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### LIST OF SYMBOLS

a	local sound speed
d	nozzle exit diameter
g	gravity acceleration
L	shock tube length, jet call length
M	flow mach number
Ms	shock mach number
Msr	reflected shock mach number
R	gas constant
P	pressure
t	time coordinate
u	particle speed
X	coordinate along the shock tube length

### GREEK SYMBOLS

$\gamma$	ratio of specific heat
$\tau$	equal to $\frac{at}{L}$

## CHAPTER I

### INTRODUCTION

Studies of fluidic ignitors are mainly experimented by resonance tube arrangement where a continuous gas supply through a nozzle is directed axially against an open end of cylindrical tube (one end is open and the other end is closed). The result is that the gases are excited within the tube and strong longitudinal oscillation is produced. These oscillations are found to occur when the open end of the tube is placed in the compression region of an underexpanded sonic jet.

#### i) Background

Hartmann<sup>1</sup>, who did the first reported work in this field, found that by placing a tube in one of the intervals of instability (the compression regions of an underexpanded sonic jet) a self-sustaining system of oscillations is created and the gas in the tube is driven into resonance.

The first systematic experimental investigation of heating effects in resonance tubes was reported by Sprenger in 1954. His work consisted of extensive measurements of the average pressure, average wall temperature and sound intensities as a function of geometric parameters and jet stagnation pressures.

Following Sprenger's<sup>2</sup> publication in 1954, there was a flurry of experimental and limited theoretical research conducted and reported by various investigators: Cassidy; Thompson

and Slawsky at NBS verified the dependence of maximum temperature rise on the jet stagnation pressure and the experimental geometry employed. Recent experimental investigators have reported resonance tube end-wall temperature-time histories for a variety of geometric and operating conditions.

Because of the complexity of the flow, only recently has any detailed theoretical analysis of the flow in a resonance tube has been published. Thompson<sup>3,4</sup>, working on his doctoral dissertation under Professor A. Shapiro at MIT, constructed wave diagrams for the internal flow in a resonance tube and verified his results experimentally with a cylindrical resonance tube. In addition, he presented some results on the heating effects estimated from the computed entropy production. Kang<sup>5</sup>, working under Thompson, at RPI, presented a rather detailed theoretical model of a cylindrical tube excited by a fully expanded supersonic airjet.

McAlevy and Pavlak at Stevens Institute investigated and compared the cylindrical resonance tube and a two dimensional tapered tube. Phillips, Pavli and Conrad<sup>6,7</sup> at NASA-Lewis investigated the use of the resonance heating phenomena to ignite a stoichiometric mixture of gaseous, hydrogen and oxygen.

The oscillation phenomena associated with Hartmann tube excited by an underexpanded sonic jet have been examined theoretically and experimentally by Neeme<sup>8</sup> and found that the pressure amplitude of the oscillation inside the tube

is related to the jet static pressure at the nozzle exit end and that the frequency of oscillation is a function of the jet sonic speed and the tube length.

Resonance tube studies are also being conducted in other countries. At the FFA, Aeronautical Research Institute of Sweden, Knoos has studied two dimensional twin tube resonator using a supersonic bistable air jet alternately driving the two tubes. In France, Brocher, Maresca and Bournay have investigated the fluid dynamics of the tube.

ii) Mechanism of Heating

When the flow emerges from a converging nozzle, it accelerates first to supersonic speed and then readjusts to subsonic speed by compression through a shock wave. The process creates a series of diamond-shaped cells of alternate supersonic and subsonic flow. These cells or conical shock waves (mach diamonds) intersect the jet axis throughout the length of the jet. Fig. (1)

Placing a tube in the compression region of an under-expanded jet a self-sustaining system of oscillations is created in the tube. Although there is continuous flow into and out of the tube, a portion of the gas remains trapped at the closed end of the tube. Periodic compression and expansion of the trapped gas at the closed end of the tube produces irreversible temperature increases, which may be several times the initial adiabatic temperature head. Resonance tubes were used as fluidic ignitors because the gas oscillations may be violent. When properly excited, the internal flow will include shock waves of moderate strength.

These shock waves effect a strong heating of the gas trapped within the tube. This heating raises the temperature of gas at the closed end to a high temperature depending on the conditions used during the test. Temperatures in excess of  $1,500^{\circ}\text{K}$  within a short time (less than a tenth of a second) have been reached in resonance tube experiments.

The temperatures obtained at the closed end of the resonance tube depends on the various parameters such as pressure ratio across the nozzle, the stagnation temperature of the jet, the shapes and size of the nozzle and tube, etc. Because of these high temperatures reached in resonance tubes, the thought of using a combustible gas instead of air has struck many resonance tube experimenters<sup>6,7</sup>.

Using a combustible gas mix, for example hydrogen-oxygen as gas supply in the nozzle, ignition can be produced at the closed end of the tube. Such apparatus could be used to produce ignition of hydrogen-oxygen mixtures of rocket engines or other engines that use a combustible gas.

### iii) Ignition Studies

Presently, ignition studies are being conducted in laboratories by the use of Hartmann resonance tube and a continuous jet supply of combustible mixtures<sup>6,7</sup>. These mixtures are supplied at high pressure in order to sustain the oscillations. Fig. (1) shows a typical apparatus used for such studies. The use of such an apparatus may cause explosion within the test room. Also, the combustible mixtures are not best controlled due to surrounding atmosphere.

iv) Proposed Model

Because of the deficiencies and complexity of the above model, we propose a new model Fig. (2) where a shock tube with a convergent nozzle, placed at the downstream end of the driven section, is used to form the underexpanded jet required for any resonance tube arrangement. Both tube and nozzle are enclosed in a gas tight chamber. The advantages of using such a model are as follows:

- a) Ignition is established in a confined chamber.
- b) Combustible mixtures need not be stored at high pressures. Shock tube will be first evacuated, then supplied with combustible mixtures which may be at any pressure depending on the desired shock wave strength.
- c) The amount of combustible gases used in the proposed model are much smaller than that required with the continuous jet model.
- d) The pressure, temperature and concentrations of the gases can be better controlled in the proposed model.

The present investigation will be concentrated mainly on determining the feasibility of such an arrangement for use in ignition studies.

This will be judged based on the previous results obtained using the continuous jet model.

## CHAPTER II

### THEORETICAL ANALYSIS AND RESULTS

#### 2.1 SHOCK TUBE THEORY<sup>9,10,11</sup>

The shock tube consists of two lengths of straight, uniform cross section tubing separated by a gas tight diaphragm Fig. (3). One half of the tube, known as the compression chamber, contains a gas at a pressure  $P_4$ , which is in excess of the pressure  $P_1$  of the gas in the other half of the tube known as the expansion chamber. The gases of either side of the diaphragm need not be of the same chemical type.

When the diaphragm is caused to shatter, a shock wave travels into the low pressure chamber and a rarefaction wave travels back into the compression chamber. Behind the shock front is a flow of gas and in order to avoid pressure variations building up the flow velocity is uniform in the region between the shock front and the tail of the rarefaction wave. This is also a region of a constant pressure. The distribution of pressure along the tube before and after the diaphragm is shattered is shown in Fig. (4). The dotted vertical line between state 2 and 3 denotes the position occupied by that gas which was originally at the diaphragm. The gas to the right has been compressed and heated by the shock wave but the gas to the left of this line has expanded from the compression chamber and has therefore been cooled. At this position there will be, therefore, in general, a change of type, temperature and

density, velocities are the same on both sides. Such a point is known as a contact discontinuity or contact surface. A plot in the  $x-t$  (distance/time) of the processes occurring in a shock tube with both ends closed is shown in Fig. (5).

The shock front position is represented by the line OA with slope  $\frac{dx}{dt} = U$  and the contact discontinuity by OB,  $\frac{dx}{dt} = u$ .

This meets the reflected shock AB at B, where the shock wave once again undergoes a reflection. The slope of the contact surface curve in the  $x, t$  plane is then very steep because the flow velocity is low in that region. The head of the rarefaction wave travelling from the diaphragm is represented by the line OC and the tail of rarefaction by OD. The interest of this report is to look at the right half of Fig. (5) with respect to the pressure distribution, the shock velocities obtained, the duration time of flow and the temperature behind shock wave.

The performance analysis of a simple shock tube can be made, assuming the gases in the high pressure and the low pressure sections are ideal gases having constant specific heats and assuming an instantaneous rupture of the diaphragm separating them. The parameters which determine the performance of a simple shock tube are the mach numbers, the temperatures (or speed of sound), and the pressures in flow regions 2 and 3. The parameters should be known as functions of the initial conditions, namely, the initial pressure and speed of sound ratio  $P_4/P_1$ , and  $a_2/a_1$ , and the values of  $\gamma_1$  and  $\gamma_4$ . The expression which relates the mach number in region 3,

$M_3$ , can be derived with the initial pressure and speed of sound ratios  $P_4/P_1$  and  $a_2/a_1$ , and with the ratio of specific heats of the gases.

From moving shock wave relations

$$\frac{u_2 - u_1}{a_1} = \frac{2}{\gamma_1 + 1} \left( \frac{u}{a_1} - \frac{a_1}{u} \right)$$

$$\frac{u}{a_1} = M_s \text{ shock wave mach number}$$

The equation can be written in the form

$$\frac{u_2}{a_1} = \frac{2}{\gamma_1 + 1} \left( M_s - \frac{1}{M_s} \right) \quad (1)$$

Also

$$\frac{P_2}{P_1} = 1 - \left[ \frac{2\gamma_1}{\gamma_1 - 1} \left( M_s^2 - 1 \right) \right] \quad (2)$$

Eliminating  $M_s$  from equations (1) and (2)

$$\frac{u_2}{a_1} = \frac{\frac{P_2}{P_1} - 1}{\frac{\gamma_1}{2\gamma_1} \sqrt{1 + (\gamma_1 + 1) \left( \frac{P_2}{P_1} - 1 \right)}} \quad (3)$$

For the centered expansion wave in regions 3 and 4

$$\frac{u_3}{a_4} = \frac{2}{\gamma_4 - 1} \left[ 1 - \left( \frac{P_3}{P_4} \right)^{\frac{\gamma_4 - 1}{2\gamma_4}} \right] \quad (4)$$

Because  $u_4 = 0$ . Since  $u_2 = u_3$  and  $P_2 = P_3$

Equation (3) and (4) could be combined to give

$$\frac{\frac{a_1}{a_4} \left( \frac{P_2}{P_1} - 1 \right)}{\gamma_1 \sqrt{1 + \frac{\gamma_1 + 1}{2\gamma_1} \left( \frac{P_2}{P_1} - 1 \right)}} = \frac{2}{\gamma_4 - 1} \left[ 1 - \left( \frac{P_2}{P_4} \right)^{\frac{\gamma_4 - 1}{2\gamma_4}} \right] \quad (5)$$

For the mach number behind an expansion wave propagating in a gas at rest is given by

$$\frac{P_2}{P_4} = \frac{P_2}{P_4} = \left[ 1 + \left( \frac{\gamma_4 - 1}{2} \right) M_3 \right]^{\frac{2\gamma_4}{\gamma_4 - 1}} \quad (6)$$

From which

$$\frac{2}{\gamma_4 - 1} \left[ 1 - \left( \frac{P_2}{P_4} \right)^{\frac{\gamma_4 - 1}{2\gamma_4}} \right] = \frac{M_3}{1 + \left( \frac{\gamma_4 - 1}{2} \right) M_3} \quad (7)$$

Equation (6) can be rearranged by multiplying the equation by  $\frac{P_1}{P_2}$

$$\frac{P_1}{P_4} = \frac{P_1}{P_2} \left[ 1 + \left( \frac{\gamma_4 - 1}{2} \right) M_3 \right]^{\frac{2\gamma_4}{\gamma_4 - 1}}$$

or

$$\frac{P_4}{P_1} = \frac{P_2}{P_1} \left[ 1 + \left( \frac{\gamma_4 - 1}{2} \right) M_3 \right]^{\frac{2\gamma_4}{\gamma_4 - 1}} \quad (8)$$

The equation relating the mach number behind the shock wave, moving in a gas at rest the pressure ratio across it is

$$M_2^2 = \frac{2}{\gamma_1(\gamma_1 - 1)} \left[ \frac{\frac{P_2}{P_1} + \frac{P_1}{P_2} - 2}{\frac{\gamma_1 + 1}{\gamma_1 - 1} + \frac{P_2}{P_1}} \right] \quad (9)$$

Also the shock wave mach number is given in terms of the initial pressure ratio  $P_4$ , the speed of sound ratio  $\frac{a_4}{a_1}$

$$\frac{P_4}{P_1} = \frac{a_4}{a_1}$$

and the specific heats  $\gamma_1$  and  $\gamma_4$  by

$$(M_s - \frac{1}{M_s}) = \left( \frac{\gamma + 1}{\gamma - 1} \right) \left( \frac{a_4}{a_1} \right) \left[ 1 - \left( \frac{2\gamma M_s^2 - (\gamma - 1)}{\gamma - 1} \right) \left( \frac{P_1}{P_4} \right) \right]^{\frac{(\gamma - 1)}{2\gamma}} \quad (10)$$

Assuming  $M_s = 1$  initially.

For the purpose of this report a range of shock wave mach numbers is given in order to determine the following:

- The initial pressure ratios required.
- The mach numbers in regions 2 and 3.
- The reflected shock mach numbers in regions 5, 6, and 7.
- The duration time of flow for reflected shock in region 5.
- The stagnation pressure in region 5. Refer to Fig. (5).

The shock mach numbers are as follows:

$$M_s = 1.2, 1.25, 1.30, 1.35, 1.40, 1.45, 1.5, 1.55, 1.6, 1.65, 1.70, 1.75.$$

Also given:  $P_1 = 14.7 \text{ PSI}$   
 $a_1 = a_4$   
 $\gamma_1 = \gamma_4 = 1.4$

## 2.2 THEORITICAL RESULTS

### 2.2.1 THE FLOW MACH NUMBERS BEHIND THE INCIDENT SHOCK

#### Calculations

Using equation (10) the initial pressure ratio  $P_4/P_1$  can be obtained

$$\left( M_s - \frac{1}{M_s} \right) = \left( \frac{\gamma + 1}{\gamma - 1} \right) \left( \frac{a_4}{a_1} \right) \left[ 1 - \left( \frac{2\gamma M_s^2 - (\gamma - 1)}{\gamma - 1} \right) \frac{P_1}{P_4} \right]^{\frac{\gamma - 1}{2\gamma}}$$

Given  $M_s = 1.2, \gamma_1 = \gamma_4 = 1.4, a_1 = a_4$

Using equation (2) one can obtain  $P_2/P_1$  which is the pressure ratio across the shock

$$\frac{P_2}{P_1} = 1 - \left( \frac{2\gamma}{\gamma - 1} \right) (M_s^2 - 1)$$

$$\frac{P_2}{P_1} = 1 + \frac{2(1.4)}{(1.4-1)} \left[ (1.2)^2 - 1 \right]$$

$$\frac{P_2}{P_1} = 1.513$$

Also the value of  $\frac{P_2}{P_1}$  could be obtained from table 1a at  
 $M_s = 1.2$ . See appendix.

Using equation (9) the mach number  $M_2$  in region 2 can be obtained

$$M_2^2 = \frac{2}{1.4(1.4-1)} \left[ \frac{1.513 + \frac{1}{1.513} - 2}{\left( \frac{1.4+1}{1.4-1} \right) + 1.513} \right]$$

$$M_2 = 0.847$$

OR  $\frac{P_4}{P_1} = 2.34$  using equation (10)

Using equation (8) the mach number in region 3 can be obtained

$$\frac{P_4}{P_1} = \frac{P_2}{P_1} \left[ 1 + \frac{(8-1)}{2} M_3 \right]^{\frac{2(8)}{8-1}}$$

$$2.3 = 1.513 \left[ 1 + \left( \frac{1.4-1}{2} \right) \right]^{\left( \frac{2(1.4)}{1.4-1} \right)}$$

or  $M_3 = 0.32$

For different shock mach number,  $M_s$ , similar calculation has been carried out and table (1) has been constructed.

Also, a curve has been plotted showing the initial pressure ratio,  $P_4/P_1$ , required to produce the given shock wave mach numbers, Fig. (6)

The pressure ratio across the shock  $\frac{P_2}{P_1}$  is a measure of the shock strength. Using equation (9) one can obtain the max. mach number  $M_2$  that can be obtained in a shock tube utilizing an ideal gas in the low pressure section having a specific heat,  $\gamma_1 = 1.4$

$$M_2^2 = \frac{2}{\gamma_1(\gamma_1-1)} \left[ \frac{\frac{P_2}{P_1} + \frac{P_1}{P_2} - 2}{\frac{\gamma_1+1}{\gamma_1-1} + \frac{P_2}{P_1}} \right]$$

As  $\frac{P_2}{P_1} \rightarrow \infty$ ,  $M_2 \rightarrow (M_2)_{\text{Lim.}} = \sqrt{\frac{2}{\gamma_1(\gamma_1-1)}}$

or  $M_2 = 1.89$

Now one can observe that for a given gas combination and initial conditions, the duration of uniform flows in regions 2 and 3 and in the region behind the reflected shock wave depends on the lengths of the high and low pressure sections of a shock tube. The maximum theoretical duration of uniform flow in region 2 behind the shock wave, for a given length of the high pressure section of the tube, gas combination, and initial conditions, can be attained in a shock tube having the low pressure section of such a length that the head of the reflected expansion wave arrived at the contact

surface before the reflected shock does. The location in the low pressure tube section, at which the duration of the uniform flow behind the shock wave is maximum, corresponds to the point at which the head of the reflected expansion wave catches up with the contact surface.

