-indoor temperature control.

#### 4.1.8 Light schedule:

1)	week days	0 - 7 AM	5% of 3W/sq.ft.: 0.15W
		7 - 8	10%
		8 - 9	20%
		9 - 12	90%
	(2 PM)	12 - 14	80%
		14 - 17	90%
		17 - 19	70%
		19 - 20	60%
		20 - 21	40%
		21 - 22	30%
		22 - 24	5%

#### 2) Week ends and holidays:

0 - 24      5% (0.15 watt/sq.ft.)

#### 4.1.9 Fan system operation following building occupancy:

.1- week days start 8 AM

stop 5 PM - unless reset

.2 week ends and holidays:

0-24 hours off unless reset

4.1.10 Shading coefficient (Table 3.17 - Ref.1)

.1 Solar bronze heat absorbing; sealed unit with  
clear glass inside,

no interior shading SC: 0.58

.2 Triple clear glass, sealed unit

no interior shading SC: 0.71

.3 Double clear glass, sealed unit

No. interior shading SC: 0.88

.4 Double clear glass, sealed unit

with bronze reflective film

on inside of outer pane SC: 0.38

(no inside shading)

4.1.11 The computer program has shading of wall capability, but no shadows or shading by adjacent buildings have been calculated as no data was given.

Shading of walls as sun rotates around the building is found in hour by hour calculations and by total ton-hour calculations, this item is included in the ERE.

Chillers are shut down:

.1 when building is unoccupied, i.e.

- between 5 PM and 8 AM week days
- Saturdays, Sundays and holidays

.2 when outside air is below 50°F.

Economizer cycle and enthalpy control are provided  
for in the ERE program.

The Enthalpy control will operate as long as outdoor air conditions permit free cooling or use of the economizer cycle, then the chiller will cut in, or the chiller will cut out if outdoor conditions permit the use of the economizer cycle. This will operate even on a cool morning in spring or fall.

An altitude of 2,200 Ft. A.S.L. (Above Sea Level) for psychrometric temperature and humidity conditions are reflected in the weather tapes.

.1 Indoor temperature setpoint in summer is 78°F.

ambient outdoor design 85°F, DB, 66°F. DP

.2 Indoor temperature setpoint in winter is 68°F.

DP, 34°F. dewpoint to prevent condensation on the windows.

.3 Heating comes on when outdoor temperature is below 65°F. and operates on an outdoor-indoor schedule.

0% heating at 65°F. outside air

100% heating at -25°F. outside air  
with a straight line relationship.

There is no specific humidification equipment in the building. It takes heat to evaporate water. There are air washers in the cooling system. Therefore, in this process, there will be some humidification as a by-product of the cooling. The heat extracted in the process must be included in the heat balance. Thus, calculation is included in the ERE program.

4.1.12 The chilled water pumps, condenser water pumps and cooling tower fans are interlocked with the chillers. These auxiliaries follow the same schedule as the chillers as discussed above.

4.1.13 Heating boilers and pumps are on an outdoor-indoor control and follow transmission loss demand when outside air temperature is below 65°F., this can happen even in Summer.

They are not shut down except when heating load is satisfied and outside temperature is above 65°F.

4.1.14 The total (kW) power of the supply and return fan systems is:

- Total upper supply	268.4 kW
- Total upper return	74.4 kW
- Total lower supply	268.4 kW
- total lower return	<u>74.4 kW</u>

685.6 kW

4.1.15 Wind factors used were:

- Winter	15 mph
- Summer	ho : 4 (7.5 mph)
	ho : 3 (less than 7.5 mph)

4.1.16 "U" factors for wall do not differ materially from summer to winter. Glass "U" factors will vary slightly from summer to winter and these have been taken into account in calculating the U-factor for the composite wall (see Table 1).

4.1.17 The U-factor for double clear glass with bronze reflective film on the inside of the outer pane is:

- 0.43 in winter with 15 mph wind
- 0.37 in summer with 15 mph wind
- 0.40 in summer with 5 mph wind

The major heat gain in summer is not by transmission but solar.

4.1.18 When calculating the chiller electrical input to the program, include the whole chiller train that is interlocked with it, i.e. condenser and chilled water pumps, cooling tower etc...

4.1.19 People load is taken at 180 sq. ft. of floor area/person and at 450 Btu/hr/person with 44% latent heat load and 56% sensitive heat load.

4.1.20 The program takes into account equipment power input in kW, and calculates system efficiencies within the envelope or conditioned space.

System losses due to low power factor or poor utility power regulation are not included in the program.

4.1.21 The total building infiltration is taken at 16000 cfm (see calculation, Table 5).

4.1.22 For these calculations, the normal exhaust rates would be the difference between supply and exhaust to maintain the building pressurization.

This would be 18000 cfm approximately at full load VAV - as minimum VAV is 40%, then the minimum exhaust rate would be 7200 CFM when systems are running.

The actual exhaust rate could be obtained by taking the sum of the toilet exhausts for the 25 floors. These exhaust fans should be scheduled to shut off during non-occupied periods, nights, week-ends, holidays.

4.1.23 Outside or make-up air quantities will vary:

.1 on the economizer cycle it could go as high as 100%.

.2 the ventilation air required for the people occupying the building at 15 CFM/person during the heating and cooling cycle is about 10% full load.

TABLE 5

SAMPLE CALCULATIONCURTAIN WALL INFILTRATION FOR THE ENTIRE BUILDING

(Reference: Chapter 5, ASHRAE, Cooling and Heating Load - Calculation manual).

P = Building perimeter = 512 lin. ft.

H = Building height = 300 ft.

A = Area of entire curtain wall of building = 153,280 sq. ft.

Q = CFM infiltration

K = Leakage coefficient of curtain wall (dimensionless) .66

F<sub>d</sub> = Thermal draft factor : .87 $\Delta t =$ 

= Winter design temperature difference = 93 °F.

 $\frac{Q}{A \cdot K \cdot F_d} = 0.18$  from Figure 5.6Q = A K F<sub>d</sub> ( Q / A K F<sub>d</sub> )

= 153,280 X .66 X .87 X .18 = 15,842 CFM

= 16,000 CFM

REQUIRED OUTSIDE AIR

Tower Floor Area Total = 322,787 sq. ft.

Floor Area per occupancy = 180 sq. ft

No occupants =  $\frac{\text{Total Area}}{\text{SQ. Ft. / person}} =$ Outside Air Required = Occupants X 15 CFM/occupancy  
(maximum)=  $\frac{322,787 \times 15}{180} = 26,900 \text{ CFM}$

4.1.24 Lighting was assumed by fluorescent fixtures with HPF (high power factor) ballasts calculated with 15% to plenum. The lighting kW input is converted to thermal gain for cooling load calculation. The energy input for lights appears in lights and power calculation separately.

Energy for cooling to compensate for light input appears as a cooling cost and/or as a heating credit.

4.1.25 The program adjusts its power energy requirements for fans, chiller, etc... as a function of the VAV system's demand.

4.1.26 The ERE Meriwether program for energy requirement estimate was used.

Full explanations as to procedure and input instructions are contained in the Reference Manual for Energy Systems Analysis series of programs (Meriwether) Public Works Canada (1980) - (Ref. 2).

4.1.27 The full tabulation of the input data required for the ERE analysis of each thermal block or zone is given in Table 6.

TABLE 6

ERE ANALYSIS

INPUT DATA NEEDED

FOR EACH THERMAL BLOCK OR ZONE

Floor area  
System data  
    Type of air distribution system (dual-duct, terminal reheat, variable volume, etc..)  
    Heating capacity  
    Cooling capacity  
Solar load  
    Reference value  
    Time of occurrence  
    Z distribution of glass area by facing direction  
Internal loads  
    Peak from people  
    Peak from lights  
    Peak from equipment  
    Peak from miscellaneous  
    24-hour profiles for each of these loads for each type of day  
Transmission loads  
    Heating peak and design temperature  
    Cooling peak and design temperature  
Process loads  
    Peak indirect (such as domestic hot water or other steam loads)  
    Peak direct  
    24-hour profiles for each of these loads for each type of day  
Return air loads  
    Roof solar and transmission  
    Heat of lights  
    Return fan temp. rise  
    Supply fan temp. rise  
Airflow quantities  
    Supply air  
    Outside air  
Control temperatures  
    Cold air supply temp.  
    Hot air supply temp.  
    Reset schedules for hot and/or cold air temperatures and humidification  
    Economizer cycle temperature limits  
Heat recovery efficiencies for energy wheel

TABLE 6.

page 2..

Time schedules for each type of day

    Outside air shutoff

    Heating system shutoff or setback

    Cooling system shutoff or setback

    Off-peak electrical service

Gas curtailment temperatures or schedules

Supplementary heating system data

    Baseboard radiation schedule (temp. and/or time)

Holiday schedule

Description of profiles and operating schedules to be used with each  
type of day.

#### 4.2 Limitations of the program

4.2.1 In the enumeration of the assumptions and data used as input to the various computer runs, several limitations were at least implicit (see articles 4.1.5,13).

4.2.2 Design inputs for lighting levels are set and applied on an occupancy schedule. There is no provision to account for lesser power consumption should certain tenants have lower lighting levels as in landscaped offices with task lighting.

4.2.3 There is a great deal of flexibility in the program but it cannot be in all ways matched to existential conditions. It remains a simulation, no matter how skillfully the simulation is carried out.

4.2.4 Inherent problems with each of these programs (Meriwether ERE, Calerda, Blast, etc...) are not only making the correct input to allow the program to carry out the simulation, it is designed to do, but once this simulation is complete, for the analyst to correctly interpret the resulting print-out.

Unless the analyst has a great deal of experience in the design of building HVAC systems, as well as trouble shooting in these areas, he or she will not be able to use all the information that is made available by these simulations.

4.2.5 ERE were calculated with Summer design room temperatures of 78°F. and Winter of 68°F. If should the tenants adjust thermostats locally, energy use will be affected.

4.2.6 People load was assumed at 180 sq. ft. per person. This may vary considerably.

4.2.7 Building infiltration may vary considerably as a function of the quality of the workmanship. Average figures were used in this case.

4.2.8 The building is divided into thermal blocks or zones i.e. North, South, East, West and Core.

Each of these zones are analyzed separately and are considered as if not interacting upon each other.

4.2.9 Many of the variables, ventilation rate, infiltration, lighting levels, internal heat gains (population) are held constant during fixed schedules (i.e. occupied or unoccupied periods).

4.2.10 The intent of the program is to get as close as possible to the overall building energy used without excessively detailed input.

4.2.11 Summary of the input data and output were shown in Table 6 and 7 respectively.

Sample output data were shown in Appendix "A".

TABLE 7

OUTPUT DATA SUMMARY

Peak heating, MBH, by month

Peak cooling, tons, by month

Total base electric load in kW

Monthly and annual loads

- Heating in MBTU
- Heating hours
- Cooling ton-hours
- Cooling hours
- Base A electric load in kWh
- Base C electric load in kWh
- Total base electric load in kWh
- Minimum room temperature °F
- Maximum room temperature °F

Unit values of building energy peaks and consumption

- Heating peak                      Btu/Hr./ Sq. Ft.
- Heating consumption                MBTU/Sq. ft.
- Full load heating                 Hours (equivalent)/yr
- Cooling peak                      Tons/1000 Sq. ft.
- Full load cooling                 Hours/hr (equivalent)
- Electric peak                     Watts/Sq. ft.
- Electric consumption             kWh/Sq. ft.
- Full load electric                Frs/yr (equivalent)

**CHAPTER 5**  
**SUMMARY AND ANALYSIS OF THE RESULTS**

## 5.1 Summary and Discussion of the Results

The zone-by-zone and total annual energy consumption summary for the 11 variations considered is shown in Table 5.1 and Figure 5.1.