### 2.2.2 CALCULATIONS TO DETERMINE THE REFLECTED SHOCK MACH NUMBER IN REGION 5

Using shock table 1a, see appendix.

At  $M_s = 1.2$  we can obtain  $\frac{\Delta u}{a}$  and also  $\frac{a'}{a}$ .

$$i/ \frac{\Delta u}{a} = \frac{u_2 - u_1}{a_1} = \frac{u_2}{a_1}, u_1 = 0 \text{ initially}$$

$$ii/ \frac{a'}{a} = \frac{a_2}{a_1}$$

Therefore by dividing i by ii we get the following

$$\frac{u_2}{a_1} \times \frac{a_1}{a_2} = \frac{u_2}{a_2}$$

$$\text{But } \frac{u_2}{a_2} = \frac{u_5 - u_2}{a_2}, u_5 = 0 \text{ at the closed end.}$$

Therefore  $\frac{u_2}{a_2} = \frac{\Delta u}{a}$ , knowing this value one can determine

the reflected shock mach number from the table 1a.

### Sample Calculation

At  $M_s = 1.2$

$$\frac{\Delta u}{a} = 0.306 \text{ approximately}$$

$$\frac{a'}{a} = 1.061$$

$$\frac{\Delta u}{a_2} = \frac{0.306}{1.061} = 0.288$$

Using table 1a (ref) at  $\frac{\Delta u}{a} = 0.288$ ,  $M_{sr} = 1.188$  and also

$$\frac{P_1}{P} = \frac{P_5}{P_2} = 1.48$$

$P_2$

Using this calculation table (2) has been constructed.

#### 2.2.3 CALCULATIONS TO DETERMINE THE REFLECTED SHOCK MACH NUMBERS IN REGION 6 AND 7

~~Knowing~~ the following values:

$$M_s, \frac{P_2}{P_1}, \frac{P_4}{P_1} = 14.7 \text{ Psi}, \frac{a_2}{a_1}$$

$$M_{sr}, \frac{P_5}{P_2}, \frac{a_5}{a_2}$$

$$a_3 = \left( \frac{P_2}{P_1} \times \frac{P_1}{P_4} \right)^{\frac{\gamma-1}{2\gamma}} a'_4$$

One can assume  $M_{sr} 5,6$  to be a certain value.

Then  $\left| \frac{u_6 - u_5}{a_5} \right|$  and  $\frac{P_6}{P_5}$  can be obtained from table 1a.

$u_6$  and  $P_6$  can be calculated knowing that  $P_6 = P_7$ .

$P_7$  can be calculated.

$$\frac{P_7}{P_3}$$

$Msr_{3,7}$  and  $\left| \frac{u_7 - u_3}{a_3} \right|$  can be obtained from table 1a, then  $u_7$

can be calculated but since  $u_7 = u_6$ , the obtained  $u_6$  from  
 $Msr_{5,6}$  should be equal to the calculated value of  $u_7$ .

If the  $u_6 \neq u_7$  then  $Msr_{5,6}$  assumed is wrong and another value  
should be assumed until  $u_6 = u_7$ .

#### Sample Calculation

The calculations were carried out for the range of mach  
numbers 1.2 to 1.75. A sample calculation for  $M = 1.2$  is  
carried out below.

$$Ms' = 1.2 \text{ given}$$

$$\frac{P_2}{P_1} = 1.513 \text{ (calculated and also from table 1a at } Ms = 1.2)$$

$$\frac{P_4}{P_1} = 2.34 \text{ calculated}$$

$$Msr_5 = 1.188 \text{ calculated}$$

$$\frac{P_5}{P_2} = 1.48 \text{ from table 1a}$$

$$\frac{a_2}{a_1} = 1.061 \text{ from table 1a at } Ms = 1.2$$

$$\frac{a_5}{a_2} = 1.058 \text{ from table 1a at } Msr_5 = 1.188$$

$$\frac{u_2 - u_1}{a_1} = 0.306 \text{ from table 1a at } Ms = 1.2$$

$$\text{Also } a_3 = \left[ \frac{P_2}{P_1} \times \frac{P_1}{P_4} \right]^{\frac{g-1}{2\gamma}} a_4$$

$$P_1 = 14.7 \text{ Psi given, } a_1 = a_4$$

Then,

$$P_2 = 1.513 \times 14.7 = 22.4 \text{ Psia}$$

$$P_5 = 1.48 \times 22.4 = 33.15 \text{ Psia}$$

$$a_2 = 1.061 \times 1 = 1.061$$

$$a_5 = 1.058 \times 1.061 = 1.122$$

$$u_2 = a_1 \times 0.306 = 0.306$$

$$a_3 = \frac{(1.513)}{2.34} 0.143 = 0.939$$

Now, assume  $Msr_{5,6} = 1.01$

From table 1a at  $Msr_{5,6} = 1.01$

$$\left| \frac{u_6 - u_5}{a_5} \right| = 0.017$$

Since  $u_5 = 0$

$$u_6 = 0.017 \times 1.122 = 0.019$$

Also from table 1a  $P_6 = 1.025$

$$\frac{P_6}{P_5}$$

Or  $P_6 = 1.025 \times 33.15 = 34. \text{ Psia}$

But  $P_6 = P_7$  and  $P_2 = P_3$

Therefore  $\frac{P_7}{P_3} = \frac{34}{22.4} = 1.517$

From table 1a at  $\frac{P_7}{P_3} = 1.517$ ,  $Msr_{317} = 1.199$

Also  $\Delta u = \frac{u_7 - u_3}{a_3} = 0.306$

Or  $u_7 = 0.306 \times 0.939 = 0.306$

$u_7 = 0.019 = u_6$  OK

Using this iteration procedure, table (3) has been constructed.

#### 2.2.4 DETERMINATION OF RUNNING TIME

In order to determine the duration time of flow it is useful first to now dimensionalize the independent and dependent variables by introducing a reference value of the speed of sound,  $a_1$ , and a reference length L (shock tube length).

The following dimensionless variables are formed. <sup>3</sup>

Dimensionless Distance Variable	$\frac{X}{L}$
Dimensionless Time Variable	$\frac{a_1 t}{L}$
Dimensionless Flow Speed	$\frac{M}{a_1}$
Dimensionless Speed of Sound	$\frac{a}{a_1}$

In order to determine the duration time one must construct  $(X, t)$  plot similar to Fig. (5). This will require a scale such that  $X/L = \frac{a_1 t}{L} = 1$  unit, therefore by knowing the gradient of each shock wave, one can plot these lines and then measure duration time from the graph.

The gradients are given by the following relationship:

$$\text{Line OA produced shock } M_s, \frac{d(X/L)}{d(at/L)} = M_s + \frac{u_1}{a_1}$$

$$\text{Line AB reflected shock } M_{sr5}, \frac{d(X/L)}{d(at/L)} = M_{sr5} \frac{(a_2)}{a_1} + \frac{u_2}{a_1}$$

$$\text{Line BG reflected shock } M_{sr5,6}, \frac{d(X/L)}{d(at/L)} = M_{sr5,6} \frac{(a_5)}{a_1} + \frac{u_5}{a_1}$$

$$\text{Line OB contact surface} = u_2$$

$$\text{Line BH reflected shock } M_{sr3,7}, \frac{d(X/L)}{d(at/L)} = M_{sr3,7} \frac{(a_3)}{a_1} + \frac{u_3}{a_1}$$

We also know that  $u_1 = 0$  initial condition

$u_2 = -ve$  relative to line AB

$u_3 = -ve$  relative to line BH

$u_5 = 0$  closed end of tube, no flow

Knowing these gradients one can determine the angle of each line and an  $(X, t)$  plot can be made. Table (4) has been made and  $(X, t)$  plot has been constructed, as shown in Fig. (6)  
(plot 1 to 13)

## CHAPTER III

### EXPERIMENTAL APPARATUS AND RESULTS

#### 3.1 EXPERIMENTAL APPARATUS

##### 3.1.1 THE TWO BY TWO SHOCK TUBE

A simple air-air shock tube with a square cross section 2.06 x 2.06 inches was employed for all tests. This apparatus was constructed from seamless structural steel tubing with a 1/4 inch wall thickness and consisted of a driver section five feet long followed by a driven section of equal length. The comparatively long driver section was chosen to ensure that neither the contact surface nor the expansion wave originating at the diaphragm would enter the test section until long after the events of interest had been completed. Mylar plastic sheets of 0.007, 0.003 and 0.0015 inch thickness were used as the diaphragm material. A schematic of the shock tube facility is presented in Fig (8).

Compressed air was supplied from a storage tank filled by a reciprocating compressor. The driven section pressure was atmospheric (14.7 Psi).

##### 3.1.2 PRESSURE MEASUREMENT

Pressure measurements were accomplished by piezoelectric gauges (PCB Piezotronics, type 101A, rise time 2 microseconds,

sensitive surface of about 0.21 inch in diameter). The gauges have an integrated circuit amplifier coupled to the crystal, and show a very good side wall response to a shock wave.

Shock velocity measurement were performed by employing the transducers in groups of two with the first transducer used to trigger an oscilloscope (Tektronix 555) beam. The second gauge was connected to the input terminal of the oscilloscope vertical deflection plate. Knowing the transducer constant,  $\frac{\Delta V}{\Delta P}$ , and the oscilloscope time scale, shock velocity was easily computed from the oscilloscope traces which were recorded on polaroid film. The location and spacing of the transducers were 3.500 and 9.4375 inch from the closed end of the driven section (low pressure section). See Fig. (8).

### 3.1.3 THE SCHLIEREN SYSTEM

Photographic studies were carried out to determine the jet structure produced by a different diaphragm thickness using a time delayed spark schlieren optical system which was triggered by a pressure transducer located just up stream of the test section.

As shown in Fig. (8), a high voltage 2KV spark was used as a light source for a conventional double mirror schlieren

optical system using 48 1/2 inch focal length. All photographs were recorded on 3000 ASA polaroid film (type 47). The square aperture at the light source had dimensions of less than 1 MM, and this gave excellent resolution. Thus, one schlieren photograph was obtained for each run of the shock tube.

### 3.2 EXPERIMENTAL RESULTS

#### 3.2.1 SHOCK VELOCITIES AND RUNNING TIME

Typical pressure traces as recorded from the pressure transducers are shown in Fig. (9). As seen in the figure there are two sudden changes in pressure - one due to the incident shock, the other due to the reflected shock. After the reflected shock, the pressure remains constant for a period of time until the reflected shock from the contact surface arrives to the transducer. This is clearly noted from the photographs. The running time is therefore the time required for the reflected shock from the contact surface to arrive at the closed end of the shock tube.

#### CALCULATIONS OF SHOCK VELOCITIES AND RUNNING TIME

##### 0.007 inch diaphragm

Burst pressure 204.7 Psia (190 Psig)

##### Running Time

$t_1 = 0.001 \text{ sec}$  - From picture

$t_2 = 0.00147$  sec - From picture

$L = 10$  ft Shock tube length

$$\text{Time } \tau_1 = \frac{a_1 t_1}{L}$$

$$a_1 = \sqrt{8grt}$$

$$a_1 = \sqrt{1.4 \times 32.2 \times 53.34 \times 535}$$

$$a_1 = 1134.2 \text{ ft/sec}$$

Therefore  $\tau_1 = \frac{a_1 t_1}{L}$

$$= \frac{1134.2 \times 0.001}{10} = 0.1134$$

$$\tau_2 = \frac{1134.2 \times 0.00147}{10} = 0.1667$$

### Shock Velocity

Knowing the distance between the transducers (5.9375 inch),  $\Delta X$ , also from the picture we can measure the time,  $\Delta t$ , using the oscilloscope scale, then we can calculate the velocity  $\frac{\Delta X}{\Delta t}$

$$\text{Therefore } \Delta V = \frac{\Delta X}{\Delta t} = \frac{5.9375 \times 1000}{12 \times 0.2704}$$

$$\Delta V = 1829.85 \text{ ft/sec.}$$

But the shock velocity  $M_s = \frac{\Delta V}{a_1}$

$$= \frac{1829.85}{1134.2}$$

$$M_s = 1.61$$

### 0.003 inch Diaphragm

Oscilloscope Sensetivity 1 Msec/cm, 0.5 volt/cm

Burst Pressure 114.7 Psi (100 Psig)

Measurement from picture

Running Time

$$t_1 = \frac{1.6 \text{ cm} \times 1 \text{ Msec/cm}}{1000} = 0.0016 \text{ sec}$$

$$t_2 = \frac{2.6 \times 1}{1000} = 0.0026 \text{ sec}$$

$$\text{Therefore, } \tau_1 = \frac{a_1 t_1}{L} = \frac{1134.2 \times 0.0016}{10}$$

$$\tau_1 = 0.181$$

$$\tau_2 = \frac{1134.2 \times 0.0026}{10}$$

$$\tau_2 = 0.294$$

Shock Velocity

$$\Delta X = 5.9375 \text{ inch}$$

$$\Delta t = 6.0 \text{ cm} \times 0.05 \text{ Msec/cm}$$

Therefore

$$\begin{aligned} \Delta V &= \frac{\Delta X}{\Delta t} \\ &= \frac{5.9375 \times 1000}{12 \times 6.0 \times 0.05} \end{aligned}$$

$$\Delta V = 1649.3 \text{ ft/sec}$$

$$\text{Shock Velocity } M_s = \frac{\Delta V}{a_1} = \frac{1649.3}{1134.2}$$

$$M_s = 1.454$$

0.0015 inch Diaphragm

Oscilloscope Sensitivity 1 Msec/cm, 0.5 V/cm

Burst Pressure.

60 Psig

Running Time

$$t_1 = \frac{2.5x1}{1000} = 0.0025 \text{ sec}$$

$$t_2 = \frac{4.0x1}{1000} = 0.004 \text{ sec}$$

Therefore

$$\tau_1 = \frac{a_1 t_1}{L} = \frac{1134.2 \times 0.0025}{10}$$

$$\tau_1 = 0.283$$

$$\tau_2 = \frac{1134.2 \times 0.0004}{10}$$

$$\tau_2 = 0.453$$

Shock Velocity

$$\Delta X = 5.9375 \text{ inch}$$

$$\Delta t = 6.78 \text{ cm} \times 0.05 \text{ Msec/cm}$$

$$\Delta V = \frac{\Delta X}{\Delta t} = \frac{5.9375 \times 1000}{12 \times 6.78 \times 0.05}$$

$$\Delta V = 1459.6 \text{ ft/sec}$$

$$\text{Shock velocity } M_s = \frac{\Delta V}{a_1}$$

$$= \frac{1459.6}{1134.2}$$

$$M_s = 1.287$$

**3.3 CALIBRATION OF TRANSDUCER**

The pressure transducers manufacturer specification gives

transducers constant  $\frac{\Delta V}{\Delta P}$  30.6m volts/PSI. For the purpose of accuracy, a check should be made in order to find the transducers constant and use it to determine the stagnation pressure  $P_5$  in Region 5. See Fig. (5)

0.007 inch Diaphragm

$$\text{Shock velocity } M_s \text{ (experimental)} = 1.61$$

$$\text{From table 1a at } M_s = 1.61, \frac{P_2}{P_1} = 2.86$$

$$\text{Since } P_1 = 14.7 \text{ Psi}$$

$$P_2 = 14.7 \times 2.86 = 42.04 \text{ PSIA}$$

$$\text{Then, } \Delta P = P_2 - P_1 = 42.04 - 14.7$$

$$\underline{\Delta P = 27.34 \text{ PSIA}}$$

From Picture

$$\Delta V = 0.987 \text{ Volts}$$

$$\text{Therefore transducer constant } \frac{\Delta V}{\Delta P} = \frac{0.987 \times 1000}{27.34}$$

$$\text{or } \frac{\Delta V}{\Delta P} = \underline{36.1 \text{ MV/PSI}}$$

0.003 inch Diaphragm

$$M_s \text{ (shock velocity)} = 1.454$$

$$\text{From table 1a at } M_s = 1.454, \frac{P_2}{P_1} = 2.3$$

$$P_1 = 14.7 \text{ PSIA}$$

$$P_2 = 2.3 \times 14.7 = 33.81$$

$$\Delta P = P_2 - P_1 = 33.81 - 14.7$$

$$\underline{\Delta P = 19.11 \text{ PSIA}}$$

From Picture

$$\Delta V = 1.38 \text{ cm} \times 0.5 \text{ V/cm}$$

$$\Delta V = 0.69 \text{ Volts}$$

Therefore the transducer constant

$$\frac{\Delta V}{\Delta P} = \frac{0.69 \times 1000}{19.11}$$

$$\frac{\Delta V}{\Delta P} = 36.1 \text{ MV/PSI}$$

0.0015 inch Diaphragm

$$M_s (\text{shock velocity}) = 1.287$$

$$\text{From table 1a at } M_s = 1.287, \frac{P_2}{P_1} = 1.76$$

$$P_1 = 14.7 \text{ PSIA}$$

$$P_2 = 14.7 \times 1.76 = 25.87 \text{ PSIA}$$

$$\Delta P = P_2 - P_1$$

$$\Delta P = 25.87 - 14.7$$

$$\Delta P = 11.17 \text{ PSI}$$

From Picture

$$\Delta V = 0.8 \text{ cm} \times 0.5 \text{ V/cm}$$

$$\Delta V = 0.40 \text{ volts}$$

$$\text{Transducer constant } \frac{\Delta V}{\Delta P} = \frac{0.40 \times 1000}{11.17}$$

$$\frac{\Delta V}{\Delta P} = 35.8 \text{ MV/PSI}$$

Since the transducer constant varied according to the calculations above, a graph has been plotted showing the diaphragm thickness against the transducer constant.

Then an approximate value of the transducer constant was found 36.0 MV/PSI. Fig. (12)

### 3.4 DETERMINATION OF STAGNATION PRESSURE P<sub>5</sub> IN REGION (5)

Using the approximate value of the transducer's constant found, 36.0 MV/PSI, one can calculate the pressure P<sub>5</sub>, using the total voltage  $\Delta V$ , from the pictures.

#### 0.007 inch Diaphragm

$\Delta V = 2.468$  volts measured since the transducer's constant is equal to  $\frac{\Delta V}{\Delta P}$ , then:

$$\begin{aligned}\Delta P &= \frac{\Delta V}{\frac{\Delta V}{\Delta P}} \\ &= \frac{2.468 \times 1000}{36}\end{aligned}$$

$$\Delta P = 68.55$$

But  $\Delta P = P_5 - P_1$

Or  $P_5 = \Delta P + P_1$   
 $= 73.23 + 14.7$

$$P_5 = 83.3 \text{ PSIA}$$

#### 0.003 inch Diaphragm

$$\begin{aligned}\Delta V &= 3.8 \text{ cm} \times 0.5 \text{ V/cm} \\ &= 1.9 \text{ Volts}\end{aligned}$$

$$\Delta P = \frac{1.9 \times 1000}{36} = 52.8 \text{ PSI}$$

Then  $P_5 = 56.37 + 14.7$

$$P_5 = 67.5 \text{ PSIA}$$

#### 0.0015 inch Diaphragm

$$\Delta V = 1.6 \text{ cm} \times 0.5 \text{ V/cm}$$

$$\Delta V = 0.8 \text{ V.}$$

$$\Delta P = \frac{0.8 \times 1000}{36} = 22.2$$

Then  $P_5 = 23.74 + 14.7$

$$P_5 = 36.9 \text{ PSIA}$$

#### 3.5 JET STRUCTURE PRODUCED BY SHOCK TUBE

Fig. (13) and (14) shows jets obtained by 0.003 and 0.0015 inch diaphragms with measured  $l/d = 1.138$  and 1.8 respectively. Also the stagnation pressures were determined by the transducer constant and found to be 67.5 and 36.9 PSIA.

#### 3.6 JET STRUCTURE PRODUCED BY CONTINUOUS JET

Steady continuous jet by Powell and Smith has been presented in Fig. (15) for comparison with jet structure produced by shock-tube arrangement,

## CHAPTER IV

### DISCUSSION

#### 4.1 RUNNING TIME

As shown in Fig. (16) the experimental running time is not a single valued curve but found to be a range. This is because diffusion is taking place at the contact surface. The theoretical results are found to be within the experimental range for the incident shock mach numbers used.

#### 4.2 JET LENGTH

A plot of the total pressure of the jet formed in the shock tube and the ratio of the jet length to the nozzle diameter measured from the photographs of Fig. (13) and (14) is presented in Fig. (17). These results are found to be in close agreement with those obtained previously from steady jets.

#### 4.3 REGIONS OF INSTABILITIES

From the recorded pressure measurement presented for the 0.003 inch diaphragms in Fig. (18), (19), (20) and (21), we can see that at 0.3 inch (gap between the nozzle and the tube), the oscillation is damped and decays rapidly.

As the gap was increased to a value higher than the length of the first cell (0.72"), the oscillations are found to maintain its amplitude. These results are in agreement with those obtained previously with a steady jet.

#### 4.4. PROPOSED TEST SECTION

The proposed test section is similar to that utilized in this experiment Fig. (8) except the resonance tube will be totally enclosed by a chamber for evacuation of air and the supply of combustible gases. See Fig. (22). A thermocouple along with temperature recorder are proposed for temperature recordings. A mechanism for positioning the resonance tube is installed for external adjustment of resonance tube position without dismantalling of combustion chamber. Also, glass plate windows are recommended for visualization of flow and ignition.

##### 4.4.1. OPERATION OF APPARATUS

First the shock tube and the ignition chamber will be evacuated by means of a vacume pump. Then a flow of combustible mixture of known concentration will be supplied to the driven section of the shock tube. The driver section will be filled with air at high pressures. The pressures in the driver and driven sections will be chosen according to the required jet structure.

As the pressure in the driver section of the shock tube is increased to required value, the diaphragm burst and a shock wave is produced. The shock wave travels into the driven section. The reflections of the shock at the downstream end of the tube will result in a formation of an under-expanded jet. If the resonance tube is placed in one of the regions of instabilities, violent oscillations of the gas trapped in the tube will take place. The temperature at the closed end will automatically increase to values much higher than the stagnation temperature of the jet and will keep on rising until ignition temperature of the mixture is reached. Ignition can be sensed by either a thermocouple placed at the end of the tube or by a Kistler pressure transducer placed at the same position as the thermocouple. Ignition can also be verified via Schlieren photography.

CHAPTER V  
CONCLUDING REMARKS

In the present report the flow in the shock tube has been successfully analyzed by the method of characteristics. Good agreement was obtained between the theory and experiment for incident shock velocities and running time. The jet structure produced by the present apparatus after the passage of diffracted shock is found to be similar to that obtained from a continuous jet.

Regions of instabilities were noted in the present experiment. Maximum pressure amplitude of the oscillation was noted when the open end of the resonance tube was placed near the end of the first cell which is in agreement with those of a continuous jet operation.

Since both jet structure and regions of instabilities noted in the present shock tube arrangement do not differ from those obtained from a continuous jet operation, the following conclusions are drawn:

The dynamics of resonance tube can be successfully studies by the use of a shock tube.

Ignition studies can be achieved by confining the resonance tube in a chamber which is more economical, easier to operate and less explosion-hazard than the previously proposed continuous jet model.

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- 4 Distance/time plot of events in a shock tube.
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- 18 Pressure traces recorded at the closed end of a square resonance tube placed at different locations.
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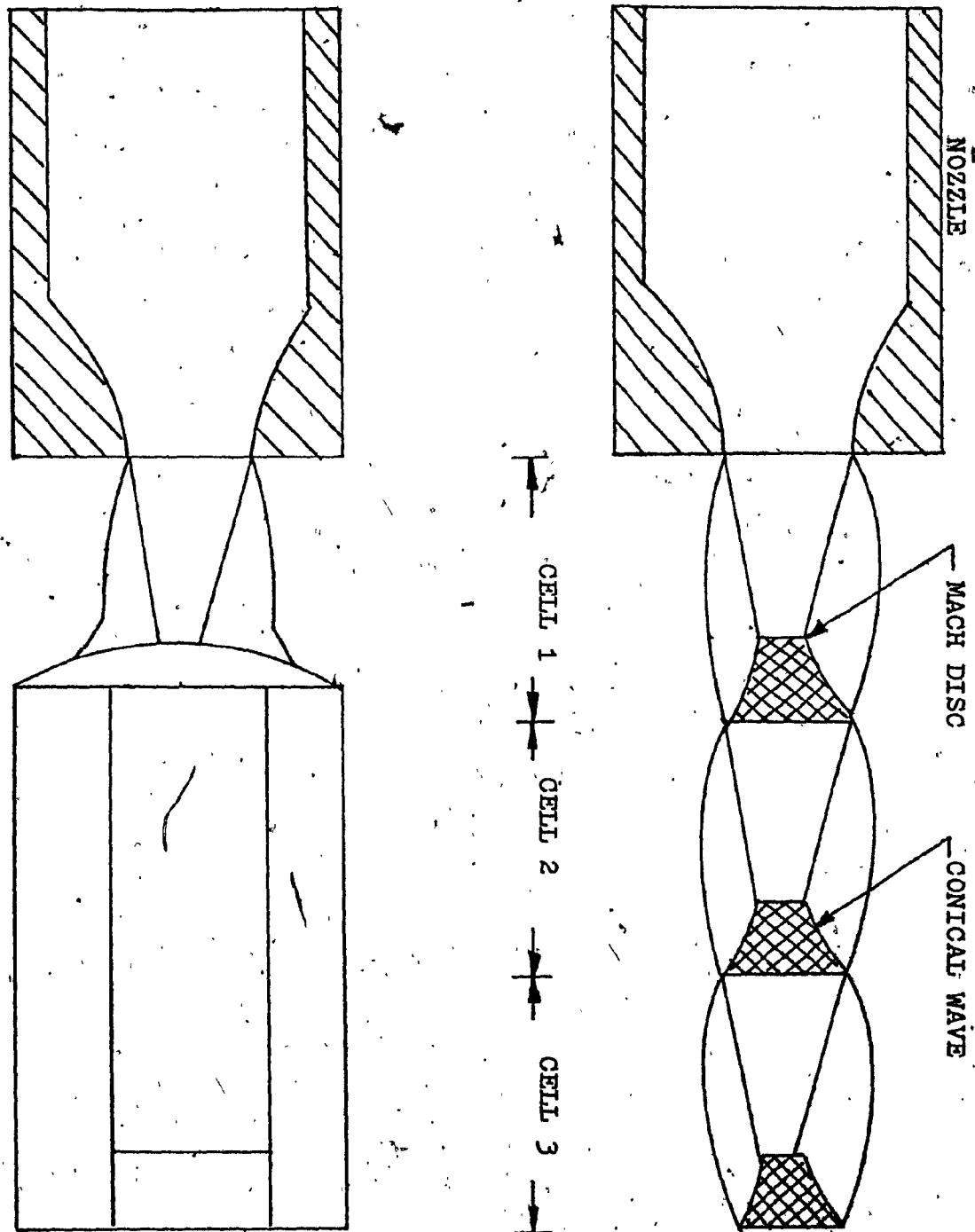


Fig. (1) A schematic of a simple resonance tube and jet structure.

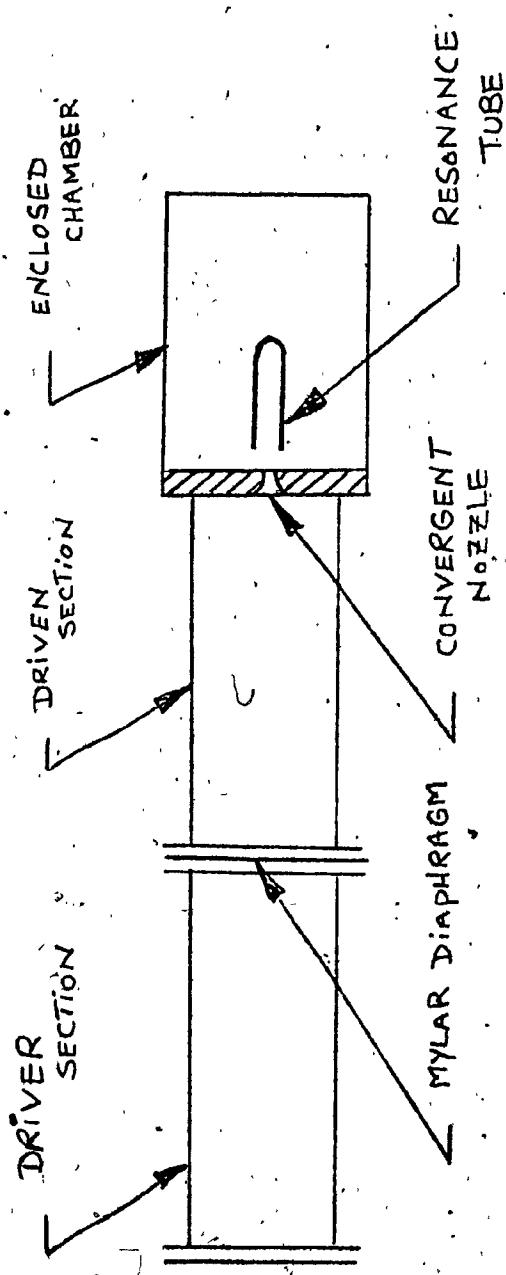


Fig. (2) Schematic of the new model

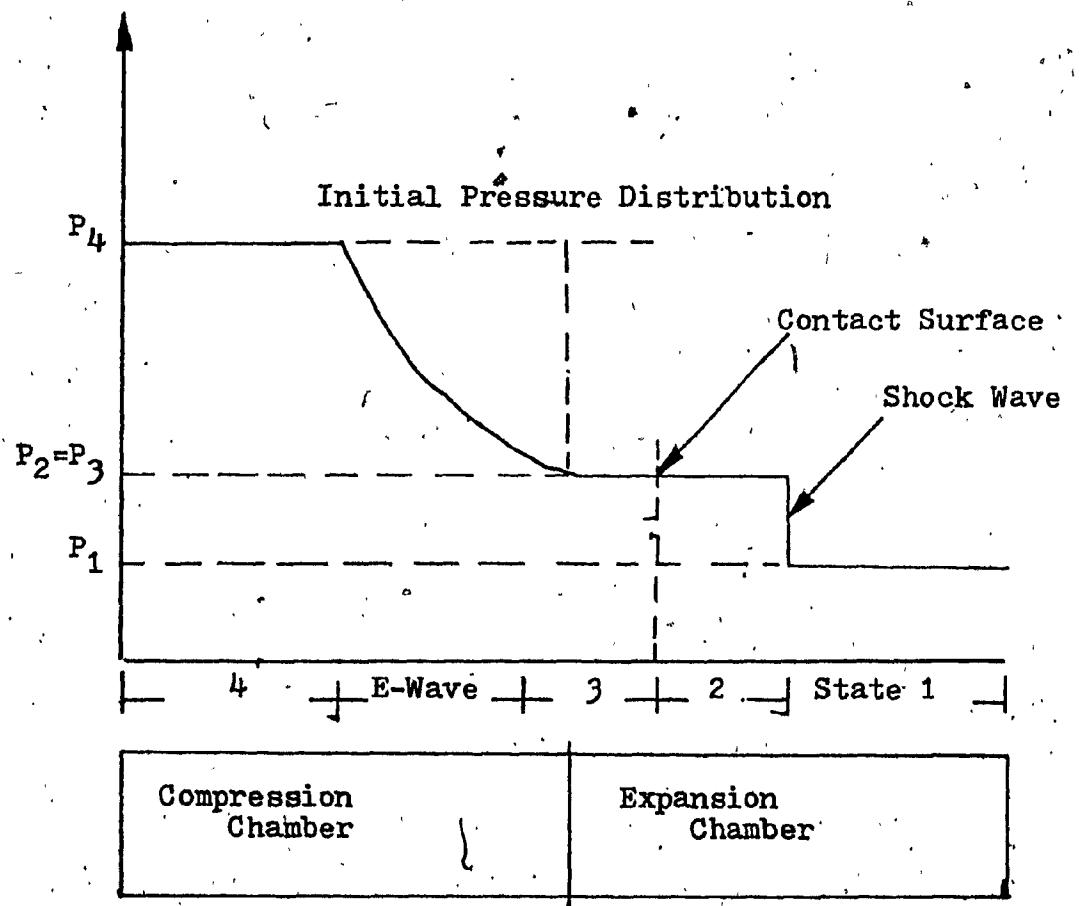


Fig. (3) Pressure variation in a shock tube shortly after the diaphragm is ruptured.

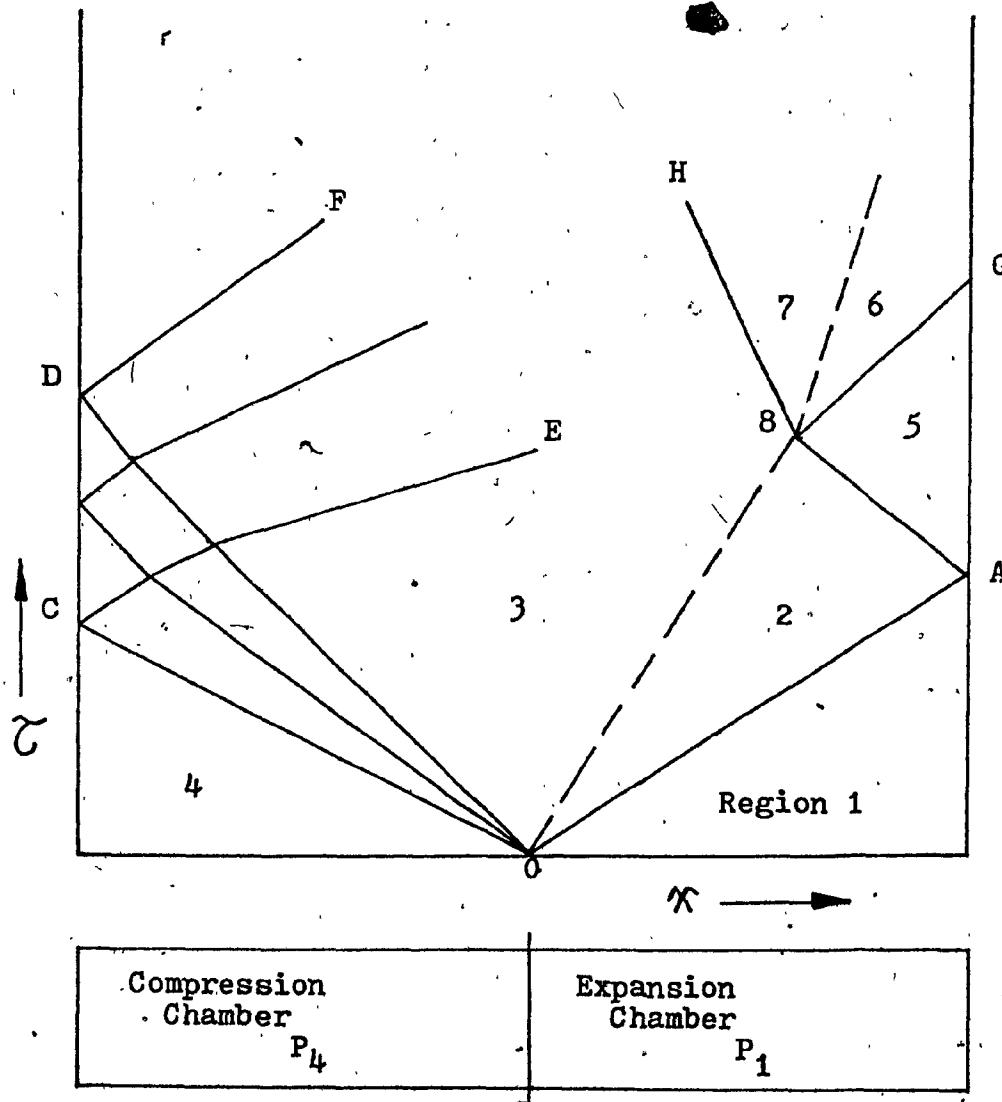


Fig. (4) Distance/time plot of events in a shock tube.