### 5.1.1 Heating Energy Requirement

From Fig. 5.1, for annual heating energy consumption, alternatives 2, 4, 6 and 11 resulted in an increase, while 5, 7 resulted in a decrease, and 3, 9 and 10 resulted in a marginal decrease with respect to the base case. The percentage changes in the heating energy and cost in comparison with the base case, are shown in Table 5.2. The alternative #8 provides the least heating energy consumption. The natural gas costs were based on the 1980 rates, for Montreal, Canada.

### 5.1.2 Cooling Energy Requirement

The annual cooling energy consumption and cost data are shown in Table 5.3. The alternative #9 provides the least cooling energy consumption. The cooling costs are calculated assuming a coefficient of performance (C.O.P.) of 3.75, and electrical consumption and demand charges for Montreal, Canada, in 1980.

TABLE 5.1  
ANNUAL ENERGY CONSUMPTION SUMMARY  
FOR VARIOUS WALL ASSEMBLIES ( SEE FIG. 5.1 )

ZONE	ITEM	1	2	3	4	5	6	7	8	9	10	11	enclosure
North	HTC, HAN	2,601,111	3,125,695	2,726,940	3,183,714	2,134,516	3,474,375	2,425,201	2,026,560	2,863,913	2,735,130	3,079,683	alternative
	incl. HAN	5,510	-	-	-	-	-	-	-	-	-	-	
	CUC, TV/WK	52,720	51,878	53,807	68,255	64,886	67,769	64,443	65,057	38,275	39,639	66,606	
	TOT. EL. WNK	976,571	979,834	983,454	1,018,169	1,007,765	1,016,583	1,006,032	1,008,395	954,722	956,003	1,009,860	
East	"	-	-	-	-	-	-	-	-	-	-	-	
	HTC, HAN	2,450,098	2,730,165	2,355,965	2,754,609	1,843,657	3,006,536	2,095,753	1,749,709	2,452,885	2,358,660	2,660,556	
	incl. HAN	87,319	87,313	87,342	87,659	86,506	87,851	87,848	87,858	80,771	80,891	85,858	
	CUC, TV/WK	1,042,452	1,073,434	1,042,933	1,055,310	1,080,875	1,054,605	1,054,695	1,055,370	1,003,729	1,004,430	1,046,021	
South	"	-	-	-	-	-	-	-	-	-	-	-	
	HTC, HAN	2,814,909	3,125,695	2,726,940	3,183,714	2,134,516	3,474,375	2,425,201	2,026,560	2,834,909	2,726,940	3,075,739	
	incl. HAN	6,200	106,616	107,012	112,645	111,130	112,509	110,924	111,205	86,763	86,921	109,876	
	CUC, TV/WK	105,765	106,616	1,330,903	1,351,562	1,344,890	1,350,748	1,344,023	1,345,209	1,274,266	1,274,882	1,349,474	
West	"	-	-	-	-	-	-	-	-	-	-	-	
	HTC, HAN	2,450,098	2,730,165	2,355,965	2,754,609	1,843,657	3,006,536	2,095,753	1,749,709	2,452,885	2,355,935	2,660,480	
	incl. HAN	83,061	82,595	83,240	86,715	87,857	86,373	86,095	86,669	73,158	73,349	84,683	
	CUC, TV/WK	1,067,403	1,046,007	1,067,925	1,083,900	1,055,151	1,062,738	1,079,634	1,081,479	1,032,263	1,032,940	1,061,839	
Core	"	-	-	-	-	-	-	-	-	-	-	-	
	HTC, HAN	5,143	5,143	5,143	5,143	5,143	5,143	5,143	5,143	5,143	5,143	5,143	
	incl. HAN	41,331	41,331	41,331	41,331	41,331	41,331	41,331	41,331	41,331	41,331	41,331	
	CUC, TV/WK	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	1,573,370	
TOTAL	HTC, incl.	10,576,216	11,603,571	10,165,810	11,876,646	7,956,746	12,961,022	9,071,908	7,552,528	10,581,860	10,972,724	11,476,468	
	WAN, HAN HTC	-	-	-	-	-	-	-	-	-	-	-	
	CUC, TV/WK	372,216	372,452	399,523	394,430	398,553	393,161	394,660	323,718	324,831	324,831	391,014	
	EL or ELECT	5,980,229	6,022,211	5,998,731	6,082,449	6,062,181	6,078,194	6,057,884	6,064,173	5,898,550	5,841,845	6,060,764	

HTC Heating (MBH)  
HAN Humidification (MBH)  
CUC Cooling  
EL or ELECT Electrical ( kwh/yr )

Column 1 indicated the base case

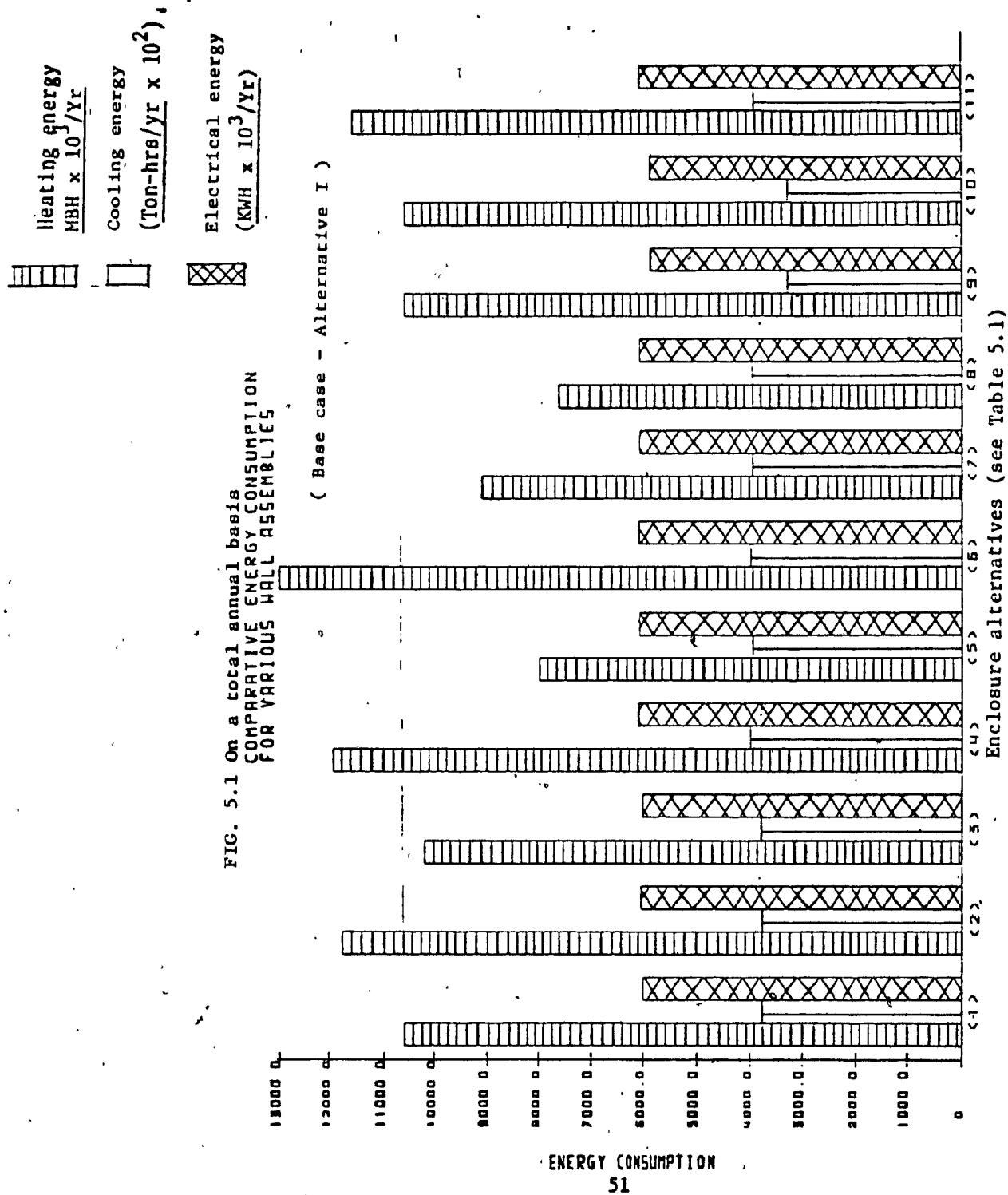


TABLE 5.2

## COMPARATIVE ANNUAL HEATING ENERGY AND COST FOR VARIOUS WALL ASSEMBLIES

ALT	ANNUAL ENERGY MBH	$\Delta\%MBH$	TOTAL COST* PER YEAR	$\Delta\%COST$
1	10,576,216	-----	39,132	-----
2	11,683,571	10.47	43,229	10.47
3	10,165,810	-3.88	37,613	-3.88
4	11,876,646	12.30	43,944	12.30
5	7,945,746	-24.77	29,440	-24.77
6	12,961,822	22.56	47,959	22.56
7	9,071,908	-14.22	33,566	-14.22
8	7,552,528	-28.59	27,944	-28.59
9	10,581,840	0.05	39,153	0.05
10	10,572,724	-0.03	39,119	-0.03
11	11,476,468	8.51	42,463	8.51

\* 1 MCF =  $10^3$  MBTU at \$3.70/MCF (1980 costs - Montreal, Canada)

TABLE 5.3  
COMPARATIVE ANNUAL COOLING ENERGY AND  
COST FOR VARIOUS WALL ASSEMBLIES

ALT	ANNUAL ENERGY ton - hr	$\Delta\%$ ton - hr	TOTAL COST ** PER YEAR	$\Delta\%$ COST
1	372,916	----	3,729	----
2	372,453	- 0.12	3,725	- 0.12
3	375,452	0.68	3,755	0.62
4	399,525	7.14	3,995	7.14
5	394,430	5.77	3,944	5.77
6	398,553	6.87	3,986	6.87
7	393,361	5.48	3,934	5.48
8	394,860	5.88	3,949	5.88
9	323,718	-13.19	3,237	-13.19
10	324,851	-12.89	3,249	-12.89
11	391,074	4.87	3,911	4.87

\*\* 1 ton-hr = 0.94 kWh  
(C.O.P. = 3.75)

1981 Hydro-Québec electrical rates have been applied

### **5.1.3      Electrical Energy Requirement**

The electrical energy consumption and cost data are shown in Table 5.4. The electrical energy included the lighting, fan and pump energy input. The estimates demand for the building (from the simulation program) is 1,808 kW/yr. The consumption and demand costs are calculated from the Hydro-Quebec rates, for Montreal, 1980.

### **5.1.4      Total Energy Requirement**

The total energy consumption is calculated by the summation of the annual heating, cooling and electrical energy requirements (see table 5.5 and Fig. 5.2). The variation of the total energy requirement for the alternatives considered (in comparison with the design condition) is observed to be very limited (-8.29% to +8.70%) indicating the insignificance of the effects of the selected enclosure system variations.

### **5.1.5      Effects of the Wall Insulation Thickness and Glazing Types**

The effects of the variation of the wall insulation thickness on the annual heating and cooling energy are shown in Fig. 5.3 and 5.4 respectively.

TABLE 5.4  
COMPARATIVE ELECTRICAL ENERGY CONSUMPTION  
FOR VARIOUS WALL ASSEMBLIES

ALT	ELECTRICAL ENERGY CONSUMPTION* kWh	$\Delta\%$ kWh	TOTAL COST PER YEAR**	$\Delta\%$ COST
1	5,980,829	----	80,593	----
2	6,022,211	0.71	81,007	0.51
3	5,998,737	0.3	80,772	0.22
4	6,082,499	1.7	81,610	1.26
5	6,062,181	1.4	81,407	1.01
6	6,078,194	1.6	81,567	1.2
7	6,057,884	1.3	81,364	0.96
8	6,064,173	1.4	81,427	1.03
9	5,838,550	-2.4	79,170	-1.76
10	5,841,845	-2.3	79,203	-1.72
11	6,060,764	1.3	81,392	0.99

\* Lighting plus power input for fans, pumps (excluding chiller energy consumption)

\*\* Billing demand is 1,808 kW/year (from the simulation program)

1981 Hydro-Quebec rates have been applied (for consumption and demand)

TABLE 5.5  
COMPARATIVE TOTAL ENERGY CONSUMPTION  
AND COST FOR VARIOUS WALL ASSEMBLIES.