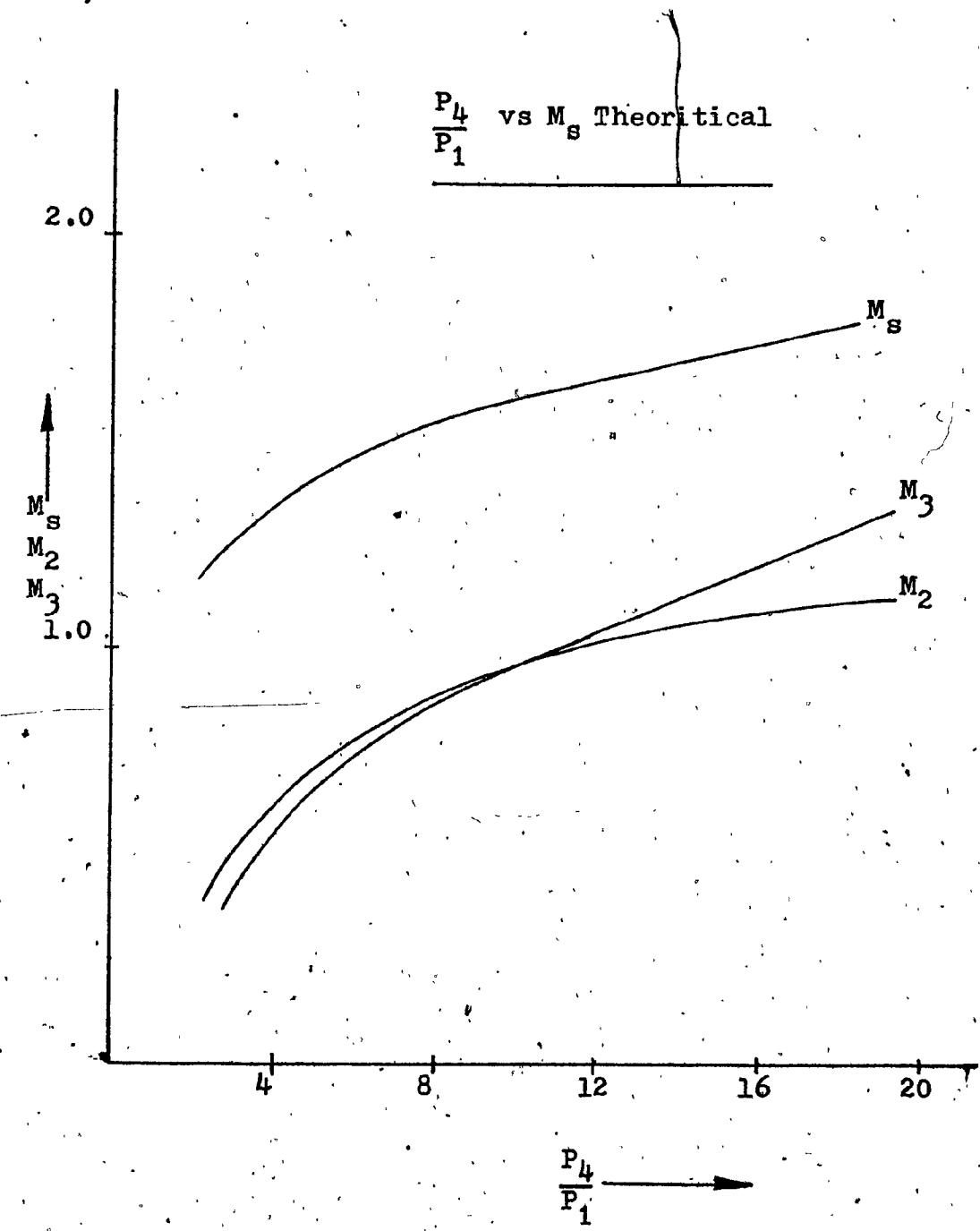


Fig. (5) Shock and flow mach numbers vs initial pressure ratio.

PLOT (1)

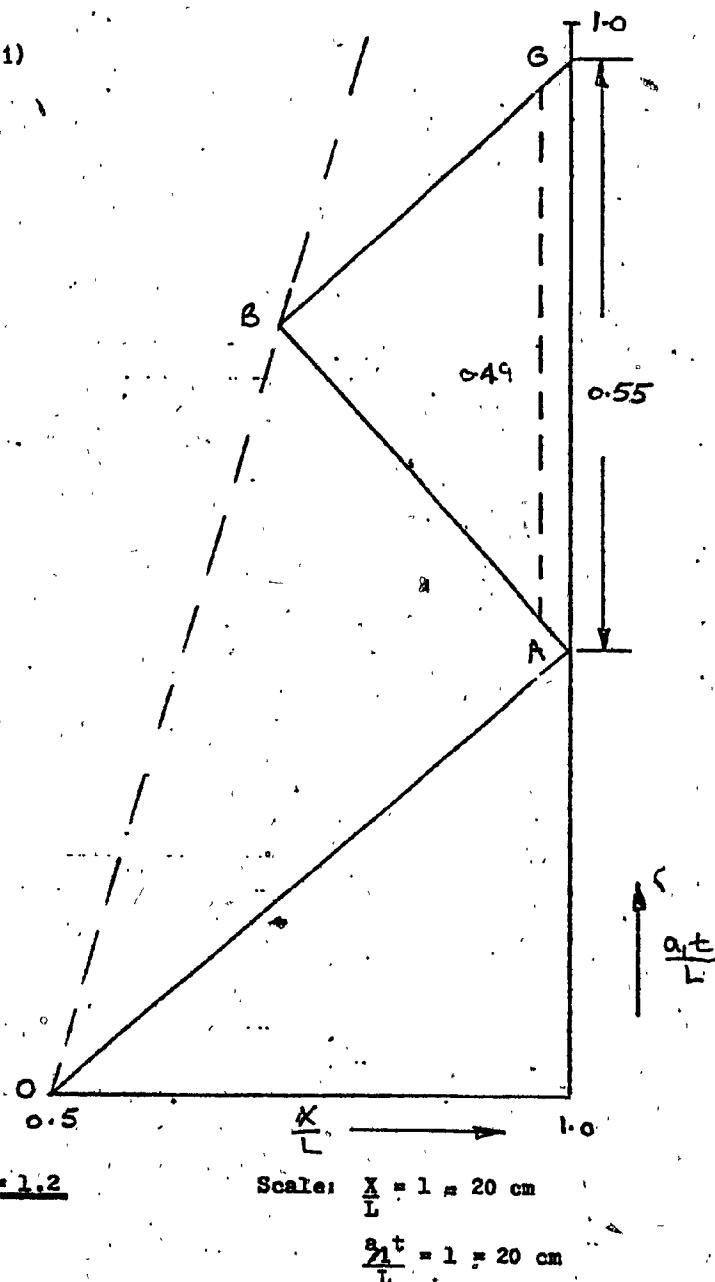
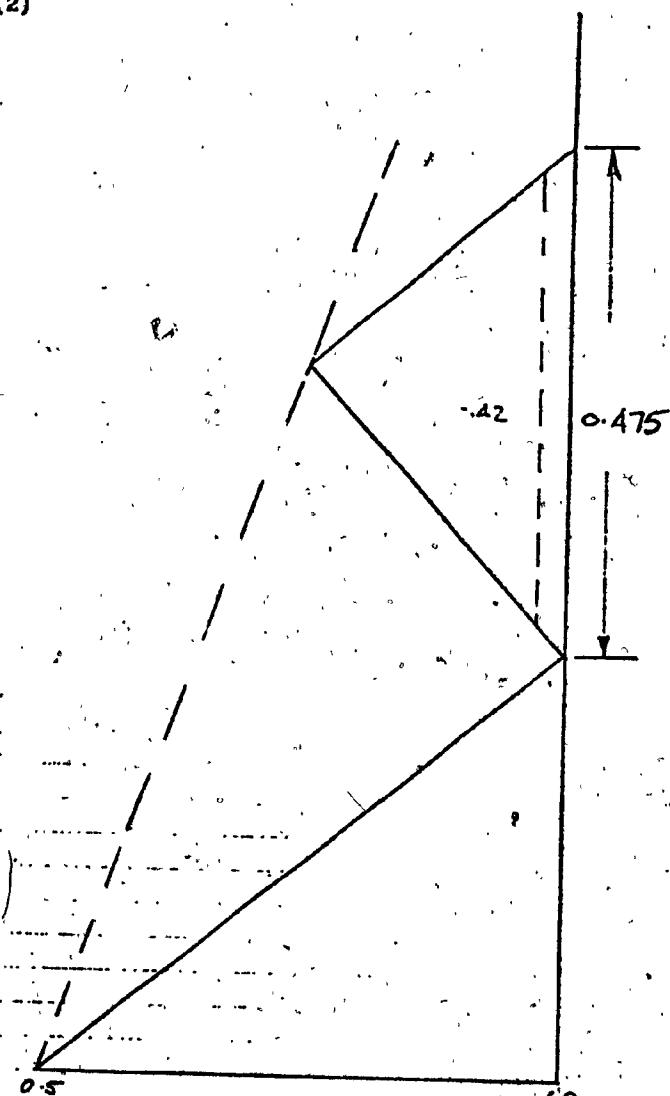


Fig. (6) Wave diagrams to determine theoretically the running time at different incident shock mach numbers.

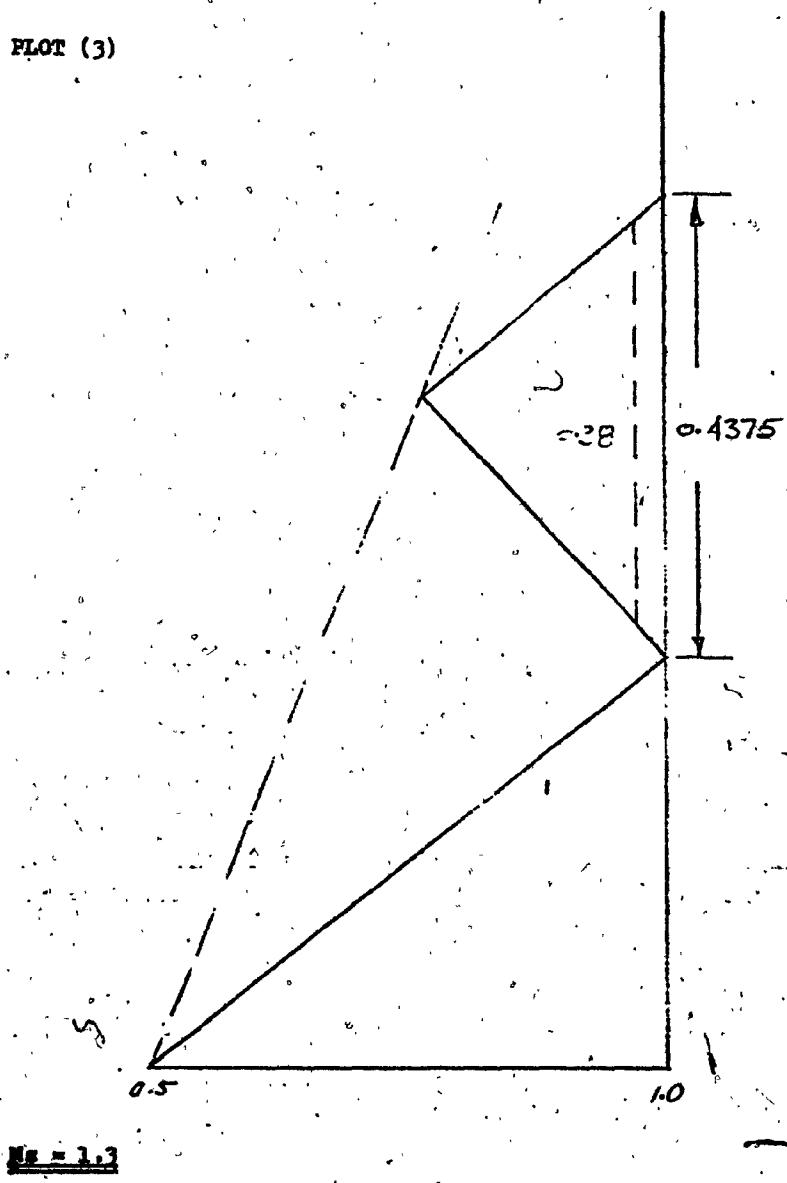
PLOT (2)



$M_2 = 1.25$

Fig. (6) continued

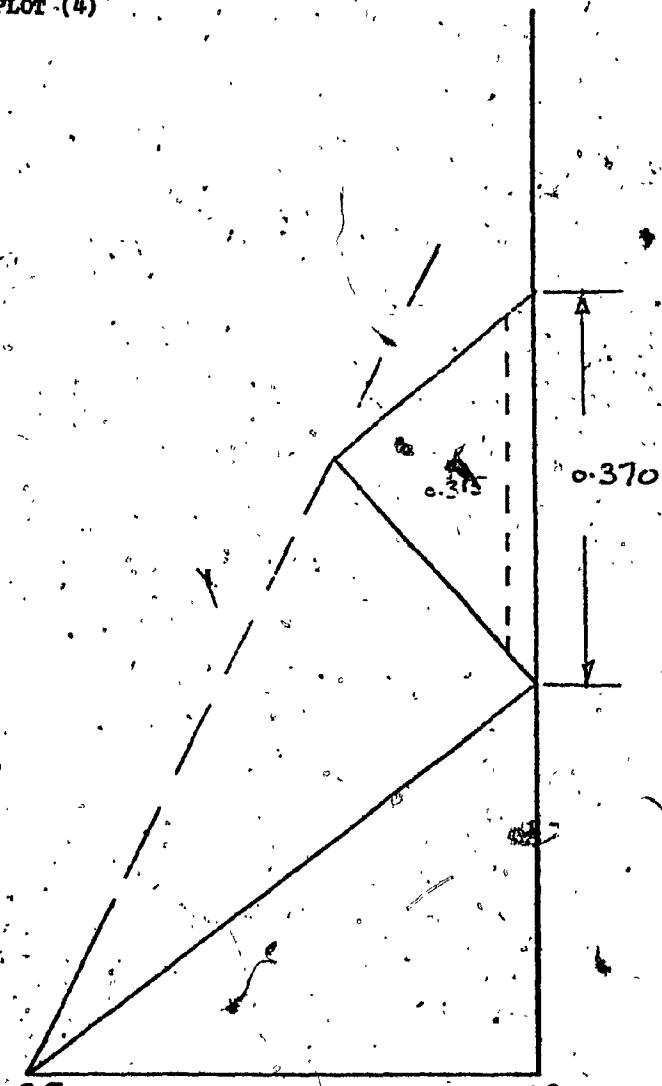
PLOT (3)



$M_2 = 1.3$

Fig. (6) continued

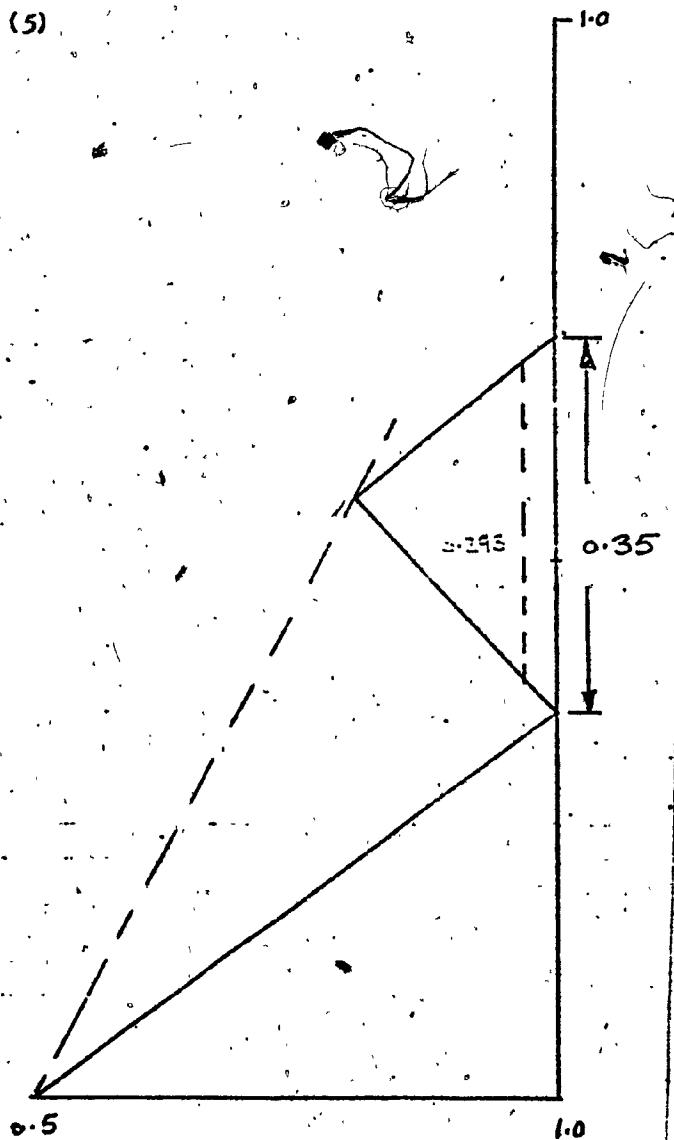
PLOT - (4)



$M_p = 1.35$

Fig. (6) continued

PLOT (5)



Hs = 1.40

Scale: 1 = 20 cm

E = 1 = 20 cm

Fig. (6) continued

PLOT (6)

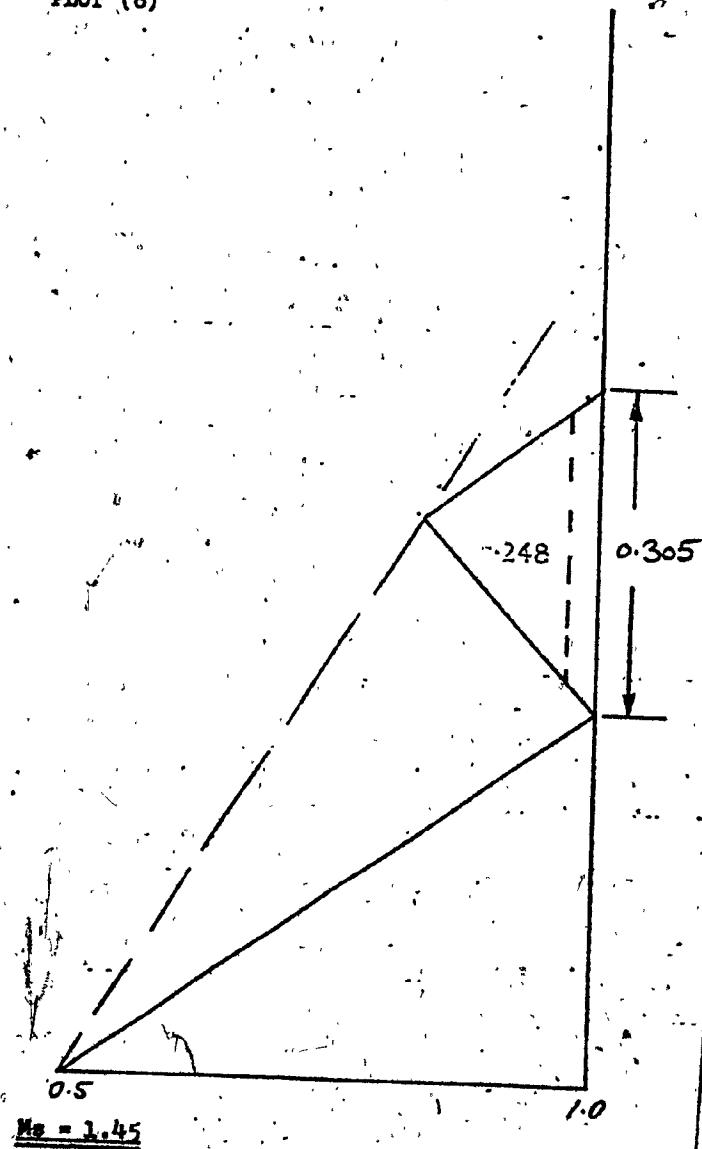
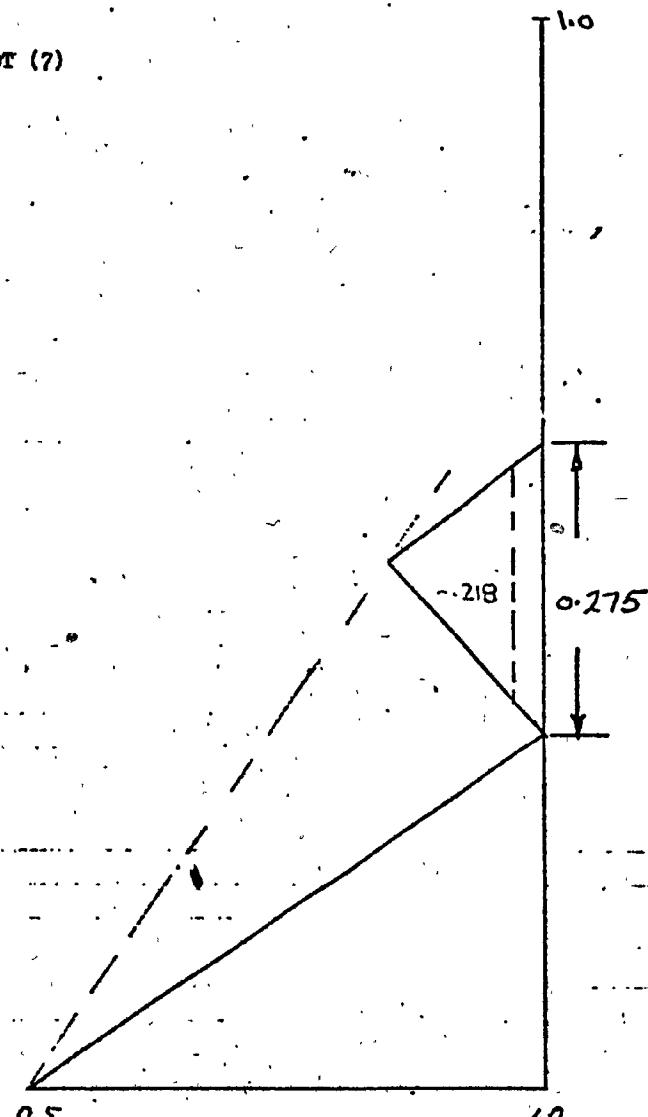


Fig. (6) continued

PLOT (7)



$M_a = 1.5$

Fig. (6) continued

PLOT (8)

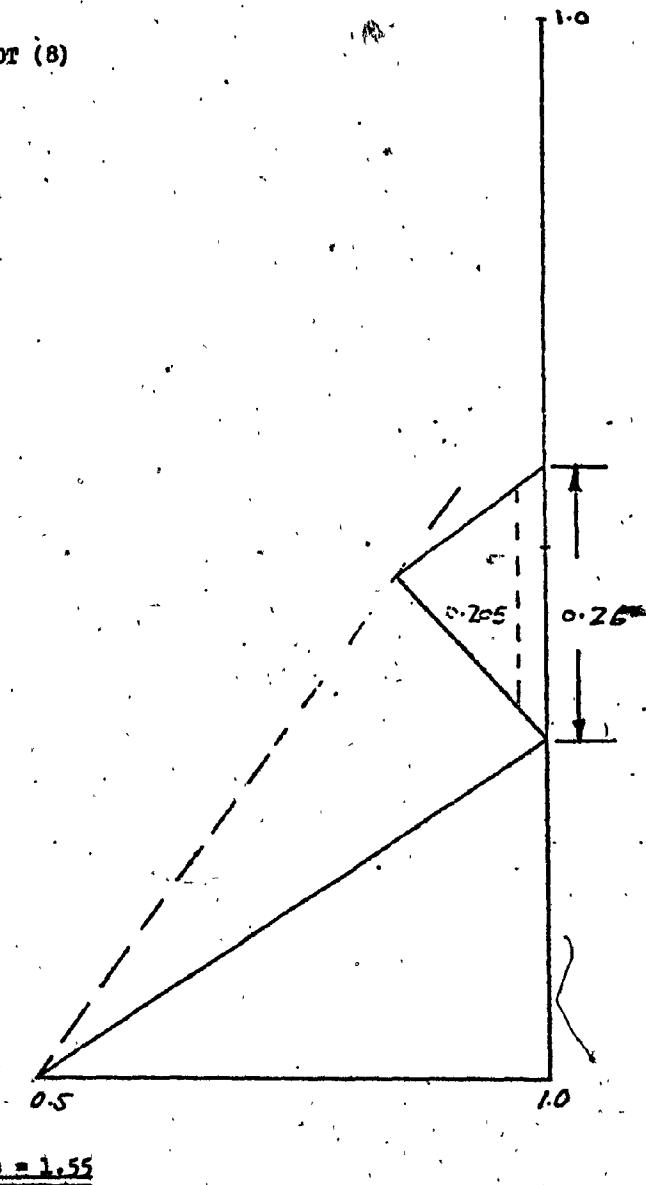
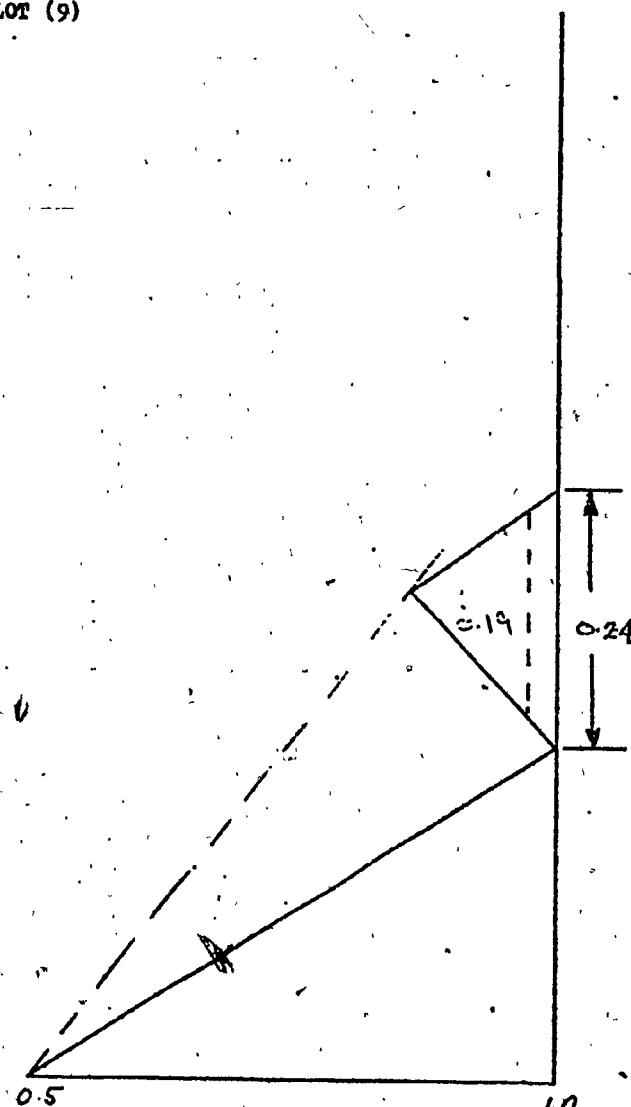


Fig. (6) continued

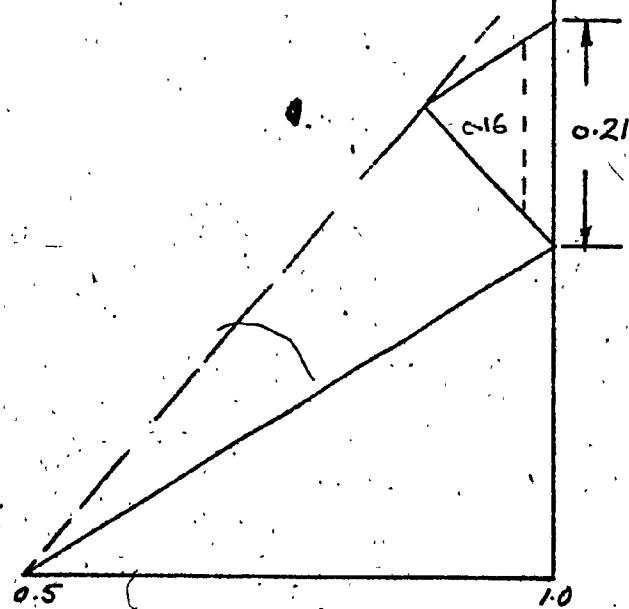
PLOT (9)



$M_2 = 1.60$

Fig. (6) continued

PLOT (10)



$M_\infty = 1.65$

Fig. (6) continued

PLOT (11)

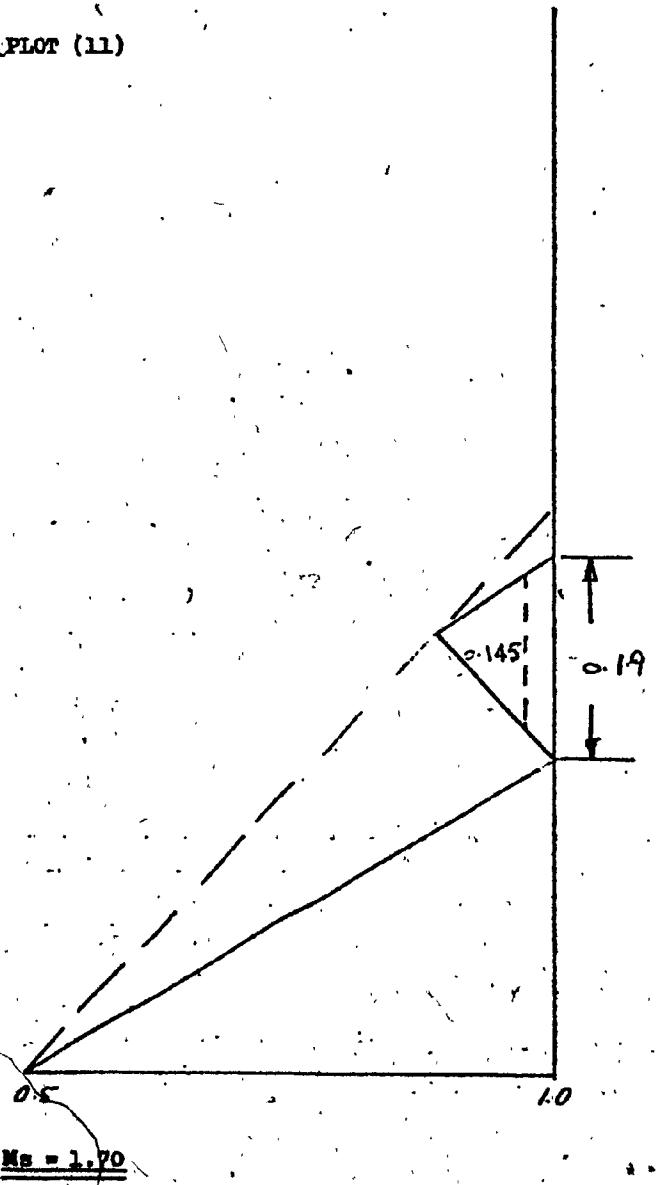
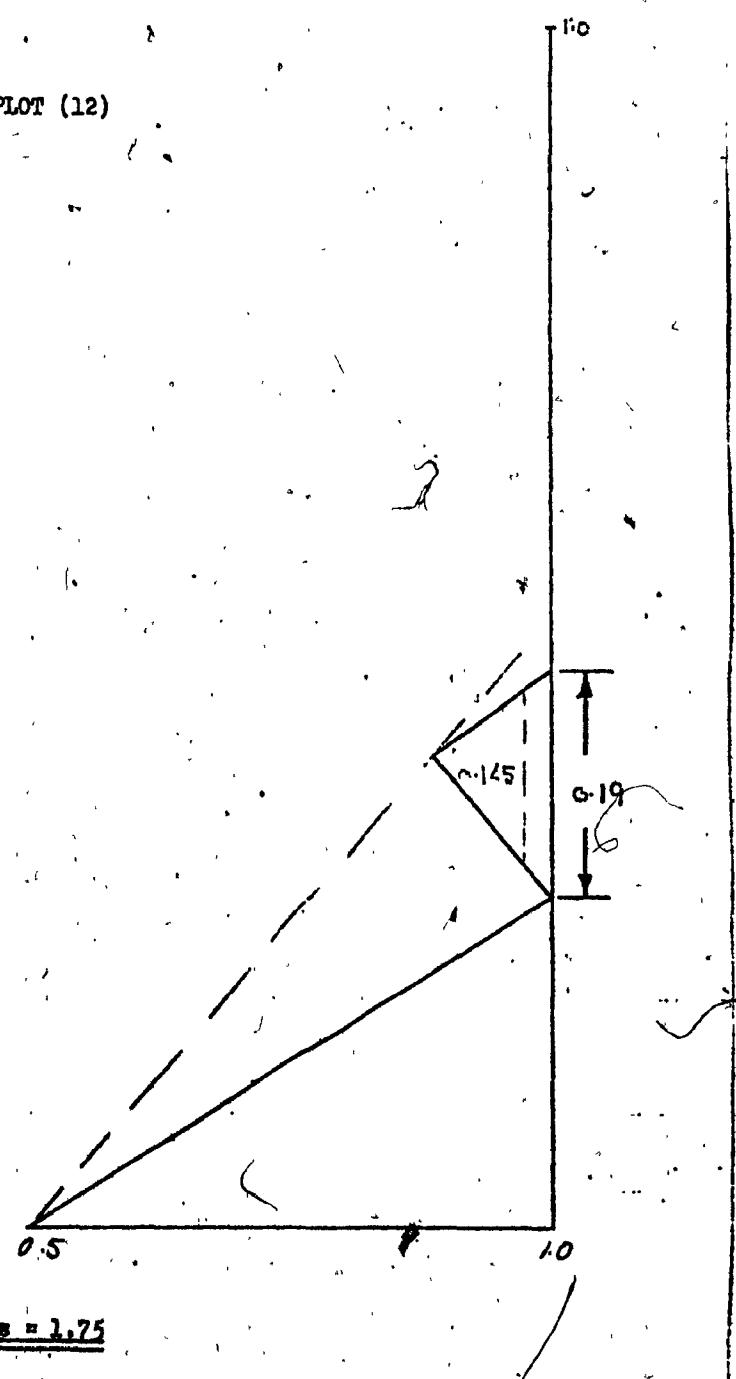