ALT	ANNUAL ENERGY* kWh	$\Delta\%$ kWh	TOTAL COST** \$/YEAR	$\Delta\%$ COST	X kWh/Ft <sup>2</sup>
1	9,429,278	0	123,454	0	28.33
2	9,794,678	3.88	127,961	3.65	29.43
3	9,329,316	-1.06	122,140	-1.06	28.03
4	9,936,915	5.38	129,549	4.94	29.86
5	8,763,304	-7.06	114,791	-7.02	26.33
6	10,249,653	8.70	133,512	8.15	30.88
7	9,084,744	-3.65	118,864	-3.72	27.30
8	8,647,265	-8.29	113,320	-8.21	25.98
9	9,242,519	-1.98	121,560	-1.53	27.77
10	9,244,205	-1.96	121,571	-1.53	27.78
11	9,790,010	3.83	127,766	3.49	29.42

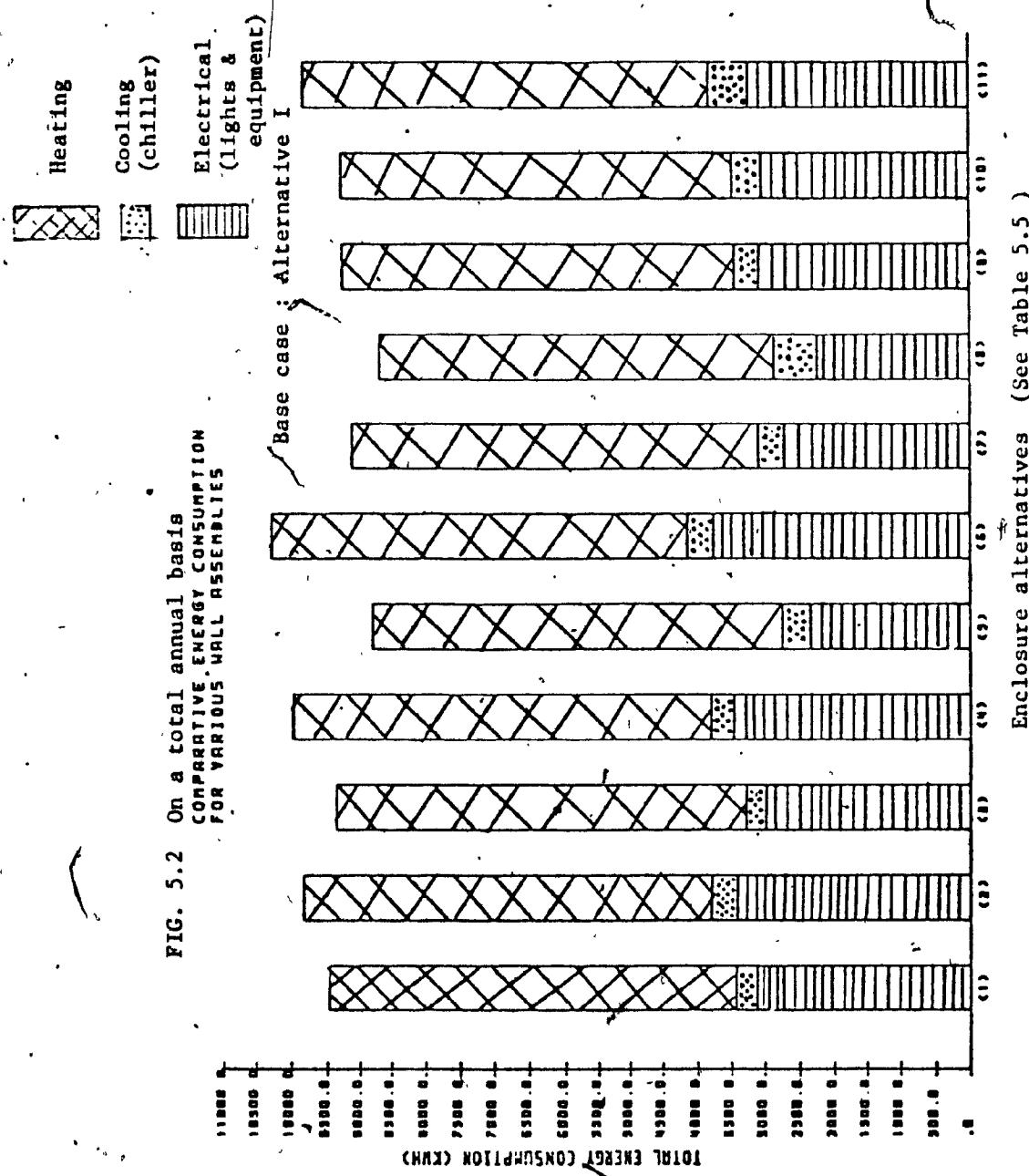
\* For heating - 1 kWh = 3.413 MBTU

For cooling - 1 kWh = (ton Hr x 12)/(3.413 C.O.P.)  
C.O.P. = 3.75

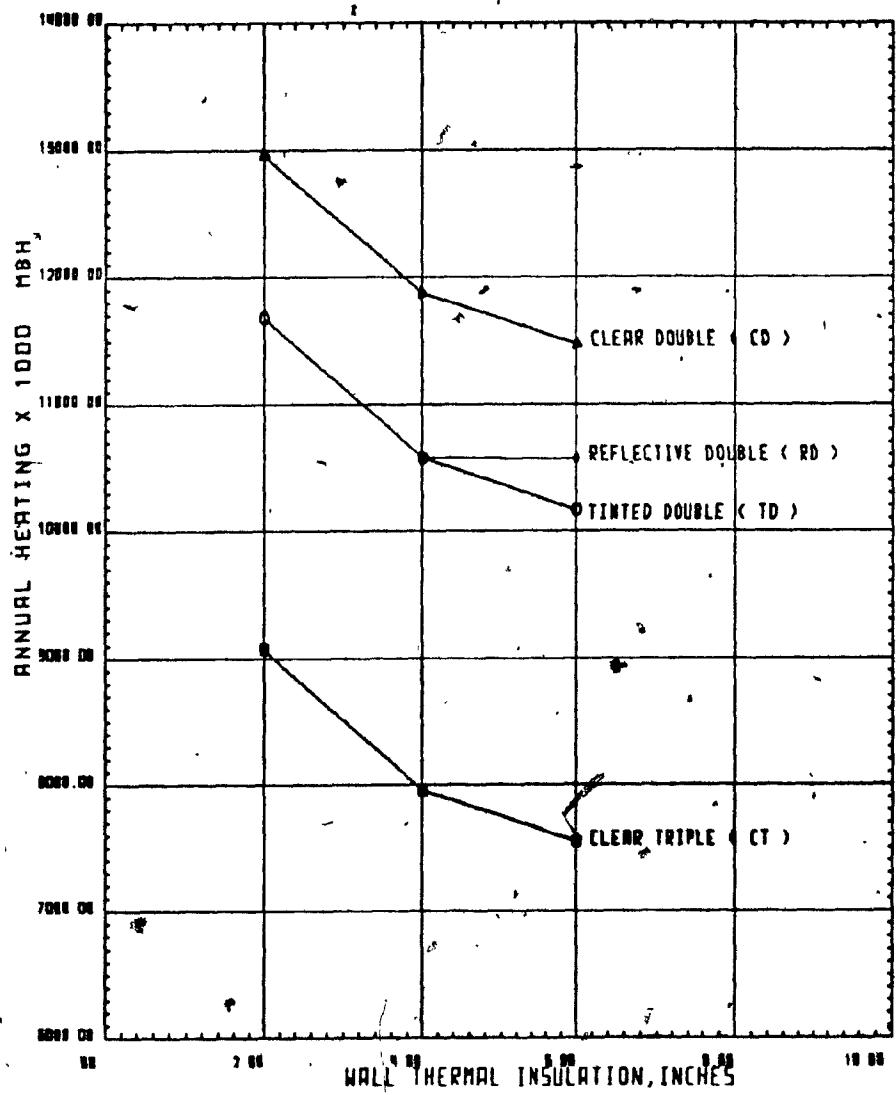
\*\* Sum of the heating, cooling and electrical costs (from tables ..., ..., ..., respectively)

X Floor area = 332,787 sq. Ft.

**FIG. 5.2 On a total annual basis  
COMPARATIVE ENERGY CONSUMPTION  
FOR VARIOUS WALL ASSEMBLIES**



Enclosure alternatives (See Table 5.5 )



**FIG. 5.3 EFFECTS OF THE WALL INSULATION THICKNESS  
ON THE ANNUAL HEATING ENERGY CONSUMPTION**

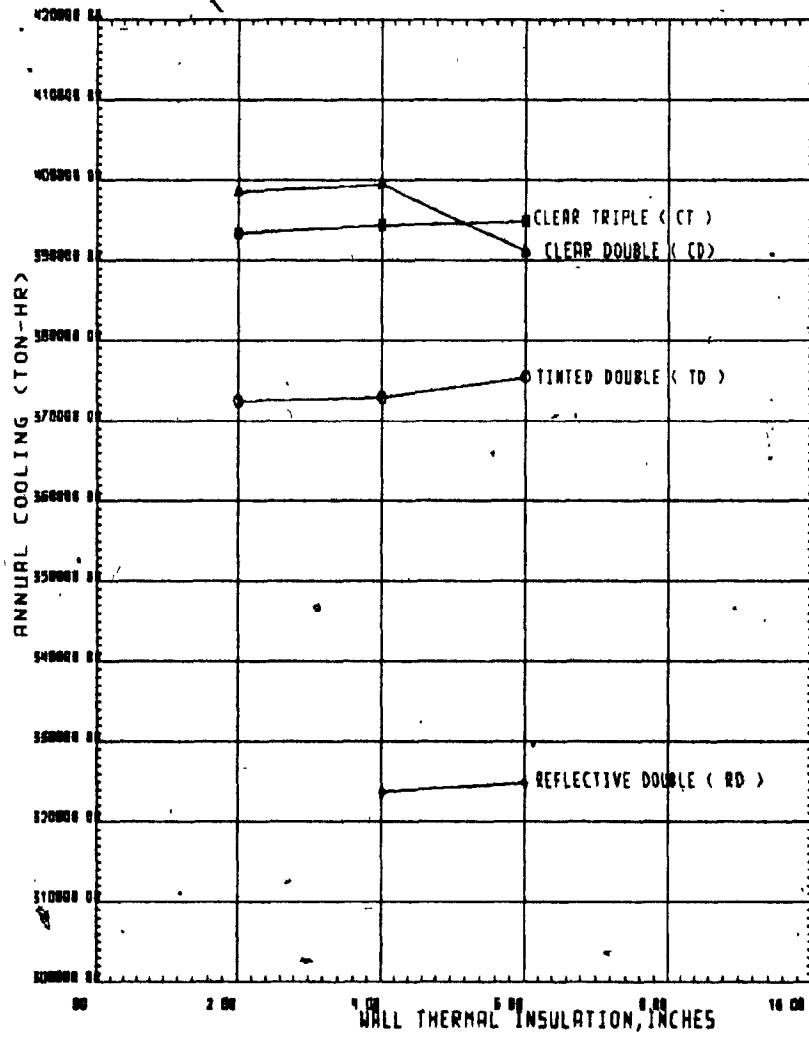


FIG. 5.4 EFFECTS OF THE GLAZING VARIATIONS  
ON THE ANNUAL ENERGY CONSUMPTION

As expected, the annual heating energy requirement decreased with the increase in the wall insulation thickness and the clear triple glazing ( $U = 0.31$ ; S.C. = 0.71) provided the least heating requirement.

For the cooling application, the reflective double glazing ( $U = .37$ ; S.C. = .38) provided the least cooling energy; this reduction can be attributed to the characteristics of the glazing, which reflects the solar radiation (minimizing the solar heat gain), while providing a path for some of the internal heat gains, more effectively than the other types of the glazing considered in this analysis.

#### 5.1.6 Monthly Load Requirements

Typical monthly heating and cooling load variations (alternative #11) are shown in Fig. 5.5. As expected, peak heating load occurs in January and the cooling load in July, and the variations were reasonably uniform (no sudden changes).

Typical monthly zone-by-zone cooling load variations are shown in Fig. 5.6. Due to relatively high glazing area (52%), the south zone has the highest cooling load, from the solar heat gains, in comparison with the other zones.

FIG. 5.5 TYPICAL MONTHLY LOAD VARIATIONS  
FOR ALTERNATIVE No. 11

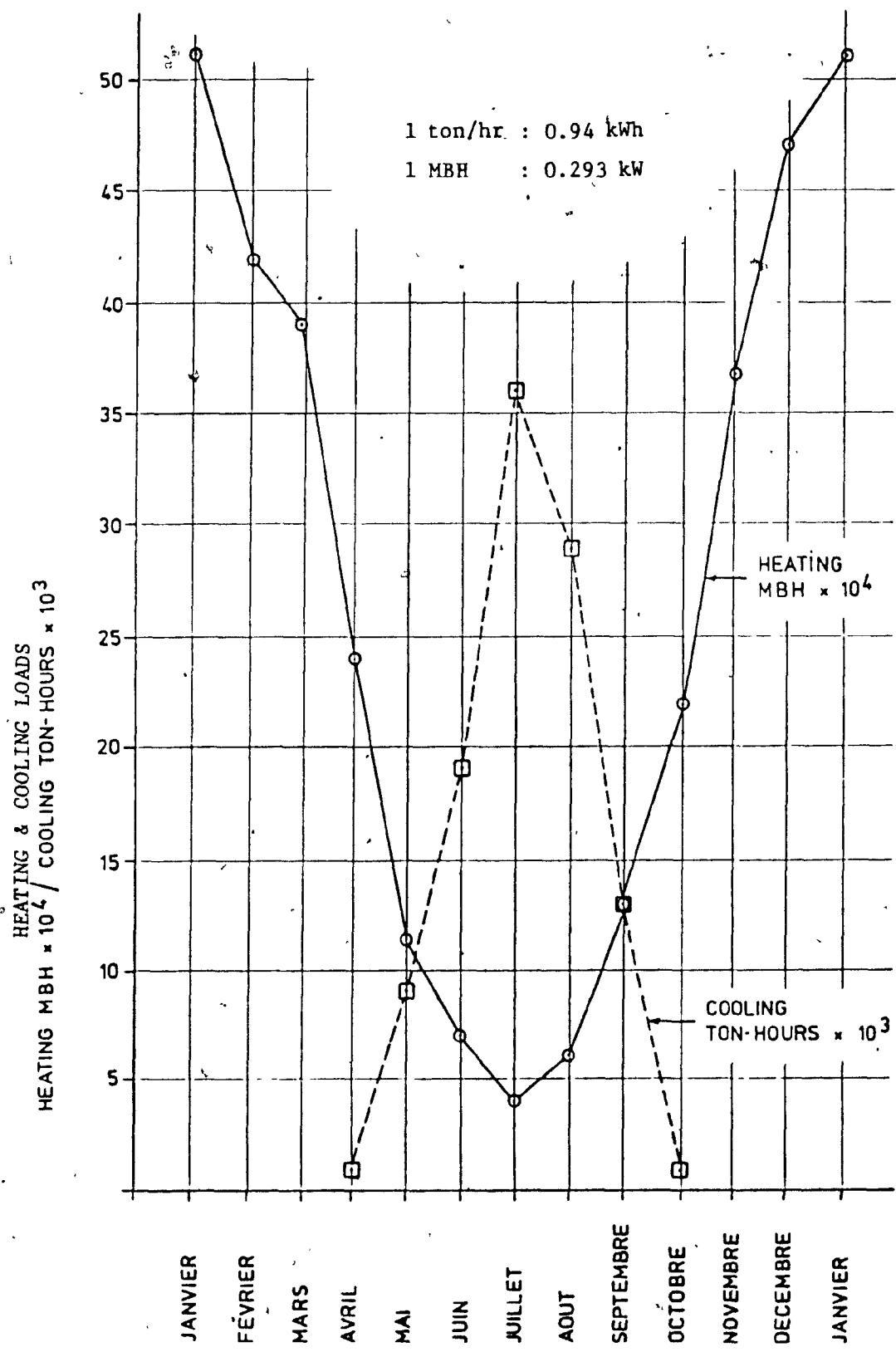
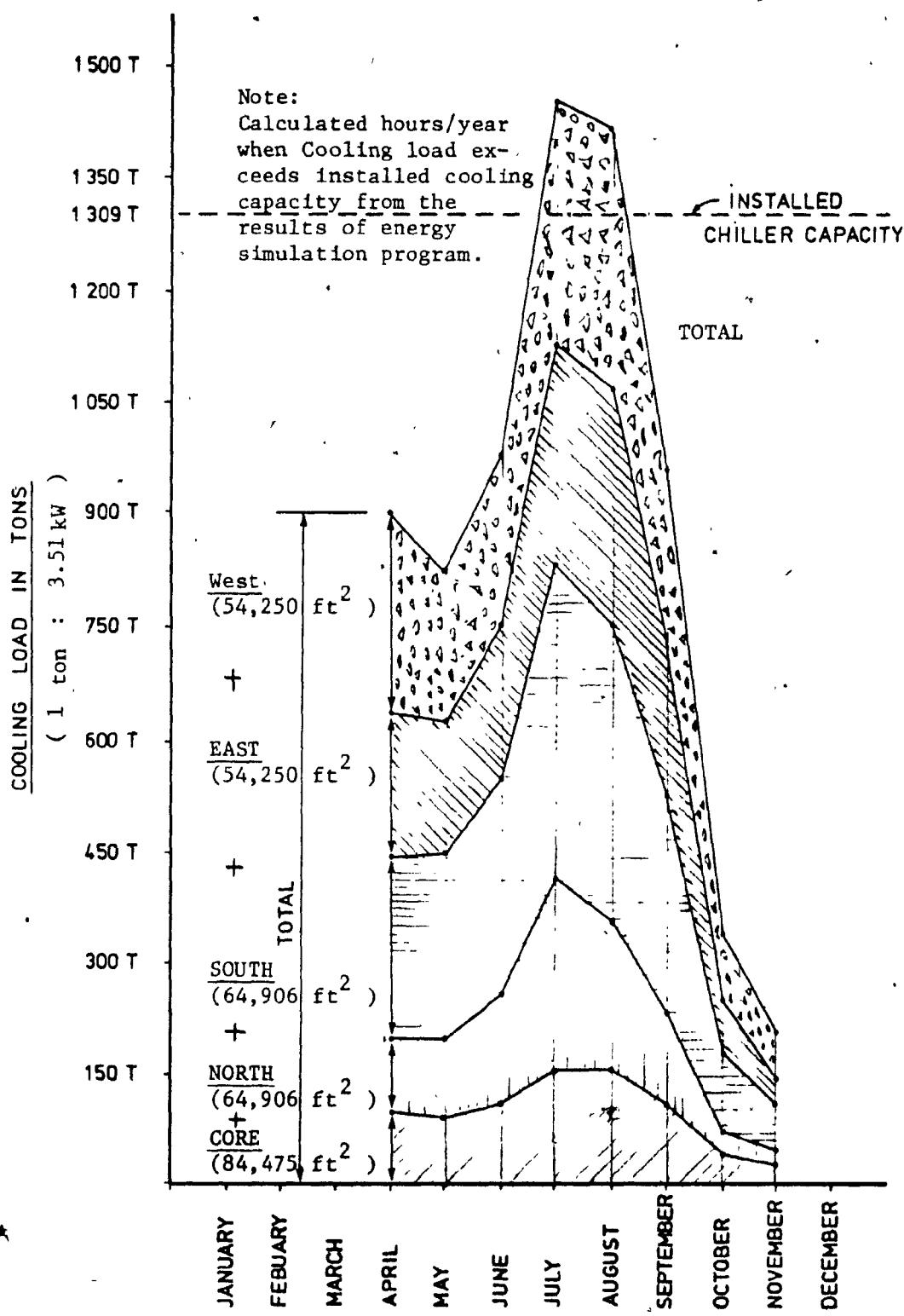


FIG. 5.6 TYPICAL MONTHLY ZONE-BY-ZONE COOLING LOAD VARIATIONS  
FOR ALTERNATIVE No. 1



## 5.2 Cost Analysis

In this section, the capital cost estimates and capital investment for various alternatives is presented. Then, the discounted payback period calculations are performed with the capital and energy-cost data.

### 5.2.1 Heating Plant Costs

The heating plant capacity and cost data for the alternatives, is shown in Table 5.6. To determine the boiler output capacity, the heat loss from the piping (15% of the building heating requirement), and the boiler efficiency (80% between 70-100% load) were considered. Two boilers, one for 60% and the other for 40% of the required heating load were selected. The plant costs were obtained from the boiler manufacturers, for various load conditions.

### 5.2.2 Chiller Costs

The chiller costs for various types of glazing considered are shown in Table 5.7. In establishing the chiller capital costs, the availability of a standard chiller of a given capacity was one of the factors. For the cooling tonnage between 1660 and 1466 (for clear double and clear triple glazing types), the selection of a single

TABLE 5.6 - HEATING PLANT CAPACITY AND COST DATA

Art.	Peak heating MBH	15% Pipe Loss	Input required using 80% efficiency	Unit required		Available unit MBH	Installed Cost	Cost/ sq. ft.
				60%	40%			
1	4435	665	6,375	3,825	2,550	4000	3000	61,760 .191
2	4601	690	6,613	3,968	2,645	4000	3000	61,760 .191
3	3672	550	5,278	3,167	2,111	3000	3000	59,175 .183
4	4290	643	6,166	3,700	3,488	4000	3000	61,760 .191
5	2874	431	4,131	2,479	1,652	3000	2000	55,687 .173
6	4682	702	6,730	4,038	2,692	4000	3000	61,760 .191
7	3266	490	4,695	2,817	1,878	3000	2000	55,687 .173
8	2728	409	3,921	2,352	1,569	3000	2000	55,687 .173
9	5392	808	7,750	4,650	3,100	5000	3000	64,345 .199
10	5246	787	7,541	4,524	3,017	5000	3000	64,345 .199
11	4775	716	6,864	4,118	2,746	4000	3000	61,760 .191

TABLE 5.7  
GLASS TYPE vs CHILLER CAPITAL COSTS  
1981

GLASS TYPE	DOUBLE CLEAR	TRIPLE CLEAR	SOLAR BRONZE	REFLECTIVE FILM
Tons Cooling	1660	1466	1288	1034
<u>One Machine</u>		Poor engineering practise to use one only machine for these two selections.		
\$ Capital Cost			\$208,000	\$180,000
\$/ ton			\$161.50	\$174.08
\$/ SQ. Ft.			\$ 0.64	\$ 0.56
<u>Two Machines (Equal)</u>				
\$ Capital Cost	\$312,000	\$304,000	\$244,000	\$224,000
\$/ ton	\$187.95	\$207.37	\$184.49	\$216.63
\$/ SQ. Ft.	\$ 0.97	\$ 0.94	\$ 0.76	\$ 0.69

Note: Building area: 322,787 Sq. Ft.

chiller was not considered to be a good practice. For lower capacities (1288 and 1034 tons) a single chiller may be used. The cost data, for the use of two appropriate chillers, for each of the 4 glazing types considered, was developed.