Fig. (6) continued

PLOT (12)



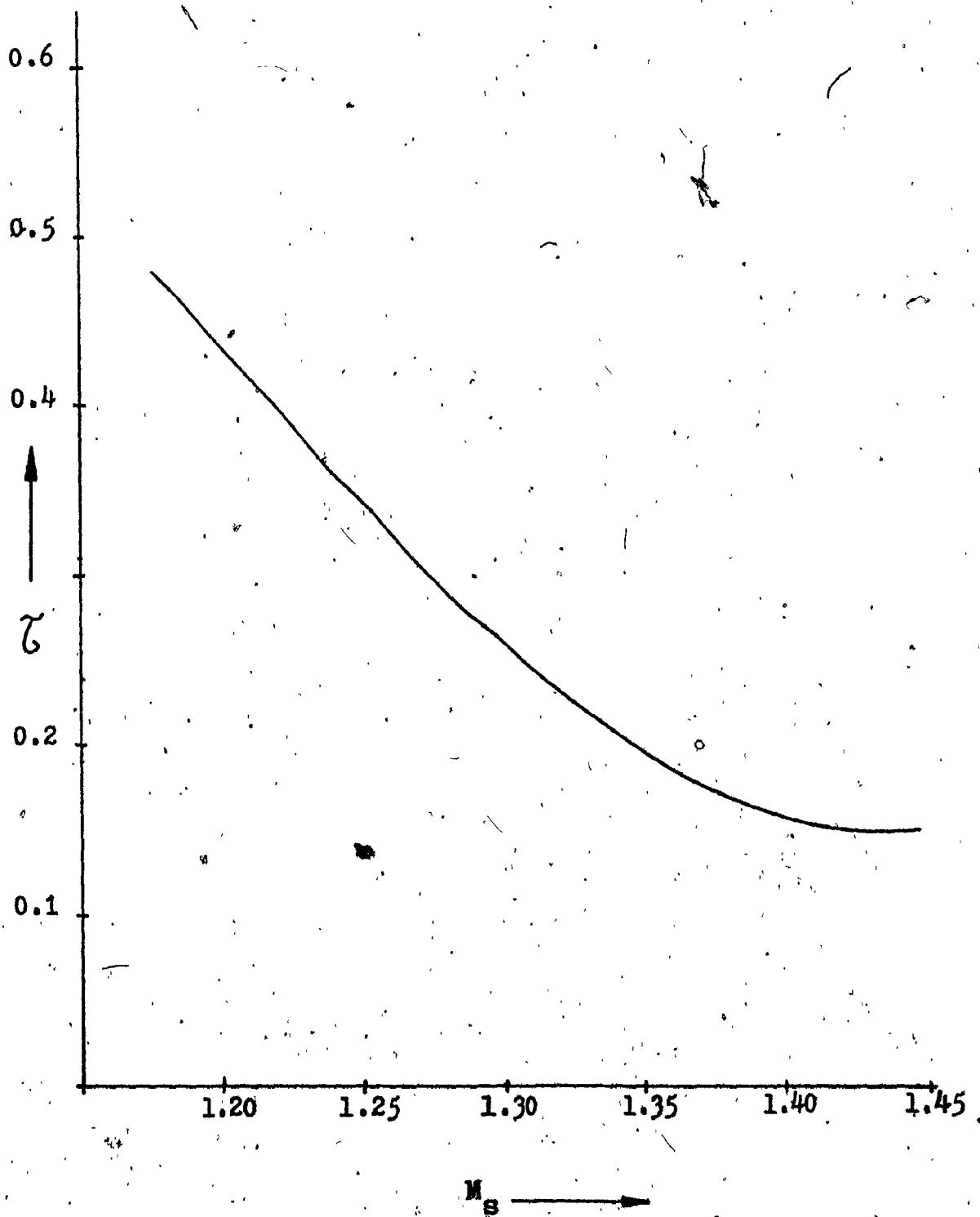


Fig. (7) Theoretical running time vs incident shock mach numbers.

Driver Section

Driven Section

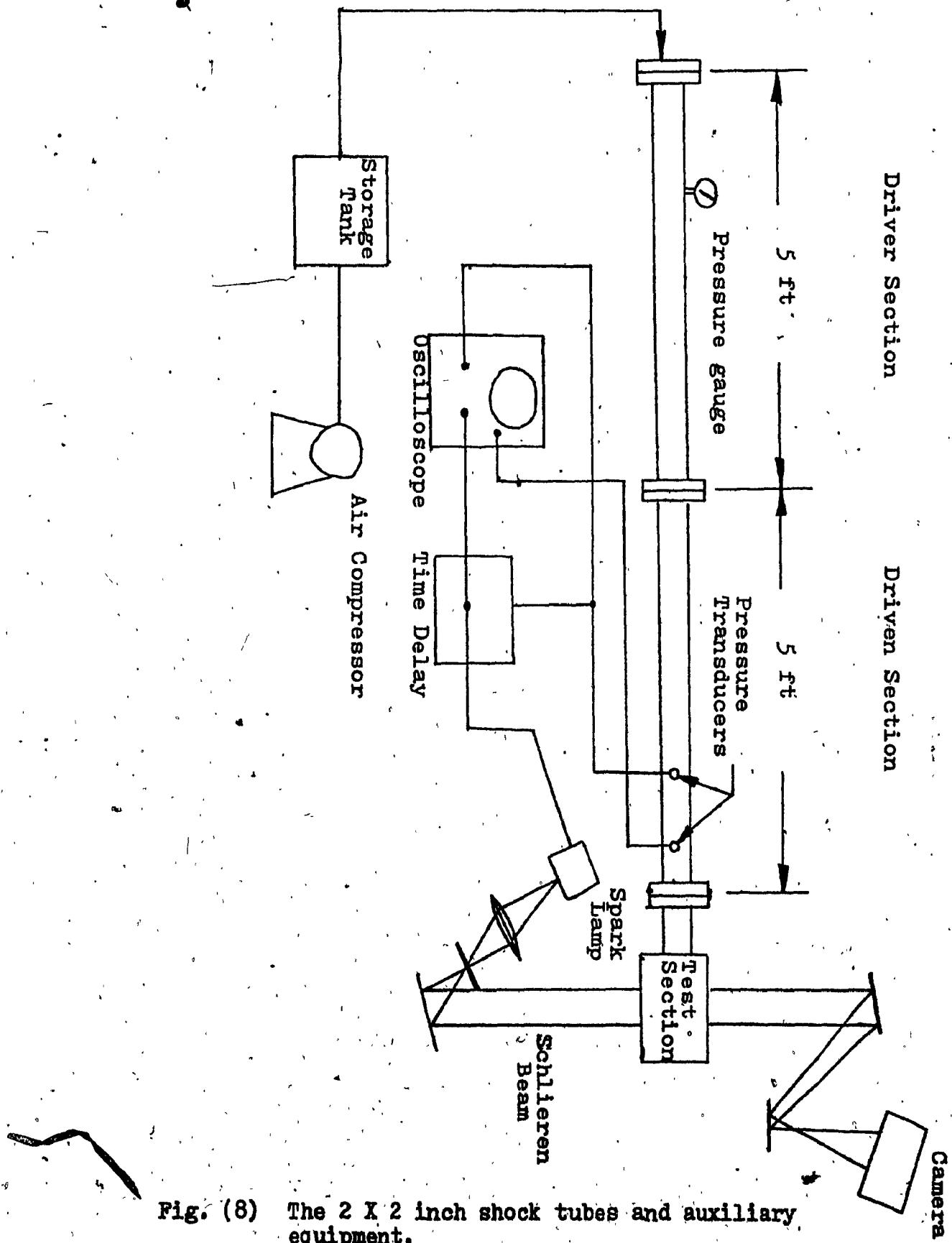
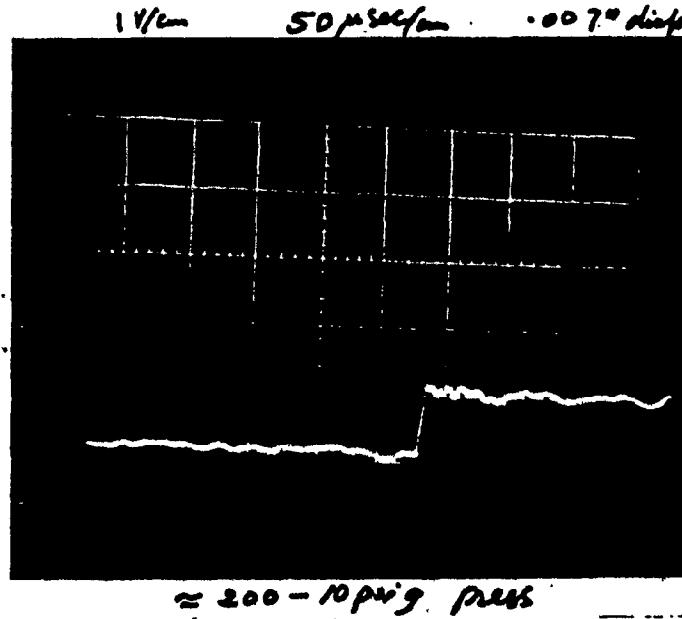
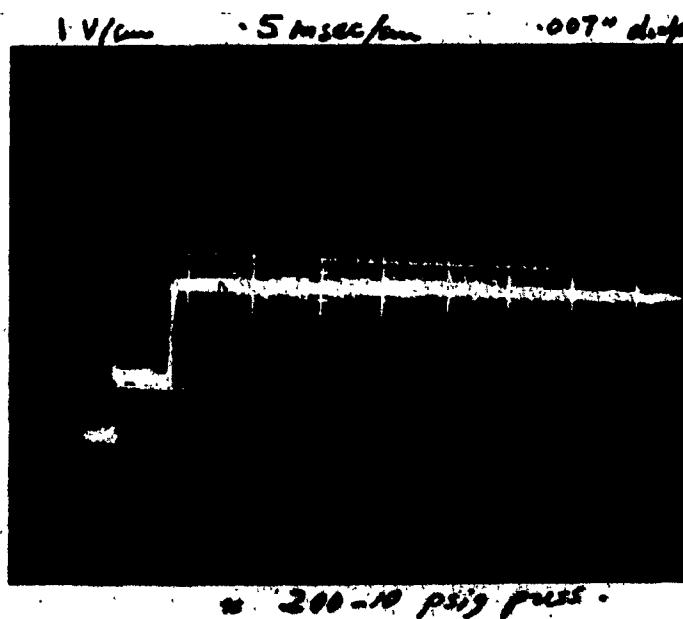


Fig. (8) The 2 X 2 inch shock tubes and auxiliary equipment.

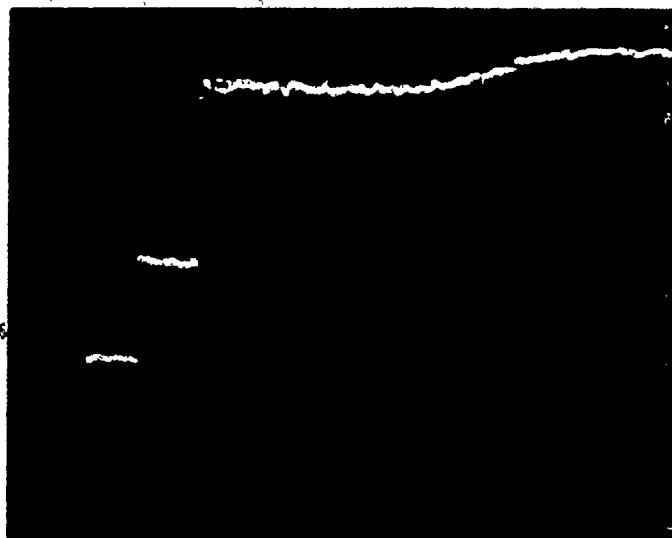


1 v/cm  
50  $\mu$  sec/cm



1 v/cm  
0.5 m sec/cm

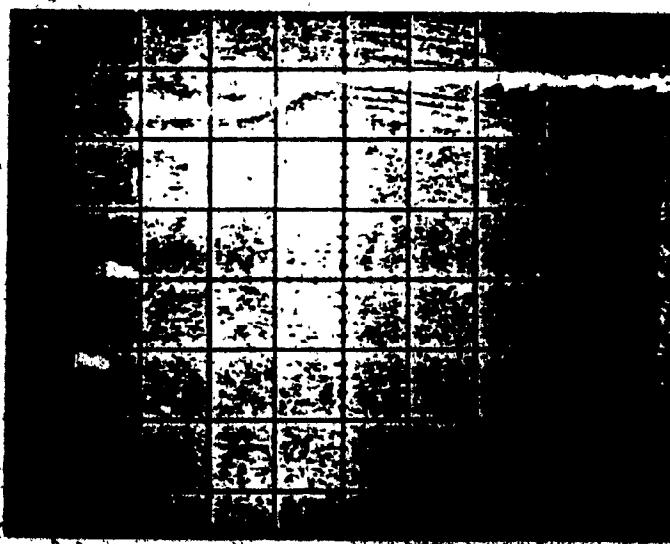
Fig. (9) Pressure traces for shock velocity, running time and stagnation pressure measurements for the 7 mil diaphragm.



0.5 v/cm

0.5 m sec/cm

0.5 msec/in 0.5V/in 100 psig



0.5 v/cm

1 m sec/cm

1 msec/in 0.5V/in 100 psig

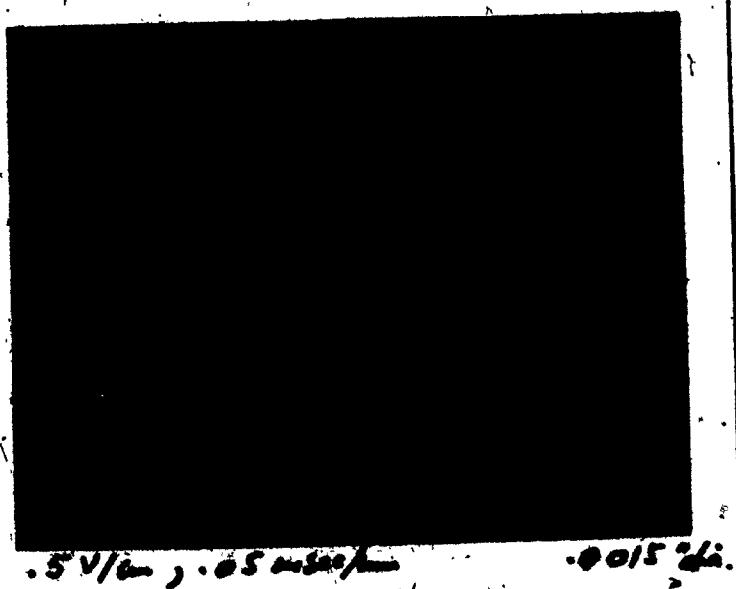
Fig. (10) Pressure traces for running time measurements for the 3 mil diaphragm.



0.5 v/cm  
0.05 m sec/cm

.05 msec/cm, .5V/cm .003" dia.

Fig. (10) cont'd...



0.5 v/cm  
0.05 msec/cm

Fig. (11) Pressure traces used for shock velocity measurements for 1.5 mil diaphragm.