#### 5.2.3 Wall Panel Assembly Costs

The wall panel assembly costs for the combinations of the wall insulation and glazing types considered, are shown in Table 5.8. This information was obtained from the wall panel manufacturers.

#### 5.2.4 Payback Period

The simple' payback period summary, for the alternatives considered, is shown in Table 5.9. Due to relatively low energy costs in Canada, the payback period for the alternatives considered are well over 20 years; consequently, none of the alternatives are economic, in this case.

#### 5.2.5 Discounted Payback Period

The discounted payback period for each of the alternatives is calculated as follows:

TABLE 5.8  
COMPARATIVE COSTS (1981)  
WALL PANEL ASSEMBLY

Panel 48%	Glass 52%	Clear Double	Tinted Double (base case)	Reflective Double	Clear Triple
2" Insul. glass fibre		11.10 22.02	11.94 23.70	12.55 24.90	12.55 24.90
4" Insul. (base case)		11.25 22.32	12.09 24.00	12.70 25.20	12.70 25.20
6" Insul.		11.40 22.62	12.24 24.30	12.85 25.50	12.85 25.50

Floor area: 322,787

Wall area : 162,654

Cost \$ /  
/ SQ. Ft

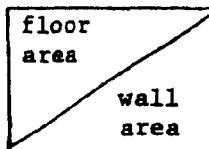


TABLE 5.9 - COMPARATIVE COST DATA AND PAYBACK PERIOD

Alternative N	Capital Costs (\$/ft <sup>2</sup> )				Change in Capital Cost	Total Energy Cost	Change in the Energy Cost	Operation (\$/ft <sup>2</sup> /yr)	Simple Payback Period (yrs)
	Heating Plant Unit Cost	Cooling Plant Unit Cost	Enclosure Unit Cost	Total Unit Cost					
1	0.191	0.760	12.09	13.041	-	0.376	-	-	
2	0.191	0.760	11.94	12.891	-0.15	0.390	+0.014	-	
3	0.183	0.760	12.24	13.182	+0.142	0.372	-0.004	35.5	
4	0.191	0.970	11.25	12.411	-0.63	0.374	-0.002	-	
5	0.173	0.940	12.70	13.812	+0.772	0.348	-0.028	27.57	
6	0.191	0.970	11.10	12.261	-0.780	0.407	+0.031	-	
7	0.173	0.940	12.55	13.663	+0.662	0.361	-0.015	44.13	
8	0.183	0.940	12.85	13.973	+0.932	0.344	-0.032	29.13	
9	0.191	0.690	12.70	13.581	+0.540	0.371	-0.005	-	
10	0.191	0.690	12.85	13.731	+0.690	0.371	-0.005	-	
11	0.191	0.97	11.40	12.561	-0.480	0.382	+0.006	-	

\* Simple payback period (yrs) : Capital Cost Investment (\$/ft<sup>2</sup>)  
 Operation Cost Saving (\$/ft<sup>2</sup> yr)

### Assumptions

Return on investment ( $R$ ) = 15%

Inflation rate for natural gas costs ( $\phi$ ) = 16%  
gas

Inflation rate for electrical energy costs ( $\phi$ ) = 17%  
elec

$$\sum_{Y=1}^{Y:P} (\text{Energy Savings})/\text{yr} = \Delta \$c$$

Y - year

P - payback period (discounted)

$\Delta \$c$  - additional capital costs (in comparison with  
base case)

The discounted payback periods for the alternatives considered, for different levels of energy costs, are shown in Table 5.10. From this table, it is obvious that the payback periods with the present energy costs are very long; consequently, for this period, none of the alternatives are economically justified.

The alternatives 5 and 8 could be economical if the total energy costs were four times the 1980 rates used in this analysis.

TABLE 5.10  
DISCOUNTED PAYBACK PERIOD FOR THE ALTERNATIVES  
FOR DIFFERENT ENERGY COST LEVELS

ALT.	1 <sub>x</sub> ENERGY COST	2 <sub>x</sub> ENERGY COST	3 <sub>x</sub> ENERGY COST	4 <sub>x</sub> ENERGY COST
1	--	--	--	--
2	--	--	--	--
3	26	13	9	7
4	--	--	--	--
5	22	12	8	6
6	--	--	--	--
7	>30	18	12	10
8	22	12	8	6
9	>30	27	18	15
10	>30	>30	24	19
11	--	--	--	--

\* Energy costs indicated in Tables 5.3 to 5.4 (1980 rates)

**CHAPTER 6**  
**CONCLUSIONS AND RECOMMENDATIONS**

## 6.

CONCLUSIONS AND RECOMMENDATIONS

The major objective of this study is to perform the energy requirement and cost analysis for an existing office buildings, for selected enclosure system alternatives.

1. As expected, the heating requirements for the building constituted a major portion of the total annual energy requirement, due to the climatic conditions for the location; consequently, for a given glazing percentage, the wall insulation thickness is the most significant parameter in determining the total energy requirement.
2. Among the glazing types considered, as expected, the clear triple glazing (best thermal resistance) with 6" - wall insulation (highest thermal resistance - alternative #8 which is the alternative for the least heating requirement) provided the least total energy. However, the least cooling energy requirement was provided by the alternative #9 (reflective double glazing with 4" insulation), due to the solar heat reflection characteristics of the glazing. Consequently, one can conclude that, for cases where the cooling load is predominant, the reflective double glazing may be a better choice.

3. Based on the discounted payback period analysis, for this project, none of the enclosure alternatives considered are economical (at the energy costs considered) which also indicates that the original design of the building (base case - alternative #1) is energy effective.

Recommendation:

For buildings with predominant heating load, analysis of the alternate heating, cooling and electrical system operation strategies may result in better energy and cost savings in comparison with the analysis of alternate enclosure systems, at the present energy costs in Canada; consequently, such an analysis is recommended for this project.

REFERENCES

REFERENCES

1. Cooling and Heating Load Calculation Manual  
ASHRAE, 1979.
2. Reference Manual for Energy Systems Analysis -  
series of Programs (Mariwether) Public Works  
Canada, 1980.
3. ASHRAE Handbook of fundamentals 1977 & 1981.
4. ASHRAE Handbook, Equipment 1979 & 1983.
5. ASHRAE Handbook, systems 1980.
6. ASHRAE Handbook, Applications 1978 & 1982.

APPENDIX A-1

APPENDIX A-1

Computer print-out showing input data and results of ERE analysis for the East zone of Run No. 7 (2" glass fibre wall insulation and triple clear glazing) and for the West zone of Run No. 7 (2" glass fibre wall insulation and triple clear glazing).

RUN No. 7

EAST ZONE



PREDATIVE MAINTENANCE PAPERS

NUMBER 12 M 1AM 2AM 3AM 4AM 5AM 6PM 7PM 8PM 9PM 10PM 11PM

SHINTO AND THE STATE

**POOR COPY**  
**COPIE DE QUALITEE IMPERIEURE**



## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

EAST RUN 7' TR GL 2PS

MONTHLY AND ANNUAL PEAK LOAD VALUES

MAR. 29/80

	PEAK HEATING, HRW	PEAK COOLING, TONS	HUMIDIFY HRW	INDIRECT PROCESS HRW	DIRECT PROCESS HRW	TOT BASE ELECTRIC KWH
	RH LOSS	RH GAIN	COILS	COILS	COILS	
60 JAN 80	839.	707.	382.2	0.0	0.	276.
DAY OF MONTH	3	3	10AM			10AM
TIME OF DAY	8AM	8AM				
DAY TYPE	DRY	DRY				
DRY BULB TEMP	51°	51°				
DRY POINT TEMP	51°	51°				
60 FEB 80	846.	856.	878.6	0.0	0.	276.
DAY OF MONTH	11	11	10AM			10AM
TIME OF DAY	8AM	8AM				
DAY TYPE	DRY	DRY				
DRY BULB TEMP	51°	51°				
DRY POINT TEMP	51°	51°				
60 MAR 80	1137.	860.	700.5	0.0	0.	276.
DAY OF MONTH	1	1	10AM			10AM
TIME OF DAY	8AM	8AM				
DAY TYPE	DRY	DRY				
DRY BULB TEMP	51°	51°				
DRY POINT TEMP	51°	51°				
60 APR 80	1211.	488.	623.2	102.1	0.	276.
DAY OF MONTH	7	7	10AM			10AM
TIME OF DAY	8PM	8AM				
DAY TYPE	DRY	DRY				
DRY BULB TEMP	51°	51°				
DRY POINT TEMP	51°	51°				
60 MAY 80	1176.	244.	647.1	175.4	0.	276.
DAY OF MONTH	10	10	10AM			10AM
TIME OF DAY	8AM	8AM				
DAY TYPE	DRY	DRY				
DRY BULB TEMP	51°	51°				
DRY POINT TEMP	51°	51°				
60 JUN 80	1127.	193.	660.5	205.8	0.	276.
DAY OF MONTH	23	23	8AM			8AM
TIME OF DAY	8AM	8AM				
DAY TYPE	DRY	DRY				
DRY BULB TEMP	51°	51°				
DRY POINT TEMP	51°	51°				

## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

EAST RUN 7 TIR SL 2PG

## MONTHLY AND ANNUAL PEAK LOAD VALUES

MARCH 29/80

				PEAK HEATING, MBH	PEAK COOLING, TONS	HUMIDCN MBH	INDIRECT PROCESS MBH	DIRECT PROCESS MBH	TOT BASE ELECTRIC KW
				RH LOSS COILS	RH GAIN COILS				
SE JUL 00	1112.	7	185.	1	602.0	200.3	20	0	270.
DAY OF MONTH	TIME OF DAY			4AM	4AM	SAH	4PM		9AH
DAY TYPE	DRY BULB TEMP			43	43	67	78		57
DEW POINT TEMP				41	41	54	60		46
SE AUG 00	961.	227.	227.	27	600.7	316.8	61	0	276.
DAY OF MONTH	TIME OF DAY			4AM	4AM	SAH	12N		9AH
DAY TYPE	DRY BULB TEMP			36	36	65	71		70
DEW POINT TEMP				34	34	53	62		58
SE SEP 00	821.	311.	207.1	207.1	205.8	205.8	61	0	276.
DAY OF MONTH	TIME OF DAY			3AM	1AM	9AM	2PM		9AH
DAY TYPE	DRY BULB TEMP			28	28	54	61		56
DEW POINT TEMP				23	23	38	41		32
SE OCT 00	689.	14	326.	27	600.1	75.5	64	0	276.
DAY OF MONTH	TIME OF DAY			6AM	2AM	10AM	1PM		9AH
DAY TYPE	DRY BULB TEMP			30	31	56	61		51
DEW POINT TEMP				28	28	50	52		43
SE NOV 00	707.	26	616.	24	300.8	45.2	60	0	276.
DAY OF MONTH	TIME OF DAY			6AM	6AM	10AM	2PM		9AH
DAY TYPE	DRY BULB TEMP			30	31	51	53		54
DEW POINT TEMP				28	28	41	44		34
SE DEC 00	891.	1	787.	27	300.1	67.1	67	0	276.
DAY OF MONTH	TIME OF DAY			4PM	10AM	11AM	4		9AH
DAY TYPE	DRY BULB TEMP			31	32	52	54		51
DEW POINT TEMP				29	29	43	46		36
SE ANN 00	1311.	1	1974.	602.0	316.8	61	0	0	276.
MONTH OF YEAR	DAY OF MONTH			4PM	10AM	11AM	4		9AH
TIME OF DAY				8PM	10AM	12N	1		10AH
DAY TYPE	DRY BULB TEMP			32	32	54	57		51
DEW POINT TEMP				30	30	45	48		38