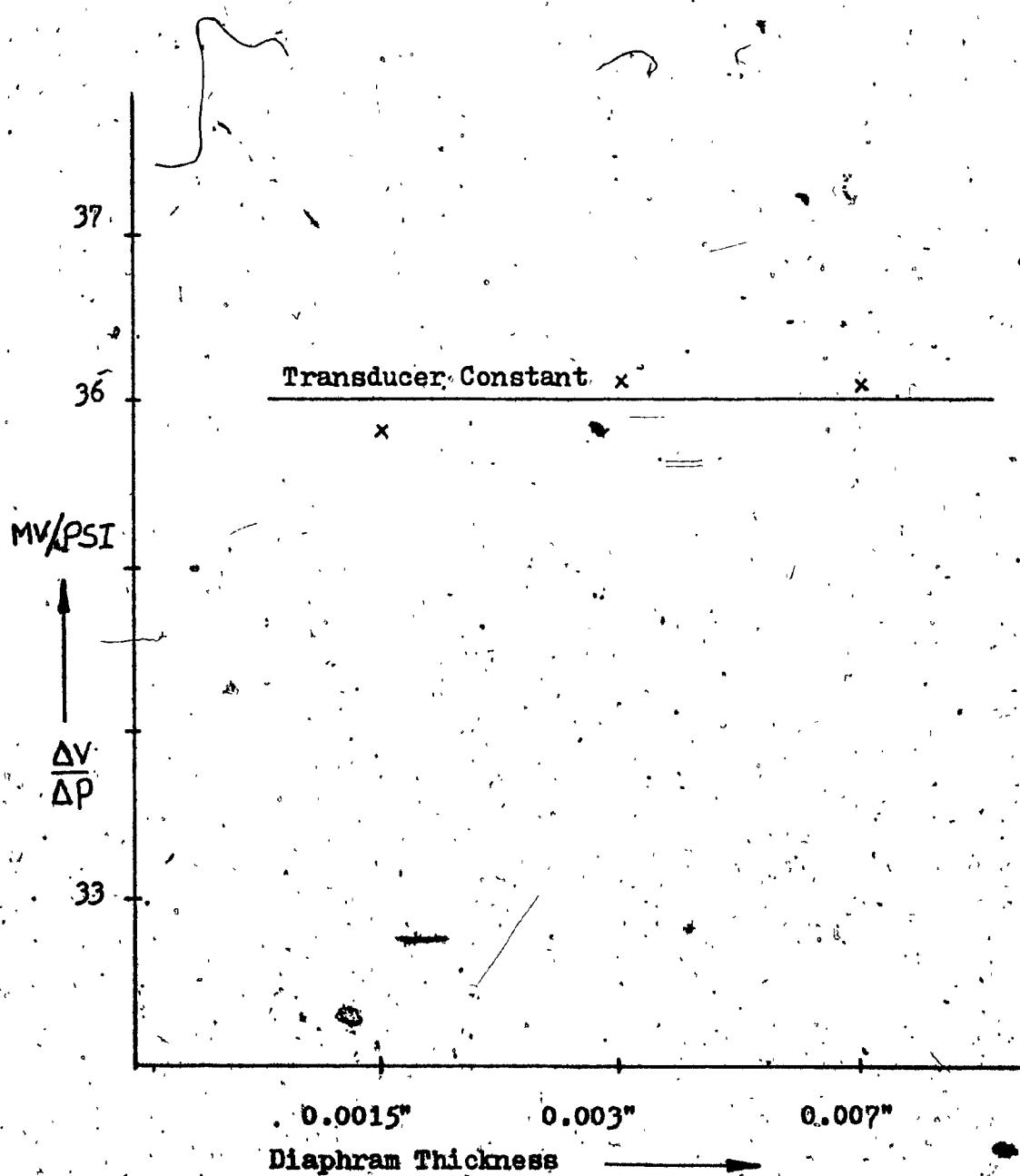


Fig. (12) - Transducer output constant curve.

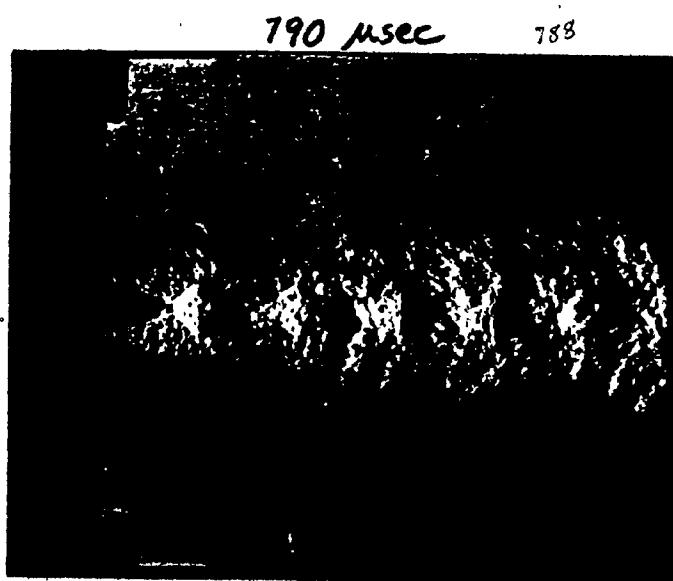
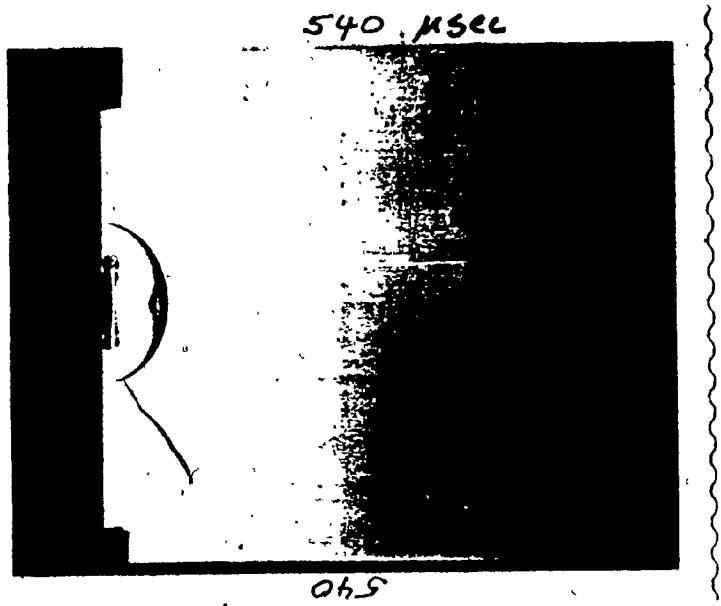


Fig. (13) Spark Schlieren photographs of the jet produced by the shock tube for the 3 mil diaphragm.

640  $\mu$ sec



680  $\mu$ sec

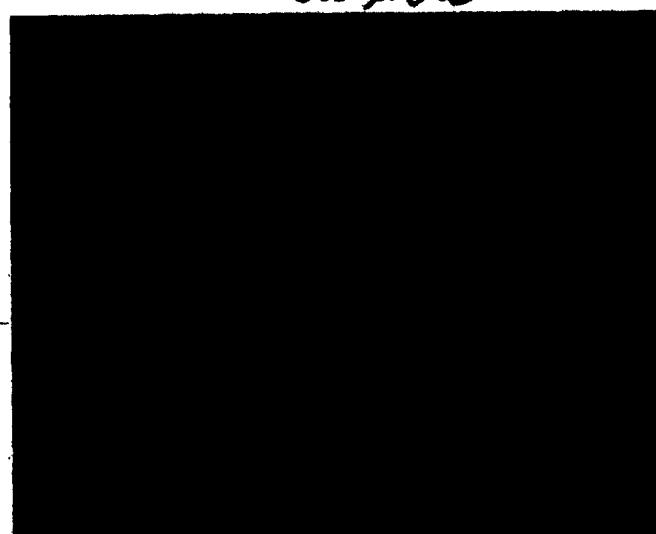
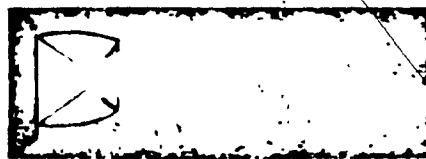


Fig. (14) Spark Schlieren photographs for the jet produced by the shock tube for the 1.5 mil diaphragm.

$P_{\text{stag.}} = 20 \text{ lb/in}^2$  Gauge



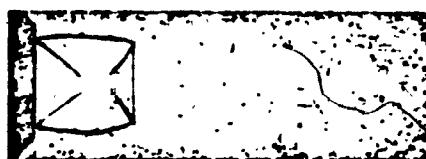
$D = 1.2 \text{ cm}$   
 $L = 1.1 \text{ cm}$

$\bullet 25 \text{ lb/in}^2$



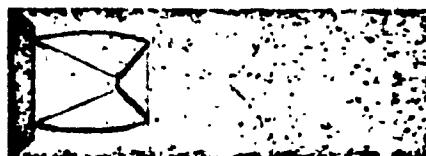
$D = 1.2 \text{ cm}$   
 $L = 1.3 \text{ cm}$

$\bullet 30 \text{ lb/in}^2$



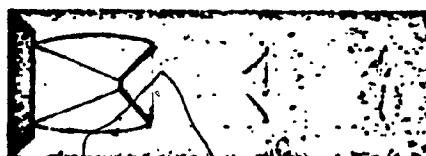
$D = 1.2 \text{ cm}$   
 $L = 1.5 \text{ cm}$

$\bullet 35 \text{ lb/in}^2$



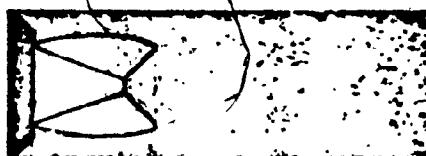
$D = 1.2 \text{ cm}$   
 $L = 1.7 \text{ cm}$

$\bullet 40 \text{ lb/in}^2$



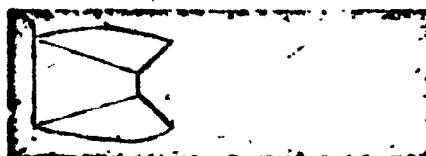
$D = 1.2 \text{ cm}$   
 $L = 1.75 \text{ cm}$

$\bullet 45 \text{ lb/in}^2$



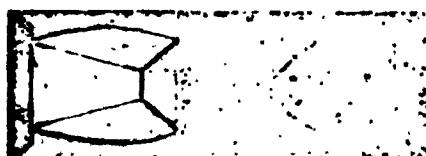
$D = 1.2 \text{ cm}$   
 $L = 1.85 \text{ cm}$

$\bullet 50 \text{ lb/in}^2$



$D = 1.2 \text{ cm}$   
 $L = 2.05 \text{ cm}$

$\bullet 55 \text{ lb/in}^2$



$D = 1.2 \text{ cm}$   
 $L = 2.1 \text{ cm}$

### SPARK-SHADOWGRAPHS OF PERIODIC JET EFFLUXES

FIGURE 15

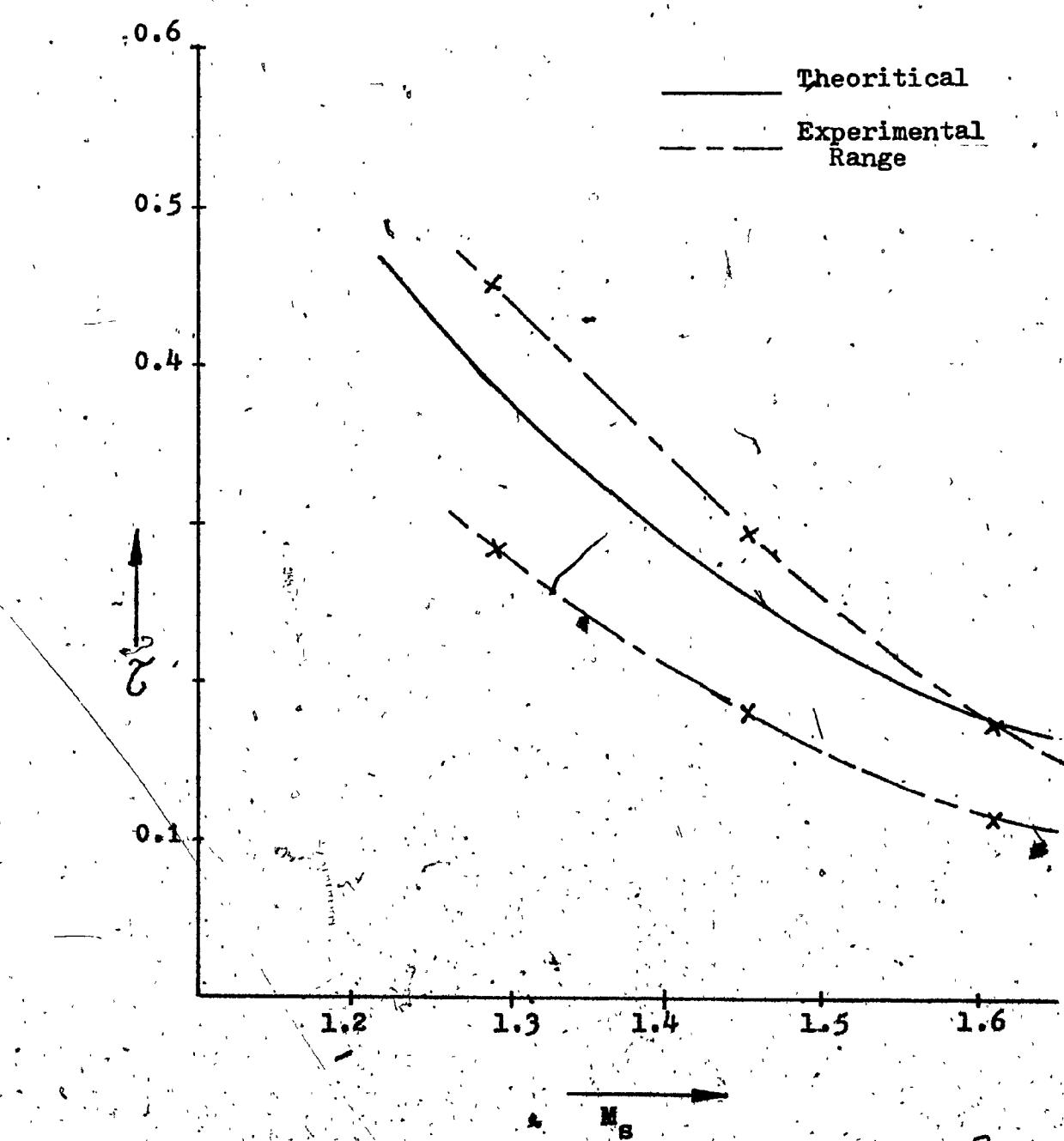


Fig. (16) Theoretical and experimental running time vs the incident shock mach numbers.

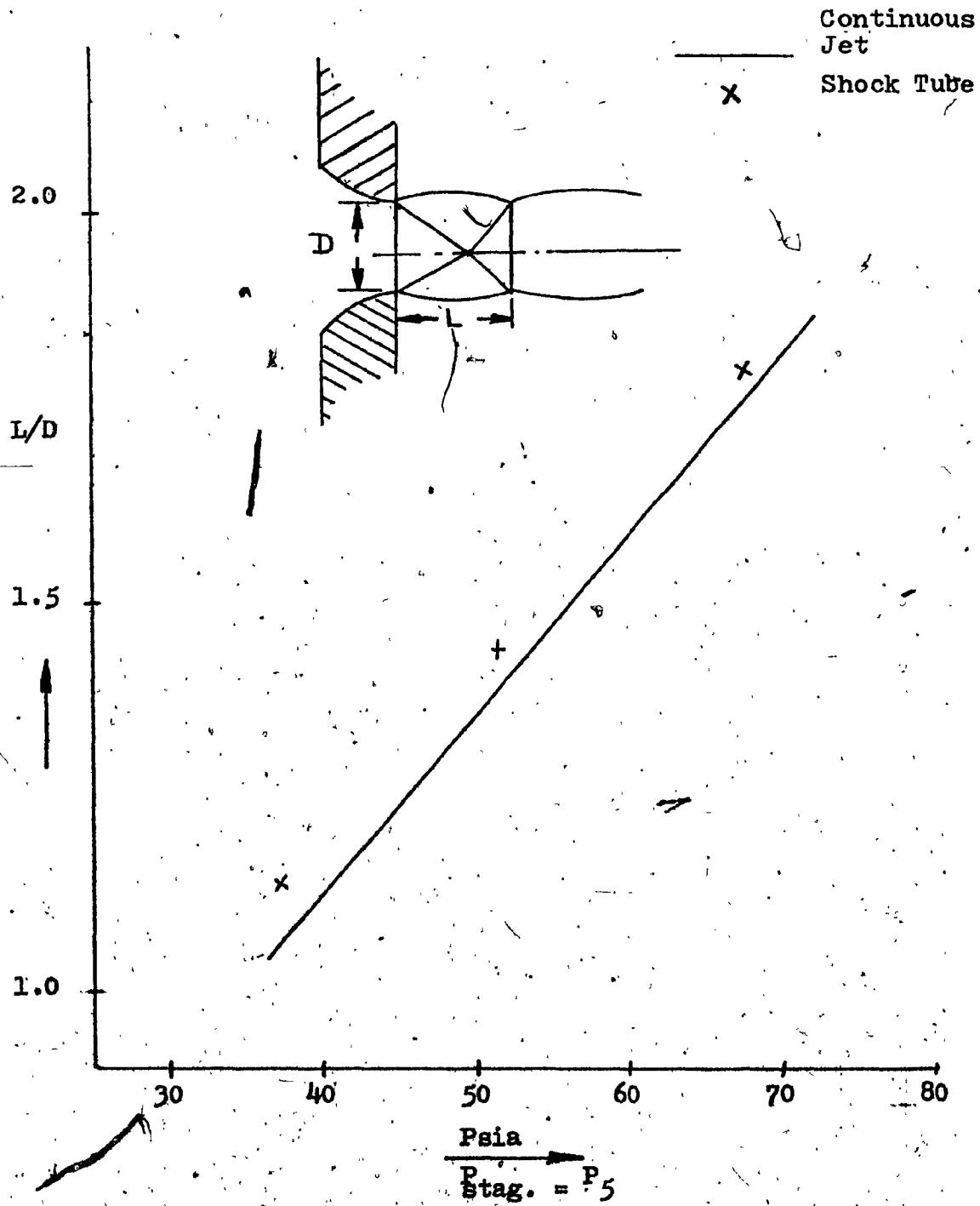


Fig. (17). Cell length at different stagnation pressures.

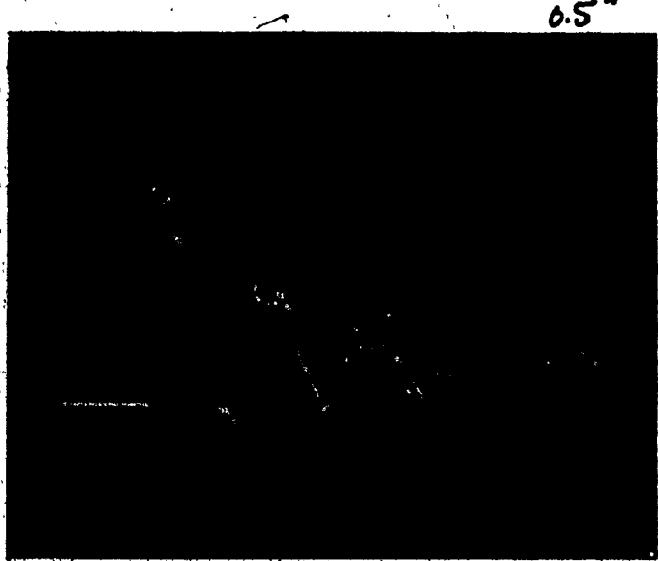
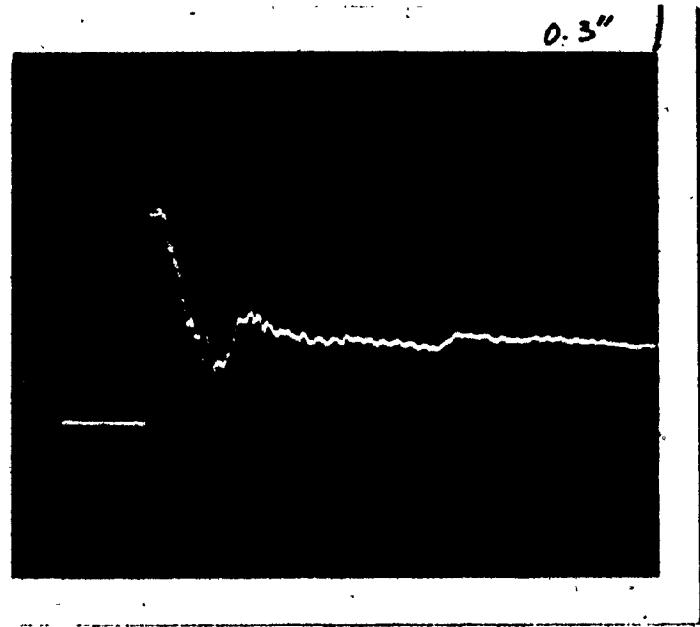
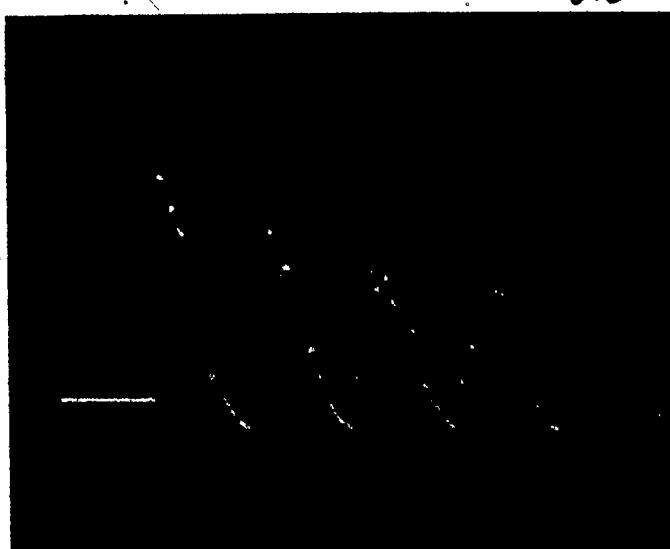
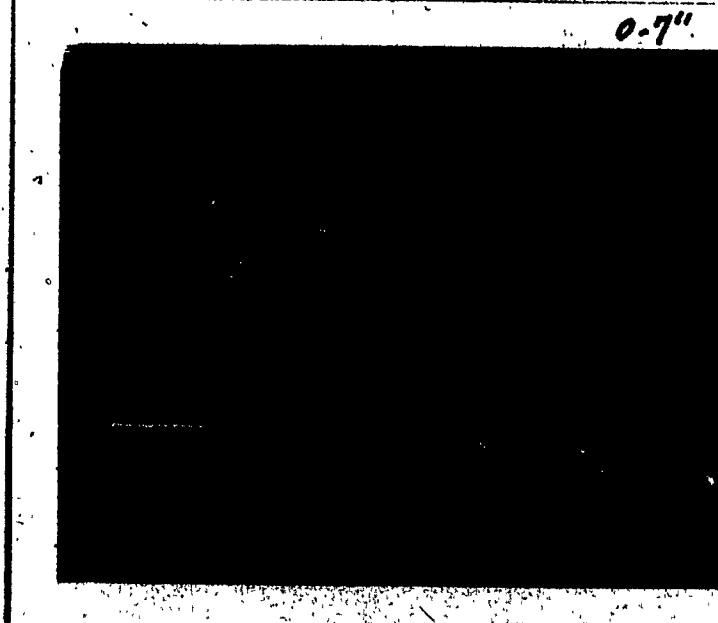


Fig. (16) Pressure traces recorded at the closed end of a square resonance tube placed at different locations.

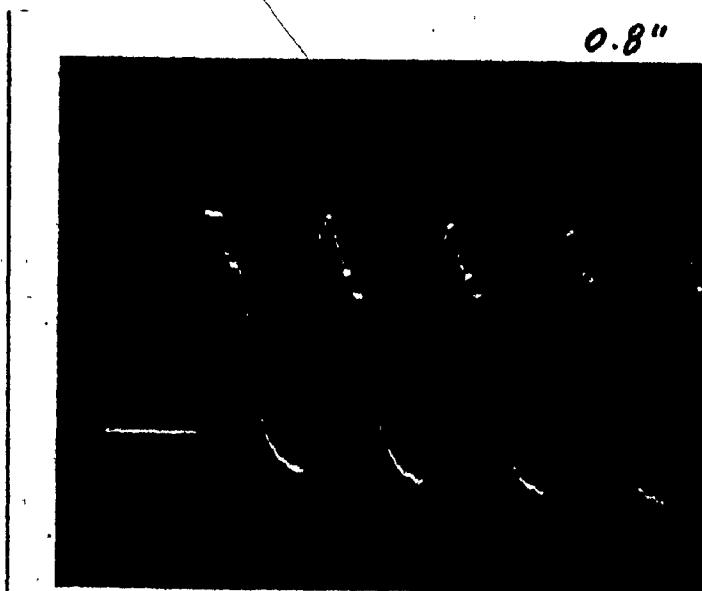


0.6"

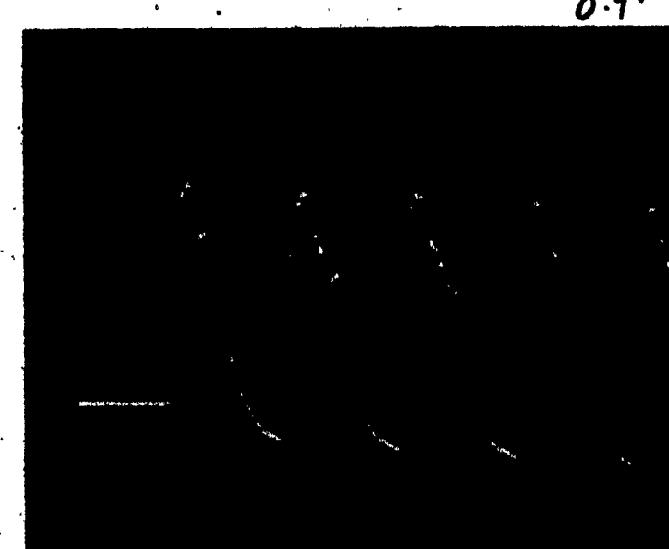


0.7"

Fig. (18) cont'd...



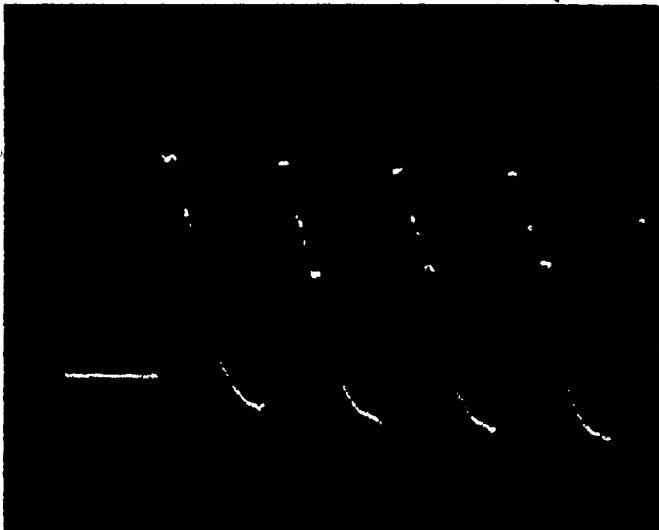
0.8"



0.9"

Fig. (18) cont'd...

1.0

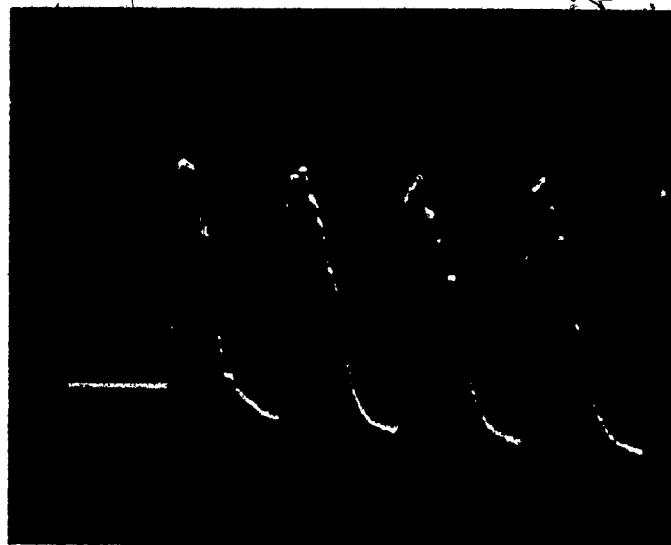


1.1



Fig. (18) cont'd...

1.2°

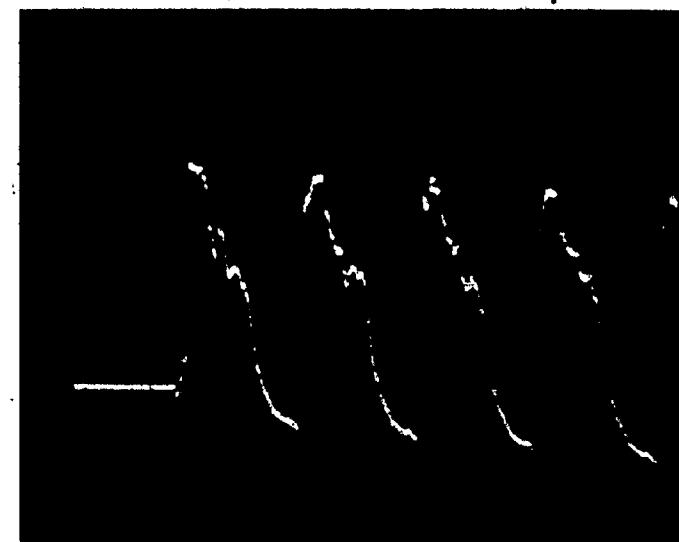


1.3°



Fig. (18) cont'd.

1.5"



1.7



Fig. (18) cont'd...

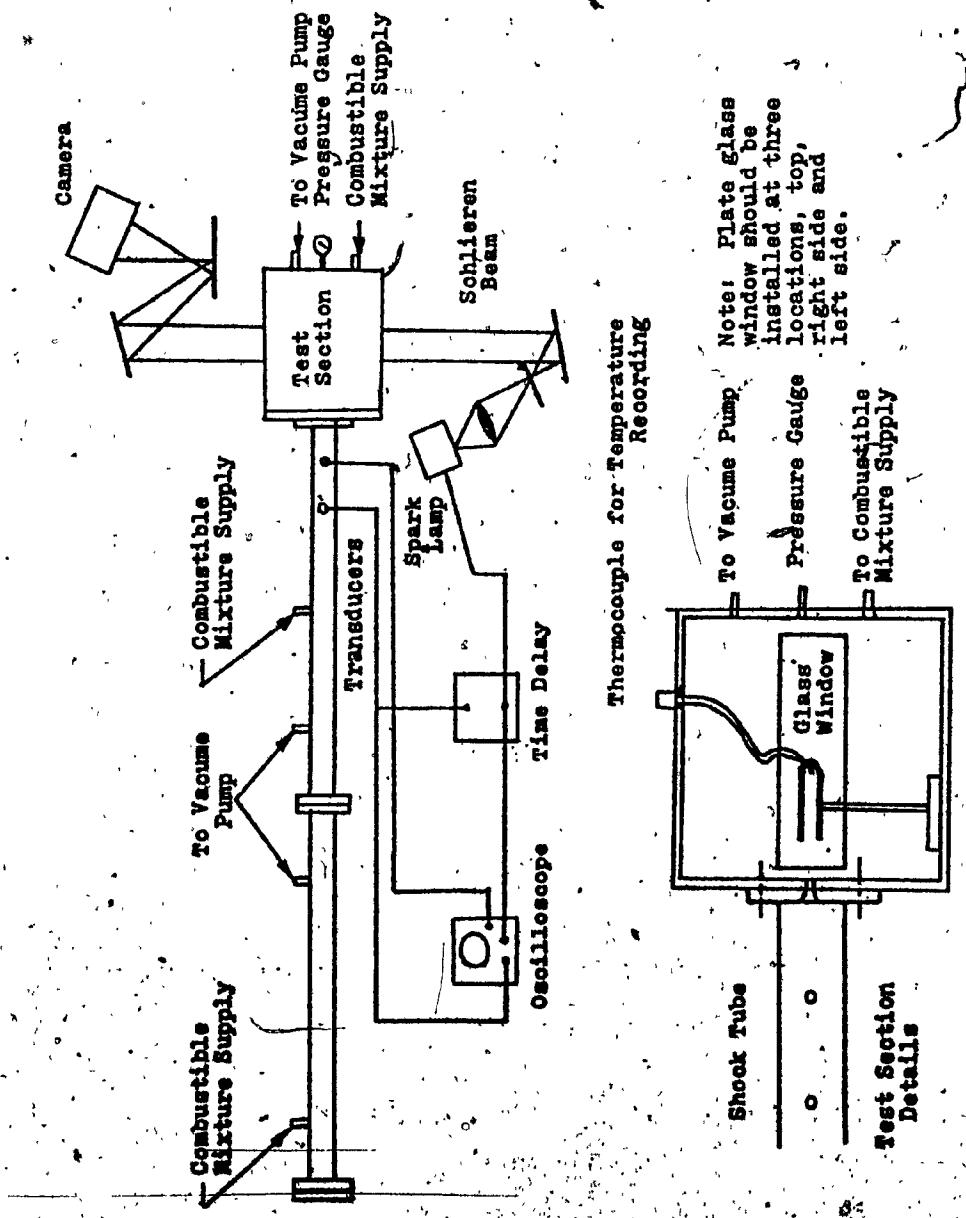


Fig. (19) Proposed Test Section

TABLE (1)

$M_s$	$P_{26}/P_1$	$P_4/P_1$	$M_2$	$M_3$
1.2	1.513	2.34	0.403	0.32
1.25	1.656	2.88	0.487	0.41
1.30	1.805	3.467	0.567	0.489
1.35	1.960	4.16	0.64	0.568
1.40	2.12	4.986	0.714	0.65
1.45	2.29	5.93	0.780	0.728
1.5	2.46	7.07	0.847	0.815
1.55	2.637	8.36	0.907	0.897
1.60	2.82	9.80	0.965	0.974
1.65	3.01	11.57	1.02	1.06
1.7	3.205	13.50	1.07	1.14
1.75	3.407	15.82	1.125	1.227

TABLE (2)

Ms	Msr <sub>5</sub>	$\frac{P_5}{P_2}$
1.2	1.188	1.48
1.25	1.23	1.6
1.30	1.266	1.7
1.35	1.32	1.84
1.40	1.351	1.96
1.45	1.351	1.96
1.45	1.389	2.08
1.50	1.425	2.2
1.55	1.46	2.33
1.60	1.495	2.44
1.65	1.528	2.55
1.70	1.56	2.68
1.75	1.59	2.79

TABLE (3)

$M_s$	$M_{sr5}$	$a_2$	$a_3$	$a_5$	$U_2$	$M_{sr5,6}$	$M_{sr3,7}$
1.2	1.188	1.061	0.939	1.122	0.306	1.01	1.199
1.25	1.23	1.077	0.924	1.153	0.375	1.015	1.25
1.3	1.266	1.091	0.911	1.179	0.433	1.021	1.297
1.35	1.32	1.106	0.898	1.213	0.509	1.027	1.351
1.40	1.351	1.120	0.885	1.238	0.570	1.034	1.395
1.45	1.389	1.135	0.873	1.267	0.635	1.039	1.455
1.5	1.425	1.149	0.860	1.290	0.694	1.044	1.496
1.55	1.46	1.163	0.850	1.322	0.753	1.049	1.561
1.6	1.495	1.178	0.837	1.352	0.811	1.056	1.614
1.65	1.528	1.192	0.825	1.389	0.868	1.068	1.641
1.70	1.56	1.207	0.814	1.408	0.927	1.077	1.695
1.75	1.59	1.222	0.802	1.437	0.982	1.084	1.744

TABLE (4)

LINE OA	LINE AB	LINE BG	LINE OB	LINE BH
39.8	46.3	41.4	73	50.6
38.7	46.5	40.5	69.4	52
37.6	46.5	39.7	66.6	53
36.5	46.44	38.7	63	54.8
35.5	46.7	58	60.3	56.4
34.6	46.7	37	57.6	57.6
33.7	46.8	36.6	55	59.4
32.8	46.6	35.8	53	60
32.0	46.5	35	51	61.6
31.2	46.4	34	49	64
30.5	46.3	33.4	47	65.6
29.7	46.1	32.7	45.5	67