POOR COPY  
COPIE DE QUALITE IMPERIEUSE

ENERGY REQUIREMENT ESTIMATE PROGRAM FOR  
EAST RUN 7 TR GL 270  
MONTHLY AND ANNUAL LOADS

MAR. 29/80

	HEATING BTU/HRS	HEAT HOURS	COOLING TON-HRS	HUMIDFCN BTU/HRS	IND PROC BTU/HRS	DIA PROC BTU/KWH	IND APU BTU/KWH	IND KWH	TOE ELEC C	TOE ELEC H	AUX F WHS
JAN	352642.	744	0	0	0	39601.	0	45569.	45144.	0	
FEB	267627.	672	0	0	0	37492.	0	44232.	81628.	0	
MAR	267325.	744	0	0	0	39601.	0	50565.	90159.	0	
APR	164926.	703	1228.	20	0	37813.	0	49772.	87570.	0	
MAY	81954.	680	8830.	147	0	39601.	0	51676.	91870.	0	
JUN	46344.	556	15930.	184	0	39395.	0	50507.	89895.	0	
JUL	29395.	436	27686.	226	0	42766.	0	53142.	95900.	0	
AUG	41456.	531	23819.	199	0	39601.	0	51677.	91870.	0	
SEP	69978.	637	9363.	119	0	39395.	0	49497.	86466.	0	
OCT	152657.	744	1185.	38	0	41163.	0	49255.	90321.	0	
NOV	259112.	726	1074	0	0	37813.	0	42677.	80866.	0	
DEC	323936.	744	0	0	0	39601.	0	42482.	88077.	0	
ANN	2099783.	750	97868.	937	0	473769.	0	561091.	108695.	0	

POOR COPY  
COPIE DE QUALITEE INFERIEURE

ENERGY REQUIREMENT ESTIMATE PROGRAM FOR EAST RUN 7 TR GL ERS		MARCH 29/80	
MIN ROOM TEMP	68.21 F	1/ 1/12 H	68.21 F 1/ 1/12 H
MAX ROOM TEMP	104.86 F	5/27/ 7AM	106.86 F 5/27/ 7AM
MAX ROOM HUMIDITY RATIO	.65.20 gm	1/ 3/ 7AM	
COUNT OF HOURS ON LIMITING CONDITIONS			
SPACE LOAD NOT MET (INCL. PICKUP LOAD)	0	1966	
SPACE LOAD NOT MET (W/O PICKUP LOAD)	0	976	
SYSTEM CAPACITY EXCEEDED	0	0	
LONGEST PICKUP PERIOD			
LAST HOUR OF OCCURRENCE	0/ 0/	4/ 1/ 4PM	
UNIT VALUES OF BUILDING ENERGY PEAKS AND CONSUMPTION			
HEATING PEAK	13.95 BTU/HGT		
HEATING CONSUMPTION	29.92 BTU/HGT		
FULL LOAD HEATING	29.92 HRS		
CLOUDING PEAK	5.80 TONS/1000 SQFT		
ORIG. CONSUMPTION	172.34 BTU/TON		
COOLING CONSUMPTION	171.62 BTU/HGT		
FULL LOAD COOLING	276 HRS		
PROCESS PEAK	0.0 BTU/HGT		
PROCESS CONSUMPTION	0.0 BTU/HGT		
INDIRECT	0.0 BTU/HGT		
DIRECT	0.0 BTU/HGT		
PULL LOAD PROCESS	0.0 HRS		
ELECTRIC PEAK	5.00 WATTS/SQFT		
ELECTRIC CONSUMPTION	19.34 KWH/3252 HRS		
FULL LOAD ELECTRIC			
MIN SUPPLY AIRFLOW IS	176230. CFM	1/ 3/10AM	MIN SUPPLY AIRFLOW IS 30492. CFM 1/ 1/ 1AM

POOR COPY  
COPIE DE QUALITEE INFERIEURE

## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

EAST RUN 7 TR 6L 2PG				MARCH 20/86			
YEAR	MO	DT	OP	LOAD	SHED	DECM	MUMID
TYPE	DY	CC	DP	WATTS	THRS	DELM	DEPROC
000 JAN 000				0	0	0	0
-026 1 -19	10991322	-07099822	0	730275	0	0	0
-024 3 -21	921925	-63579	0	1501863	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0
000 FEB 000				0	0	0	0
1297 2 0 -17	24765	-771051	0	73847285	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0
000 MAR 000				0	0	0	0
1423 3 -2 24	24765	-0412986	0	0508221	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0
000 APR 000				0	0	0	0
2216 4 1 7	7644514	-0740617	0	7153226	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0
000 MAY 000				0	0	0	0
3109 5 0 -30	24765	-0403397	0	0458632	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0
000 JUN 000				0	0	0	0
3824 6 -1 42	24765	-0712148	0	5697284	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0
000 JUN 000				0	0	0	0
3687 6 1 80	1613259	-0459971	0	2271072	0	0	0
0 0 0	0	0	0	0	0	0	0
0 0 0	0	0	0	0	0	0	0

POOR COPY  
COPIE DE QUALITEE INFERIEURE

## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

EAST RUN 7 TA 8L 2PM

YAHR	MO	DT	DA	SORAD	TRLOAD	BHSHAC	72H0	DACIN	ATEMP	AMAT	COST	TSA/IVFC	CCLOAD	CLI	D-PROC	MUMIO	
TIME	YY	CC	OP	HR13	HR13	HRSHR	72H0	DECRH	RHT	RHTA	HOST	2VFR	ACLOAD	WT	I-PROC	SELCT	
*** JUL ***																	
4:34P - 7	1	43		24765.	0.	-779441.		0.	141.7	141.7	30492.		18504.	0	0	0	
4:4AH - 1	0	41		24765.	208.			0.	141.7	141.7	30492.		18504.	0	0	57.	
5:00P 7	1	78		1402701.		-815215.		35035824.	1400410.	76230.	136.9	78.0	76230.	3596062.	1	0	0
5:04P 28	2	60		521925.		26040096.		2503480.	23.71.	154.7	55.0	55.0	0.	0.	0	276.	
*** AUG ***																	
6:11AM 8	1	36		24765.	208.			0.	1400162.		30492.		22710.	0	0	0	
6:54AM 27	0	56						0.	2668572.		0.	104.8	144.2	174.8		0	
8:34A 1	0	77		2449386.		-724717.		46736336.	2163367.	76230.	160.8	77.0	76230.	3777314.	1	0	0
12:4N 11	1	82		4267700.		27260.		34373808.	36268192.	29.47	179.6	54.3	0.	0.	0	259.	
*** SEP ***																	
6:12T 9	2	29		24765.	208.			0.	1447907.		30492.		31121.	0	0	0	
6:54AH 13	1	23						0.	134188.		0.	88.8	93.8	97.8		0	
8:51A 9	1	80		1264316.		-414316.		89265920.	1371925.	76230.	80.7	80.7	76230.	2669851.	1	0	0
9:2PM 29	0	41		521925.		63576.		19963664.	21651048.	0.0	144.1	39.3	55.0	0.	0	0	
*** OCT ***																	
7:17A 10	1	27		24765.	208.			0.	1466197.		30492.		319620.	0	0	0	
7:24A 27	0	16						0.	1342070.		0.	114.0	116.4	116.4		0	
7:21A 10	1	61		831231.		-7242684.		7206298.	111072.	76230.	88.9	61.0	76230.	905612.	1	0	0
7:2PM 28	4	32		521925.		63579.		63579.	6168624.	0.0	103.7	21.6	55.0	0.	0	0	
*** NOV ***																	
7:50A 11	1	68						0.	14707388.		30492.		30492.	0	0	0	
7:54AM 26	0	11		24765.				0.	1460889.		0.	114.0	116.4	116.4		0	
7:43A 11	1	55		833312.		-173609.		889875.	881426.	762904.	76.0	50.0	82904.	963492.	0	0	0
7:2PM 26	4	24		521925.		63579.		63579.	1563127.	0.0	154.4	19.8	55.0	0.	0	0	
*** DEC ***																	
6:54A 12	2	26		2342031.		-987321.		0.	12080373.		0.	76.0	98.8	104.8		0	
7:0AM 27	0	30		24765.				0.	2139970.		0.	98.8	137.5	103.8	30492.	0	
6:54A 0	0	0						0.	0.		0.	0.	0.	0.	0.	0.	
6:54A 0	0	0						0.	0.		0.	0.	0.	0.	0.	0.	

POOR COPY  
COPIE DE QUALITEE INFERIEURE

RUN NO. 7

WEST ZONE

## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

WEST RUN 7 TR GL 2P0

SYS TP	FLOOR	SOLAR	-P1P	P1P1	P1L1A	P1L1H	P1L	P1M	P1M1	P1M1L	P1M1H	P1P	P1P1	P1P1L	P1P1H	
8	94250.	3021.0	136.0	46.	945.0	15.	0.	27.7	0.	0.	0.	171.	0.	171.	0.	171.

SOLAR AREA PERCENTAGES	0.02	0.03	0.05	0.07	0.08	0.09	0.10	0.11	0.12	0.13	0.14	0.15	0.16	0.17	0.18	0.19
0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

HEATING LOADS	WIND WIND															
68.	0.	-25.	0.	747.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

BATTERY BASSOLN	SKOT SKOT	SKOT SKOT	HSP HSP													
0.0	34.0	1.6	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

AIRFLOW DATA	TGA TGA	ZVPM ZVPM	DAVMAX DAVMAX	DAMINW DAMINW	DAMINW DAMINW	TLEU TLEU	TLCBU TLCBU	CDTU CDTU								
44384.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

ADDITIONAL SYSTEM DATA	RECA NORMAL OPERATION	HT HT														
0.0	64	0.	-25	757.	0.	0.	-25	757.	0.	0.	0.	0.	0.	0.	0.	0.

SHUTOFF AND SETBACK TIME SCHEDULES	NO HRDR HRDR	NO MO MO	HRDR HRDR													
1	1.0	1	1.0	1	1.0	1	1.0	1	1.0	1	1.0	1	1.0	1	1.0	1
2	1.24	0	1/12	0	0	0	0	0	0	0	0	0	0	0	0	0

PERCENTAGE VARIATION PROFILES	1	0.018	100.0	3	80.0	1	80.0	1	100.0	3	100.0	4	0.01	0	0.01	0
2	0.024	10.0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
3	0.027	10.0	0	20.0	1	20.0	1	20.0	2	20.0	2	20.0	1	20.0	1	20.0
4	0.024	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

DAY NUMBER FOR JANUARY 1 IS 1	LEAP YEAR KEY IS 1	1	1	1	1	1	1	1	1	1	1	1	1	1	1	1
MOL	1/1	1	4/4	0	14/4	0	1	4/4	0	14/4	0	1	4/4	0	14/4	0

DAY TYPE DESCRIPTION AND DISTRIBUTION	KEY	TIME SCHEDULES	SAT	SUN	MON	TUE	WED	THU	FRI	SAT	SUN	MON	TUE	WED	THU	FRI	
DT PCT VARIATION PROFILE NUMBERS	NO. SEC'D	SEC'D	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

SPECIAL HOURLY PRINTOUT	MO DAY/DAY															
NO. SEC'D	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

MAN. 29/80  
TIME FEATURES - NO. OF DAYS

SHUTOFF AND PRIMARY PULC INTERRUPTIONS

COOLING LOADS

CDTB CDDB CDDE

CDTB COST MRCO SCST SHRC

POOR COPY  
COPIE DE QUALITEE INFERIEURE

## PERCENTAGE VARIATION PROFILES

	NUMBER	J2	W	1AM	2AM	3AM	4AM	5AM	6AM	7AM	8AM	9AM	10AM	11AM	12N	1PM	2PM	3PM	4PM	5PM	6PM	7PM	8PM	9PM	10PM	11PM
1	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	
2	3.	5.	5.	5.	5.	5.	5.	5.	5.	10.	20.	50.	90.	80.	60.	40.	20.	100.	100.	100.	100.	100.	100.	100.	100.	
3	5.	5.	5.	5.	5.	5.	5.	5.	5.	10.	20.	50.	90.	80.	60.	40.	20.	100.	100.	100.	100.	100.	100.	100.	100.	
4	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	5.	

## SHUTOFF AND SETBACK TIME SCHEDULE

	12 M	1AM	2AM	3AM	4AM	5AM	6AM	7AM	8AM	9AM	10AM	11AM	12N	1PM	2PM	3PM	4PM	5PM	6PM	7PM	8PM	9PM	10PM	11PM
1	JAN																							
2	FEB																							
3	MAR																							
4	APR																							
5	MAY																							
6	JUN																							
7	JUL																							
8	AUG																							
9	SEP																							
10	OCT																							
11	NOV																							
12	DEC																							

## SHUTOFF AND SETBACK TIME SCHEDULE

	12 M	1AM	2AM	3AM	4AM	5AM	6AM	7AM	8AM	9AM	10AM	11AM	12N	1PM	2PM	3PM	4PM	5PM	6PM	7PM	8PM	9PM	10PM	11PM
1	JAN																							
2	FEB																							
3	MAR																							
4	APR																							
5	MAY																							
6	JUN																							
7	JUL																							
8	AUG																							
9	SEP																							
10	OCT																							
11	NOV																							
12	DEC																							

POOR COPY  
COPIE DE QUALITEE INFERIEURE

## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

WEST RUN 7 TR GL 2PG

SYSTEM TYPE 8 STD. VARIABLE VOLUME

UNIT VALUE OF INPUT DATA FLOOR AREA = 54250. SQFT

MAR. 29/86

	LATENT	SENSIBLE	
SOLAR LOAD	55.69	BTUH/SQFT	
TRANSMISSION LOSS	0.148	BTUH/SF-SQFT	
TRANSMISSION GAIN	0.150	BTUH/F-SQFT	
TOTAL INTERNAL LOAD	1.10	BTUH/SQFT	
PEOPLE	1.07	BTUH/SQFT	
LIGHTS	1.023	BTUH/SQFT	
EQUIPMENT	0.0	BTUH/SQFT	
MISCELLANEOUS	0.43	BTUH/SQFT	
MINIMUM O/A - COOLING	4.54	BTUH/SQFT	
OR 0.00	0.156	BTUH/SQFT	
OR 0.00	10.00	PCY OF TOTAL	
PEAK COOLING LOAD	9.60	BTUH/SQFT	
ROOM SENS ONLY	67.427	BTUH/SQFT	
TOTAL	67.427	BTUH/SQFT	
OR 0.00	154.14	TONS/1000 SQFT	
COOLING CAPACITY	24.00	TONS/1000 SQFT	
ROOM SENS ONLY	31.62	SQFT/ZONATION	
MINIMUM O/A - HEATING	0.0	BTUH/SQFT	
OR 0.00	0.156	BTUH/SQFT	
OR 0.00	10.00	PCY OF TOTAL	
PEAK HEATING LOAD	0.0	BTUH/SQFT	
HEATING CAPACITY	16433.16	BTUH/SQFT	
TOTAL PROCESS LOAD	0.0	BTUH/SQFT	
INDIRECT	0.0	BTUH/SQFT	
DIRECT	0.0	BTUH/SQFT	
TOTAL BASE ELEC LOAD	5.66	WATTS/SQFT	
SOURCE A	2.15	WATTS/SQFT	
SOURCE B	2.01	WATTS/SQFT	
SOURCE C	2.51	WATTS/SQFT	
FOR PRIMARY AIRFLOW	1.555	CFM/SQFT	
OR 0.00	164.8	CFM/VTON	
HEATING INSIDE = 60. / 0. OUTSIDE = 25. / 0.			
COOLING INSIDE = 70. / 0. OUTSIDE = 85. / 0.			

POOR COPY  
COPIE DE QUALITEE INFERIEURE



## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

WEST RUN 7 TR 0L 2PG

MONTHLY AND ANNUAL PEAK LOAD VALUES

		PEAK HEATING, WASH		PEAK COOLING, TONS		INDIRECT PROCESS WASH		DIRECT PROCESS WASH		TOT. BASE ELECTRIC KWH	
		RH LOSS	COILS	RH GAIN	COILS	MUMIDFCN WASH	MUMIDFCN WASH	RH	RH	RH	RH
** JUL 68	1312.	185.	1	834.9	326.0	28	0	0	0	0	290.
DAY OF MONTH		SAM	4AM		SPM	4PM					
TIME OF DAY											
DAY TYPE											
DRY BULB TEMP		51	43		51	78					10AM
DEW POINT TEMP		51	41		47	60					11AM
** AUG 68	1239.	227.	1	805.7	348.5	0	0	0	0	0	290.
DAY OF MONTH		11	27		4PM	12N					2PM
TIME OF DAY		4AM	SAM								
DAY TYPE											
DRY BULB TEMP		50	36		71	77					76
DEW POINT TEMP		50	36		51	62					52
** SEP 68	1123.	311.	1	823.1	227.8	0	0	0	0	0	290.
DAY OF MONTH		SAM	6AM	13	29	29					11AM
TIME OF DAY											
DAY TYPE											
DRY BULB TEMP		37	23		35	41					62
DEW POINT TEMP											
** OCT 68	926.	326.	1	652.4	83.5	28	0	0	0	0	290.
DAY OF MONTH		14	27		21	28					2PM
TIME OF DAY		6AM	2AM		3PM	2PM					
DAY TYPE											
DRY BULB TEMP		30	21		51	61					65
DEW POINT TEMP		30	19		23	32					50
** NOV 68	830.	31A.		AAA.1	14	80.5					290.
DAY OF MONTH		30	26		1	1					2PM
TIME OF DAY		6AM	6AM		30	50					
DAY TYPE											
DRY BULB TEMP		2	1		23	24					35
DEW POINT TEMP		1	0								26
** DEC 68	1072.	757.	1	326.6	0	0	0	0	0	0	290.
DAY OF MONTH		22	22								
TIME OF DAY		4PM	10AM		2PM						2PM
DAY TYPE											
DRY BULB TEMP		619	-26		35	0					29
DEW POINT TEMP		626	-30		21	0					17
** ANN 68	1515.	757.	1	846.0	5	348.5					290.
MONTH OF YEAR		42	12		14	5					
DAY OF MONTH		SAM	10AM		SPM	12N					2PM
TIME OF DAY											
DAY TYPE											
DRY BULB TEMP		31	-26		68	77					35
DEW POINT TEMP		30	-30		26	62					26

POOR COPY.  
COPIE DE QUALITEE, INFERIEURE)

ENERGY REQUIREMENT ESTIMATE PROGRAM FOR  
WEST RUN 7 TR GL 2P0  
MONTHLY AND ANNUAL LOADS

	HEATING WATU	HEAT TON-HRS	COOLING HOURS	HUMIDFCN MBTU	IND PROT MBTU	DIR PROT MBTU	BS ELECTC KWH	BS ELECTC KWH	TOT AS ELS	AUX F WATU
JAN	3478442.	744	0.	0.	0.	0.	398016.	0.	51267.	0.
FEB	297827.	672	0.	0.	0.	0.	37402.	0.	66799.	0.
MAR	267325.	744	0.	0.	0.	0.	39801.	0.	51793.	0.
APR	166629.	703	1320.	20.	0.	0.	37413.	0.	49907.	0.
MAY	81954.	660	8883.	147	0.	0.	39801.	0.	50405.	0.
JUN	48344.	899	14616.	184	0.	0.	39198.	0.	48850.	0.
JUL	293395.	436	27363.	226.	0.	0.	36276.	0.	54156.	0.
AUG	41456.	931	21765.	169	0.	0.	39401.	0.	51114.	0.
SEP	166670.	637	10158.	119	0.	0.	36395.	0.	51224.	0.
OCT	192647.	744	1326.	38	0.	0.	41163.	0.	52161.	0.
NOV	235112.	720	199.	4.	0.	0.	17813.	0.	48616.	0.
DEC	323936.	744	0.	0.	0.	0.	36601.	0.	49566.	0.
ANN	2099793.	7916	86095.	937	0.	0.	473769.	0.	605921.	107036.

POOR COPY  
COPIE DE QUALITEE INFERIEURE

## ENERGY REQUIREMENT ESTIMATE PROGRAM, FOR

WEST RUN, Y	TR 01 2PM	30 OR 58	HO/DY/TIME	NORM OPEN	HO/DY/TIME
MIN ROOM TEMP	68.423 F	1/1/12 H	ABOVE	1/1/12 H	
MAX ROOM TEMP	140.93 F	4/6/11AM	180.93 F	4/6/11AM	
MAX ROOM HUMIDITY RATIO	.65.15 GR	1/3/7 AM			
COUNT OF HOURS ON LIMITING CONDITIONS			HEATING	COOLING	
SPACE LOAD NOT MET (INCH PICKUP LOAD)	0	1269			
SPACE LOAD NOT MET (W/O PICKUP LOAD)	0	1162			
SYSTEM CAPACITY EXCEEDED	0	0			
LONGEST PICKUP PERIOD	0	7			

LAST HOUR OF OCCURRENCE 07/07 27/12 3PM

## UNIT VALUES OF BUILDING ENERGY PEAKS AND CONSUMPTION

HEATING PEAK CONSUMPTION	13.95	HTRU/SQFT	189500. BTU/SQFT/YR
HEATING CONSUMPTION	36.63	HTRU/SQFT	
FULL LOAD HEATING	276A.	HRS	
COOLING PEAK OR CONSUMPTION	6.42 TONS/1000 SQFT		
COOLING CONSUMPTION	55.68 SQFT/TON		
FULL LOAD COOLING	2159 HRS		
PROCESS PEAK CONSUMPTION	0.0 HTRU/SQFT		
PROCESS CONSUMPTION	0.0 HTRU/SQFT		
DIRECT	0.0 HTRU/SQFT		
PULL LOAD PROCESS	0.0 HRS		
ELECTRIC PEAK CONSUMPTION	5.35 HTRU/SQFT		
ELECTRIC CONSUMPTION	16.96 HTRU/SQFT		
FULL LOAD ELECTRIC	3723. HRS		

MAX SUPPLY AIRFLOW IS 64300. CFM 1/3/32 3PM MIN SUPPLY AIRFLOW IS 23794. CFM 1/1/12 1AM

## ENERGY REQUIREMENT ESTIMATE PROGRAM FOR

WEST RUN 7 TR 9L 2PG

YEAR	MO	DT	DR	SORAD	TRLOAD	RHML	RHSHRC	RHSHRI	TZSHC	DACFM	ATEMP	COST	TSA/ZVFC	CCLOAD	CLT	DLLOAD	CLT	D-PROC	I-PROC	HUMID	AIRLEC
1988	JAN	1	-19	219993.	-949549.	0.	-205330.	14454.	78.0	50.0	11786.	0.	70633.	0.	0.	0.	0.	0.	0.	0.	
	2	-21	521929.	63570.	0.	543156.	0.0	101.7	17.3	53.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	205.	
	3	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
	4	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
1988	MAR	1	-13	1397.	66.	-932230.	0.	4907465.	0.0	104.7	104.7	104.7	33754.	0.	65601.	0.	0.	0.	0.	0.	
	2	-17	24765.	205.	0.	-2058901.	0.0	104.3	109.7	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	63.	
	3	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
	4	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
1988	APR	1	-14	1423.	66.	-806702.	205.	5761936.	0.0	78.0	98.1	98.1	33754.	0.	880367.	0.	0.	0.	0.	0.	
	2	-16	24765.	205.	0.	5169074.	0.0	98.1	137.3	193.1	0.	0.	0.	0.	0.	0.	0.	0.	0.	63.	
	3	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
	4	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
1988	MAY	1	-17	27216.	77.	995777.	-9975029.	0.	-370922.	21962.	78.0	59.0	33754.	0.	68784.	0.	0.	0.	0.	0.	
	2	-1	495229.	415.	0.	121917.	0.0	130.0	130.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	72.	
	3	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
	4	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
1988	JUN	1	-18	29225.	78.	983346.	-970516.	38795560.	9884761.	84365.	149.4	76.0	64365.	2891601.	1.	0.	0.	0.	0.	0.	0.
	2	0	521929.	633979.	633979.	34945296.	37062336.	0.0	161.4	244.2	85.0	0.	0.	0.	0.	0.	0.	0.	0.	290.	
	3	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
	4	0	0	0	0.	0.	0.	0.	0.	0.0	0.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
1988	JUL	1	-19	3100.	79.	-780353.	0.	0.	-498588.	0.	178.0	127.3	127.3	33754.	0.	243922.	0.	0.	0.	0.	0.
	2	0	30	24765.	205.	0.	-449168.	0.	127.3	132.3	0.	0.	0.	0.	0.	0.	0.	0.	0.	63.	
	3	1	76	9007812.	2496534.	32606640.	9031201.	84365.	149.5	76.0	64365.	2366938.	1.	0.	0.	0.	0.	0.	0.	0.	290.
	4	29	521929.	633979.	633979.	31960192.	31960192.	0.0	150.5	244.2	85.0	0.	0.	0.	0.	0.	0.	0.	0.	0.	
1988	JUN	2	36	34416.	42.	-928506.	0.	0.	-593741.	0.	75.0	179.2	159.2	33754.	0.	103455.	0.	0.	0.	0.	0.
	3	0	40	24765.	205.	0.	-5767788.	0.0	159.2	147.6	164.2	0.	0.	0.	0.	0.	0.	0.	0.	63.	
	4	1	69	9614865.	470670.	39112168.	5665619.	84365.	136.8	80.0	85.0	84365.	2156073.	1.	0.	0.	0.	0.	0.	0.	
	5	1	41	921929.	633979.	26442722.	26442722.	0.0	151.0	244.2	85.0	0.	0.	0.	0.	0.	0.	0.	0.	290.	

POOR COPY  
COPIE DE QUALITEE INFERIEURE

## ENRACK REQUIREMENT ESTIMATE PROGRAM, FOR

WEST RUN 7 TR GL 270				TRLOAD				BHSHRC BHSHRAJ		29SH T29SH		DACPW DELHR		AVAT HRRA		COST-TSA/ZVPC HOST-ZVPH		CCLOAD MCLOAD		CLT MLT		D-PROC I-PROC		HUMID RELEC	
YR/MR	MO	DT	DB	SORAD	RAIL	TIME	DY	CC	OP	TIME	DY	CC	OP	TIME	DY	CC	OP	TIME	DY	CC	OP	TIME	DY	CC	OP
655 JUL 1	43	0	10003067.	-10003067.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
4349 1 1 0	41	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
5009 7 1 76	9007812.	-9007812.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
4PM 2A 4 60	521925.	521925.	633579.	3678740.	368884480.	36916168.	36942756.	369503648.	36957192.	369603648.	36962064.	3696331.	3696365.	3696485.	3696585.	3696685.	3696785.	3696885.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.
655 AUD 000	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
5710 5 1 38	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
5710 5 27 0	38	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
5341 6 1 77	3014599.	-3014599.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
12N 11 2 62	426770.	426770.	27260.	39579.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	41675232.	
655 DEC 000	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
6127 9 1 23	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
6519 9 1 40	6573674.	-6573674.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
2PM 29 0 41	521925.	521925.	633579.	36962064.	3696331.	3696365.	3696485.	3696585.	3696685.	3696785.	3696885.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	
655 OCT 000	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
7127 10 1 27	1	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
2PM 27 0 19	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
7215 10 1 61	5477742.	-5477742.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
2PM 29 4 32	521925.	521925.	633579.	36962064.	3696331.	3696365.	3696485.	3696585.	3696685.	3696785.	3696885.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	
655 NOV 000	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
7127 10 1 27	1	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
7215 10 1 61	5477742.	-5477742.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
2PM 29 4 32	521925.	521925.	633579.	36962064.	3696331.	3696365.	3696485.	3696585.	3696685.	3696785.	3696885.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	
655 DEC 000	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
7003 11 1 76	24765.	24765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
7431 11 1 56	4399047.	-4399047.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
2PM 29 0 24	521925.	521925.	633579.	36962064.	3696331.	3696365.	3696485.	3696585.	3696685.	3696785.	3696885.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	3696985.	
655 DEC 000	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
10AH 27 5 30	224765.	224765.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

POOR COPY  
COPIE DE QUALITEE INFERIEURE

**APPENDIX A-2**

**GLOSSARY OF TERMS USED IN ENERGY REQUIREMENT ESTIMATE (ERE)**

APPENDIX A-2

GLOSSARY OF TERMS USED IN ENERGY REQUIREMENT ESTIMATE (ERE)

<u>Term</u>	<u>Location on Input Sheets</u>		<u>Remarks</u>
	<u>Card</u>	<u>Cols.</u>	
AR KY	5A	73	Key for air handling unit fan operation during setback.
BEA	10	15 - 15	Percentage variation profile number for the source A base electric load.
BEB	10	16 - 17	Percentage variation profile number for the source B base electric load.
BEC	10	18 - 19	Percentage variation profile number for the source C base electric load.
CDDB	4	38 - 40	Ambient dry bulb on the cooling cycle, F.
CDDP	4	41 - 43	Ambient design dew point on the cooling cycle, F.

CDST	4	51 - 53	Cold deck supply temperature or primary cold coil leaving temperature, F.
CDTG	4	44 - 50	Cooling design transmission gain, MBh.
CDTL	5	65 - 67	Cold air supply temperature at the lower ambient limit, F.
CDTU	5	59 - 61	Cold air supply temperature at the upper ambient limit, F.
CDTUO	5	68 - 70	cold air supply temperature to be in effect during unoccupied periods (determined by outside air shutoff schedule), F
CIDB	4	32 - 34	Indoor design dry bulb on the cooling cycle, F
CIDP	4	35 - 37	Indoor design dew point on the cooling cycle, F.
CS	10	22	Number of the shutoff schedule to be used for this day type for the cooling system.

CSCAP 3 41 - 45 cooling system capacity, tons.

DPR 10 12 - 13 Percentage variation profile number for the direct process load.

DR 6 Duration of a shutoff period in hours.

EFEL 5 77 - 80 Latent efficiency of outside air/exhaust air heat recovery system.

EFFS 5 73 - 76 Sensible efficiency of outside air/exhaust air heat recovery system.

EL KY 5-A 39 Key for the supplemental heating to be converted to electricity on the ERE output tape.

EQP 10 6 - 7 Percentage variation profile number for the internal load from equipment.

FLOOR 2 9 - 15 Air conditioned floor area, sq.ft.

HDDB 4 7 - 9 Ambient design dry bulb on the heating cycle, F.

HDDP        4    10 - 12    ambient design dew point on the heating cycle, F.

HDSTL       4    29 - 31    Hot deck supply temperature at the lower ambient limit, F.

HDSTU       4    23 - 25    Hot deck supply temperature at the upper ambient limit, F.

HDTL        4    13 - 19    Heating design transmission loss, MBH

HI AM       5-A    68 - 69    Ambient temperature for highest room dp

HI DP       5-A    66 - 67    Highest room dew point in humidification reset schedule.

HIDB       4    1 - / 3    Indoor design dry bulb on the heating cycle, F.

HIDP       4    4 - 6    Indoor design dew point on the heating cycle, F.

HM KY       5-A    72       Sprayed coil humidification key.

HR           6       Beginning hour number of a shutoff period.

HRCD	4	54 - 56	Humidity ratio of the cold deck or primary cold coil, grains/lb,
HRS	3	55 - 56	Number of hours of interruption duration.
HS	10	21	Number of the shutoff schedule to be used for this day type for the heating system.
HSCAP	3	35 - 40	Heating capacity, MBH.
HSF	4	76 - 80	Heat storage factor, BTu/F-sq.ft.
JN KY	5-A	80	Infiltration key
INTCST	3	49 - 51	Cooling system shutoff temperature, F.
INTHST	3	46 - 48	Heating system shutoff temperature, F.
IPR	10	10 - 11	Percentage variation profile number for the indirect process load.
KEYOA	10	73	Outside air key
KHSF	4	80	Key for duration of pickup of heat storage.

LO AM      5-A 70 - 71      Lowest ambient temperature for  
humidification.

LTS      10      4 - 5      Percentage variation profile number for  
the internal load from lights.

MO/MO      6      Month numbers beginning with and thru  
which preceding shutoff schedule shall  
apply.

MSC      10      8 - 9      Percentage variation profile number for  
the internal load from any miscellaneous  
source.

OA      10      20      Number of the shutoff schedule to be used  
for this day type for outside air.

OAMINC      5      29 - 35      Minimum outside airflow on the cooling,  
cfm.

OAMINH      5      36 - 42      Minimum outside airflow on the heating  
cycle, cfm.

OAMINS      5      43 - 49      Minimum outside airflow during "shutoff"  
period, cfm.

OAVMAX	5	22 - 28	Maximum outside airflow, cfm.
OP	10	45	Time schedule number for off-peak electrical usage.
PBEA	2	66 - 70	Peak base electric load for source A, kw
PBEB	2	71 - 75	Peak base electric load for source B, kw
PBEC	2	76 - 79	Peak base electric load for source C, kw
PCILTR	2	35 - 37	Percent of lighting load which passes directly into the return air stream.
PCIML	2	51 - 53	Percent of miscellaneous internal load which is latent.
PCIPL	2	27 - 29	Percent of internal load from people which is latent.
PCLTRH	2	38 - 40	A percent of the percent of the lighting load which passes directly into the return air stream which is available for reheat.
PFIT	3	52 - 54	Primary fuel interruption temperature, F.

<u>PIE</u>	2	41 - 45	Peak internal load from equipment MBh
<u>PILT</u>	2	30 - 34	Peak internal load from lights, MBh
<u>PIM</u>	2	46 - 50	Peak internal load from any miscellaneous source, MBh
<u>PIP</u>	2	22 - 26	Peak internal load from people, MBh
<u>PPD</u>	2	60 - 65	Peak direct-fired process load, MBh
<u>PPI</u>	2	54 - 59	Peak indirect process load, MBh
<u>PR KY</u>	5-A	65	Process key indicating which process load is to be reduced by preceding percentages.
<u>RASOLM</u>	4	66 - 69	Return air solar gain at summer design, MBh
<u>RATRLM</u>	4	73 - 75	Return fan delta T or temperature rise, F
<u>SB</u>	10	23	Number of the setback time schedule to be used for this day type.
<u>SBT SCH</u>	10	24	Number of the setback temperature schedule to be used for this day type.

SCDST	4	57 - 59	Secondary cold deck supply temperature, F.
SFDT	4	70 - 72	Supply fan delta T or temperature rise, F.
SHRCD	4	60 - 61	Humidity ratio of the secondary cold coil, grains/lb.
SOLAR	2	16 - 21	Reference solar load, MBh
SY KY	5-A	40	Supplemental system operation key
SYS TP	2	6	System Type.
TLCDL	5	62 - 64	Lower ambient temperature limit for cold supply air temperature schedule, F.
TLCDU	5	58 - 58	Upper ambient temperature limit for cold supply air temperature schedule, F.
TLEL	5	53 - 55	Lower temperature limit for the economizer cycle, F.
TLEU	5	50 - 52	Upper temperature limit for the economizer cycle, F.

TLHDL	4	26 - 28	Lower ambient temperature limit for hot deck temperature schedule, F.
TLHDU	4	20 - 22	upper ambient temperature limit for hot deck temperature schedule, F.
TSA	5	1 - 7	total supply airflow, cfm
VOA	10	71 - 72	Profile number for variable outside air schedule.
VVMAF	5	71 - 72	Minimum percentage of supply airflow for variable volume systems.
ZVFCM	5	8 - 14	Maximum cold deck airflow, cfm
ZVFHM	5	15 - 21	Maximum hot deck airflow, cfm.